

Sta # 4
 IIII 26 1998

15

ENGINEERING DATA TRANSMITTAL

Page 1 of 1
 1. EDT 622232

2. To: (Receiving Organization) Distribution		3. From: (Originating Organization) TWRS Projects/SST Retrieval		4. Related EDT No.: n/a	
5. Proj./Prog./Dept./Div.: D2991 W-320 TWRS/TCPN # D2MAT		6. Design Authority/ Design Agent/Cog. Engr.: JW Bailey, NHC		7. Purchase Order No.: n/a	
8. Originator Remarks: For approval and release of a new supporting document. This document has been generated to ensure retrievability of the Project W-320 "Piping Calculations, Vol. 4".				9. Equip./Component No.: n/a	
				10. System/Bldg./Facility: 241-C-106	
11. Receiver Remarks: 11A. Design Baseline Document? [X] Yes [] No				12. Major Assm. Dwg. No.: n/a	
				13. Permit/Permit Application No.: n/a	
				14. Required Response Date:	

15. DATA TRANSMITTED					(F)	(G)	(H)	(I)
(A) Item No.	(B) Document/Drawing No.	(C) Sheet No.	(D) Rev. No.	(E) Title or Description of Data Transmitted	Approval Designator	Reason for Transmittal	Originator Disposition	Receiver Disposition
1	HNF-2474	-	0	Project W-320, 241-C-106 Sluicing, Piping Calculations, Vol. 4	NA			-

16. KEY

Approval Designator (F)	Reason for Transmittal (G)	Disposition (H) & (I)
E, S, Q, D or N/A (see WHC-CM-3-5, Sec.12.7)	1. Approval 2. Release 3. Information 4. Review 5. Post-Review 6. Dist. (Receipt Acknow. Required)	1. Approved 2. Approved w/comment 3. Disapproved w/comment 4. Reviewed no/comment 5. Reviewed w/comment 6. Receipt acknowledged

17. SIGNATURE/DISTRIBUTION
 (See Approval Designator for required signatures)

(G) Reason	(H) Disp.	(J) Name	(K) Signature	(L) Date	(M) MSIN	(G) Reason	(H) Disp.	(J) Name	(K) Signature	(L) Date	(M) MSIN
2	1	Design Authority	JW Bailey	7/23/98	S2-48						
		Design Agent	MC Davenport	7/23/98	S2-48						
2	1	Cog. Eng.	BE Greaves	7/23/98	S2-48						
2	1	Cog. Mgr	JW Bailey	7/23/98	S2-48						
		QA									
		Safety									
		Env.									

18. MC Davenport Signature of EDT Originator Date 7/23/98		19. Authorized Representative Date for Receiving Organization		20. JW Bailey Signature of Design Authority/Cognizant Manager Date 7/23/98		21. DOE APPROVAL (if required) Ctrl. No. <input type="checkbox"/> Approved <input type="checkbox"/> Approved w/comments <input type="checkbox"/> Disapproved w/comments	
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Project W-320, 241-C-106 Sluicing Piping Calculations, Vol. 4

John W. Bailey

Numatec Hanford Co., Richland, WA 99352
U.S. Department of Energy Contract DE-AC09-96RL13200

EDT/ECN: 622232 UC: 506
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Key Words: W-320, Sluicing, Tank 241-C-106, Tank 241-AY-102, WRSS,
calculations, piping.

Abstract: This supporting document has been prepared to make the FDNW
calculations for Project W-320, readily retrievable.

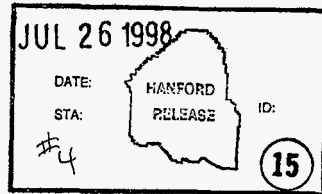
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[Signature]
Release Approval

7/24/98
Date



Release Stamp

Approved for Public Release

Project W-320, 241-C-106 Sluicing Piping Calculations, Vol. 4

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W320-27-015

Support Structural Analysis

This sheet shows the status and description of the attached Design Analysis sheets.

Discipline 27, Piping and Vessels WO/Job No. ER4319 Calculation No. W-320-27-015
 Project No. & Name Project W 320 Waste Retrieval for Tank 241-C-106
 Calculation Item Support Structural Analysis

These calculations apply to:

Dwg. No. H-2-818548 Shts 1 & 2 of 2 Rev. No. 0
 Dwg. No. _____ Rev. No. _____
 Other (Study, CDR) _____ Rev. No. _____

The status of these calculations is:

- Preliminary Calculations
- Final Calculations
- Check Calculations (On Calculation Dated _____)
- Void Calculation (Reason Voided _____)

Incorporated in Final Drawings? Yes No
 This calculation verified by independent "check" calculation? Yes No

Original and Revised Calculation Approvals:

	Rev. 0 Signature / Date	Rev. 1 Signature / Date	Rev. 2 Signature / Date
Originator	D. L Stone 5/20/94 J. R. Booth 5/31/94	D. L Stone 8/8/95	<i>D. L Stone</i> 10/13/96
Checked by	M.M. Ahmed 5/20/94	C.D. Jones 8/10/95	<i>[Signature]</i> 10/03/96
Approved by	C.D. Jones 6/3/94	C.D. Jones 8/10/95	<i>[Signature]</i> 10/03/96
Checked Against Approved Vendor Data			

INDEX

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<u>Appx A</u>	<u>"POP" Analysis</u>
<u>Appx B</u>	<u>Drawings</u>

This sheet shows the status and description of the attached Design Analysis sheets.
 Discipline: (27) Piping and Vessels WO/Job No.: ER4319
 Project No. & Name: Project W-320 Waste Retrieval for Tank 241-C-106
 Calculation Item: Support Structural Analysis

Calculation No.: W320-27-015

These calculations apply to:

Dwg. No.: See Calculation Cross Index

Rev. No.

Dwg. No.:

Rev. No.

Other (Study, CDR):

Rev. No.

The status of these calculations is:

- Preliminary Calculations
 Final Calculations
 Check Calculations (On Calculation Dated)
 Void Calculation (Reason Voiced)

Incorporated in Final Drawings?

 Yes No

This calculation verified by independent "check" calculations?

 Yes No

Original and Revised Calculation Approvals:

	Rev. 3 Signature/Date	Rev. 4 Signature/Date	Rev. 5 Signature/Date
Originator	<i>Kelly K. Hagan</i> 11/24/97		
Checked by	<i>M. Ahmed</i> 3.13.98		
Approved by	<i>J. P. Evans</i> 4/9/98		
Checked Against Approved Vendor Data	<i>M. Ahmed</i> 4.10.98		

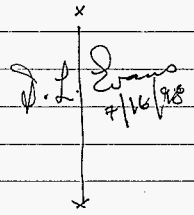
INDEX

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ii	Calculation Cross Index
1	Objective, Design Inputs, Design Methodology
2	Calculations, References and Conclusions
3 - 29	Calculations
Appx A	"POP" Analysis
Appx B	Drawings
	Rev. 3 Incorporates As-Built Conditions. Page ia added. Pages ii, 1, 2, 6, and 16 revised.

CALCULATION CROSS INDEX (Typical)

Subject Calculation No.: W320-27-014

Page ii of 11

Subject Calculation Revision No.	Superceded by Calculation No.	These interfacing calculation/documents provide input to the subject calculation, and if revised may require revision of the subject calculation.		Results and conclusions of the subject calculation are used in these interfacing calculations and/or documents		Does the output interface calculation/documents require revision?		Has the output interface calculation/documents been revised?		Discipline manager's signature and date indicating evaluation complete.
		Calculation/Document No.	Revision No.	Calculation/Document No.	Revision No.	Yes	No	Yes	No	
3	NA	H-2-818532, Sheet 1	Ø 1				x		x	
		H-2-818532, Sheet 2	Ø 1							
		H-2-818548, Sheet 1	Ø 1	▲ 3						
		H-2-818548, Sheet 2	Ø 1	U 3.13.98						
		Calc # W320-27-013	3			x		x		
		Calc # W320-27-014	3						x	

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DESIGN ANALYSIS

Calc No. W-320-27-015

Revision No. 23

Page No. 1 of 29

Client WESTINGHOUSE HANFORD COMPANY
Subject SUPPORT STRUCTURAL ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319
Date 9 / 13 / 96
Checked 10 / 3 / 96
Revised 11 / 24 / 97
Filename SSA-TEXT. WP
By D.L.STONE
By N.J.JUPLO
By K. Hayase

CHKD: [Signature]
3.8.93

OBJECTIVE:

The objective of this calculation is to perform the structural analysis of the Pipe Supports designed for Slurry & Supernate transfer pipe lines in order to meet the requirements of Code ASME Code B31.3.

DESIGN INPUTS:

Design Criteria:

1. ASME Code B31.3-93 with B31.3a-93 Addenda.
2. Uniform Building Code, 1991 Edition.
3. Functional Design Criteria WHC-SD-W320-FDC-001, Rev. 3.
4

DESIGN METHODOLOGY:

Support Design Loads:

The pipe support design loads are obtained from the piping stress calculations W320-27-14 (Ref.7) and W320-27-15 (Ref.8). These loads are the total summation of the gravity, pressure, thermal and seismic loads.

Since standard typical designs are used for each type of pipe support such as Y-Stop, Guide and Anchors, each type of support is evaluated for the maximum loads to which this type of supports are subjected. These loads are obtained from the AutoPipe analysis (Ref.7) and used to check the structural adequacy of these supports. These loads are tabulated on Page 3.

Since the intermediate anchor is designed to serve as an anchor for both primary pipe and encasement pipe, it is evaluated for the combined effect of the loads from the primary pipe (AutoPipe analysis) and encasement pipe loads. The only Encasement pipe load this anchor will be subjected to is a thermal axial load. This load is calculated on Page 3. In the event of earth quake, since anchor structure will move in tandem with ground motion, it will not experience any seismic loads.

DESIGN ANALYSIS

Calc No. W-320-27-015

Revision No. 23

Page No. 2 of 29

Client WESTINGHOUSE HANFORD COMPANY
Subject SUPPORT STRUCTURAL ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319
Date 9 / 13 / 96
Checked 10 / 3 / 96
Revised 11 / 24 / 97
Filename SSA-TEXT. WP
By D.L.STONE
By N.J.JUPLO
By *K. Hargreave*

CHKD: *[Signature]*
3.3.98

STRUCTURAL ANALYSIS:

The structural analysis (member stresses & weld stresses) is performed using the principles of statics as illustrated in "Design of Welded structures" by Blodgett (Ref.5) & "Formulas for Stress and Strain" by Roark (Ref.6).

The local stresses in the pipe wall due to welded attachments such as trunnions are computed based on Bijlaard principles promulgated in WRC Bulletin 107 (Ref.10). The "POP" computer program (Ref.11), a program based on WRC Bulletin, is used for computing the local stresses for Trunnion attachment. The calculated pipe local stresses are added to corresponding pipe stresses obtained from the AutoPipe analysis in order to satisfy the requirements of Code B31.3.

The base plates are analyzed using the moment of inertia method. The concrete anchors are qualified based on criteria in SDC 4.2, Rev-0, (Ref.14).

CALCULATIONS:

See pages 3 thru 16 for calculations.
See Appendix-A for "POP" program results.

REFERENCES:

1. ASME B31.3-93 Code and ASME B31.3.3a-93 Addenda for "Chemical and Petroleum Refinery".
2. Uniform Building Code, 1991 Edition.
3. Crane Technical Paper No: 410, 22nd Printing-1985.
4. AISC "Steel Construction Manual" 9th Edition.
5. "Design of Welded Structures" by Blodgett, 8th Printing.
6. "Formulas for Stress and Strain" by Roark & Young, 5th Edition.
7. Calculation: W320-27-14, ~~Rev-0~~ "Encasement Pipe Stress Analysis". Rev. 3
8. Calculation: W320-27-13, ~~Rev-0~~ "Process Pipe Stress Analysis". Rev. 3
9. Pipe Support Detail Drawings: H-2-818548 sh 1 & 2 of 2, Rev-0
10. WRC Bulletin no: 107, March 1979 "Local Stresses in Spherical and Cylindrical Shells due to External Loadings" by K.R. Wichman, A.G.Hopper and J.L. Mershon.
11. Computer program "POP" (PIPE ON PIPE) Version-0, Release Date: Feb 12, 1993.
12. W-930-Project Specs: "WHC-SD-W320-FDC-001, Rev-2" 4
13. "Hand Book of Engineering Fundamentals" Third Edition by O. W. Eshback and M. Souders.
14. SDC 4.2, Rev-0, "Design & Installation of Expansion Anchors"
(Now FOUW Practice # 154.215.1206, Design of Expansion Anchors)

3
3
3

CONCLUSION:

The review of succeeding calculations indicates, pipe supports structures meet ASME Code B31.3 requirements.

Rev. 3 of this calculation verified the analysis against the as-built conditions and found the changes did not have any adverse impact on analysis.

3

ICF KAISER HANFORD COMPANY

Calc. No. W-320-27-015

Revision No. 2

Page No. 3 of 29

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD CO.
Subject PROJECT W-320 WASTE RETRIEVAL
for Tank 241-C-106 Support Structural Analysis
Location C TANK FARM 200 EAST AREA

WO/Job No. ER4319
Date 9/13/96
Checked 10/03/96
Revised

Filename FORCES.XLS
By D. L. STONE
By A. J. JUPLD
By

SLURRY LINE

From Autopipe Analysis for Inner Process Pipe

(Ref. 8)

(Ref. W320-27-013 Appdx. A)		FORCES (lb)			MOMENTS (ft-lb)		
Anchor	Node	x	y	z	x	y	z
	A39	508	259	688	88	273	362
Guide	Node	x	y	z	x	y	z
	A01	0	514	1218	/	/	/
Incline	Node	x	y	z	x	y	z
	A74		283		/	/	/

SUPERNATE LINE

(Ref. W320-27-013 Appdx. B)		FORCES (lb)			MOMENTS (ft-lb)		
Anchor	Node	x	y	z	x	y	z
	A39	422	259	612	88	227	362
Guide	Node	x	y	z	x	y	z
	A41	670	532	670			
Incline	Node	x	y	z	x	y	z
	A25		286		/	/	/

ENCASEMENT PIPE AXIAL LOADS FOR INTERMEDIATE ANCHOR

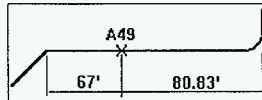
(Ref. 7)

(Ref. W320-27-013 Appdx. C)		FORCES (lb)			MOMENTS (ft-lb)		
Thermal Loads	Node	x	y	z	x	y	z
	A49			3844 *			

* Note: Where Encasement Pipe segments on either side of anchors are equal, axial friction forces at the anchor are equal in magnitude and opposite in direction, resulting in zero axial load on the anchors. The maximum difference between pipe segment lengths on either side of an anchor occurs at Node A49.

Axial load at A49:

$$\Delta L = 13.82 \quad F = 23.18 \quad F_2 = F \times \Delta L = 3844.1712$$



(Ref. 7)

DESIGN ANALYSIS

Client: WESTINGHOUSE HANFORD COMPANY
 Subject: PROJECT W-320 WASTE RETRIEVAL
 PIPE SUPPORT STRUCTURAL ANALYSIS
 Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
 Date: 9 / 13 / 96
 Checked: 10/03/96
 Revised:

Filename: SA-01.mcd
 By: D.L. STONE
 By: J. J. JUPLO

DEAD WEIGHT SUPPORT ANALYSIS (Details 2 and 4 Ref.9)

The purpose of this calculation is to determine a minimum plate thickness required for the Process Pipe Dead Weight Support.

Static bending analysis to determine maximum allowable stress for a minimum plate thickness.

Worst case loadings determined from *Autopi* analysis. (See Page 3)

ASTM A36 Plate Material (Ref. 9)

$S_h = 16900$ -psi Code B31.3 Allowable Stress at 200 deg F (Ref. 1)

$F_y = 286$ -lbf Worst case loading for Process Pipe Dead Weight Support.

$b = 3.3125$ -in Design width of support plate. (Ref. 9)
 (Simple support beam span - conservative)

$d = 3$ -in Plate Length

$M = F_y \frac{b}{4}$ Moment at F_y (Ref. 5)

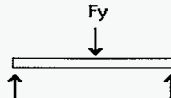
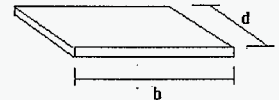
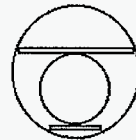
$M = 236.844$ ·lbf·in

$t = 0.25$ -in Design thickness of support plate.

$c := \left(\frac{t}{2}\right)$
 $c = 0.125$ -in Distance between centroid and outerfiber. (Ref. 5)

$I := \left(\frac{d \cdot t^3}{12}\right)$
 $I = 0.004$ ·in⁴ Moment of inertia of the plate

$\sigma_{BpL} := \frac{M \cdot c}{I}$
 $\sigma_{BpL} = 7.579 \cdot 10^3$ ·psi Calculated bending stress at F_y



DESIGN ANALYSIS

Client: **WESTINGHOUSE HANFORD COMPANY**
 Subject: **PROJECT W-320 WASTE RETRIEVAL**
PIPE SUPPORT STRUCTURAL ANALYSIS
 Location: **C TANK FARM - 200 EAST AREA HANFORD**

WO/Job No.: **ER4319**
 Date: **9 / 13 / 96**
 Checked: **10/03/96**
 Revised:
 By: **D.L. STONE**
D. J. JUPLO
 By:

$$\sigma_{all} := 1.33 \cdot S_h$$

(Ref. 1)

$$\sigma_{all} = 2.248 \cdot 10^4 \text{ psi}$$

Allowable stress for ASTM A-36 Plate

$$\left[\frac{M \left(\frac{t}{2} \right)}{d \cdot t_{min}^3} \right] \longrightarrow \frac{6 \cdot M}{d \cdot t_{min}^2}$$

Governing Equation

(Ref. 5)

$$t_{min} := \sqrt{\frac{6 \cdot M}{d \cdot \sigma_{all}}}$$

Minimum required plate thickness.

$$t_{min} = 0.145 \text{ in}$$

Design Thickness

(Ref. 9)

$$t = 0.25 \text{ in}$$

1/4" Plate is adequate for worst case loadings.

Client: WESTINGHOUSE HANFORD COMPANY
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PIPE SUPPORT STRUCTURAL ANALYSIS
Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
Date: 9/13/96
Checked: 10/03/96
Revised:

Filename: SA-03A.mcd
By: D.L. STONE
By: J.J. JUPLO
By:

Process Pipe Guide Lug Design

(Detail 3 Ref. 9)

Assumptions: Lugs see Process Pipe Loads, only.

Each lug is subjected to total loading (conservative.)
Critical Process Pipe Loads located at Nodepoint A41 on
Autopipe Model TRAN-SP are determined as critical.

(Ref.8)

Loads (See Page 3)

$F_x := 0 \cdot \text{lbF}$ Longitudinal Force

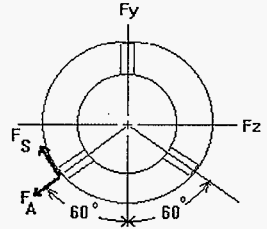
$F_y := 532 \cdot \text{lbF}$ Vertical Force

$F_z := \sqrt{670^2 + 670^2} \cdot \text{lbF}$ $F_z = 947.523 \cdot \text{lbF}$ Horizontal Force

$F_L := \frac{F_x}{3}$ $F_L = 0 \cdot \text{lbF}$ Longitudinal Loading

$F_A := F_y \cdot \cos(60 \cdot \text{deg}) + F_z \cdot \sin(60 \cdot \text{deg})$ $F_A = 1.087 \cdot 10^3 \cdot \text{lbF}$ Axial Loading

$F_S := F_y \cdot \sin(60 \cdot \text{deg}) + F_z \cdot \cos(60 \cdot \text{deg})$ $F_S = 934.487 \cdot \text{lbF}$ Shear Loading



Properties

Material: ASTM A240 TP 304L (Ref. 9)

$S_h := 16700 \cdot \text{psi}$ Allowable Stress at 200 deg F (Ref.1)

$L := 4 \cdot \text{in}$ Lug Length

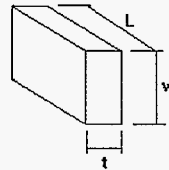
$w := 0.72 \cdot \text{in}$ Lug Width

$t := 0.375 \cdot \text{in}$ Lug thickness

$c_y := \frac{t}{2}$

$c_z := \frac{L}{2}$ (Ref. 5)

$A := L \cdot t$ $A = 1.5 \cdot \text{in}^2$ Lug Area



DESIGN ANALYSIS

Calc. No.: **W-320-27-015**
Revision No.: **2**
Page No.: **8** of **29**

Client: **WESTINGHOUSE HANFORD COMPANY**
Subject: **PROJECT W-320 WASTE RETRIEVAL**
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: **C TANK FARM - 200 EAST AREA HANFORD**

WO/Job No.: **ER4319**
Date: **9 / 13 / 96**
Checked: **10 / 03 / 96**
Revised:

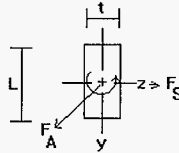
Filename: **SA-03A.mcd**
By: **D.L. STONE**
J.J. JOPLD
By:

$I_y := \frac{1}{12} \cdot t^3 \cdot L$ $I_y = 0.018 \cdot \text{in}^4$ Moment of Area about y (Ref. 5)

$I_z := \frac{1}{12} \cdot t \cdot L^3$ $I_z = 2 \cdot \text{in}^4$ Moment of Area about z

$z_y := \frac{I_y}{c_y}$ $z_y = 0.094 \cdot \text{in}^3$ Section Modulus

$z_z := \frac{I_z}{c_z}$ $z_z = 1 \cdot \text{in}^3$ Section Modulus



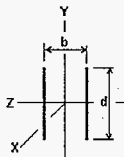
Tension and Bending

$\sigma_n := \frac{F_A}{A} + \frac{F_S \cdot w}{(z_y)}$ $\sigma_n = 7.901 \cdot 10^3 \cdot \text{psi}$

$S_{all} := 1.33 \cdot S_h$ $S_{all} = 2.221 \cdot 10^4 \cdot \text{psi}$

$|\sigma_n| = 7.901 \cdot 10^3 \cdot \text{psi} < S_{all} = 2.221 \cdot 10^4 \cdot \text{psi}$ (Ref. 1)

WELD CALCULATIONS FOR JOINT BETWEEN LUGS AND PROCESS PIPE:



WELD PROPERTIES: (Ref. 5)

$b := 0.375 \cdot \text{in}$ $d = 4 \cdot \text{in}$

$C_y := \frac{b}{2}$ $C_y = 0.187 \cdot \text{in}$

$C_z := \frac{d}{2}$ $C_z = 2 \cdot \text{in}$

$A_w := 2 \cdot d$ $A_w = 8 \cdot \text{in}$

$Sw_y := b \cdot d$ $Sw_y = 1.5 \cdot \text{in}^2$

$Sw_z := \frac{d^2}{3}$ $Sw_z = 5.333 \cdot \text{in}^2$

$J_w := \frac{d \cdot (3 \cdot b^2 + d^2)}{6}$ $J_w = 10.948 \cdot \text{in}^3$

DESIGN ANALYSIS

Client: **WESTINGHOUSE HANFORD COMPANY**
Subject: **PROJECT W-320 WASTE RETRIEVAL**
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: **C TANK FARM - 200 EAST AREA HANFORD**

WO/Job No.: **ER4319** Filename: **SA-03A.mcd**
Date: **9 / 13 / 96** By: **D.L. STONE**
Checked: **10/02/96** By: **N.J. JURELO**
Revised: By:

JOINT LOADS:

Tensile	$F_x := F_A$	$F_y := 0\text{-lbf}$	$F_z := 0\text{-lbf}$
Moment	$M_x := 0\text{-in-lbf}$	$M_y := F_S \cdot w$	$M_z := 0\text{-in-lbf}$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5} \quad f_w = 584.376 \frac{\text{lbf}}{\text{in}}$$

FILLET WELD SIZE REQUIRED:

$S_h := 16700\text{-psi}$

$S := 1.2 \cdot 0.75 \cdot S_h \quad S = 1.503 \cdot 10^4 \text{-psi} \quad \text{Allowable Weld Stress per ASME Code B31.3} \quad (\text{Ref. 1})$

$w := \frac{f_w}{0.707 \cdot S} \quad w = 0.055 \text{-in} \quad \text{Minimum Fillet Weld Required}$

$w_{\text{Design}} := \left(\frac{3}{16} \text{-in} \right) \quad \text{Design Fillet Weld Size}$

$SF_{f_w} := \frac{w_{\text{Design}}}{w} \quad SF_{f_w} = 3.409 \quad \text{Design exceeds requirement by a factor of } 3.409 - \underline{OK}$

PIPE LOCAL STRESSES DUE TO WELDED LUGS:

As is evident from above, since weld stresses are very low, local stresses in pipe will be insignificant.

Client: WESTINGHOUSE HANFORD COMPANY
Subject: PROJECT W-320 WASTE RETRIEVAL
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
Date: 9 / 13 / 96
Checked: 10 / 03 / 96
Revised:
Filename: SA-07mcd
By: D.L. STONE
By: J. J. JUPLO

INTERMEDIATE ANCHOR: (Detail 1 Ref. 9)

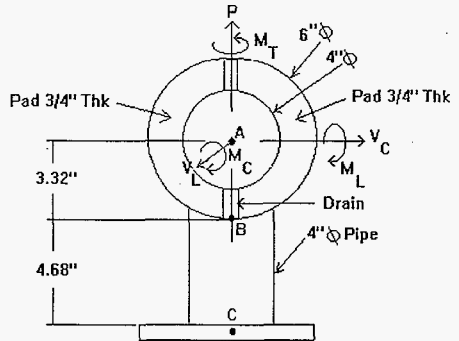
AUTOPIPE Loads for Process Pipe (See Page 3)

$F_x := 508 \cdot \text{lbf}$ $M_x := 88 \cdot \text{ft} \cdot \text{lbf}$
 $F_y := 259 \cdot \text{lbf}$ $M_y := 273 \cdot \text{ft} \cdot \text{lbf}$
 $F_z := 688 \cdot \text{lbf}$ $M_z := 362 \cdot \text{ft} \cdot \text{lbf}$

AUTOPIPE global axes oriented at 45 degree angle to pipe local axes.

PIPE LOADS at A

$P := F_y$	$P = 259 \cdot \text{lbf}$	Radial Load
$V_C := F_z \cdot \cos(45 \cdot \text{deg}) + F_x \cdot \cos(45 \cdot \text{deg})$	$V_C = 845.7 \cdot \text{lbf}$	Circumferential Shear
$V_L := F_x \cdot \sin(45 \cdot \text{deg}) + F_z \cdot \sin(45 \cdot \text{deg})$	$V_L = 845.7 \cdot \text{lbf}$	Longitudinal Shear
$M_T := M_y$	$M_T = 3.276 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	Torsional Moment
$M_C := M_x \cdot \cos(45 \cdot \text{deg}) + M_z \cdot \cos(45 \cdot \text{deg})$	$M_C = 3.818 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	Circumferential Moment
$M_L := M_x \cdot \sin(45 \cdot \text{deg}) + M_z \cdot \sin(45 \cdot \text{deg})$	$M_L = 3.818 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	Longitudinal Moment
$F_{AL} := 3844 \cdot \text{lbf}$		Encased pipe Axial Load (See page 3)



PIPE LOADS at B (Checking pipe local stresses and weld between pad and pipes.)

$P_B := P$	$P_B = 259 \cdot \text{lbf}$
$V_{CB} := V_C$	$V_{CB} = 845.7 \cdot \text{lbf}$
$V_{LB} := V_L + F_{AL}$	$V_{LB} = 4.69 \cdot 10^3 \cdot \text{lbf}$
$M_{TB} := M_T$	$M_{TB} = 3.276 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$
$M_{CB} := M_C + (3.32 \cdot \text{in}) \cdot V_C$	$M_{CB} = 6.626 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$
$M_{LB} := M_L + (3.32 \cdot \text{in}) \cdot V_L$	$M_{LB} = 1.159 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$

DESIGN ANALYSIS

Client: WESTINGHOUSE HANFORD COMPANY
 Subject: PROJECT W-320 WASTE RETRIEVAL
 PIPE SUPPORT STRUCTURAL ANALYSIS
 Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
 Date: 9 / 13 / 96
 Checked: 10 / 03 / 96
 Revised:

Filename: SA-07mcd
 By: D.L. STONE
 By: J. J. JOFFLO
 By:

PIPE LOADS at C (Checking base plate and anchor bolts.)

$P_C := P$

$P_C = 259 \cdot \text{lbf}$

$V_{CC} := V_C$

$V_{CC} = 845.7 \cdot \text{lbf}$

$V_{LC} := V_L + F_{AL}$

$V_{LC} = 4.69 \cdot 10^3 \cdot \text{lbf}$

$M_{TC} := M_T$

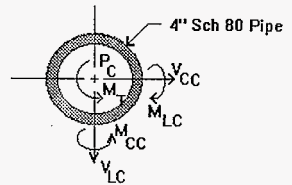
$M_{TC} = 3.276 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$

$M_{CC} := M_C + (8 \cdot \text{in}) \cdot V_C$

$M_{CC} = 1.058 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$

$M_{LC} := M_L + (8 \cdot \text{in}) \cdot V_{LC}$

$M_{LC} = 4.134 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$



TRUNION STRESSES:

Material:

4" Schedule 80 A 106 GR-B Pipe

$A := 4.407 \cdot \text{in}^2$

Area of metal of 4" pipe.

$z := 4.217 \cdot \text{in}^3$

Section Modulus

$\sigma_n := \frac{P_C}{A} + \frac{M_{CC} + M_{LC}}{z} \quad \sigma_n = 1.237 \cdot 10^4 \cdot \text{psi}$

Tension and Bending

$S_h := 20000 \cdot \text{psi}$

$S_{all} := 1.33 \cdot S_h$

Allowable Stress

$\sigma_n = 1.237 \cdot 10^4 \cdot \text{psi} < S_{all} = 2.66 \cdot 10^4 \cdot \text{psi}$

Shear stresses are insignificant.

DESIGN ANALYSIS

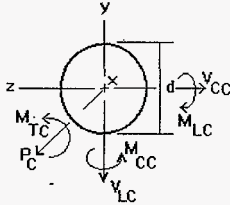
Client: WESTINGHOUSE HANFORD COMPANY
Subject: PROJECT W-320 WASTE RETRIEVAL
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
Date: 9 / 13 / 96
Checked: 10/03/96
Revised:

Filename: SA-07.mcd
By: D.L. STONE
By: A.J. JUPLO
By:

WELD CALCULATIONS FOR TRUNION TO BASEPLATE JOINT:

WELD PROPERTIES:



$d := 4.5\text{-in}$	(Ref. 5)
$C_y := \frac{d}{2}$	$C_y = 2.25\text{-in}$
$C_z := \frac{d}{2}$	$C_z = 2.25\text{-in}$
$A_w := \pi \cdot d$	$A_w = 14.137\text{-in}$
$S_{wy} := \frac{\pi \cdot d^2}{4}$	$S_{wy} = 15.904\text{-in}^2$
$S_{wz} := S_{wy}$	$S_{wz} = 15.904\text{-in}^2$
$J_w := \frac{\pi \cdot d^3}{4}$	$J_w = 71.569\text{-in}^3$

JOINT LOADS at C:

$F_x := P_C$	$F_y := V_{LC}$	$F_z := V_{CC}$
$F_x = 259\text{-lbf}$	$F_y = 4.69 \cdot 10^3\text{-lbf}$	$F_z = 845.7\text{-lbf}$
$M_x := M_{TC}$	$M_y := M_{CC}$	$M_z := M_{LC}$
$M_x = 3.276 \cdot 10^3\text{-in-lbf}$	$M_y = 1.058 \cdot 10^4\text{-in-lbf}$	$M_z = 4.134 \cdot 10^4\text{-in-lbf}$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5} \quad f_w = 3.315 \cdot 10^3 \frac{\text{lbf}}{\text{in}} \quad (\text{Ref. 5})$$

FILLET WELD SIZE REQUIRED:

$S_h := 17800\text{-psi}$ ALLOWABLE WELD STRESS PER CODE B31.3 for A36 Plate

$S := 1.2 \cdot 0.75 \cdot S_h \quad S = 1.602 \cdot 10^4\text{-psi}$ (Ref. 1)

$w := \frac{f_w}{0.707 \cdot S} \quad w = 0.293\text{-in}$ Minimum Fillet Weld Required

$w_{\text{Design}} := \left(\frac{S}{16} \right)\text{-in}$ Design Fillet Weld Size

$SF_{f_w} := \frac{w_{\text{Design}}}{w} \quad SF_{f_w} = 1.068$ HNF-2474, Rev. 0, Page 4 of 15
Design exceeds requirement by a factor of 1.068 - **OK**

DESIGN ANALYSIS

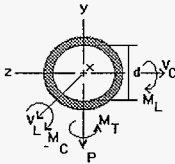
Client: WESTINGHOUSE HANFORD COMPANY
Subject: PROJECT W-320 WASTE RETRIEVAL
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
Date: 9 / 13 / 96
Checked: 10 / 03 / 96
Revised:

Filename: SA-07.mcd
By: D.L. STONE
By: J. J. JUREL
By:

WELD CALCULATIONS FOR JOINT BETWEEN PAD AND PIPES:

Note: 1" gaps on top and bottom have been omitted to allow room to weld drain and vent . These gaps are insignificant compared to the total weld length and are, therefore, not considered in the analysis.



WELD PROPERTIES:

$d := 4.5 \cdot \text{in}$	(Ref. 5)
$C_y := \frac{d}{2}$	$C_y = 2.25 \cdot \text{in}$
$C_z := \frac{d}{2}$	$C_z = 2.25 \cdot \text{in}$
$A_w := 2 \cdot (\pi \cdot d)$	$A_w = 28.274 \cdot \text{in}$
$S_{wy} := 2 \cdot \left(\frac{\pi \cdot d^2}{4} \right)$	$S_{wy} = 31.809 \cdot \text{in}^2$
$S_{wz} := S_{wy}$	$S_{wz} = 31.809 \cdot \text{in}^2$
$J_w := 2 \cdot \left(\frac{\pi \cdot d^3}{4} \right)$	$J_w = 143.139 \cdot \text{in}^3$

JOINT LOADS at B:

$F_x := V_{LB}$	$F_y := P_B$	$F_z := V_{CB}$
$F_x = 4.69 \cdot 10^3 \cdot \text{lbf}$	$F_y = 259 \cdot \text{lbf}$	$F_z = 845.7 \cdot \text{lbf}$
$M_x := M_{CB}$	$M_y := M_{TB}$	$M_z := M_{LB}$
$M_x = 6.626 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	$M_y = 3.276 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	$M_z = 1.939 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5} \quad f_w = 895.75 \cdot \frac{\text{lbf}}{\text{in}} \quad (\text{Ref. 5})$$

FILLET WELD SIZE REQUIRED:

$S_h = 16700 \cdot \text{psi}$	ALLOWABLE WELD STRESS PER CODE B31.3	(Ref. 1)
	for A312 TP306 Pipe	
$S = 1.2 \cdot 0.75 \cdot S_h$	$S = 1.503 \cdot 10^4 \cdot \text{psi}$	(Ref. 1)

$w := \frac{f_w}{0.707 \cdot S} \quad w = 0.084 \cdot \text{in}$ **Minimum Fillet Weld Required**

$w_{\text{Design}} := \left(\frac{3}{16} \right) \cdot \text{in}$ **Design Fillet Weld Size**

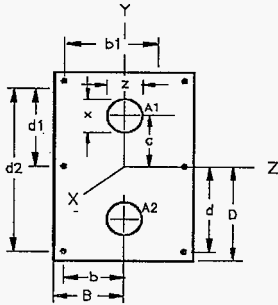
$SF_{f_w} := \frac{w_{\text{Design}}}{w} \quad SF_{f_w} = 2.224$ **HNF-2474, Rev. 0, Page A-16**
Design exceeds requirement by a factor of 2.224 - OK

Client: WESTINGHOUSE HANFORD COMPANY
Subject: PROJECT W-320 WASTE RETRIEVAL
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
Date: 9/13/96
Checked: 10/03/96
Revised:

Filename: SA-07.mcd
By: D.L. STONE
By: J.J. JUPLO
By:

BASE PLATE ANALYSIS:



DIMENSIONS:

$b := 7.75 \text{ in}$	$B := 9 \text{ in}$	
$d := 11.75 \text{ in}$	$D := 13 \text{ in}$	
$c := 7 \text{ in}$	$t := 1 \text{ in}$	Plate Thickness
$x := 4.5 \text{ in}$	$z := 4.5 \text{ in}$	Attachment Diam. (circular)
$b1 := b + \frac{z}{2} + 2t$	$b1 := 12 \text{ in}$	
$d1 := c + \frac{x}{2} + 2t$	$d1 := 11.25 \text{ in}$	
$d2 := d + d1$	$d2 := 23 \text{ in}$	

LOADS @ ATTACHMENT "A1": (See Page 11)

$Fx1 := P_C$	$Fy1 := V_{CC}$	$Fz1 := V_{LC}$
$Fx1 = 259 \cdot \text{lbf}$	$Fy1 = 845.7 \cdot \text{lbf}$	$Fz1 = 4.69 \cdot 10^3 \cdot \text{lbf}$
$Mx1 := M_{TC}$	$My1 := M_{LC}$	$Mz1 := M_{CC}$
$Mx1 = 3.276 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	$My1 = 4.134 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$	$Mz1 = 1.058 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$

LOADS @ ATTACHMENT "A2":

$Fx2 := P_C$	$Fy2 := V_{CC}$	$Fz2 := V_{LC}$
$Fx2 = 259 \cdot \text{lbf}$	$Fy2 = 845.7 \cdot \text{lbf}$	$Fz2 = 4.69 \cdot 10^3 \cdot \text{lbf}$
$Mx2 := M_{TC}$	$My2 := M_{LC}$	$Mz2 := M_{CC}$
$Mx2 = 3.276 \cdot 10^3 \cdot \text{in} \cdot \text{lbf}$	$My2 = 4.134 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$	$Mz2 = 1.058 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$

LOADS @ CENTROID OF BOLT PATTERN:

$Fx = Fx1 + Fx2$	$Fx = 518 \cdot \text{lbf}$
$Fy = Fy1 + Fy2$	$Fy = 1.691 \cdot 10^3 \cdot \text{lbf}$
$Fz = Fz1 + Fz2$	$Fz = 9.379 \cdot 10^3 \cdot \text{lbf}$
$Mx := Mx1 + Mx2 + c \cdot (Fz1 + Fz2)$	$Mx = 7.221 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$
$My := My1 + My2$	$My = 8.267 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$
$Mz := Mz1 + Mz2 + c \cdot (Fx1 + Fx2)$	$Mz = 4.134 \cdot 10^4 \cdot \text{in} \cdot \text{lbf}$

Client: WESTINGHOUSE HANFORD COMPANY
Subject: PROJECT W-320 WASTE RETRIEVAL
PIPE SUPPORT STRUCTURAL ANALYSIS
Location: C TANK FARM - 200 EAST AREA HANFORD

WO/Job No.: ER4319
Date: 9 / 13 / 96
Checked: LD / 03 / 96
Revised:
Filename: SA-07.mcd
By: D.L. STONE
By: J.J. SUPLO
By:

BOLT TENSION & SHEAR:

$$T := \frac{F_x}{6} + \frac{M_y}{3 \cdot b1} + \frac{M_z}{2 \cdot d1 + 2 \cdot d2}$$

$$T = 2.745 \cdot 10^3 \cdot \text{lbF}$$

$$S := \left[\left[\frac{F_y}{6} + \frac{M_x}{4 \cdot (b^2 + d^2)^{0.5}} \right]^2 + \left[\frac{F_z}{6} + \frac{M_x}{4 \cdot (b^2 + d^2)^{0.5}} \right]^2 \right]^{0.5}$$

$$S = 3.247 \cdot 10^3 \cdot \text{lbF}$$

BOLT INTERACTION (1) - BOLT SIZE & EMBEDMENT:

$$A_s := 0.462$$

Tensile Stress Area for 7/8 Diam. Bolt

(Ref. 4)

$$Sh := 13700$$

Allowable Tensile stress for A307 Bolts at 200° F.

Ref. AISC Pg. 4-3

$$T_a := 1.33 \cdot A_s \cdot Sh$$

$$T_a = 8.418 \cdot 10^3$$

TENSION Allowable

$$S_a := 1.33 \cdot A_s \cdot 0.8 \cdot Sh$$

$$S_a = 6.734 \cdot 10^3$$

SHEAR Allowable

$$I := \frac{T}{T_a} + \frac{S}{S_a}$$

$$I = 0.808 \cdot \text{lbF} < 1.0$$

BOLTS ARE ACCEPTABLE

BASE PLATE STRESS:

$$M_b := 3 \cdot T \cdot \left(b - \frac{z}{2} \right)$$

$$M_b = 4.529 \cdot 10^4 \cdot \text{in} \cdot \text{lbF}$$

Maximum Bending Moment on Plate

(Ref. 5)

$$Z := \frac{2 \cdot B \cdot t^2}{6}$$

$$Z = 3 \cdot \text{in}^3$$

$$\sigma_b := \frac{M_b}{Z}$$

$$\sigma_b = 1.51 \cdot 10^4 \cdot \text{psi}$$

Maximum Bending Stress

$$S_h := 17800 \cdot \text{psi}$$

For A36 Plate

$$\sigma_{\text{all}} := 1.33 \cdot S_h$$

$$\sigma_{\text{all}} = 2.367 \cdot 10^4 \cdot \text{psi}$$

Code B31.3 Allowable

(Ref. 1)

$$\sigma_b = 1.51 \cdot 10^4 \cdot \text{psi}$$

$$< \sigma_{\text{all}} = 2.367 \cdot 10^4 \cdot \text{psi}$$

MAXIMUM STRESS LEVEL ACCEPTABLE

DESIGN ANALYSIS

Client: Numatec

WO/Job No.: ER4319/W-320

Subject: Support Structural Analysis
Project W-320 Waste Retrieval for Tank 241-C-106

Date: 11/24/97 By: K. Hayase
Checked: 3.13.98 By: M. Ahmed

Location: C Tank Farm 200 East Area

Revised: By:

TOTAL PIPE STRESSES

Includes local pipe stresses due to welded trunion. Assumes the same stress state in the process and encasement pipe due to the trunion.

Process Pipe: (ASTM A312 TP304L) $S_h = S_c = 16700 \text{ psi}$

Max Sustained Stress (Autopipe, Ref. 8) 8128 psi

Appdx - A page 103

Local Stress (Sus. + Ther. + Occ.) 7813 psi

(From 'POP', Pg. 6, Appdx A) Total 15941 psi < 16700 psi allowable

Max Displacement Stress (Autopipe, Ref. 8) 16758 psi

Local Stress

7813 psi

Total 24571 psi < 25050 psi allowable

$$= [1.25 S_h + 0.25 S_c]$$

Max Occasional Stress (Autopipe, Ref. 8) 13427 psi

Appdx - A page 104

Local Stress 7813 psi

Total 21240 psi < 22211 psi allowable = (1.33 S_h)

Note: The allowable stress exceeds two-thirds of yield strength at temperature (Ref. B31.3 Para 302.3.6(a) and should be reduced slightly. However, the actual stresses at the intermediate anchor points are significantly less than the max occasional stress used here, therefore this is acceptable.

Encasement Pipe: (ASTM A53 Type - E GR-B) $S_h = 20000 \text{ psi}$

Max Sustained Stress (STAB, Ref. 7) 1922 psi

Local Stress

7813 psi

Total 9735 psi < 20000 psi allowable (S_h)

Max Displacement Stress (STAB, Ref. 7) 23800 psi

(Page - 10 of the Calc)

Local Stress 7813 psi

Total 31613 psi < 40265 psi allowable

Note: allowable = 1.25(20000 psi + 20000 psi) - 9735 psi = 40265 psi (Ref. B31.3)

Max Occasional Stress (STAB, Ref. 7) 5904 psi

Local Stress

7813 psi

Total 13717 psi < 26600 psi allowable (1.33 S_h)

DESIGN ANALYSIS

Client	WHC	WO/Job No.	ES432 / W-320
Subject	Transfer Line - Concrete Base	Date	5/20/94
Design		By	[Signature]
Location	ZOG	Checked	5/20/94
		Revised	
		By	

Determine the Resteel Required in the Concrete Base for the Transfer Line Pipe Supports

1. Determine if size of footing is adequate to resist movement from forces imposed by the transfer lines.

$$F_x = 2(905 \text{ lb}) = 1810 \text{ lbs} \\ = 1.8 \text{ kips}$$

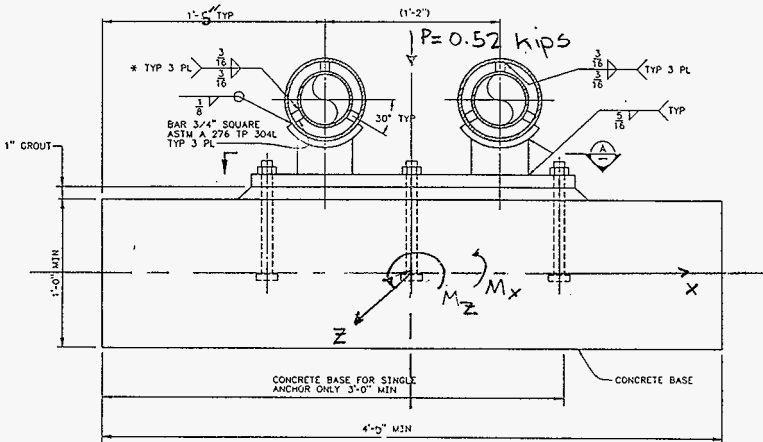
$$M_x = 2(79850) = 159700 \text{ in}\cdot\text{lb} \\ = 159.7 \text{ in}\cdot\text{kips}$$

$$F_y = 2(259 \text{ lb}) = 518 \text{ lbs} \\ = 0.52 \text{ kips}$$

$$M_y = 2(6816) + 2(4752 \text{ lb})(7 \text{ in}) \\ = 80160 \text{ in}\cdot\text{lb} \\ = 80.2 \text{ in}\cdot\text{kips}$$

$$F_z = 2(4752) = 9504 \text{ lbs} \\ = 9.5 \text{ kips}$$

$$M_z = 2(18298) = 36596 \text{ in}\cdot\text{lb} \\ = 36.6 \text{ in}\cdot\text{kip}$$



a) $M_z = M_o = 36.6 + 0.57(7 \text{ in}) = 40.2 \text{ in}\cdot\text{kip}$ (Overturning Moment)

Determine if eccentricity is greater than kern ($\frac{w}{6}$).

$w = \text{width of Footing} = 4.17 \text{ ft.}$

DESIGN ANALYSIS

Calc. No. 1/220-1
Revision 0
Page No. 18 of 22

Client	WHC	WO/Job No.	2007-02-20
Subject		Date	By
		Checked	5/20/12 By
Location	220E	Revised	By

a. continued

wt of Footing = (1 Ft)(3 Ft)(4.2 Ft)(150 pcf) = 1999 lbs = 1.9 kip

P = D + L = 1.9 + 0.52 + 5.7 = 8.12 kip (Ws = wt. of soil = 5.7 kips)

$e = \frac{M}{P} = \frac{40.2 \text{ in} \cdot \text{kip}}{8.12 \text{ kip}} = 5.0 \text{ in}$ $\gamma = 115 \text{ pcf}$, Table 2 of Shannon & Wilson Report KEH-8008-2

$k = \frac{.36 \text{ in}}{6} = 6 \text{ in}$

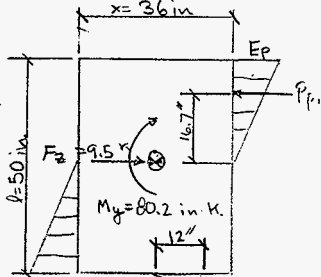
∴ Since $e < k$ (kern) determine $f_{max} = \frac{P}{BL} (1 + \frac{6e}{B})$

P = 8.12 kip e = 5 1/2 Ft
B = 3 Ft $f_A = 4000 \text{ psf}$
L = 4.2 Ft

$f_{max} = \frac{8.12}{(3)(4.2)} (1 + \frac{6(5\frac{1}{2})}{3}) = 1.18 \text{ ksf} < 4 \text{ ksf}$ OK

* $f_A = 4000 \text{ psf}$ from Shannon & Wilson Soil Report contract # KEH-8008-2

b. Check lateral resistance to sliding & rotation.



Allowable Passive earth pressure E_p at min depth
 $E_p = \frac{1}{2} (115 \text{ pcf})(4.5 \text{ Ft})^2 (1/1.5)$
 $E_p = 3105 \text{ lbs/Ft}$

Assume M_y is resisted by the 2 faces of the base.

$P_{p1} = \left[\frac{E_p \cdot 1/2}{2} \right] = \left[\frac{3105(1/2)}{2} \right]$

= 3260 lbs

$P_{p1} = 3.3 \text{ kip}$

Note: E_p is total force from soil acting against the footing at that depth and acts as a triangular distribution from centroid.

Client <u>W/H</u>	WO/Job No. <u>ER4312/W-320</u>
Subject <u>Foundation - Concrete Soil</u>	Date <u>5/27/78</u> By <u>[Signature]</u>
Location <u>[Blank]</u>	Checked <u>5/27/78</u> By <u>[Signature]</u>
	Revised _____ By _____

b. Continued

$M_p \geq M_y$; required to resist torsion.

$$M_p = P_p(16.7)z = 110.2 \text{ in} \cdot \text{kip}$$

$$\therefore M_p > M_y, \underline{OK}$$

Note: For the equation $e_p = \frac{1}{2} \sigma h^2 K_p$, $K_p = 4.0$ from S&W report. S&W recommended S.F. of 1.5.
 E_p is Total force acting along the base. (10/Ft)

c. Check sliding resistance:

$$F_z = 9.5 \text{ kip}$$

$$F = \frac{F_r}{F_z} \geq 1.5, \text{ for cohesionless soils.}$$

where

$$F_r = R \tan \phi + E_p l$$

where ϕ = Friction ϕ ($\phi = 37^\circ$, from S&W Report)
 $R = W_c + W_s = 7.6 \text{ kip}$

$$W_c = 1.9 \text{ kip}$$

$$W_s = 115 \text{ pcf} \times 4.2' \times 3.0' \times 3.92' = 5680 \text{ lbs}$$

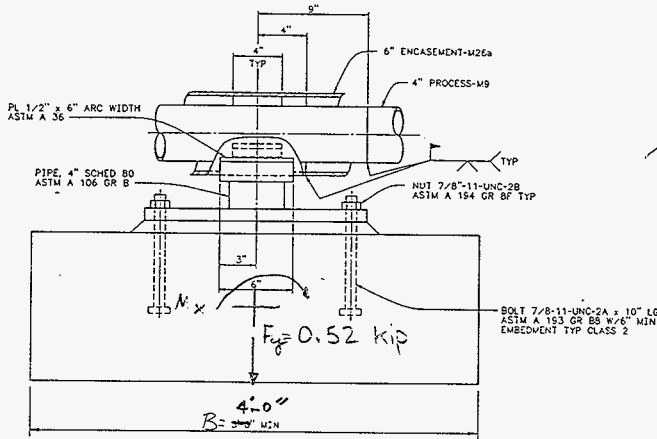
$$F_r = 7.6(\tan 37^\circ) + 3.1(4.2) = 18.7 \text{ kip}$$

$$\frac{F_r}{F_z} = \frac{18.7}{9.5} = 2.0 > 1.5 \quad \therefore \underline{OK}$$

DESIGN ANALYSIS

Client	<u>WHC</u>	WO/Job No.	<u>ER4319/W-320</u>
Subject	<u>Transfer Line - Concrete Base</u>	Date	<u>5/20/19.4</u>
	<u>W. H. H.</u>	Checked	<u>5/20/19.4</u>
Location	<u>2002E</u>	Revised	
		By	<u>W. H. H.</u>

d. Check moment from other view.



$$M_o = M_x = 159.7 \text{ in} \cdot \text{kip}$$

$$P = F_y + W_c + W_b = 8.12 \text{ kips}$$

$$e = \frac{M_o}{P} = \frac{159.7}{8.12 \text{ kip}} = 19.7 \text{ in} \quad k = \frac{50 \text{ in}}{6} = 8.3 \text{ in}$$

Increase B to 3'-6", which will increase P by 315 lb to P = 8.44 kips

$$e = \frac{159.7}{8.44} = 18.9 \text{ in}$$

$$f_{ult} = \frac{2P}{3L(B/2 - e)} \text{ , since } e > k$$

$$f_{ult} = \frac{2(8.44)}{3(4.2)(3.5/2 - 18.9/12)} = 7.7 \text{ ksf} \quad N.G$$

DESIGN ANALYSIS

Client <u>WHC</u>	WO/Job No. <u>FD4319 / W-320</u>
Subject <u>Transfer Line - Concrete Base Design</u>	Date <u>5/24/94</u> By <u>J.P. Smith</u>
	Checked <u>Shu</u> By <u>W. Smith</u>
Location <u>2100 E</u>	Revised _____ By _____

d. Continued:

Try $B = 4.0'$

P will increase by 315 lb to $P = 8.76 \text{ kip}$,

$e = \frac{159.7}{8.76} = 18.23 \text{ in}$

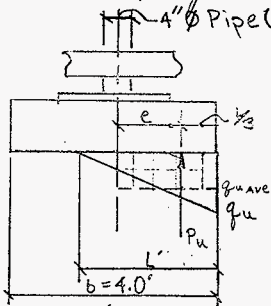
$q_{ult} = \frac{2(8.76)}{3(4.0)(\frac{4}{2} - \frac{18.23}{12})} = 3.0 \text{ ksf} < q_A \text{ OK}$

Note: Stability not a concern since the pipe lines will prevent the footing from overturning.

From a constructibility stand point
make concrete base square (4.0' x 4.0')

e. Determine Re steel Required for Slab.

$L' = (\frac{b}{2} - e) 3$
 $= (\frac{4}{2} - 18.23) 3$
 $= 17.31 \text{ in}$
 $= 1.44 \text{ ft}$



$q_{max} = 3.0 \text{ ksf}$
Use q_A to determine bending for Re steel to take into account the small moment in the other direction.

Ave. the distributed load to be more conserv. and take the moment about the center.

$q_u = 1.6 q_A = 1.6(4000 \text{ plf}) = 6400 \text{ plf}$

Averaging the q_u into $q_{u \text{ ave}}$.

$q_{u \text{ ave}} = \left(\frac{6.4}{1.44} \right) 2 + \left[6.4 - \left(\frac{6.4}{1.44} \right) 2 \right] = 6.4 \text{ plf}$

$M_u = \frac{6.4(2)^2}{2} = 12.8 \text{ ft} \cdot \text{kip}$

Client	<u>W/HC</u>	WO/Job No.	<u>ER4319 / W-320</u>
Subject	<u>Transfer Line - Concrete Base Design</u>	Date	<u>5/24/94</u>
		Checked	<u>SLW</u>
		By	<u>SLW</u>
Location	<u>200E</u>	Revised	
		By	

e. Continued:

- Use design procedure 3 of ACI SP-17 (91) vol. 1, "ACI Design Handbook."

$$d = 12 - 3 - \frac{0.625}{2} = 8.69 \text{ } \phi 11 \approx 11$$

$$F = \frac{bd^2}{12000} = \frac{(12)(9)^2}{12000} = 0.081$$

$$K_n = M_u / F = 12.8 / 0.081 = 158.02$$

$K_n < K_{n,max}$, select K_n based on $\frac{1}{3} f_{max} < \rho^{2/3} f_{max}$

$$\therefore K_n = 370$$

- Determine flexural steel required:

$$A_s = \frac{12.8}{(9)(4.21)} = 0.35 \text{ in}^2/\text{ft}$$

$$A_{s,min} = 0.0018(12)(12) = 0.26 \text{ in}^2/\text{ft} \quad \therefore A_s > A_{s,min}$$

$$\text{Use } A_s = 0.35 \text{ in}^2/\text{ft}$$

- Check Shear: $\phi V_c \geq V_u$ @ d from face of support

$$\phi V_c = \phi (2 \sqrt{f_c'}) b w d = (0.85)(2)(4000) \frac{1}{2} (12)(9) = 11.16 \text{ kip}$$

$$V_u = q_u \left(\frac{b-a}{2} - d \right) = 6.4 \left(\frac{4 - \frac{1}{2}}{2} - \frac{9}{2} \right) = 7 \text{ kip} \quad \frac{1}{2} \text{ ' is for } 4" \phi \text{ Pipe support}$$

$$\phi V_c > V_u \quad \text{OK for shear for one-way shear}$$

Two-Way Shear (Punching): Compute b_o required and compare to b_o (supplied)

$$b_o = 2(26 + 9) + 2(18 + 9) = 124 \text{ in (Supplied)}$$

DESIGN ANALYSIS

Client	W/C	WO/Job No.	ER4319/W-320
Subject	Transfer Line - Concrete Base	Date	6/24/94
	Design	By	AP Smith
Location	208E	Checked	AP Smith
		Revised	By

e) Continued:

$$\frac{K_v V_u}{1.02} = 1.72$$

for $d = 8''$ conserv.
 $f_s = 6.67$ from SP-17, pg-259.

$$a_b = 26.6$$

$$A_{FV} = (4)^2 - \left(\frac{26+9}{12} \times \frac{18+9}{12} \right) = 9.4 \text{ ft}^2$$

$$V_u = A_{FV} \cdot f_s = (9.4 \text{ ft}^2)(6.67 \text{ ksi}) = 62.7 \text{ kips}$$

$$\beta_c = \frac{26}{18} = 1.44 \quad K_v = 1.0 \quad \text{for } \beta_c \leq 2.0$$

$$b_o = \frac{(1)(62.7)}{1.72} = 36.5 \text{ in}$$

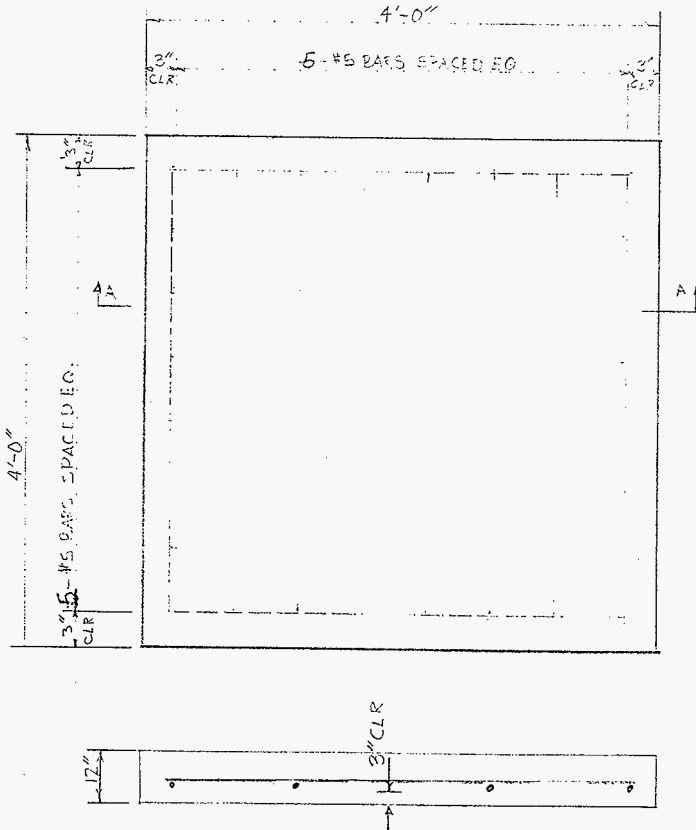
$$b_o < b_o \text{ (supplied)}$$

Bar Selection: Use 5-#5 bars spaced equally
EW w/ 3" c/c on bottom and sides or at 10.5 in
spacing.

DESIGN ANALYSIS

Client <u>WHC</u>	WO/Job No. <u>ER4319 / W-320</u>
Subject <u>Transfer Line - Concrete. Base</u>	Date <u>6/30/94</u> By <u>J.R. Booth</u>
<u>Design</u>	Checked <u>SP/PL</u> By <u>Rev. [Signature]</u>
Location <u>300F</u>	Revised _____ By _____

f. Plan View of 5.0c Ductwork Base



DESIGN ANALYSIS

Client	<u>WHC</u>	WO/Job No.	<u>ER4319/W-320</u>
Subject	<u>Transfer Line - Concrete Base</u>	Date	<u>5/20/94</u> By <u>JR Booth</u>
Location	<u>200 E</u>	Checked	<u>SL</u> By <u>KLW</u>
		Revised	By

g. Check Pull-Out Strength of Anchor Bolts:

Unfactored loads From pg. 15 of calc.

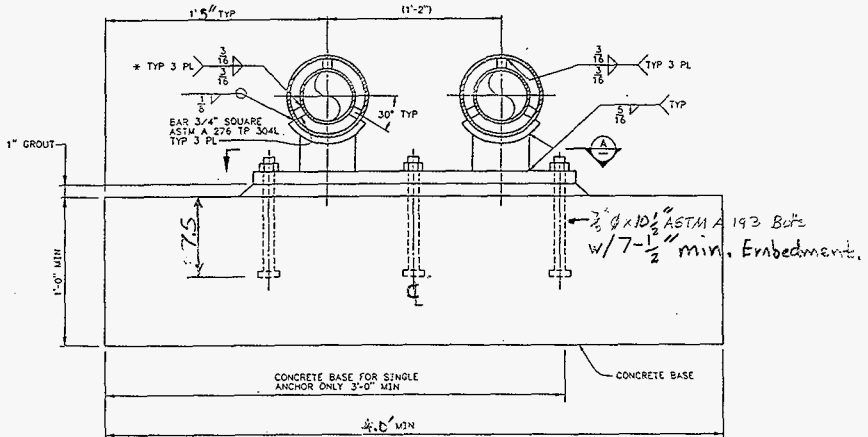
$$T = 2786 \text{ lbs}$$

$$S = 3467 \text{ lbs}$$

Adequacy of bolt has already been determined.

Using method presented in A/E STD. # GC-ANCB-01.

Check embedment depth of anchor bolts



SECTION

$$L_e = 7 \text{ in}$$

$$C = 1.44 \text{ in}$$

$$P_u = \pi (L_e^2 + C L_e) (4 \phi_c \sqrt{f'_c}) = 26431 \text{ lb}$$

$$A_t = 0.462 \text{ in}^2$$

$$\phi_c = 0.65$$

$$f'_c = 3000$$

$$F_u = \frac{26431}{0.462} = 57210 \text{ psi}$$

DESIGN ANALYSIS

Client	<u>WHC</u>	WO/Job No.	<u>ER 4319 / W320</u>
Subject	<u>Transfer Pipe - Concrete Base</u>	Date	<u>5/20/94</u> By <u>JR Booth</u>
	<u>Design</u>	Checked	<u>5/20/94</u> By <u>R.W. Dowd</u>
Location	<u>208E</u>	Revised	By

g. Continued:

$$L_e = \frac{\sqrt{C^2 + \frac{1.33 F_u A_t}{\pi \phi_c \sqrt{F_{ic}}}}}{2} - C$$

$$= \frac{\sqrt{(1.44)^2 + \frac{1.33 (57210)(0.462)}{\pi (0.65) \sqrt{3000}}}}{2} - 1.44 \text{ in} = 8.04 \text{ in}$$

$K_{ns} = 1.0$
 $K_{\phi_c} = 0.86$
 $K_{F_{ic}} = 0.93$

$$L'_e = L_e (K_{ns} K_{\phi_c} K_{F_{ic}}) = (8.04)(1.0)(0.86)(0.93) = 6.43 \text{ in}$$

$L'_e > L_e$ embedment + depth OK

Edge Distance for Tension Load:

$F_u = 105 \text{ ksi}$ (ASME B31.3, Table A-2, 1993 ED.)
 $F_y = 65 \text{ ksi}$

$D = \frac{7}{8} \text{ in}$
 $F_{ult} = 105 \text{ ksi}$
 $F_{ic} = 4000 \text{ psi}$

$$m_t = D \sqrt{\frac{F_{ult}}{56 + F_{ic}}} = \left(\frac{7}{8}\right) \sqrt{\frac{105000}{56 + 4000}} = 4.76 \text{ in}$$

m_t is min. edge distance

(and) Actual edge distance = 11.75 in OK

$$V_D = 223.7 (m_v)^2 = 223.7 (13)^2 = 37805 \text{ lbs}$$

Client	WHC	WO/Job No.	ER4319 / W-320
Subject	Transfer Pipe - Concrete Base	Date	5/20/94 By ATR Bonth
	Design	Checked	By [Signature]
Location	200E	Revised	By

g. Continued:

Edge distance for Shear Load:

$$m_v = D \sqrt{\frac{F_{uit}}{7.5 f'_c}} = \left(\frac{7}{8}\right) \sqrt{\frac{105000}{7.5 \cdot 4000}} = 13 \text{ in}$$

$m_d < m_v$, N.G. Going to FIG. 4

$$V_c = 4 \sqrt{f'_c} A_c / 2 A_m = 46634 \text{ lbs}$$

$$V_o/V_c = 0.8$$

$$m_d/m_v = 0.9, \text{ zone I}$$

OK for shear

Check anchor bolt spacing:

$$\begin{aligned} r_{m'} &= \text{min. anchor bolt spacing} \\ &= 2 l_e + C \\ &= 2(7) + 1.44 = 15.44 \text{ in} > 11.75 \quad \text{NG} \end{aligned}$$

Re-evaluate l_e $p = 11.75"$

Note: Increase in min. embedment depth required because anchor bolt spacing less than min. req. w/ $\frac{7}{8}" \phi$ bolts.

$$\frac{r_{m-p}}{r_m} = 0.24 \quad K_{le} = 0.93$$

$$l_e = \sqrt{\frac{(7)^2}{(0.93)}} = 7.3 \text{ in.}$$

Go to $\frac{7}{8}" - 11 - \text{UNC} - 2A \times 10.5" \text{ LG, ASTM A 193}$
Anchor bolts. (Min. emb. = 7.5")

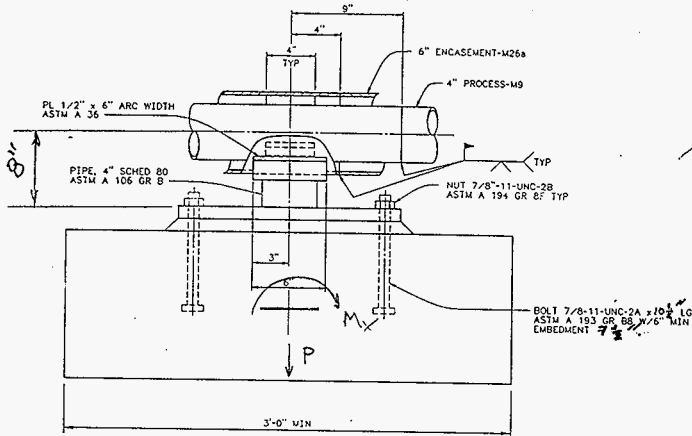
Conclusions:

- 1) Concrete Base dimensions now 4'-0" x 4'-0" x 1'-0"
- 2) Resteel - 5 - #5 bars equally spaced w/ 3" clear spacing from sides & bottom.
- 3) Use $\frac{7}{8}" \phi$ anchor bolts now $10\frac{1}{2}"$ long.

Client	WHC	WO/Job No.	ER 4319 / W-320
Subject	Transfer Line - Concrete Base	Date	5/23/94 By J.R. Booth
	Design	Checked	5/23/94 By J.R. Booth
Location	200E	Revised	By

Check Size Required for Concrete Base w/ only one Pipe

a) Check Worst Case Overtuning Moment:



$$M_x = 79.85 \text{ in} \cdot \text{kip}$$

$$P = F_y + W_c + S$$

where

$$W_c = 150 \text{ pcF} (1 \times 3 \times 3) = 1350 \text{ lb} \quad (\text{Concrete Wt})$$

$$S = 115 \text{ pcF} (3 \times 3 \times 3.92) = 4057 \text{ lb} \quad (\text{Soil Wt})$$

$$F_y = 259 \text{ lbs}$$

$$P = \frac{259 + 1350 + 4057}{1000} = 5.67 \text{ kip}$$

$$e = \frac{79.85 \text{ in} \cdot \text{kip}}{5.67} = 14.1 \text{ in}$$

$$k = \frac{36 \text{ in}}{6} = 6 \text{ in}$$

$$\text{since } e > k, \quad f_{ut} = \frac{2P}{3L \left(\frac{1}{2} - e \right)} = \frac{2(5.67)}{3(3) \left(\frac{1}{2} - 1.10 \right)} = 3.9 \text{ ksf}$$

$$f_{ut} < f_A = 4.0 \text{ ksf} \quad \underline{\underline{OK}}$$

DESIGN ANALYSIS

Calc. No. W320-27-03
Revision 0.1 X 2
Page No. 29 of 32

Client	W/C
Subject	Transfer Case - Concrete Run
Date	5/23/94
Checked By	W. B. Smith
Revised By	W. B. Smith
Location	200F

W/O Job No. ERA 319/W-320
Date 10/3/90
By W. B. Smith

b) For a concrete base supporting only one pipe the twisting action will not be a factor and maintaining approximately the same raster spacing by inspection will be adequate. The raster will have 3" clear from the sides and bottom and be 4" #5 spaced equally in each direction or a 10 inches.

The base will be 3' x 3' x 1'.

Note: The loads used to design the footing were based on preliminary pipe stress information which were greater than the final values calculated by D.L. Stone. Therefore, no changes are necessary in the design of the footing.

Based upon revised loads presented in revision 1 of pages 1 thru 16, the above conclusion is still valid.

Based upon the revised loads presented in revision 2 of this calculation, page 14, the conclusion provided in revision 0 is still valid. No further analysis is required.

P O P

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(PIPE ON PIPE)

=====

PIPE WALL LOCAL STRESSES DUE TO WELDED TRUNNION

=====

SOURCE:

WELDING RESEARCH COUNCIL BULLETIN NO. 107 MARCH 1979 REVISION. LOCAL
STRESSES IN SPHERICAL AND CYLINDRICAL SHELL DUE TO EXTERNAL
LOADING" BY K.R. WICHMAN A.G. HOPPER AND J.L. MERSHON

PROGRAM AUTHOR: MOHAMMED M. AHMED

FILE NAME: W320POP. mcd

WORK ORDER/ JOB NO: ER4319

INPUT BY: Dianna L. Stone

CHECKED BY:

MODEL 7400

Janus
(Supersedes N.

WAFER-MAG™ ELECTROMAGNETIC FLOWMETER

DESCRIPTION

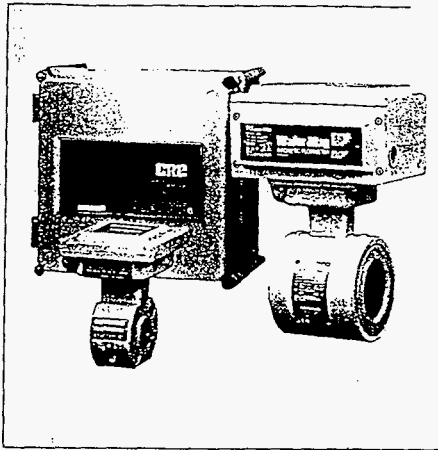
The Brooks® Wafer-Mag electromagnetic flowmeter is a solid state, obstructionless flow rate measuring device which is capable of measuring a wide range of conductive liquids, clear or slurred.

The Wafer-Mag is comprised of two basic parts: a flow transmitter which becomes a part of the pipeline and a signal converter. A magnetic field is generated by compact high density field coils mounted to a short section of straight pipe. As a conductive liquid passes through the pipe section and its magnetic field, a voltage is generated that is directly proportional to the velocity of the conductive liquid. The voltage induced is sensed by electrodes mounted at right angles to the magnetic field and in contact with the conductive liquid or slurry. The electrodes are insulated from the pipe by a non-conductive liner that encapsulates all wetted parts of the pipe. From the electrodes, the voltage is transmitted to a signal conditioner where it is conditioned to the desired output signal(s).

The Brooks Wafer-Mag employs a keyed bi-directional dc magnetic field and digital electronics to establish new criteria in wide rangeability, high accuracy and low power consumption. The Wafer-Mag generates an open collector frequency (proportional to liquid velocity) of 10 kHz at maximum flow rate. Convenient terminal connections are provided to permit local or remote verification of proper meter operation. Optional analog current or factored/scaled pulse outputs are available for integral or remote mounting.

DESIGN FEATURES

- *Accuracy of ±0.5% of rate*
- *Lay length:* wafer body design for installation between mating flanges of ANSI, DIN or BS bolt pattern, to 300 lb. ANSI rating. Minimum length for maximum ease of installation.
- *Light weight:* the wafer body design minimizes weight to maximize ease of handling.
- *Unobstructed flow:* no restrictions are in the liquid path. Pressure drop is equal to an equivalent length of pipe of the same inside diameter.
- *Stable operation:* flow measurement is unaffected by changes in temperature, pressure, specific gravity,



viscosity and/or conductivity above a minimum of 3 micromhos/cm over a wide range of ambient temperatures.

- *DC magnetic field:* bi-directionally driven magnetic field.
- *Automatic zero:* no adjustments required. Signal voltage is dynamically verified each time the coil current reverses up to 75 times per second.
- *Low power consumption:* approximately 15 watts for sizes 0.15" to 8".
- *Interchangeable electronics:* all wafer-mag electronics are interchangeable. No test equipment required.
- *Potted coil housing:* coil housing internals are completely potted for maximum environmental and vibration resistance.
- *Separate sealed wiring compartments:* provide easy customer access for hook-up with maximum protection for electronics.
- *Digital analog span:* analog output has digital span setting for re-ranging without test equipment and no electrical power required for setting.
- *Optional Smart electronics (3580):* configurable through local or remote operator interface.

Brooks Instrument

FISHER-ROSEMOUNT

Electrical Approvals:

- Flowheads with integral electronics: FM approved Class I, Division 2, Groups A, B, C and D, Class II, Division 1, Groups E, F and G and Class III.
- Flowheads for remote hook-up to 3560, 3570 or 3580 electronics: FM approved Class I, Division 2, Groups A, B, C and D, Class II, Division 1, Groups E, F and G and Class III.
- Optional: FM approved Class I, Division 1, Groups C and D. Flowhead can be used with Brooks remote electronics Models 3560, 3570 and 3580.

Accuracy

System $\pm 0.5\%$ of the rate from one unit per second to ten units per second. (one unit = one foot or meter) and 0.005 units per second below one unit per second.

Analog output

Add $\pm 0.1\%$ full scale to above accuracy specifications.

Repeatability

0.25% of rate

Pressure Limits

740 psig @ 100°F (40°C) Tefzel and Teflon tubes
 285 psig @ 100°F (40°C) Ryton™ tubes (1" and smaller)
 150 psig @ 100°F (40°C) Ryton tubes 1-1/2" and 2"

Temperature

Ambient: 0 to 150°F (-18 to 65°C)

Process: Refer to Table 2.

Power Requirements

Selectable by jumper plug -115 Vac or 230 Vac. $\pm 15\%$
 47-63 Hz, 10-30 Vdc

Power Consumption

15 Watts maximum

SPECIFICATIONS

WARNING: Do not operate this instrument in excess of the specifications listed below. Failure to heed this warning could result in personal injury and/or damage to the equipment.

Capacities

Full Scale Meter Range: 1 to 10 meters/sec or 1 to 10 feet/sec, jumper selectable with no special calibration required (refer to Table 1).

Table 1 Capacity Table - Velocity-to-Flow Conversion Factors 0.15" to 8" *

Meter Size		Flow in gpm Units = Feet						Flow in gpm Units = Meters						Guidelines Flow 6 ft/sec.	
		1,000 ft/sec		10.00 ft/sec		Low Flow Cut-off ⁽¹⁾		1,000 m/sec		10,000 m/sec		Low Flow Cut-off ⁽²⁾			
Inches	mm	gpm	lpm	gpm	lpm	gpm	lpm	gpm	lpm	gpm	lpm	gpm	lpm	gpm	lpm
0.15 ⁽¹⁾	4	0.055	0.21	0.55	2.1	0.001	0.004	0.18	0.68	1.8	6.8	0.004	0.02	0.33	1.3
0.3 ⁽¹⁾	8	0.22	0.83	2.2	8.3	0.005	0.02	0.72	2.7	7.2	27	0.02	0.06	1.3	4.9
1/2	15	0.86	3.3	8.6	33	0.02	0.08	2.8	10.6	28	106	0.06	0.23	5	19
1	25	2.3	8.7	23	87	0.05	0.20	7	28	75	284	0.16	0.6	14	53
1-1/2	40	5.5	21	55	208	0.13	0.48	18	68	180	681	0.41	1.6	36	136
2	50	9.1	34	91	344	0.21	0.79	30	113	300	1,136	0.69	2.6	55	208
3	80	21	80	213	806	0.48	1.8	70	265	700	2,650	1.6	6	128	485
4	100	37	138	365	1,382	0.85	3.2	120	454	1,200	4,542	2.8	10	220	833
6	150	83	314	830	3,142	1.9	7.2	270	1,022	2,700	10,220	6.2	24	495	1,874
8	200	152	575	1,520	5,753	3.5	13	500	1,893	5,000	18,925	11.5	44	915	3,463

* All Values are approximate — actual values will depend on flow tube K-factor

⁽¹⁾ These meter sizes are only offered with Ryton flow tubes, maximum pressure 285 psig at 100°F (40°C)

⁽²⁾ Low flow cut-off is 2.3% of the selected analog setting adjustable from 23 Hz as listed in the above table through 230 Hz at the MAXIMUM FULL SCALE ANALOG SETTING AT 10 FEET OR 10M PER SECOND.

Table 2 Maximum Working Pressures and Temperatures

Materials (Liner)	Characteristics (Liner)	Available in Meter Sizes	Pressure psig (kPa)			Maximum Allowable Liquid Temperature (1)
			0-100°F (40°C)	200°F (93°C)	300°F (149°C)	
Tefzel	Chemical Resistant	1/2 through 8 inch	740 psig 5202 kPa	675 psig 4745 kPa	655 psig 4605 kPa	300°F (149°C)
Teflon	3A Food Services	1/2 through 6 inch	740 psig 5202 kPa	575 psig 4745 kPa	655 psig 4605 kPa	350°F (149°C)
Ryton	Fractional Sizes	.15 and .30 inch	285 psig 2004 kPa	185 psig 1300 kPa	Not Applicable	200°F (93°C)
Kynar	Consult Factory	Consult Factory	CF	CF	CF	CF

Note (1): Temperature limits may vary depending on liquid being metered.

PROGRAMMED BY:
M. M. AHMED

VERSION: 0
PAGE NO: 2

INPUT DATA:
=====

1. APPLIED LOADS:

NOTE: THESE ARE THE LOADS ACTING AT PIPE WALL &
ATTACHMENT JUNCTURE.

(Ref. File SA-07.mcd,
"Pipe Loads at B")

P = 259 RADIAL LOAD IN POUNDS
M_c = 6626 CIRCUMFERENTIAL MOMENT IN INCH-POUNDS
M₁ = 19390 LONGITUDINAL MOMENT IN INCH-POUNDS
M_t = 3276 TORSIONAL MOMENT IN INCH-POUNDS
V_c = 846 SHEAR LOAD ACROSS PIPE LENGTH IN POUNDS
V₁ = 4690 SHEAR LOAD ALONG PIPE LENGTH IN POUNDS

2. GEOMETRY:

T = 1.267 PROC. PIPE 0.237 + ENC. PIPE 0.28 + PAD 0.75 = 1.267"
R_m = 2.68 (ENC. OD 6.625 - T 1.267)/2
ro = 2.25 ATTACHMENT PIPE (TRUNNION) RADIUS IN INCHES
w = 0.3125 FILLET WELD SIZE IN INCHES
r = 0.707 · w FILLET WELD RADIUS IN INCHES
r = 0.221

3. GEOMETRIC PARAMETERS:

$\Gamma = \frac{R_m}{T}$ PIPE/VESSEL PARAMETER

$\Gamma = 2.115$

$\beta = (0.875) \frac{ro}{R_m}$ ATTACHMENT PARAMETER

$\beta = 0.735$

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M.M.AHMED

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PAGE NO: 3

4. STRESS CONCENTRATION FACTORS:

$$K_n = 1 + \left(\frac{2 \cdot T}{5.6 \cdot T} \right)^{0.65} \quad \text{TENSILE FACTOR}$$

$$K_n = 2.594$$

$$K_b = 1 + \left(\frac{2 \cdot T}{9.4 \cdot T} \right)^{0.8} \quad \text{BENDING FACTOR}$$

$$K_b = 2.173$$

BIJLAARD'S NONDIMENSIONAL CURVES:

THESE CURVES ARE PLOTTED FOR ' β ' VERSUS A NONDIMENSIONAL CONSTANTS FOR DIFFERENT VALUES OF ' Γ ' AND APPLIED LOADS. THIS PROGRAM ADOPTS A SIMPLE AND CONSERVATIVE APPROACH FOR A SPECIFIC, MOST OFTEN USED RANGE OF VALUES FOR ' β ' & ' Γ '. SINCE MOST OF THE PIPE ATTACHMENTS DESIGNED HAVE THE PARAMETERS ' β ' & ' Γ ' WITHIN THE RANGES LISTED BELOW, THE MOST CRITICAL NONDIMENSIONAL CONSTANTS ARE CHOSEN FROM THE APPLICABLE CURVES AND USED IN COMPUTING STRESSES CONSERVATIVELY. THUS ELIMINATING THE USE OF BIJLAARD CHARTS EACH TIME.

IF VALUE OF " Γ " IS BETWEEN 5 AND 15 $\Gamma = 2.115$

AND " β " = OR > 0.5, $\beta = 0.735$

USE THE CONSTANTS GIVEN BELOW.

(IF Γ & β DO NOT FALL WITHIN ABOVE RANGE, READ CONSTANTS FROM SPECIFIC CURVES AND USE THEM IN PLACE OF CONSTANTS LISTED BELOW.)

SINCE "b

SINCE VALUES ARE OUT OF ABOVE RANGE, APPROPRIATE BIJLAARD CONSTANTS ARE USED.

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PAGE NO: 4

CONSTANTS FOR CIRCUMFERENTIAL STRESSES:

$A\phi = 0.70$ FROM FIG 4C: VALUE OF ' $N\phi/P/Rm$ ' FOR 'P'
 $B\phi = 0.04$ FROM FIG 2C-1: VALUE OF ' $M\phi/P$ ' FOR 'P'
 $C\phi = 0.20$ FROM FIG 3A: VALUE OF ' $N\phi/Mc/Rm^2\beta$ ' FOR Mc
 $D\phi = 0.12$ FROM FIG 1A: VALUE OF ' $M\phi/Mc/Rm\beta$ ' FOR Mc
 $E\phi = 0.50$ FROM FIG 3B: VALUE OF ' $N\phi/ML/Rm^2\beta$ ' FOR ML
 $F\phi = 0.04$ FROM FIG 1B: VALUE OF ' $M\phi/ML/Rm\beta$ ' FOR ML

CONSTANTS FOR LONGITUDINAL STRESSES

$Ax = 0.60$ FROM FIG 4C: VALUE OF ' $Nx/P/Rm$ ' FOR 'P'
 $Bx = 0.08$ FROM FIG 1C-1: VALUE OF ' Nx/P ' FOR 'P'
 $Cx = 0.30$ FROM FIG 4A: VALUE OF ' $Nx/Mc/Rm^2\beta$ ' FOR Mc
 $Dx = 0.07$ FROM FIG 2A: VALUE OF ' $Mx/Mc/Rm\beta$ ' FOR Mc
 $Ex = 0.15$ FROM FIG 4B: VALUE OF ' $Nx/ML/Rm^2\beta$ ' FOR ML
 $Fx = 0.08$ FROM FIG 2B: VALUE OF ' $Mx/ML/Rm\beta$ ' FOR ML

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M.M. AHMED

VERSION-0
PAGE NO:5

CIRCUMFERENTIAL ($\sigma\phi$) STRESSES:

$$\sigma\phi_a = K_n \cdot A\phi \cdot \frac{P}{R_m \cdot T} \qquad \sigma\phi_a = 138.48$$

$$\sigma\phi_b = K_b \cdot B\phi \cdot \frac{P}{T^2} \qquad \sigma\phi_b = 84.125$$

$$\sigma\phi_c = K_n \cdot C\phi \cdot \frac{M_c}{R_m^2 \cdot \beta \cdot T} \qquad \sigma\phi_c = 514.14$$

$$\sigma\phi_d = K_b \cdot D\phi \cdot \frac{M_c}{(R_m \cdot \beta) \cdot T^2} \qquad \sigma\phi_d = 3.28 \cdot 10^3$$

$$\sigma\phi_e = K_n \cdot E\phi \cdot \frac{M_1}{R_m^2 \cdot \beta \cdot T} \qquad \sigma\phi_e = 3.761 \cdot 10^3$$

$$\sigma\phi_f = K_b \cdot F\phi \cdot \frac{M_1}{(R_m \cdot \beta \cdot T^2)} \qquad \sigma\phi_f = 3.199 \cdot 10^3$$

CIRCUMFERENTIAL STRESS ($\sigma\phi$) SUMMATION:

NOTE: USE THE MOST CRITICAL STRESS OF THE FOLLOWING SUMMATIONS.

$$\sigma\phi_{Cu} = \sigma\phi_a + \sigma\phi_b + \sigma\phi_c + \sigma\phi_d \qquad \sigma\phi_{Cu} = 4.016 \cdot 10^3$$

OR

$$\sigma\phi_{Au} = \sigma\phi_a + \sigma\phi_b + \sigma\phi_e + \sigma\phi_f \qquad \sigma\phi_{Au} = 7.183 \cdot 10^3$$

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VERSION:0
PAGE NO:6

LONGITUDINAL (σ_x) STRESSES:

$$\sigma_{xa} = K_n \cdot A_x \cdot \frac{P}{Rm \cdot T} \qquad \sigma_{xa} = 118.697$$

$$\sigma_{xb} = K_b \cdot B_x \cdot \frac{P}{T^2} \qquad \sigma_{xb} = 168.25$$

$$\sigma_{xc} = K_n \cdot C_x \cdot \frac{M_c}{Rm^2 \cdot \beta \cdot T} \qquad \sigma_{xc} = 771.209$$

$$\sigma_{xd} = K_b \cdot D_x \cdot \frac{M_c}{(Rm \cdot \beta \cdot T^2)} \qquad \sigma_{xd} = 1.913 \cdot 10^3$$

$$\sigma_{xe} = K_n \cdot E_x \cdot \frac{M_1}{Rm^2 \cdot \beta \cdot T} \qquad \sigma_{xe} = 1.128 \cdot 10^3$$

$$\sigma_{xf} = K_b \cdot F_x \cdot \frac{M_1}{(Rm \cdot \beta \cdot T^2)} \qquad \sigma_{xf} = 6.398 \cdot 10^3$$

LONGITUDINAL (σ_x) STRESS SUMMATION:

NOTE: USE THE MOST CRITICAL OF THE FOLLOWING SUMMATIONS.

$$\sigma_{Cu} = \sigma_{xa} + \sigma_{xb} + \sigma_{xc} + \sigma_{xd} \qquad \sigma_{Cu} = 2.971 \cdot 10^3$$

OR

$$\sigma_{Au} = \sigma_{xa} + \sigma_{xb} + \sigma_{xe} + \sigma_{xf} \qquad \sigma_{Au} = 7.813 \cdot 10^3$$

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M.M. AHMED

VERSION:0
PAGE NO:7

SHEAR ($\Gamma_{x\phi}$) STRESSES:

$$\Gamma_{x\phi 1} = \frac{M_t}{(2 \cdot \pi \cdot r_o^2 \cdot T)} \quad \text{SHEAR STRESS DUE TO TORSION 'MT'}$$

$$\Gamma_{x\phi 1} = 81.287$$

$$\Gamma_{x\phi 2} = \frac{V_c}{\pi \cdot r_o \cdot T} \quad \text{SHEAR STRESS DUE TO SHEAR 'VC'}$$

$$\Gamma_{x\phi 2} = 94.463$$

$$\Gamma_{x\phi 3} = \frac{V_l}{\pi \cdot r_o \cdot T} \quad \text{SHEAR STRESS DUE TO SHEAR 'VL'}$$

$$\Gamma_{x\phi 3} = 523.677$$

SHEAR STRESS ($\Gamma_{x\phi}$) SUMMATION:

THE TWO COMPONENTS OF SHEAR STRESSES ARE:

$$\Gamma_{x\phi Au} = (\Gamma_{x\phi 1} + \Gamma_{x\phi 2}) \quad \Gamma_{x\phi Au} = 175.75$$

$$\Gamma_{x\phi Du} = (\Gamma_{x\phi 1} + \Gamma_{x\phi 3}) \quad \Gamma_{x\phi Du} = 604.965$$

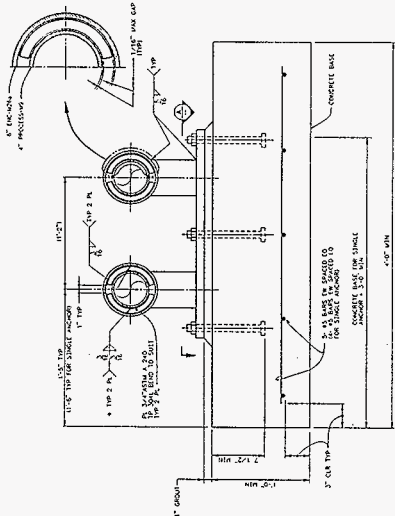
NOTE: WHILE QUALIFYING THE PIPE STRESSES FOR ASME CODES B31.1 AND B31.3, THIS PROGRAM, "POP", SHALL BE RUN TWICE, ONCE FOR THERMAL LOADS AND ONCE FOR SUSTAINED + SEISMIC LOADS. THE HIGHEST OF THE LONGITUDINAL STRESSES (σ_{xAu} or σ_{xCu}) COMPUTED ON PAGE-6 OF THIS PROGRAM SHALL BE ADDED TO THE "AUTOPIPE" PIPE STRESSES FOR RESPECTIVE LOAD CASE. THE SUMMATION PIPE NORMAL STRESSES (AUTOPIE) AND LOCAL LONGITUDINAL STRESSES (POP) SHALL BE WITHIN THE CODE ALLOWABLES FOR CORRESPONDING LOAD CASES.

C.D. JONES, PE

DESIGNED BY		C.D. JONES, PE	
CHECKED BY		[Signature]	
DATE		[Date]	
PROJECT NO.		H-2-6155-68-10	
DRAWING NO.		[Drawing No.]	
SHEET NO.		[Sheet No.]	
SUPPORT DETAILS		[Support Details]	
REV.		[Revisions]	

NOTES:

1. USE APPROVED AND LISTED SET BOLT AND WASHERS
2. BRUSH ALL SURFACES AND REMOVE ALL IMPURITIES
3. AND INSTALL ON THE PROPER TIGHTENING TORQUE
4. SUCH AS GIVEN IN THE SPECIFICATIONS, OTHERWISE TO THE



✖ INSURE PIPE ANCHOR SHOE

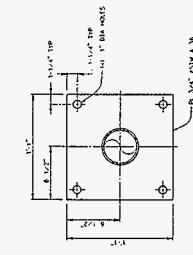
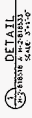


PLATE FOR SINGLE ANCHOR

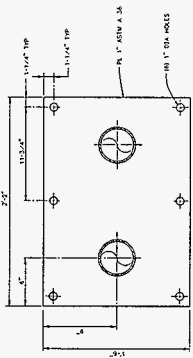
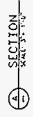
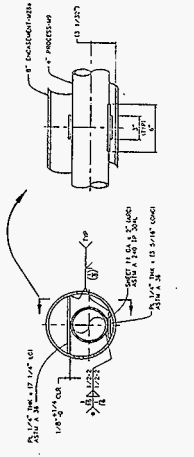
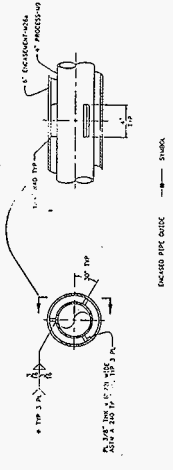


PLATE FOR DOUBLE ANCHOR

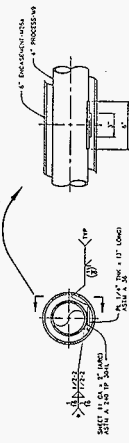




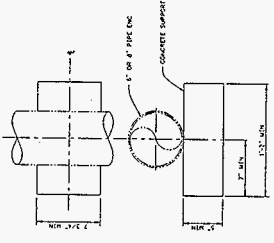
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 DETAIL
 1/4\"/>



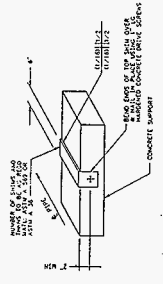
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 DETAIL
 1/4\"/>



ENCASED PIPE SUPPORT ——— SINKING
 DETAIL
 1/4\"/>



DETAIL
 1/4\"/>



DETAIL
 1/4\"/>

NOTE:
 1. FOR NOTES, SEE SHEET 1

C.D. JONES, PE

PROJECT NO.	100-100000000
DATE	11/15/88
U.S. DEPARTMENT OF ENERGY	
OFFICE OF NEUTRON PHYSICS	
PIPPING	
SUPPORT DETAILS	
FIGURE NO.	1-2-88S-48
SCALE	AS SHOWN
DESIGNED BY	C.D. JONES
CHECKED BY	
APPROVED BY	

W320-27-016

C Farm Jumper Stress Analysis

This sheet shows the status and description of the attached Design Analysis sheets.

Discipline 27, Piping and Vessels WO/Job No. ER4319 Calculation No. W320-27-016
 Project No. & Name Project W-320 Waste Retrieval for Tank 241-C-106
 Calculation Item C Farm Jumper Stress Analysis

These calculations apply to:

Dwg. No. H-2-818524, H-2-818505, and H-2-818508 Rev. No. 81
 Dwg. No. H-2-818526 and H-2-818515 Rev. No. 81
 Other (Study, CDR) _____ Rev. No. _____

The status of these calculations is:

- Preliminary Calculations VZ
- Final Calculations
- Check Calculations (On Calculation Dated _____)
- Void Calculation (Reason Voided _____)

Incorporated in Final Drawings? Yes No
 This calculation verified by independent "check" calculation? Yes No

Original and Revised Calculation Approvals:

	Rev. 0 Signature / Date	Rev. 1 Signature / Date	Rev. 2 Signature / Date
Originator	M. M. Ahmed 9-11-95	<i>M. Ahmed</i> M.M. Ahmed 6/28/96	<i>V. Harper</i> V. Harper 11/25/97
Checked by	M. K. Pal 9-11-95	<i>J. Stone</i> D.L. Stone 7-1-96	<i>M. Ahmed</i> M. Ahmed 3.17.98
Approved by		<i>CD Jones</i> CD Jones 7/1/96	<i>J.P. Evans</i> J.P. Evans 4/9/98
Checked Against			<i>M. Ahmed</i> M. Ahmed 4.10.98

REVISION 1 SUMMARY: Revised pages 1, 5, 6, 7 & 8, Appendices A & B.
 Added page-9.

	INDEX
Design Analysis	Description
Page No.	
<u>i - ii</u>	<u>Calculation Identification and Index</u>
<u>iii</u>	<u>Calculation Cross Index</u>
<u>1</u>	<u>Objective, Design Criteria ,Design Inputs</u>
<u>2 - 3</u>	<u>Design Methodology</u>
<u>4</u>	<u>Assumptions</u>
<u>4 - 5</u>	<u>References</u>
<u>5</u>	<u>Calculations and Conclusions</u>
<u>6</u>	<u>ISB Jumper Connectors qualification</u>
<u>7 - 8</u>	<u>Pit Wall Anchor Loads</u>
<u>9</u>	<u>Jumper piping Isometric 3D Model</u>

Discipline	27, Piping and Vessels	WO/Job No.	ER4319	Calculation No.	W320-27-016
------------	------------------------	------------	--------	-----------------	-------------

	Rev. 0 Signature / Date	Rev. 1 Signature / Date	Rev. 2 Signature / Date
Originator	M. M. Ahmed 9-11-95	M.M. Ahmed 6-28-96 <i>M. Ahmed</i>	V. B. Khatyaa 11/25/97
Checked by	M. K. Pal 9-11-95	D. L. Stone 7-1-96 <i>D.L. Stone</i>	M. Ahmed 3.17.98 <i>M. Ahmed</i>

INDEX

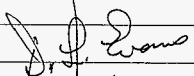
Design Analysis	Description
Page No.	
Appdx A	AutoPipe File no: MA-C06A, Stress Isometric & Drawings
Appdx B	AutoPipe File ^{MA-C06AS} MA-C06AS Static & Hydrostatic analysis // Rev 1
Appdx E	AutoPipe File no: MA-J6A, Isometric & Piping Drawings // REV 1
Appdx C	Reference Documentation // Rev 1
Appdx A	AutoPipe File : MA-C06A , and drawings
Appdx B	AutoPipe File : MA-C06AS
Appdx C	AutoPipe File : MA-J6A , Isometric , and drawings
Appdx D	Reference Documentation
	Rev. 2 - Verify As-Built Conditions.
	Page 5a added. Pages I, ii, iii, I, and 4 revised.



CALCULATION CROSS INDEX (Typical)

Subject Calculation No.: W320-27-016

Page iii of iii

Subject Calculation Revision No.	Superceded by Calculation No.	These interfacing calculation/documents provide input to the subject calculation, and if revised may require revision of the subject calculation.		Results and conclusions of the subject calculation are used in these interfacing calculations and/or documents		Does the output interface calculation/ documents require revision?		Has the output interfacerecalculation/ documents been revised?		Discipline manager's signature and date indicating evaluation complete.
		Calculation/Document No.	Revision No.	Calculation/Document No.	Revision No.	Yes	No	Yes	No	
2	NA	H-2-818505, Sheet 1	Ø	NA			x		x	 7/16/98
		H-2-818505, Sheet 2	Ø				x		x	
		H-2-818508, Sheet 1	Ø				x		x	
		H-2-818515, Sheet 1	Ø				x		x	
		H-2-818524, Sheet 1	Ø	△			x		x	
		H-2-818524, Sheet 2	Ø	△			x		x	
		H-2-818526, Sheet 1	Ø	△			x		x	
		H-2-818526, Sheet 2	Ø	△			x		x	

HNF-2474, Rev. 0
Page B-3

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY
Subject AY FARM - JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL FOR TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319
Date 9-11-95
Checked 9/11/95
Revised 1/1
Filename: C06A.WP
By: M.M. Ahmed
By: M.K. Pal
By:

Rev. 2 By: K. Hayase 11/25/97
Chk'd By: M. Ahmed 3.17.98

Rev-1: By: M. Ahmed 6-28-96
Chk'd By: D. L. Sp... 7-1-96

OBJECTIVE:

The objectives of this calculation are:

1. Perform static and dynamic piping stress analysis of the "C- Tank Farm" Jumper Assemblies in order to be in compliance with the Code ASME B31.3 requirements.
2. Qualify the Integral Seal Block (ISB) Jumper connectors for loads exerted by piping in accordance with document WHC-SD-W236A-DA-002, Rev-0
3. Prepare a report on piping loads exerted on pit wall at pipe penetration location.

REV-1:

To incorporate the changes resulting from using a different Booster pump.

Rev-2: Verify As-built conditions

1 Rev-1

DESIGN CRITERIA:

1. ASME CODE B31.3-93 with Addenda up to 1994.
2. SDC 4.1, REV-12. (Now FDNW Practice #134.245.1217)
3. PROJECT W-320 FDC: WHC-SD-W320-FDC-001, REV. 2/4
4. WHC-SD-W236A-DA-002, Rev-0

DESIGN INPUTS:

GENERAL (Applicable to all Jumpers)

Ambient Temperature	40°F	(Ref: 3)
Design Temperature:	180°F	(Ref: 3)
Design Pressure:	325 psig	(Ref: 10)
Hydrostatic Pressure	490 psig	
Pipe Materials:	ASTM A312 TP304L	(Ref: 19)
Pipe Sizes:	3 in. and 4 in. 1 Rev-1	(Ref: 19)
Pipe Wall	Schedule 40	(Ref: 5 & 19)
Safety Class category:	SC-3	(Ref: 4)
Corrosion Allowance	0.06 inches	(Ref: 7)
B31.3 Code Allowable stress	16700 psi	(Ref: 1)

JUMPER U9-A, B

PUMP PIT 241-C-06A (Autopipe Filename: MA-C06A & C06A-ST)

MA-C06AS

Process Fluid:	241-C-06A Slurry	(Ref: 20)
Specific gravity:	1.20	(Ref: 8)
Piping Dimensions:	Dwg: H-2-818505	(Ref: 19)
Flanges	300# ANSI rated	(Ref: 21)
Connector weights	4": 122 lbs 3": 77 lbs 2": 29 lbs	(Ref: 9)
Nozzle weights	4": 11 lbs 3": 6.5 lbs 2": 2.5 lbs	(Ref: 9)
Kickplate weights	4": 16.5 lbs 3": 12 lbs 2": 3.5 lbs	(Ref: 9)
Pressure Element Weight	4": 27 lbs N/A 2": 12 lbs (Appx: C)	(Ref: 18)
Pump	W _{pump} = 1000 lbs	(Appx: C) (Ref: 17) Rev-1
Pump Motor	W _{pump Motor} = 2250 lbs	(Appx: C) (Ref: 17) Rev-1
Blank 2" Connector with Counter Weight	86.98 lbs	(Appx: C) (Ref: 22)

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY
Subject AY FARM - JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319 Filename: C06A.WP
Date 9-11-95 By: M.M. Ahmed
Checked 9/11/95 By: M.K. Pal
Revised / / By:

JUMPER U6-A PUMP PIT 241-C-06C (Autopipe Filename: MA-J6A)

Process Fluid:	Tank 241-C-106 Supernate	(Ref: 20)
Specific gravity:	1.12	(Ref: 3)
Piping Dimensions:	Dwg: H-2-818515	(Ref: 19)
Connector weight	122 lbs	(Ref: 9)
Nozzle weight	11 lbs	(Ref: 9)
Kickplate weight	17 lbs	(Ref: 9)
Jumper Flushing Blank Head	71.5 lbs	(Ref: 9)

DESIGN METHODOLOGY:

C Farm Jumper Piping Assemblies include Jumper U9-A that is located in Pump Pit 241-c-06A and Jumper U6-A that is located in Pump Pit 241-C-06C.

The jumper assembly models include connectors and nozzles. The nozzles are considered to be rigidly anchored either to pit walls or cover plate for tanks (H-2-818524 and H-2-818526, Ref:19).

The jumper connectors are specified as Integral Seal Block (ISB) connectors (Ref:9) The document WHC-SD-W236A-DA-002 (Ref:22) provides all the analytical data such as connector stiffness, acceptable piping loads & seating forces for the ISB connectors. These connectors have been modeled in Autopipe as short radius elbows and flexible couplings with specified translation and rotational stiffness. The connector and nozzle weights are imposed at Flexible coupling.

The weights of structural steel assemblies and lifting bails used to connect jumper piping to nozzles during installation are calculated and imposed at appropriate points.

Stress analysis of the jumpers has been performed in accordance with ASME B31.3 for sustained, thermal and seismic loading (Ref. 1) using the computer program "Autopipe" Version-4.6 by "Engineering Design Automation" (EDA) (Ref. 11).

SUSTAINED LOADS:

The "AutoPipe" program checks the sustained stress (the summation of gravity and pressure stresses) against the B31.3 Code Allowable stress "S" for the material specified for each jumper piping.

THERMAL (DISPLACEMENT) STRESS:

The maximum design temperature for the pipe of 180°F and the ambient temperature of 40°F has been used in "AutoPipe" program to qualify the thermal expansion stress to the Code B31.3 allowable for displacement stress, "Sa".

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY
Subject AY FARM - JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL FOR TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319 Filename: C06A.WP
Date 9-11-95 By: M.M. Ahmed
Checked 9/11/95 By: M.K. Pal
Revised / / By:

SEISMIC STRESSES:

The jumper piping has been categorized as Safety Class 3. However the pit structure is classified as Safety Class 1 category. In order to perform Safety Class 3/1 analysis for this investigation, dynamic response spectra analysis is performed using Safety class-1, "Fig 3" seismic response spectra (5% Damping) of SDC 4.1, Rev-12 (Ref.2). This response spectra is created in "AutoPipe" as "SC1" and used in this dynamic analysis. Autopipe qualifies seismic stresses as occasional stress with the code allowable 1.33S.

The Jumper Connectors are qualified in accordance with WHC-SD-236A-DA-002, Rev-0 (Ref:22) Table 3.4 Acceptable piping loads. The acceptable axial force is limited to allowable seating forces for 50 ft-lb torque of Table 3.3 assuming all the actual axial loads tend to un-seat connector assembly. This results in conservative analysis. The acceptable loads are not increased by 1/3 as permitted on page-6 of Ref:22, which is conservative. The interaction of loads for critical connector is calculated (See page-6 of this calculations) as specified on page-7 of Ref:22.

HYDROSTATIC PRESSURE:

A hydrostatic pressure analysis is performed for 490 psig (1.5 times of Design pressure) and documented in Appendix-B.

The following describes the modelling and analysis techniques unique to individual jumpers.

JUMPER U9-A

Jumper U9-A includes the piping extending from the Tank C-06 Submersible Pump Assembly nozzle (A) to the Tank C-06 Booster Pump, and from the Booster Pump to the Transfer Line nozzle (U-9). The "Frame" feature of the "Autopipe" program has been used to assimilate the Booster Pump Assembly in order to consider the adequate stiffness of the pump in the system. The weight of the pump and motor is imposed at locations corresponding the centers of gravity of the pump and motor. The seismic stabilizer is represented in the model by two 3-inch schedule 40 pipes that connect to the lifting frame sub-assembly.

A Gravity support (Y-stop) has been modeled directly below the pump assembly (Node point#1) which is seated on concrete pedestal for the Sustained (Static) analysis only (See Appendix-B). The Y-stop is removed from the model for the dynamic analysis (See Appendix-A) in order to retain the linearity of the dynamic analysis and most accurately assimilate the effects of movement of the pump assembly in a Seismic event.

Weight of instrumentation has been applied at the instrument center on the pipe with flanges input as 400# ANSI rated in accordance with the 325 psig design pressure.

PIPING LOADS AT BUILDING PENETRATION ANCHORS

The spreadsheet program, "Excel" (Ref. 12), has been used to summarize Pipe Loading at Building Penetration Anchors and listed on pages 7 & 8.

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY
Subject AY FARM - JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319 Filename: C06A.WP
Date 9-11-95 By: M.M. Ahmed
Checked 9/11/95 By: M.K. Pal
Revised 11/25/97 By: K. Hayase

CHK'D BY: M. Ahmed
3.17.98

ASSUMPTIONS:

1. Since the "Autopipe" program cannot perform non-linear dynamic analysis, Y-stops (No gap below and an infinite gap above the pipe) is ignored for the dynamic analysis. This results in a conservative analysis.
2. It is assumed that the lifting bail and structural steel assemblies designed to support the jumper piping during installation will be supported by the jumper during operation.
3. Assumed the ISB jumper connectors will be used and torqued to 50 ft-lbs.
4. Building structures and tank nozzles are structurally adequate to withstand pipe loads.

REFERENCES:

1. ASME Code B31.3 and B31.3a-1994 Addenda for "Chemical and Petroleum Refinery."
2. SDC 4.1, Rev. 12, "Standard Architecture & Civil Design Criteria."
(Now FDMW Practice # 134, 215, 1217, Design Loads for Facilities)
3. WHC-SD-W320-FDC-001, Rev-2. "Functional Design Criteria for Tank 241-C-106 Waste Retrieval, Project W-320". 4
4. WHC-SD-WM-SEL-033, Rev-1, "Safety Equipment List for 241-C-106 Waste Retrieval Project W-230"
5. Construction specifications W-320-C5, Rev-0, Section 15493 for piping materials.
6. Construction Specification No: B-131-C1, Section 15415 for piping materials.
7. Calculation No: W-030-011, Rev-1 for "Fouling factor and Corrosion rate."
8. *Data Compendium*, Bruce A. Castaing
9. Jumper Design Fabrication Drawings: H-2-821325, H-2-821326 & H-2-821404.
10. DSI from C.D.Jones dated March 28, 1994 "Design Pressure for Slurry and Sluice Pumps".
11. Computer program "AutoPipe" Version 4.6 by "Engineering Design Automation."
12. Computer Program "EXCEL" Version 4.0, by Microsoft.
13. Crane Technical Paper No: 410, 22nd Printing-1986.
14. Ryerson Stock List and Data Book, 1985-86 Edition, Joseph T. Ryerson & Son, Inc.

2

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY
Subject AY FARM - JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319 Filename: C06A.WP
Date 9-11-95 By: M.M. Ahmed
Checked 9/11/95 By: M.K. Pal
Revised / / By:

Rev-1: By: *M. Ahmed* 6.28.96
Checked By: *D.L. [Signature]* 7-1-96

- 15. PBM Reference Book Sec. II (SP Series) and Sec. IV (MP Series), PBM, Inc.
- 16. Fax Transmittal Memo from Micro Motion, Inc. Fax dated 6/22/94
- 17. Fax Transmittal Memo from C.D.Jones of Reliance Dimension Sheet 611740-503 and Goulds Catalog # GPM5, Page 134 for Pump Model 3900, dated 1/23/95
- 18. Fax Transmittal Memo from Red Valve Co. Inc., on Series 40 Pressure Sensors.
- 19. Piping Arrangement and Detail drawings: H-2-818524, H-2-818505, H-2-818508, H-2-818526, and H-2-818515
- 20. Process Flow Diagrams: H-2-818558 H-2-818559, and H-2-818560
- 21. ASME/ANSI B16.5
- 22. WHC-SD-W236A-DA-002, Rev-0 "Stress Analysis of Single Port (ISB) Jumper Connectors for 2-, 3-, and 4-in Sizes".
- 23. Seismic Design White Paper Project W-320 Tank 241-C-106 Sluicing

CALCULATIONS:

The calculations are documented in this package as shown below.

AutoPipe stress analysis results & Isometric models	Appendices A, B & C
Jumper Connector load qualification	Page-6
Pit Wall Anchor Loads	Pages 7 & 8
Jumper 3D Isometric Model for AutoPipe	Page-9
	I Rev-1

CONCLUSION:

The review of the "AutoPipe" summary results attached as Appendices to this calculation indicates that all pipe stresses are in compliance with the requirements of ASME Code B31.3 requirements.

The Jumper qualification on page-6 indicates that ISB connectors are adequate for piping loads.

The Piping loads on Pit walls at piping penetrations are summarized on pages 7 & 8.

DESIGN ANALYSIS

Client: Numatec

WO/Job No.: ER4319/W-320

Subject: C Farm Jumper Stress Analysis
Project W-320 Waste Retrieval for Tank 241-C-106

Date: 11/25/97 By: K. Miyase
Checked: 3.17.98 By: *K. Ahmed*

Location: C Tank Farm 200 East Area

Revised: By:

Rev. 2

CONCLUSION (CONTINUED)

Rev. 2 of this calculation verified the analysis against the as-built conditions (~~as of 11/24/97~~^{8a}). The analysis includes the weights of a flow meter and pressure sensors. These weights are not critical to the results of the analysis. Therefore, any differences between the weights used in the analysis and the approved vendor data are negligible.

AS-BUILT CHANGES ARE INSIGNIFICANT AND WILL NOT HAVE ANY
ADVERSE IMPACT ON ANALYSIS.

Client WESTINGHOUSE HANFORD COMPANY
 Subject AY FARM - JUMPER STRESS ANALYSIS
 PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
 Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319
 Date 9-11-95
 Checked 9/11/95
 Revised / /

Filename: C06A.WP
 By: M.M. Ahmed
 By: M.K. Pal
 By:

Rev-1: By: *A. Ahmed* 6.28.96
 Checked By: *D. Shaw* 7.19.96

ISB JUMPER CONNECTOR LOADS QUALIFICATION

CONNECTOR LOCATION	AUTOPIPE FILE NAME & NODE	SIZE	CONNECTOR ACTUAL LOADS I Rev-1 /ALLOWABLE LOADS						REF NO:
			Axial X (lbs)	Shear Y (lbs)	Shear Z (lbs)	Torsion Mx (ft-lb)	Bend My (ft-lb)	Bend Mz (ft-lb)	
Connector-U9 Pump Pit C-06A	MA-C06A A03M	4"	520	276	381	180	197	236	APDX-A
			6310	2220	2220	780	1170	1170	22
Connector A Pump Pit C-06A	MA-C06A A11M	4"	483	149	337	140	365	185	APDX-A
			6310	2220	2220	780	1170	1170	22
Connector B Pump Pit C-06A	MA-C06A B04M	4"	269	258	333	30	100	84	APDX-A
			6310	2220	2220	780	1170	1170	22

SEATING FORCES BETWEEN CONNECTOR & NOZZLES:

The axial allowable load considered in table above are limited to seating forces for 4" Connectors for 50 ft-lb torque as specified in Table 3.3(Ref:22). Since the actual axial forces are less than the allowable seating forces, the connector will not be un-seated from the nozzle.

INTERACTION OF SHEAR, TORSION AND BENDING:

$I = f_{yz}/F_{yz} + [m_x]/M_x + m_z/M_z < 1.0$ Ref:22, page-7

Review of actual loads and allowable loads in the table above indicates that the Connector-U9 in the pump pit is more critical for interaction. I Rev-1

$f_{yz} = 470$ lbs (SRSS of $f_y = 276$ & $f_z = 381$). Increase Allowable by 1.33 for seismic I Rev-1

$m_{yz} = 307$ ft-lbs (SRSS of $m_y = 197$ & $m_z = 236$) I Rev-1

$I = 470/1.33 \times 2220 + 180/1.33 \times 780 + 307/1.33 \times 1170$ I Rev-1

$I = 0.53 < 1.00$ As such Jumper Connectors are adequate. I Rev-1

**ICF KAISER
HANFORD COMPANY**

DESIGN ANALYSIS

Calc. No. W-320-27-016
Revision No: 0
Page No. 7 of 9

Client WESTINGHOUSE HANFORD CO.
Subject AY FARM JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM 200 EAST AREA


WO/Job N ER4319
Date 9/11/95
Checked 9/11/95
Revised

Filename U9.xls:2
By M. M. AHMED
By M. K. PAL
By

REV-1: BY: *M. Ahmed* 6.28.96
CHECKED BY: *D. L. Stod* 7-1-96

ANCHOR LOADS AT PIT WALLS

NOTE: THESE ARE THE JUMPER PIPING LOADS EXERTED ON PIT WALL FROM INSIDE THE PIT. DOES NOT INCLUDE THE TRANSFER PIPE LINE LOADS FROM OUT SIDE THE PIT.

(Inside Pit)	JUMPER LOADS		Fx (lbs)	Fy (lbs)	Fz (lbs)	Mx (in-lb)	My (in-lb)	Mz (in-lb)
Identification:		(Seismic)	132	128	295	1548	2352	948
Dwg:H-2-B18505		(*1)						
AutoPipe File: MA-C06A		(Thermal)	23	321	233	3192	24	468
Node #: A00		(*2)						
Orientation:								
 <p>(Plan View)</p>								
(*3) ABSOLUTE SUM			155	449	528	4740	2376	1416

**ICF KAISER
HANFORD COMPANY**

DESIGN ANALYSIS

Calc. No. W-320-27-016
Revision No: 0
Page No. 8 of 9

Client WESTINGHOUSE HANFORD CO.
Subject AY FARM JUMPER STRESS ANALYSIS
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location C TANK FARM 200 EAST AREA

WO/Job N ER4319
Date 9/11/95
Checked 9/11/95
Revised

Filename U9.xls:2
By M. M. AHMED
By M. K. PAL
By

REV:1 BY: *R. Ahmed* 6-28-96
CHECKED BY: *D. L. Stone* 7-1-96

NOTES:

(*1): SEISMIC LOADS COULD BE POSITIVE OR NEGATIVE.

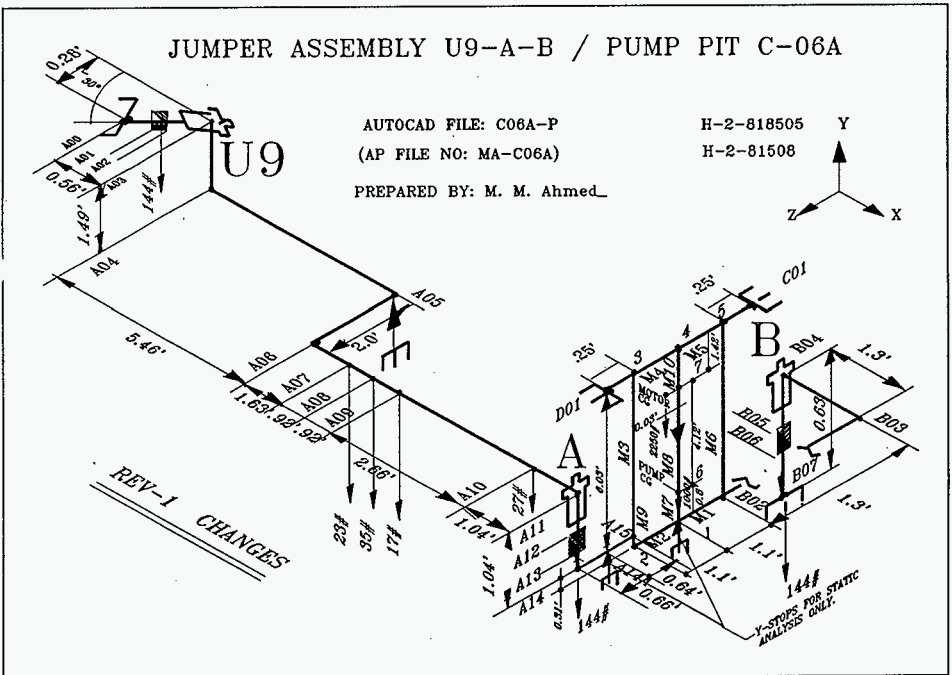
(*2): THERMAL LOADS SPECIFIED ARE THE ALGEBRIC SUM OF GRAVITY AND THERMAL LOADS.

(*3): ABSOLUTE SUM IS THE TOTAL OF THERMAL AND SEISMIC DISREGARDING THE SIGNS.

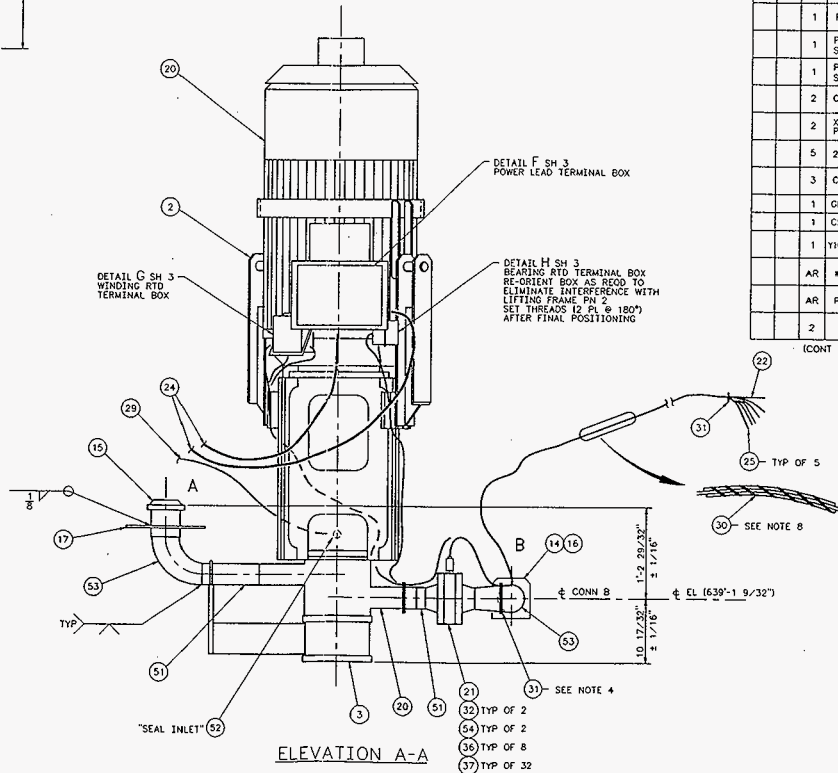
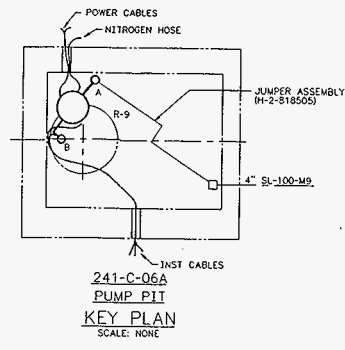
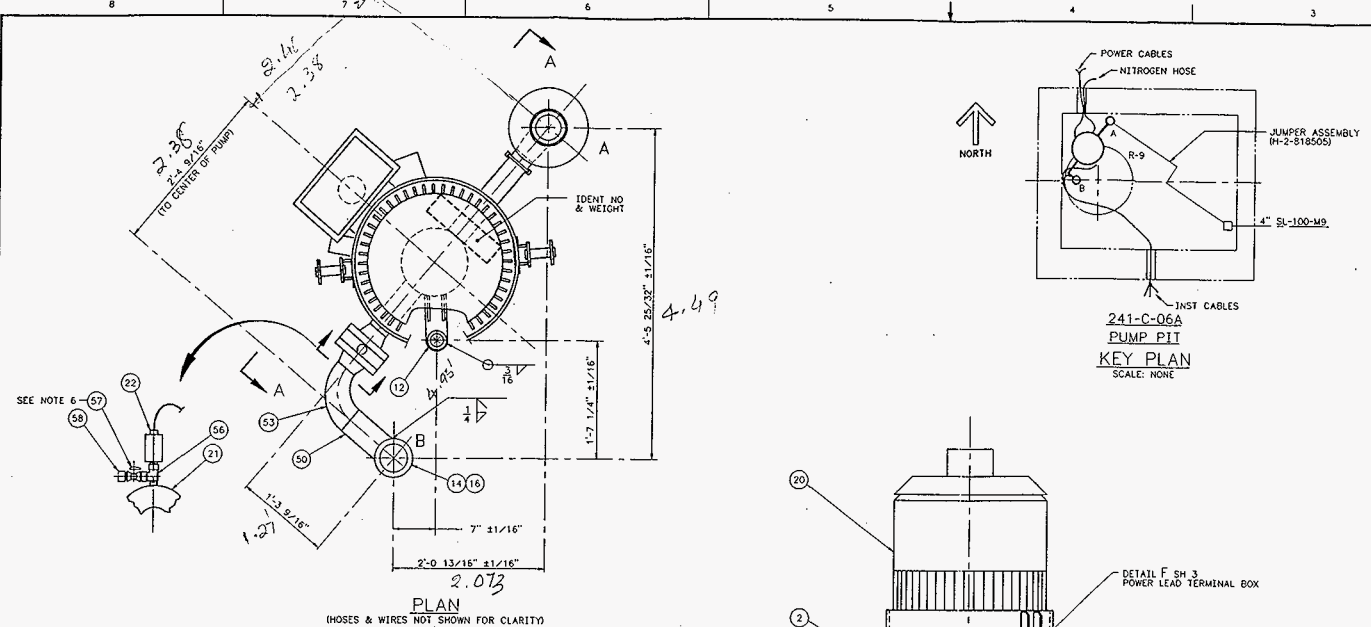
Client WESTINGHOUSE HANFORD COMPANY
 Subject AY FARM - JUMPER STRESS ANALYSIS
 PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
 Location C TANK FARM - 200 W. AREA

WO/Job No. ER4319 Filename: C06A.WP
 Date 9-11-95 By: M.M. Ahmed
 Checked 9/11/95 By: M.K. Pal
 Revised / / By:

Rev-1: By: *M. Ahmed* 6-28-96
 Checked By: *PL. S. 770* 7-1-96



APPDX - A.



PARTS/MATERIAL LIST						
QUANTITY REQUIRED	PART-DASH NUMBER	NOMENCLATURE/DESCRIPTION	MATERIAL-REFERENCE	SHEET	ITEM NO.	
X	-010	ASSEMBLY			1	
1	-020	LIFTING FRAME SUB-ASSY		2	2	
X	-030	PUMP BASE SUB-ASSY			3	
				4	4	
				5	5	
4	-001	PLATE, 3/8" THK	ASTM A 36	3	6	
4	-002	PLATE, 3/8" THK	ASTM A 36	3	7	
	-003	PLATE, 3/8" THK	ASTM A 36	3	8	
2	-004	PLATE, 1/2" THK	ASTM A 36	3	9	
1	-004	PLATE, 1/2" THK	ASTM A 36	3	9	
2	-005	PLATE, 1" THK x 11 5/8" DIA	ASTM A 276 304L	3	10	
				3	11	
1	-007	PLATE, 3/8" THK	ASTM A 36	3	12	
				3	13	
1	H-2-324-0-2	CONN, VERT 4"			14	
1	H-2-90187-2	NOZZLE, MALE 4"			15	
1	H-2-3897-8	GASKET 4" TYPE III			16	
1	H-2-3194-4	KICK-OFF FL, 4" STD ROUND			17	
					18	
1	RB350-250	CONDUIT RED BUSHING, 3 1/2" x 2 1/2"	APPLETON ELECTRIC		19	
1	P-1362	PUMP, INLINE 3" INLET & 3" OUTLET	PROCUREMENT SPEC W-320-P17		20	
1	PE-1363 SEE NOTE 6 & 9	PRESSURE SENSOR, 3"	RED VALVE CO, INC		21	
1	PE-1363 SEE NOTE 7	PRESSURE TRANSDUCER (PE) 1/4" NPT	CONSTR SPEC W-320-C5		22	
2	COB7913	CABLE FITTING, 2 1/2" MNPT x 1.375/1.625 DIA	CROUSE-HINDS		23	
2	WHW P62-0404	CABLE, 3C, 1/0 AWM W/ GROUND, RATED FOR 90°C, 30' L	ROCKWELL FIRSWALL III		24	
5	2433C	CABLE, 3C, #16 AWG W/ OVERALL SHIELD x 25' L	ALPHA		25	
3	COB293	CABLE FITTING, 3/4" MNPT x .250/.375 DIA	CROUSE-HINDS		26	
1	GL150	CABINET ADAPTER, 1 1/2"	O-Z/GEDNEY		27	
1	CSB1-15CP-1	CONDUIT SEALING BUSHING, 1 1/2"	O-Z/GEDNEY		28	
1	Y10T040H004NC-300	HOSE ASSY 1/4" ID FLEX x 25' L	IMPERIAL EASTMAN		29	
AR	#150T	CABLE TEFLON SPIRAL WRAPPING x L AS REQD	PANDUIT		30	
AR	PLT2H-	CABLE TIE, NYLON HWY CROSS SECTION x L AS REQD	PANDUIT		31	
2		GASKET, 3" FLANGE, 1/8" THK COMPRESSED FIBER GARLOCK BLUE GARD			32	

(CONT ON SHEET 2)

NOTES:

1. FABRICATION AND TESTING SHALL BE IN ACCORDANCE WITH HANFORD SPECIFICATION HS-85-0084, REV B.
2. HYDROSTATIC TEST AT 480 PSIG.
3. JUMPER WEIGHT IS 3750 LB APPROX.
4. ROUTE CABLES APPROXIMATELY AS SHOWN AND SECURE TO SUPPORT STEEL W/PN 31, AVOID IMPACT WRENCH AND LIFTING BAIL AREAS.
5. USE (1) COAT OF AMERLOCK 400 IN PLACE OF AMERCOAT 187 PRIMER AND AMERCOAT 33 WHERE REQUIRED BY JUMPER FABRICATION SPECIFICATION.
6. SEE CONSTRUCTION SPEC W-320-C5 SECTION 13400 FOR FILLING INSTRUCTION AFTER ASSEMBLY. AFTER FILLING, REMOVE HANDLE FROM BALL VALVE (PN 57).
7. TRIM CABLES FURNISHED WITH PN 22 TO 25' LONG PRIOR TO INSTALLATION.
8. TIE EACH END OF TEFLON SPIRAL WRAPPING (PN 30) WITH NYLON CABLE TIES (PN 31).
9. PRESSURE SENSOR (PN 21) SHALL BE SERIES 40, FOR CLASS 300 FLANGES, 316 SSI, EPDM SLEEVES, WITH 1/4" NPT PRESSURE TAP. FURNISH WITH 100ML OF SILICONE OIL IN A SEPARATE CONTAINER.

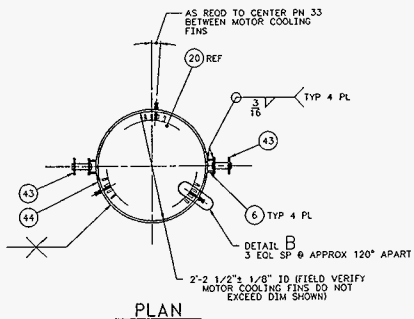


DESIGNED BY	EA MEYER	DATE	3/88	CONSTRUCTION
REVISIONS		NO.		
APPROVED BY				
U.S. DEPARTMENT OF ENERGY HEALTH, SAFETY, AND ENVIRONMENT OFFICE ICF KAMBER HANFORD COMPANY JUMPER ASSEMBLY PUMP, PIT C-06A-B-(A)				
PROJECT	W-320 TANK 241-C-106 SLUICING			
DATE	F 24HC	PLANT	8400	
DRAWING NO.	B508AR1			1
SCALE	1:1			

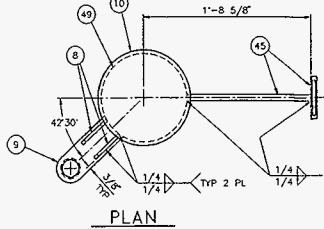
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SH 7							

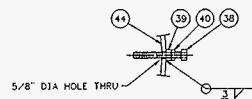
HNF-2474, Rev. 0
Page B-15



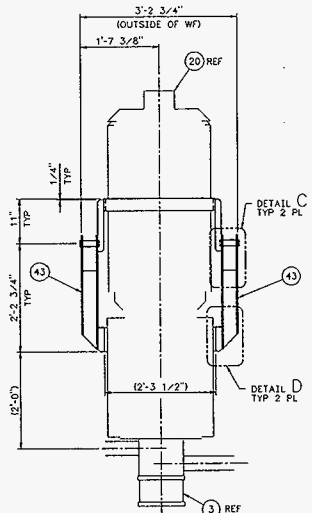
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PLAN

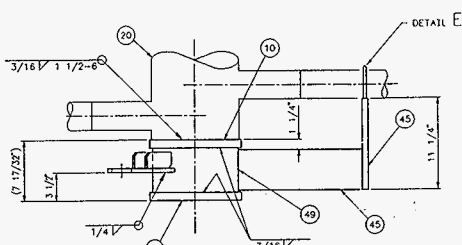


DETAIL B
SCALE: 3"=1'-0"



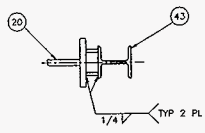
ELEVATION

2 LIFTING FRAME SUB-ASSY
SCALE: 1"=1'-0"

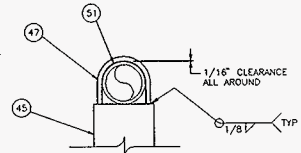


ELEVATION

3 PUMP BASE SUB-ASSY
SCALE: 2"=1'-0"

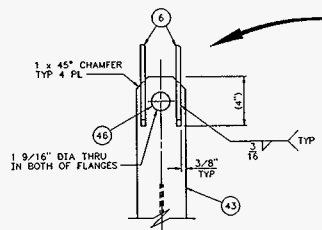


PLAN



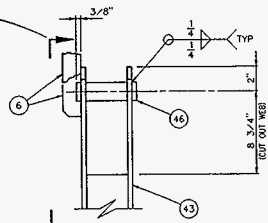
ELEVATION

DETAIL E
SCALE: 3"=1'-0"

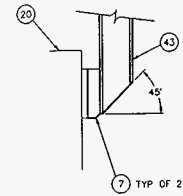


ELEVATION

DETAIL C
SCALE: 3"=1'-0"



ELEVATION



ELEVATION

DETAIL D
SCALE: 2"=1'-0"

QUANTITY REQUIRED		PARTS/MATERIAL LIST			SHEET	ITEM NO.
		PART/DASH NUMBER	NOMENCLATURE/DESCRIPTION	MATERIAL-REFERENCE		
			(CONT FROM SHEET 1)		1	1
			ASSEMBLY			
			LIFTING FRAME SUB-ASSY		2	
			PUMP BASE SUB-ASSY		3	
						33
						34
						35
	8		STUD BOLT, 3/4"-10 UNC-2A x 7 1/2" L	ASTM A 193, GR B8 304		36
	32		NUT, HWY HEX, 3/4"-10 UNC-2B	ASTM A 194, GR B8 303		37
	3		MACHINE SCREW, 1/2"-13 UNC-2A x 4 1/2" L	SAE J429 GR. 5		38
	3		NUT, HWY HEX, 1/2"-13 UNC-2B	ASTM A 563, GR. B		39
	3		NUT, JAM, 1/2"-13 UNC-2B	ASTM A 563, GR. A		40
						41
						42
	AR		W4 x 13	ASTM A 36		43
	AR		FLAT BAR, 3" x 1/2"	ASTM A 36		44
	AR		FLAT BAR, 5" x 3/4"	ASTM A 36		45
	2		BAR, 1 1/2" DIA x 4 7/8" L	ASTM A 36		46
	AR		BAR, 3/8" DIA	ASTM A 36		47
						48
	AR		PIPE, 10" SCHED 40S	ASTM A 312 GR TP 304L		49
	AR		PIPE, 4" SCHED 40S	ASTM A 312 GR TP 304L		50
	AR		PIPE, 3" SCHED 40S	ASTM A 312 GR TP 304L		51
	1	SS-8-RB-4	RED BUSHING, 1/2" NPT x 1/4" NPT	CAJON		52
	2		RDR ELB 90° LR, 4" x 3", SCHED 40S BW	ASTM A 403 WP-S 304L		53
	2		FLANGE 3", 300# RF WN	ASTM A 182 GR F 304L		54
	1		PLUG, 3/8" NPT	ASTM A 126 CCI		55
	1	SS-4-SI	STREET TEE, 1/4" NPT	CAJON		56
	1	SS-43M4	BALL VALVE, 1/4" NPT	WHITEY		57
	1	SS-4-CP	PIPE CAP, 1/4" NPT	CAJON		58

NOTE:
1. FOR NOTES, SEE SHEET 1



DESIGNED BY MEYER OF SCL	DATE	3	CONSTRUCTION
DRAWN			
CHECKED			
DATE	SEE DR/AT FORM		
SCALE	SEE DR/AT FORM		
REV	SEE DR/AT FORM		
DATE	SEE DR/AT FORM		
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REV	57	1958	
REV	58	1958	

NO.	REVISION	DATE	BY	APP	DESCRIPTION
1					
2					
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7					
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9					
10					

NO.	REVISION	DATE	BY	APP	DESCRIPTION
1					
2					
3					
4					
5					
6					
7					
8					
9					
10					

PROJECT NO.	W-320	TANK	241-C-106 SLUICING
DATE	12-1-58	SCALE	
REV	1	DATE	12-1-58
REV	2	DATE	12-1-58
REV	3	DATE	12-1-58
REV	4	DATE	12-1-58
REV	5	DATE	12-1-58
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REV	26	DATE	12-1-58
REV	27	DATE	12-1-58
REV	28	DATE	12-1-58
REV	29	DATE	12-1-58
REV	30	DATE	12-1-58
REV	31	DATE	12-1-58
REV	32	DATE	12-1-58
REV	33	DATE	12-1-58
REV	34	DATE	12-1-58
REV	35	DATE	12-1-58
REV	36	DATE	12-1-58
REV	37	DATE	12-1-58
REV	38	DATE	12-1-58
REV	39	DATE	12-1-58
REV	40	DATE	12-1-58
REV	41	DATE	12-1-58
REV	42	DATE	12-1-58
REV	43	DATE	12-1-58
REV	44	DATE	12-1-58
REV	45	DATE	12-1-58
REV	46	DATE	12-1-58
REV	47	DATE	12-1-58
REV	48	DATE	12-1-58
REV	49	DATE	12-1-58
REV	50	DATE	12-1-58
REV	51	DATE	12-1-58
REV	52	DATE	12-1-58
REV	53	DATE	12-1-58
REV	54	DATE	12-1-58
REV	55	DATE	12-1-58
REV	56	DATE	12-1-58
REV	57	DATE	12-1-58
REV	58	DATE	12-1-58

POINT DATA LISTING

POINT NAME	TYPE	-----OFFSETS (ft)-----			PIPE ID	DESCRIPTION
		X	Y	Z		
*** SEGMENT A						
A00	Run	0	0	0	4PIPE	
A01	Flex	0.22	0	-0.11		Wt= 144 lb, Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
A02	Run	0.04	0	-0.02		
A03	Bend	0.30	0	-0.15		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
A04	Bend	0	-1.49	0		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A05	Bend	5.46	0	0		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A06	Bend	0	0	2.00		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A07	Run	1.63	0	0		
A08	Run	0.92	0	0		
A09	Run	0.92	0	0		
A10	Run	2.66	0	0		
A11	Bend	1.04	0	0		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
A12	Flex	0	-0.40	0		Wt= 144 lb, Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
A13	Run	0	-0.04	0		
A14	Bend	0	-0.60	0		Long Elbow, Radius = 6.00 inch

POINT DATA LISTING

POINT NAME	TYPE	-----OFFSETS (ft)-----			PIPE ID	DESCRIPTION
		X	Y	Z		
Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272						
A14	F Far				3PIPE	
A14A	Run	0	0	-0.66		
A15	Run	0	0	-0.64		
*** SEGMENT B						
B02	Run	13.19	-2.84	-1.78	3PIPE	
B03	Bend	0	0	-1.30		Long Elbow, Radius = 4.50 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.78, Out = 1.48 Flex = 4.577
B03	F Far				4PIPE	
B04	Bend	-1.30	0	0		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
B05	Flex	0	-0.40	0		Wt= 144 lb, Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
B06	Run	0	-0.04	0		
B07	Run	0	-0.19	0		
*** SEGMENT C						
5	Run	13.19	3.50	-1.78	3.5PIPE	
C01	Run	0	0	-0.25		
*** SEGMENT D						
3	Run	13.19	3.50	0.42	3.5PIPE	
D01	Run	0	0	0.25		
Total weight of empty pipes : 254 lb						

COMPONENT DATA LISTING

POINT NAME	---COORDINATE(ft) X Y Z	--- DATA TYPE DESCRIPTION
*** SEGMENT A		
A00	0.00 0.00 0.00	ANCHOR Rigid Thermal movements : None
A01	0.22 0.00 -0.11	
A02	0.26 0.00 -0.13	
A03 N	0.26 0.00 -0.13	
A03	0.56 0.00 -0.28	TI
A03 M	0.47 -0.10 -0.24	
A03 F	0.56 -0.33 -0.28	
A04 N	0.56 -0.99 -0.28	
A04	0.56 -1.49 -0.28	TI
A04 M	0.71 -1.34 -0.28	
A04 F	1.06 -1.49 -0.28	
A05 N	5.52 -1.49 -0.28	
A05	6.02 -1.49 -0.28	TI
A05 M	5.87 -1.49 -0.13	
A05 F	6.02 -1.49 0.22	
A06 N	6.02 -1.49 1.22	
A06	6.02 -1.49 1.72	TI
A06 M	6.17 -1.49 1.57	
A06 F	6.52 -1.49 1.72	
A07	7.65 -1.49 1.72	WEIGHT 23 lb , No offsets
A08	8.57 -1.49 1.72	WEIGHT 35 lb , No offsets
		FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb
		FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb
A09	9.49 -1.49 1.72	WEIGHT 17 lb , No offsets
A10	12.15 -1.49 1.72	WEIGHT 27 lb , No offsets
		FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb
		FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb
A11 N	12.86 -1.49 1.72	
A11	13.19 -1.49 1.72	TI
A11 M	13.09 -1.59 1.72	
A11 F	13.19 -1.82 1.72	
A12	13.19 -1.89 1.72	
A13	13.19 -1.93 1.72	
A14 N	13.19 -2.03 1.72	
A14	13.19 -2.53 1.72	TI
A14 M	13.19 -2.38 1.57	
A14 F	13.19 -2.53 1.22	
A14A	13.19 -2.53 1.06	
A15	13.19 -2.53 0.42	

*** SEGMENT B		
B02	13.19 -2.84 -1.78	
B03 N	13.19 -2.84 -2.70	

COMPONENT DATA LISTING

POINT NAME	---COORDINATE(ft) X Y Z	--- DATA TYPE DESCRIPTION
B03	13.19 -2.84 -3.08	TI
B03 M	13.08 -2.84 -2.97	
B03 F	12.81 -2.84 -3.08	
B04 N	12.22 -2.84 -3.08	
B04	11.89 -2.84 -3.08	TI
B04 M	11.99 -2.94 -3.08	
B04 F	11.89 -3.17 -3.08	
B05	11.89 -3.24 -3.08	
B06	11.89 -3.28 -3.08	
B07	11.89 -3.47 -3.08	ANCHOR Rigid Thermal movements : None
*** SEGMENT C		
5	13.19 3.50 -1.78	ANCHOR Translational stiffness lb/in X= FREE Y= FREE Z= FREE Rotational stiffness ft-lb/deg X= RIGID Y= RIGID Z= RIGID Thermal movements : None
C01	13.19 3.50 -2.03	ANCHOR Rigid Thermal movements : None
*** SEGMENT D		
3	13.19 3.50 0.42	ANCHOR Translational stiffness lb/in X= FREE Y= FREE Z= FREE Rotational stiffness ft-lb/deg X= RIGID Y= RIGID Z= RIGID Thermal movements : None
D01	13.19 3.50 0.67	ANCHOR Rigid Thermal movements : None

Number of points in the system : 51

PIPE DATA LISTING

Pipe ID/ Material	Nom/ Sch	O.D. inch	-----Thickness(inch)-----				Spec Ling	Spec Grav	Weight(lb/ft)	
			W.Th.	Corr	Mill	Insu			Pipe	Other Total
4PIPE AU	4 STD	4.500	0.237	0.06	0.03	0	0	1.20	11.04	0 17.66
3.5PIPE AU	3.500 STD	4.000	0.226	0.06	0.03	0	0	0	9.32	0 9.32
3PIPE AU	3.000 STD	3.500	0.216	0.06	0.03	0	0	1.20	7.75	0 11.60

MATERIAL DATA LISTING

Material Name	Pipe ID	Density lb/cu.ft	Pois. Ratio	Temper. deg f	Modulus E6 psi	Expans. in/100ft	Allow. psi
AU	3PIPE	501.0	0.30	40.0 180.0	28.42	1.5442	16700.0 16700.0
AU	3.5PIPE	501.0	0.30	40.0 180.0	28.42	1.5442	16700.0 16700.0

FRAME POINT DATA LISTING

POINT NAME	---COORDINATE(ft)---			DATA TYPE	DESCRIPTION
	X	Y	Z		
A15	13.19	-2.53	0.42		
1	13.19	-2.84	-0.68		
B02	13.19	-2.84	-1.78		
5	13.19	3.50	-1.78	ANCHOR	Translational stiffness lb/in X= FREE Y= FREE Z= FREE Rotational stiffness ft-lb/deg X= RIGID Y= RIGID Z= RIGID Thermal movements : None
4	13.19	3.50	-0.68		
3	13.19	3.50	0.42		
6	13.19	-2.04	-0.68	WEIGHT 1000 lb	, No offsets
2	13.19	-2.84	0.42		
7	13.19	2.08	-0.68	WEIGHT 2250 lb	Offsets in ft DX = 0.00, DY = 0.00, DZ = 0.03

BEAM DATA LISTING

BEAM ID	POINT NAME	LENGTH (ft)	SECTION ID/ MATERIAL ID	BETA ANGLE	RIGID END(ft)	RELEASE		
						Ax	Y-Y	Z-Z
M2	From 2 To 1	1.10	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M9	From 2 To A15	0.31	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M1	From 1 To B02	1.10	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M6	From 5 To B02	6.34	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M3	From A15 To 3	6.03	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M4	From 3 To 4	1.10	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M5	From 4 To 5	1.10	W4X13 A36	0.00	0.00 0.00	N N	N N	N N
M7	From 1 To 6	0.80	XS12 A36	0.00	0.00 0.00	N N	N N	N N
M8	From 6 To 7	4.12	XS12 A36	0.00	0.00 0.00	N N	N N	N N
M10	From 4 To 7	1.42	XS12 A36	0.00	0.00 0.00	N N	N N	N N

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CROSS SECTION LISTING

Section ID	Area(sq.in)			Inertia (in**4)		
	Axial	Y-Shear	Z-Shear	Torsion	Y-Y Bend	Z-Z Bend
W4X13	3.83	1.16	1.86	0.2	3.9	11.3
XS12	19.20	9.60	9.60	724.0	362.0	362.0

BEAM MATERIAL LISTING

MATERIAL ID	Elastic modulus E6 psi	Poissons ratio	Yield Stress psi	Density lb/cu.ft	Expansion E-6 /F	Ultimate stress psi
A36	29.000	0.250	36000	490.00	6.50000	58000

TEMPERATURE AND PRESSURE DATA

POINT NAME	CASE 1			CASE 2			CASE 3		
	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft

*** SEGMENT A
A00 325 180 1.544
A14 F 325 180 1.544
A15 325 180 1.544

*** SEGMENT B
B02 325 180 1.544
B03 F 325 180 1.544
B07 325 180 1.544

*** SEGMENT C
5 490 180 1.544
C01 490 180 1.544

*** SEGMENT D
3 490 180 1.544
D01 490 180 1.544

ANALYSIS SUMMARY

Current model revision number : 24

Static - Date and Time of analysis Jun 26, 1996 11:08 AM
Model Revision Number 23 ***
Number of load cases 2
Load cases analyzed GR T1
Gaps/Friction/Yielding considered No
Hanger design run No
Cut short included No
Weight of contents included Yes
Pressure stiffening case 0
Water elevation for buoyancy loads Not considered

Modal - Date and Time of analysis Jun 26, 1996 11:09 AM
Model Revision Number 24
Number of modes 6
Cutoff frequency (Hz) 33.0
Weight of contents included Yes
Pressure stiffening case 0
Water elevation for buoyancy loads Not considered

Response - Date and Time of analysis Jun 26, 1996 11:09 AM
Model Revision Number 24
Number of load cases 3
Load cases analyzed R1 R2 R3
Date and time of modal analysis Jun 26, 1996 11:09 AM
Number of modes 6
Cutoff frequency (Hz) 33.0
Model revision of modal analysis 24
Weight of contents included Yes
Pressure stiffening case 0
Water elevation for buoyancy loads Not considered

*** WARNING

Current model revision does not match analysis

CODE COMPLIANCE COMBINATIONS

Combination	Category	Method	Load	Factor	Allowable	Remarks
GR + Max P	Sustain	Sum	Gravity Max Long	1.00 1.00	Automatic	Default
Cold to T1	Expansion	Sum	Thermal 1	1.00	Automatic	Default
Sus. + R1	Occasion	Abs sum	Response 1 Max Sus	1.00 1.00	Automatic	Default
Sus. + R2	Occasion	Abs sum	Response 2 Max Sus	1.00 1.00	Automatic	Default
Sus. + R3	Occasion	Abs sum	Response 3 Max Sus	1.00 1.00	Automatic	Default
Max P	Hoop		Max Hoop	1.00	Automatic	Default
SRSS	Occasion	SRSS	Response 1 Response 2 Response 3	1.00 1.00 1.00	Automatic	User

OTHER USER COMBINATIONS

Combination	Method	Load	Factor	Remarks
GR	Sum	Gravity	1.00	Default
T1	Sum	Thermal 1	1.00	Default
R1	Sum	Response 1	1.00	Default
R2	Sum	Response 2	1.00	Default
R3	Sum	Response 3	1.00	Default
GR+T1	Sum	Gravity Thermal 1	1.00 1.00	User
RESPONSE	SRSS	Response 1 Response 2 Response 3	1.00 1.00 1.00	User
TOTAL	Abs sum	GR+T1 RESPONSE	1.00 1.00	User

CODE COMPLIANCE

Y - Factor	0.40
Weld efficiency factor	1.00
Range reduction factor	1.00
Design Pressure Factor	1.00
Minimum stress ratio used in reports	0.00
Include corrosion in stress calcs.	Y
Include torsion in code stress	N
Include axial force in code stress	N
Longitudinal pressure calculation	PD/4t
Include rigorous pressure	Not analyzed

RESPONSE SPECTRUM LOAD CASES :

Number of load cases analysed : 3

Load case 1 - R1

Missing mass : Yes
 ZPA : No

Combination method : SRSS

X- Spectrum : SC1
 Multiplier : 1.00

SC1

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.03	0.250	0.06
0.400	0.10	0.600	0.16	1.100	0.28
1.640	0.42	8.000	0.42	12.000	0.34
20.000	0.26	33.000	0.20	100.000	0.20

Load case 2 - R2

Missing mass : Yes
 ZPA : No

Combination method : SRSS

Y- Spectrum : SC1
 Multiplier : 1.00

SC1

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.03	0.250	0.06
0.400	0.10	0.600	0.16	1.100	0.28
1.640	0.42	8.000	0.42	12.000	0.34
20.000	0.26	33.000	0.20	100.000	0.20

Load case 3 - R3

Missing mass : Yes
ZPA : No

Combination method : SRSS

Z- Spectrum : SC1
Multiplier : 1.00

SC1

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.03	0.250	0.06
0.400	0.10	0.600	0.16	1.100	0.28
1.640	0.42	8.000	0.42	12.000	0.34
20.000	0.26	33.000	0.20	100.000	0.20

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
*** Segment A begin ***							
A00	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
	RESPONSE TOTAL	0.000	0.000	0.000	0.000	0.000	0.000
A01	GR	0.000	0.000	0.000	0.000	0.000	-0.002
	T1	0.003	0.000	-0.002	0.000	0.000	-0.001
	GR+T1	0.003	0.000	-0.002	0.000	0.000	-0.002
	RESPONSE TOTAL	0.000	0.000	0.000	0.001	0.002	0.001
A02	GR	0.000	0.000	0.000	0.153	-0.015	-0.124
	T1	0.003	0.000	-0.002	0.058	0.000	-0.076
	GR+T1	0.003	-0.001	-0.002	0.210	-0.015	-0.200
	RESPONSE TOTAL	0.000	0.000	0.000	0.161	0.078	0.097
A03 N	GR	0.000	0.000	0.000	0.153	-0.015	-0.124
	T1	0.003	0.000	-0.002	0.058	0.000	-0.076
	GR+T1	0.003	-0.001	-0.002	0.210	-0.015	-0.200
	RESPONSE TOTAL	0.000	0.000	0.000	0.161	0.078	0.097
A03 M	GR	-0.002	-0.003	-0.002	0.151	-0.015	-0.128
	T1	0.005	-0.004	-0.004	0.055	-0.002	-0.082
	GR+T1	0.003	-0.007	-0.007	0.206	-0.017	-0.211
	RESPONSE TOTAL	0.003	0.004	0.006	0.164	0.088	0.102
A03 F	GR	-0.009	-0.004	-0.010	0.154	-0.014	-0.132
	T1	0.002	-0.009	-0.009	0.051	-0.005	-0.085
	GR+T1	-0.006	-0.012	-0.017	0.205	-0.018	-0.217
	RESPONSE TOTAL	0.008	0.005	0.015	0.167	0.091	0.106
A04 N	GR	-0.027	-0.004	-0.031	0.156	-0.015	-0.134
	T1	-0.010	-0.019	-0.015	0.050	-0.005	-0.083
	GR+T1	-0.036	-0.022	-0.046	0.206	-0.020	-0.217
	RESPONSE TOTAL	0.022	0.005	0.037	0.168	0.095	0.105
		0.058	0.028	0.083	0.373	0.115	0.321

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A04 M	GR	-0.037	-0.008	-0.042	0.159	-0.014	-0.142
	T1	-0.013	-0.026	-0.018	0.045	-0.008	-0.063
	GR+T1	-0.050	-0.034	-0.060	0.204	-0.023	-0.205
	RESPONSE TOTAL	0.029	0.008	0.051	0.166	0.097	0.097
A04 F	GR	-0.041	-0.019	-0.046	0.161	-0.014	-0.147
	T1	-0.009	-0.032	-0.019	0.041	-0.014	-0.037
	GR+T1	-0.050	-0.051	-0.064	0.202	-0.029	-0.184
	RESPONSE TOTAL	0.032	0.014	0.060	0.165	0.101	0.091
A05 N	GR	-0.041	-0.139	-0.032	0.175	-0.013	-0.093
	T1	0.060	-0.046	0.005	0.025	-0.041	0.004
	GR+T1	0.018	-0.185	-0.027	0.200	-0.054	-0.089
	RESPONSE TOTAL	0.032	0.087	0.131	0.158	0.083	0.060
A05 M	GR	-0.042	-0.150	-0.032	0.165	-0.011	-0.052
	T1	0.063	-0.046	0.011	0.020	-0.058	0.015
	GR+T1	0.022	-0.196	-0.021	0.185	-0.069	-0.036
	RESPONSE TOTAL	0.033	0.092	0.135	0.153	0.069	0.038
A05 F	GR	-0.042	-0.163	-0.031	0.155	-0.010	-0.033
	T1	0.061	-0.046	0.018	0.013	-0.070	0.021
	GR+T1	-0.018	-0.209	-0.012	0.169	-0.080	-0.012
	RESPONSE TOTAL	0.036	0.099	0.136	0.147	0.067	0.030
A06 N	GR	-0.045	-0.196	-0.031	0.154	-0.011	-0.001
	T1	0.046	-0.049	0.034	0.012	-0.068	0.027
	GR+T1	0.001	-0.244	0.003	0.166	-0.079	0.027
	RESPONSE TOTAL	0.048	0.118	0.136	0.145	0.065	0.021
A06 M	GR	-0.046	-0.206	-0.031	0.137	-0.016	0.020
	T1	0.044	-0.049	0.041	0.009	-0.047	0.032
	GR+T1	-0.002	-0.255	0.010	0.146	-0.063	0.052
	RESPONSE TOTAL	0.052	0.125	0.135	0.135	0.057	0.024

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A06 F	GR	-0.046	-0.207	-0.029	0.122	-0.023	0.068
	T1	0.048	-0.046	0.046	0.007	-0.020	0.040
	GR+T1	0.002	-0.253	0.016	0.129	-0.043	0.108
	RESPONSE TOTAL	0.053	0.126	0.132	0.128	0.062	0.045
A07	GR	-0.046	-0.187	-0.024	0.117	-0.026	0.098
	T1	0.066	-0.036	0.049	0.007	-0.006	0.044
	GR+T1	0.019	-0.224	0.025	0.124	-0.032	0.142
	RESPONSE TOTAL	0.053	0.114	0.118	0.123	0.073	0.061
A08	GR	-0.046	-0.166	-0.018	0.113	-0.028	0.124
	T1	0.080	-0.028	0.049	0.007	0.004	0.046
	GR+T1	0.034	-0.194	0.031	0.120	-0.024	0.170
	RESPONSE TOTAL	0.053	0.101	0.104	0.119	0.084	0.075
A09	GR	-0.046	-0.140	-0.013	0.109	-0.030	0.147
	T1	0.094	-0.019	0.047	0.007	0.013	0.047
	GR+T1	0.048	-0.158	0.035	0.116	-0.017	0.194
	RESPONSE TOTAL	0.053	0.086	0.088	0.115	0.095	0.088
A10	GR	-0.046	-0.044	0.005	0.097	-0.034	0.186
	T1	0.135	0.007	0.034	0.008	0.033	0.042
	GR+T1	0.089	-0.038	0.039	0.105	-0.001	0.228
	RESPONSE TOTAL	0.053	0.030	0.033	0.103	0.115	0.108
A11 N	GR	-0.046	-0.017	0.010	0.094	-0.035	0.186
	T1	0.146	0.013	0.029	0.008	0.037	0.039
	GR+T1	0.100	-0.004	0.039	0.102	0.002	0.225
	RESPONSE TOTAL	0.053	0.015	0.022	0.101	0.116	0.107
A11 M	GR	-0.046	-0.019	0.061	0.153	0.019	0.332
	T1	-0.042	-0.008	0.010	0.091	-0.038	0.176
	GR+T1	0.150	0.013	0.026	0.011	0.046	0.029
	RESPONSE TOTAL	0.108	0.005	0.037	0.102	0.007	0.205

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A11 F	GR	-0.034	-0.004	0.007	0.085	-0.040	0.160
	T1	0.153	0.010	0.025	0.013	0.049	0.023
	GR+T1	0.119	0.005	0.032	0.098	0.008	0.183
	RESPONSE	0.050	0.008	0.016	0.093	0.118	0.094
TOTAL	0.169	0.014	0.048	0.191	0.126	0.277	
A12	GR	-0.032	-0.004	0.006	0.085	-0.040	0.160
	T1	0.153	0.009	0.025	0.013	0.049	0.023
	GR+T1	0.121	0.004	0.030	0.098	0.008	0.182
	RESPONSE	0.049	0.008	0.015	0.092	0.118	0.094
TOTAL	0.171	0.013	0.046	0.190	0.126	0.276	
A13	GR	-0.031	-0.004	0.005	0.046	-0.096	0.058
	T1	0.153	0.009	0.025	0.009	0.418	0.016
	GR+T1	0.122	0.004	0.030	0.054	0.323	0.074
	RESPONSE	0.049	0.008	0.015	0.053	0.134	0.043
TOTAL	0.172	0.013	0.045	0.107	0.457	0.117	
A14 N	GR	-0.030	-0.004	0.004	0.045	-0.096	0.057
	T1	0.153	0.007	0.024	0.008	0.419	0.016
	GR+T1	0.124	0.003	0.028	0.054	0.323	0.073
	RESPONSE	0.049	0.008	0.014	0.053	0.135	0.043
TOTAL	0.173	0.011	0.043	0.106	0.458	0.116	
A14 M	GR	-0.023	-0.003	0.001	0.037	-0.092	0.041
	T1	0.142	0.002	0.022	0.007	0.420	0.020
	GR+T1	0.119	-0.001	0.023	0.044	0.329	0.061
	RESPONSE	0.046	0.007	0.013	0.044	0.134	0.036
TOTAL	0.164	0.008	0.036	0.087	0.462	0.097	
A14 F	GR	-0.015	-0.001	0.000	0.018	-0.087	0.033
	T1	0.111	0.000	0.016	0.003	0.419	0.024
	GR+T1	0.096	-0.001	0.016	0.022	0.332	0.057
	RESPONSE	0.038	0.004	0.013	0.030	0.132	0.034
TOTAL	0.134	0.005	0.029	0.052	0.464	0.091	
A14A	GR	-0.013	0.000	0.000	0.013	-0.086	0.028
	T1	0.097	0.000	0.013	0.003	0.418	0.027
	GR+T1	0.085	0.000	0.013	0.016	0.331	0.055
	RESPONSE	0.034	0.003	0.013	0.027	0.132	0.032
TOTAL	0.119	0.003	0.027	0.043	0.463	0.087	

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A15	GR	-0.001	0.000	0.000	0.000	-0.084	0.008
	T1	0.042	0.000	0.004	-0.001	0.407	0.040
	GR+T1	0.041	0.000	0.004	-0.001	0.323	0.049
	RESPONSE	0.025	0.001	0.013	0.011	0.129	0.029
TOTAL	0.065	0.001	0.017	0.012	0.452	0.078	
*** Segment A end ***							
*** Segment B begin ***							
B02	GR	0.000	0.000	0.000	-0.001	0.000	0.000
	T1	0.033	0.000	0.004	0.003	0.029	0.028
	GR+T1	0.035	0.000	0.004	0.002	0.029	0.028
	RESPONSE	0.016	0.001	0.013	0.003	0.048	0.016
TOTAL	0.049	0.001	0.017	0.005	0.077	0.044	
B03 N	GR	0.000	0.000	0.000	-0.001	0.000	0.000
	T1	0.026	0.002	-0.011	0.011	0.040	0.003
	GR+T1	0.026	0.002	-0.011	0.010	0.040	0.003
	RESPONSE	0.005	0.001	0.013	0.001	0.058	0.012
TOTAL	0.031	0.003	0.024	0.012	0.098	0.015	
B03 M	GR	0.000	0.000	0.000	-0.001	0.000	-0.001
	T1	0.022	0.003	-0.014	0.015	0.054	-0.007
	GR+T1	0.022	0.002	-0.014	0.015	0.054	-0.008
	RESPONSE	0.002	0.001	0.012	0.004	0.053	0.009
TOTAL	0.024	0.003	0.026	0.018	0.107	0.017	
B03 F	GR	0.000	0.000	0.000	-0.001	0.000	-0.001
	T1	0.016	0.004	-0.012	0.017	0.066	-0.024
	GR+T1	0.017	0.004	-0.012	0.017	0.065	-0.025
	RESPONSE	0.001	0.001	0.010	0.007	0.049	0.004
TOTAL	0.018	0.004	0.021	0.024	0.115	0.029	
B04 N	GR	0.000	0.000	0.000	-0.001	0.000	-0.001
	T1	0.007	0.007	-0.004	0.015	0.067	-0.026
	GR+T1	0.007	0.007	-0.004	0.015	0.067	-0.027
	RESPONSE	0.001	0.000	0.004	0.008	0.049	0.003
TOTAL	0.008	0.008	0.007	0.023	0.116	0.030	
B04 M	GR	0.000	0.000	0.000	-0.001	0.000	-0.002
	T1	0.003	0.007	-0.001	0.014	0.068	-0.026
	GR+T1	0.003	0.007	-0.001	0.013	0.068	-0.027
	RESPONSE	0.001	0.000	0.001	0.010	0.049	0.008
TOTAL	0.004	0.007	0.002	0.023	0.117	0.035	

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
B04 F	GR	0.000	0.000	0.000	0.000	0.000	-0.001
	T1	0.000	0.004	0.000	0.011	0.069	-0.023
	GR+T1	0.000	0.004	0.000	0.011	0.069	-0.024
	RESPONSE	0.000	0.000	0.000	0.011	0.048	0.011
	TOTAL	0.001	0.004	0.000	0.021	0.117	0.035
B05	GR	0.000	0.000	0.000	0.000	0.000	-0.001
	T1	0.000	0.003	0.000	0.011	0.069	-0.022
	GR+T1	0.000	0.003	0.000	0.011	0.068	-0.024
	RESPONSE	0.000	0.000	0.000	0.011	0.048	0.011
	TOTAL	0.000	0.003	0.000	0.021	0.117	0.035
B06	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.003	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.003	0.000	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.003	0.000	0.000	0.000	0.001
B07	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment B end ***							
*** Segment C begin ***							
5	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.003	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.003	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.003	0.000	0.000	0.000
C01	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment C end ***							
** Segment D begin ***							

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
3	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	-0.003	0.000	0.000	0.000
	GR+T1	0.000	0.000	-0.003	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.003	0.000	0.000	0.000
D01	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment D end ***							
A15	GR	-0.001	0.000	0.000	0.000	-0.084	0.008
	T1	0.042	0.000	0.004	-0.001	0.407	0.040
	GR+T1	0.041	0.000	0.004	-0.001	0.323	0.049
	RESPONSE	0.025	0.001	0.013	0.011	0.129	0.029
	TOTAL	0.065	0.001	0.017	0.012	0.452	0.078
1	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.038	0.000	0.004	-0.002	0.013	0.029
	GR+T1	0.038	0.000	0.004	-0.003	0.012	0.029
	RESPONSE	0.024	0.000	0.013	0.009	0.012	0.017
	TOTAL	0.062	0.000	0.017	0.011	0.025	0.046
B02	GR	0.000	0.000	0.000	-0.001	0.000	0.000
	T1	0.033	0.000	0.004	0.003	0.029	0.028
	GR+T1	0.033	0.000	0.004	0.002	0.029	0.028
	RESPONSE	0.016	0.001	0.013	0.003	0.048	0.016
	TOTAL	0.049	0.001	0.017	0.005	0.077	0.044
5	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.003	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.003	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.003	0.000	0.000	0.000
4	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.000	0.000	0.000	-0.002	0.009	0.029
	GR+T1	0.000	0.000	0.000	-0.002	0.008	0.029
	RESPONSE	0.000	0.000	0.000	0.009	0.008	0.018
	TOTAL	0.000	0.001	0.000	0.011	0.017	0.047

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
3	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	-0.003	0.000	0.000	0.000
	GR+T1	0.000	0.000	-0.003	0.000	0.000	0.000
	RESPONSE	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.003	0.000	0.000	0.000
6	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.033	0.000	0.003	-0.003	0.013	0.029
	GR+T1	0.033	0.000	0.003	-0.003	0.012	0.029
	RESPONSE	0.021	0.001	0.012	0.009	0.012	0.017
	TOTAL	0.054	0.001	0.015	0.012	0.024	0.046
2	GR	-0.001	0.000	0.000	-0.001	-0.005	0.008
	T1	0.044	0.000	0.004	-0.001	0.039	0.040
	GR+T1	0.044	0.000	0.004	-0.001	0.034	0.049
	RESPONSE	0.026	0.001	0.013	0.005	0.016	0.029
	TOTAL	0.070	0.001	0.017	0.007	0.051	0.077
7	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.009	0.000	0.001	-0.003	0.010	0.029
	GR+T1	0.009	0.000	0.001	-0.003	0.009	0.029
	RESPONSE	0.006	0.001	0.003	0.010	0.009	0.018
	TOTAL	0.014	0.001	0.004	0.013	0.019	0.047

SUPPORT FORCES

Point/ Supp. ID	Connect/ Type	Load Combination	Dirn	LOCAL		GLOBAL	
				Force	Deform	Dirn	Force Deform

*** End of system , no supports encountered. ***

RESTRAINT REACTIONS

Point name	Load combination	FORCES (lb)			Result	MOMENTS (ft-lb)			Result
		X	Y	Z		X	Y	Z	
A00	Anchor								
	GR	57	-358	9	362	-28	-39	-186	193
	T1	-290	377	-32	294	-11	41	-80	91
	GR+T1	-233	-321	-23	397	-39	2	-266	269
	RESPONSE	295	128	132	347	79	196	129	248
	TOTAL	528	449	155	710	118	198	395	458
B07	Anchor								
	GR	5	-167	2	167	-1	0	-4	4
	T1	8	-197	-103	222	1	21	-48	52
	GR+T1	13	-363	-102	378	1	21	-52	56
	RESPONSE	326	86	169	377	56	15	90	107
	TOTAL	339	450	270	625	56	35	142	156
5	Anchor								
	GR	0	0	0	0	50	4	2	50
	T1	0	0	0	0	-4	-72	284	293
	GR+T1	0	0	0	0	46	-69	286	298
	RESPONSE	0	0	0	0	94	60	123	166
	TOTAL	0	0	0	0	140	129	409	451
C01	Anchor								
	GR	22	-231	-6	232	29	3	0	29
	T1	-279	-14	-24319	24321	2	-35	0	35
	GR+T1	-257	-245	-24325	24328	31	-32	0	44
	RESPONSE	281	1063	316	1142	333	35	0	137
	TOTAL	538	1307	24639	24680	163	67	0	177
3	Anchor								
	GR	0	0	0	0	-80	13	-134	156
	T1	0	0	0	0	-41	-167	748	767
	GR+T1	0	0	0	0	-121	-155	614	645
	RESPONSE	0	0	0	0	200	240	397	506
	TOTAL	0	0	0	0	321	395	1011	1132
D01	Anchor								
	GR	-84	154	-5	175	19	10	0	22
	T1	561	262	24455	24463	33	-70	0	77
	GR+T1	477	416	24450	24458	52	-60	0	79
	RESPONSE	560	1180	316	1344	148	70	0	163
	TOTAL	1037	1596	24766	24839	200	130	0	238

RESTRAINT REACTIONS

Point name	Load combination	FORCES (lb)			Result	MOMENTS (ft-lb)			Result
		X	Y	Z		X	Y	Z	
5	Anchor								
	GR	0	0	0	0	50	4	2	50
	T1	0	0	0	0	-4	-72	284	293
	GR+T1	0	0	0	0	46	-69	286	298
	RESPONSE	0	0	0	0	94	60	123	166
	TOTAL	0	0	0	0	140	129	409	451

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)			Result	MOMENTS (ft-lb)			Result
		X	Y	Z		X	Y	Z	
*** Segment A begin ***									
A00	GR	-57	358	-9	362	28	39	186	193
	T1	290	-37	32	294	11	-41	80	91
	GR+T1	233	321	23	397	39	-2	266	269
	RESPONSE	295	128	23	347	79	196	129	248
	TOTAL	528	448	155	710	118	198	395	458
A01	GR	-57	209	-9	217	-11	31	108	113
	T1	290	-37	32	294	15	-2	88	89
	GR+T1	233	172	23	291	4	29	196	198
	RESPONSE	290	115	121	335	75	161	118	213
	TOTAL	523	288	145	614	80	190	314	376
A02	GR	-57	209	-9	217	-16	29	97	103
	T1	290	-37	32	294	16	5	90	91
	GR+T1	233	172	23	291	-1	34	186	190
	RESPONSE	287	109	117	329	75	155	117	208
	TOTAL	520	281	140	608	76	190	303	366
A03 N	GR	-57	209	-9	217	-17	29	97	102
	T1	290	-37	32	294	16	5	90	91
	GR+T1	233	172	23	291	-1	35	186	189
	RESPONSE	287	109	117	329	75	155	117	208
	TOTAL	520	281	140	608	76	190	303	365
A03 M	GR	-57	205	-9	212	-39	21	58	74
	T1	290	-37	32	294	23	63	69	84
	GR+T1	233	168	23	288	-17	64	128	144
	RESPONSE	287	108	117	328	68	133	96	177
	TOTAL	520	276	140	605	85	197	223	309
A03 F	GR	-57	200	-9	208	-50	18	54	76
	T1	290	-37	32	294	32	58	4	67
	GR+T1	233	163	23	286	-18	76	58	98
	RESPONSE	287	108	116	327	62	128	76	161
	TOTAL	520	271	139	602	80	204	134	257
A04 N	GR	-57	188	-9	197	-56	18	91	109
	T1	290	-37	32	294	53	58	-186	202
	GR+T1	233	151	23	279	-3	76	-95	122
	RESPONSE	285	108	114	325	66	128	213	266
	TOTAL	518	259	137	595	99	204	308	383

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)			Result	MOMENTS (ft-lb)			Result
		X	Y	Z		X	Y	Z	
A04 M	GR	-57	181	-9	190	-60	17	84	105
	T1	290	-37	32	294	65	63	-283	297
	GR+T1	233	145	23	275	5	80	-199	215
	RESPONSE	284	107	111	323	129	118	298	346
	TOTAL	517	252	135	591	135	197	498	552
A04 F	GR	-57	174	-9	184	-61	14	80	69
	T1	290	-37	32	294	70	74	-313	329
	GR+T1	233	138	23	272	9	88	-283	297
	RESPONSE	276	104	100	312	144	101	314	360
	TOTAL	509	242	123	577	153	189	598	645
A05 N	GR	-57	96	-9	112	-61	-27	-573	577
	T1	290	-37	32	294	70	219	-148	273
	GR+T1	233	59	23	242	9	192	-721	746
	RESPONSE	268	83	73	290	144	417	354	565
	TOTAL	502	141	96	530	153	609	1075	1245
A05 M	GR	-57	89	-9	106	-48	-22	-605	608
	T1	290	-37	32	294	64	188	-135	240
	GR+T1	233	52	23	240	17	166	-741	759
	RESPONSE	267	80	68	287	135	417	371	574
	TOTAL	500	132	92	526	152	583	1112	1265
A05 F	GR	-57	82	-9	100	-17	-3	-618	618
	T1	290	-37	32	294	51	90	-130	166
	GR+T1	233	45	23	239	34	87	-748	754
	RESPONSE	264	77	61	282	117	377	378	547
	TOTAL	498	122	84	519	151	464	1126	1227
A06 N	GR	-57	64	-9	86	56	54	-618	623
	T1	290	-37	32	294	14	-200	-130	239
	GR+T1	233	27	23	236	70	-146	-748	765
	RESPONSE	260	70	54	274	98	354	378	527
	TOTAL	493	98	78	508	168	500	1126	1244
A06 M	GR	-57	57	-9	81	77	72	-627	636
	T1	290	-37	32	294	1	-297	-124	322
	GR+T1	233	20	23	235	78	-225	-751	788
	RESPONSE	257	66	51	270	104	392	383	558
	TOTAL	490	86	74	503	182	617	1134	1304

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A06 F	GR	-57	50	-9	76	85	77	-646	656
	T1	290	-37	32	294	-4	-328	-111	347
	GR+T1	233	13	23	235	81	-251	-757	802
	RESPONSE	250	58	45	261	107	414	393	581
	TOTAL	484	71	68	494	188	665	1150	1342
A07 -	GR	-57	30	-9	65	85	67	-691	700
	T1	290	-37	32	294	-4	-292	-70	300
	GR+T1	233	-7	23	235	81	-225	-761	798
	RESPONSE	250	58	45	261	107	419	420	603
	TOTAL	484	65	68	493	188	644	1181	1358
A07 +	GR	-57	7	-9	58	85	67	-691	700
	T1	290	-37	32	294	-4	-292	-70	300
	GR+T1	233	-30	23	236	81	-225	-761	798
	RESPONSE	232	42	38	239	107	419	420	603
	TOTAL	465	71	61	475	188	644	1181	1358
A08 -	GR	-57	-9	-9	58	85	59	-691	698
	T1	290	-37	32	294	-4	-262	-36	264
	GR+T1	233	-46	23	239	81	-203	-726	758
	RESPONSE	232	42	38	239	107	407	420	595
	TOTAL	465	88	61	477	188	610	1146	1312
A08 +	GR	-57	-114	-9	128	85	59	-691	698
	T1	290	-37	32	294	-4	-262	-36	264
	GR+T1	233	-151	23	279	81	-303	-726	758
	RESPONSE	179	83	76	212	107	407	420	595
	TOTAL	412	234	100	484	188	610	1146	1312
A09 -	GR	-57	-130	-9	142	85	50	-578	587
	T1	290	-37	32	294	-4	-232	-2	232
	GR+T1	233	-167	23	288	81	-182	-580	613
	RESPONSE	179	83	76	212	107	339	347	697
	TOTAL	412	250	100	492	188	520	927	1080
A09 +	GR	-57	-147	-9	158	85	50	-578	587
	T1	290	-37	32	294	-4	-232	-2	232
	GR+T1	233	-184	23	298	81	-182	-580	613
	RESPONSE	158	109	97	216	107	339	347	497
	TOTAL	392	294	120	504	188	520	927	1080

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A10 -	GR	-57	-194	-9	202	85	26	-124	153
	T1	290	-37	32	294	-4	-146	96	175
	GR+T1	233	-231	23	329	81	-120	-28	147
	RESPONSE	158	109	97	216	107	92	88	166
	TOTAL	392	340	120	533	188	211	116	306
A10 +	GR	-57	-291	-9	297	85	26	-124	153
	T1	290	-37	32	294	-4	-146	96	175
	GR+T1	233	-328	23	403	81	-120	-28	147
	RESPONSE	108	136	124	213	107	92	88	166
	TOTAL	342	464	147	595	188	211	116	306
A11 N	GR	-57	-304	-9	309	85	20	86	122
	T1	290	-37	32	294	-4	-123	123	174
	GR+T1	233	-341	23	413	81	-103	208	246
	RESPONSE	105	137	125	214	107	30	78	136
	TOTAL	339	478	148	604	188	133	287	368
A11 M	GR	-57	-308	-9	314	84	18	163	185
	T1	290	-37	32	294	-1	-115	103	155
	GR+T1	233	-345	23	417	83	-98	266	296
	RESPONSE	104	138	126	213	101	42	99	148
	TOTAL	337	483	149	607	185	140	365	432
A11 F	GR	-57	-313	-9	318	82	17	207	223
	T1	290	-37	32	294	7	-112	38	119
	GR+T1	233	-350	23	421	89	-96	245	278
	RESPONSE	103	138	126	213	93	51	115	156
	TOTAL	336	488	149	611	181	147	360	429
A12	GR	-57	-458	-9	462	82	17	211	227
	T1	290	-37	32	294	9	-112	19	114
	GR+T1	233	-495	23	548	90	-96	230	265
	RESPONSE	65	154	148	223	92	51	117	157
	TOTAL	296	649	172	733	182	147	347	418
A13	GR	-57	-458	-9	462	81	17	213	229
	T1	290	-37	32	294	10	-112	7	113
	GR+T1	233	-495	23	548	91	-96	220	257
	RESPONSE	92	176	178	267	92	51	119	159
	TOTAL	325	671	201	772	183	147	339	412

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A14 N	GR	-57	-460	-9	463	80	17	219	234
	T1	290	-37	32	294	13	-112	-22	115
	GR+T1	233	-497	23	549	93	-96	197	238
	RESPONSE	94	176	179	268	95	51	124	164
	TOTAL	327	673	202	775	189	147	321	400
A14 M	GR	-57	-467	-9	470	145	9	239	279
	T1	290	-37	32	294	30	-70	-124	146
	GR+T1	233	-504	23	556	175	-61	115	218
	RESPONSE	96	177	180	270	143	40	144	207
	TOTAL	329	681	204	783	318	101	258	422
A14 F	GR	-57	-474	-9	477	310	-12	247	397
	T1	290	-37	32	294	48	33	-167	176
	GR+T1	233	-511	23	562	358	21	80	367
	RESPONSE	97	178	181	272	204	23	154	256
	TOTAL	330	688	204	790	562	44	234	610
A14A	GR	-57	549	-9	552	386	-21	247	459
	T1	290	-2	32	292	54	79	-167	192
	GR+T1	233	546	23	594	440	58	80	451
	RESPONSE	98	178	182	273	226	29	154	275
	TOTAL	331	725	205	823	666	87	234	711
A15	GR	-57	541	-9	544	37	-57	247	256
	T1	290	-2	32	292	55	265	-167	317
	GR+T1	233	539	23	588	93	208	80	241
	RESPONSE	98	178	182	273	326	83	154	370
	TOTAL	331	717	205	816	419	291	234	561
*** Segment A end ***									
*** Segment B begin ***									
B02	GR	5	.19	2	20	13	-5	4	15
	T1	8	-197	-103	222	-189	-124	213	311
	GR+T1	13	-178	-102	205	-176	-129	217	308
	RESPONSE	319	59	154	359	45	259	39	266
	TOTAL	332	237	255	441	221	388	256	515
B03 N	GR	5	8	2	10	1	0	4	4
	T1	8	-197	-103	222	-7	-117	213	243
	GR+T1	13	-188	-102	214	-7	-117	217	246
	RESPONSE	319	59	154	359	28	113	39	122
	TOTAL	332	247	256	487	35	229	256	345

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
B03 M	GR	5	5	2	7	-1	1	5	5
	T1	8	-197	-103	222	45	-103	192	222
	GR+T1	13	-192	-102	217	43	-102	196	225
	RESPONSE	319	59	155	360	39	154	44	165
	TOTAL	332	251	257	489	83	256	241	361
B03 F	GR	5	2	2	5	-2	1	5	6
	T1	8	-197	-103	222	66	-75	139	172
	GR+T1	13	-195	-102	220	65	-73	145	175
	RESPONSE	319	59	156	360	45	142	59	161
	TOTAL	332	254	257	491	109	216	204	316
B04 N	GR	5	-9	2	10	-2	0	3	4
	T1	8	-197	-103	222	66	-14	23	71
	GR+T1	13	-206	-102	230	65	-13	26	71
	RESPONSE	319	59	156	360	45	52	93	115
	TOTAL	333	264	258	497	109	65	119	174
B04 M	GR	5	-13	2	14	-1	0	0	1
	T1	8	-197	-102	222	56	11	-24	62
	GR+T1	13	-210	-102	234	55	11	-24	61
	RESPONSE	319	59	156	360	29	19	76	85
	TOTAL	333	269	258	500	84	30	100	134
B04 F	GR	5	-18	2	19	-1	0	-2	3
	T1	8	-197	-103	222	32	21	-45	59
	GR+T1	13	-215	-102	238	31	21	-48	60
	RESPONSE	319	59	156	360	8	15	7	18
	TOTAL	333	274	258	502	39	35	55	76
B05	GR	5	-163	2	163	-1	0	-3	3
	T1	8	-197	-103	222	25	21	-46	56
	GR+T1	13	-360	-102	374	24	21	-49	58
	RESPONSE	321	67	160	365	18	15	16	28
	TOTAL	334	427	262	602	42	35	65	85
B06	GR	5	-163	2	163	-1	0	-3	3
	T1	8	-197	-103	222	21	21	-46	55
	GR+T1	13	-360	-102	374	20	21	-49	57
	RESPONSE	326	86	168	377	25	15	29	40
	TOTAL	339	446	270	622	44	35	78	96

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
B07	GR	5	-167	2	167	-1	0	-4	4
	T1	8	-197	-103	222	1	21	-48	52
	GR+T1	13	-363	-102	378	1	21	-52	56
	RESPONSE	326	86	168	377	56	15	90	107
	TOTAL	339	449	270	624	56	35	142	156

*** Segment B end ***

*** Segment C begin ***

5	GR	22	-229	-6	230	-29	-3	0	29
	T1	-279	-14	-24319	24321	-2	35	0	35
	GR+T1	-257	-243	-24325	24328	-30	32	0	44
	RESPONSE	281	1062	314	1142	133	35	0	137
	TOTAL	538	1304	24639	24679	163	67	0	176

C01	GR	22	-231	-6	232	29	3	0	29
	T1	-279	-14	-24319	24321	2	-35	0	35
	GR+T1	-257	-245	-24325	24328	31	-52	0	44
	RESPONSE	281	1062	314	1142	133	35	0	137
	TOTAL	538	1307	24639	24679	163	67	0	177

*** Segment C end ***

*** Segment D begin ***

3	GR	-84	156	-5	177	-19	-10	0	22
	T1	561	262	24455	24463	-33	70	0	77
	GR+T1	477	418	24450	24458	-52	60	0	79
	RESPONSE	560	1180	316	1344	148	70	0	163
	TOTAL	1037	1598	24766	24839	200	130	0	238

D01	GR	-84	154	-5	175	19	10	0	22
	T1	561	262	24455	24463	33	-70	0	77
	GR+T1	477	416	24450	24458	-52	-60	0	79
	RESPONSE	560	1180	316	1344	148	70	0	163
	TOTAL	1037	1596	24766	24839	200	130	0	238

*** Segment D end ***

ASME B31.3c (1992) CODE COMPLIANCE

Point name	Load combination	(Moments in ft-lb)			Torsion		S.I.F		(Stress in psi)		Code Stress Allow.
		In-Pl.	Out-Pl.	Moment	In	Out	no.	type			

*** Segment A begin ***

A00	Max P								(3a) HOOP	4844	16700
	GR + Max P	179	39			1.00	1.00		(18) SUST	2947	16700
	Cold to T1	76	41			26	1.00	1.00	(17) DISP	338	25050
	SRSS	141	196				1.00	1.00	(18) OCC	1162	22211

A01	Max P								(3a) HOOP	4844	16700
	GR + Max P	92	31			1.00	1.00		(18) SUST	2531	16700
	Cold to T1	85	2			26	1.00	1.00	(17) DISP	334	25050
	SRSS	130	161				1.00	1.00	(18) OCC	992	22211

A02	Max P								(3a) HOOP	4844	16700
	GR + Max P	79	29			1.00	1.00		(18) SUST	2472	16700
	Cold to T1	87	5			26	1.00	1.00	(17) DISP	340	25050
	SRSS	128	155				1.00	1.00	(18) OCC	967	22211

A03 N-	Max P								(3a) HOOP	4844	16700
	GR + Max P	79	29			1.00	1.00		(18) SUST	2470	16700
	Cold to T1	87	5			26	1.00	1.00	(17) DISP	340	25050
	SRSS	128	155				1.00	1.00	(18) OCC	966	22211

A03 N+	Max P								(3a) HOOP	4844	16700
	GR + Max P	79	29			2.56	2.13		(18) SUST	3080	16700
	Cold to T1	87	5			26	2.56	2.13	(17) DISP	839	25050
	SRSS	128	155				2.56	2.13	(18) OCC	2235	22211

A03 M	Max P								(3a) HOOP	4844	16700
	GR + Max P	35	28			2.56	2.13		(18) SUST	2580	16700
	Cold to T1	72	23			38	2.56	2.13	(17) DISP	725	25050
	SRSS	108	99				2.56	2.13	(18) OCC	1674	22211

A03 F-	Max P								(3a) HOOP	4844	16700
	GR + Max P	26	69			2.56	2.13		(18) SUST	2842	16700
	Cold to T1	18	27			58	2.56	2.13	(17) DISP	349	25050
	SRSS	79	57				2.56	2.13	(18) OCC	1136	22211

A03 F+	Max P								(3a) HOOP	4844	16700
	GR + Max P	50	54			1.00	1.00		(18) SUST	2420	16700
	Cold to T1	32	4			58	1.00	1.00	(17) DISP	248	25050
	SRSS	62	76				1.00	1.00	(18) OCC	469	22211

Point name	Load combination	ASME B31.3c (1992) CODE COMPLIANCE (Moments in ft-lb)			S.I.F		(Stress in psi)		Code Allow.
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no.	Code type	
A04 N-	Max P						(3a) HOOP	4844	16700
	GR + Max P	56	91		1.00	1.00	(18) SUST	2580	16700
	Cold to T1	53	186	58	1.00	1.00	(17) DISP	755	25050
	SRSS	96	213		1.00	1.00	(18) OCC	1121	22211
A04 N+	Max P						(3a) HOOP	4844	16700
	GR + Max P	91	56		1.95	1.63	(18) SUST	3027	16700
	Cold to T1	186	53	58	1.95	1.63	(17) DISP	1413	25050
	SRSS	213	96		1.95	1.63	(18) OCC	2132	22211
A04 M	Max P						(3a) HOOP	4844	16700
	GR + Max P	84	30		1.95	1.63	(18) SUST	2890	16700
	Cold to T1	283	90	1	1.95	1.63	(17) DISP	2137	25050
	SRSS	298	93		1.95	1.63	(18) OCC	2889	22211
A04 F-	Max P						(3a) HOOP	4844	16700
	GR + Max P	30	14		1.95	1.63	(18) SUST	2364	16700
	Cold to T1	313	74	70	1.95	1.63	(17) DISP	2339	25050
	SRSS	314	101		1.95	1.63	(18) OCC	3050	22211
A04 F+	Max P						(3a) HOOP	4844	16700
	GR + Max P	30	14		1.00	1.00	(18) SUST	2223	16700
	Cold to T1	313	74	70	1.00	1.00	(17) DISP	1228	25050
	SRSS	314	101		1.00	1.00	(18) OCC	1585	22211
A05 N-	Max P						(3a) HOOP	4844	16700
	GR + Max P	573	27		1.00	1.00	(18) SUST	4818	16700
	Cold to T1	148	219	70	1.00	1.00	(17) DISP	1020	25050
	SRSS	354	417		1.00	1.00	(18) OCC	2624	22211
A05 N+	Max P						(3a) HOOP	4844	16700
	GR + Max P	27	573		1.95	1.63	(18) SUST	6546	16700
	Cold to T1	219	148	70	1.95	1.63	(17) DISP	1850	25050
	SRSS	417	354		1.95	1.63	(18) OCC	4783	22211
A05 M	Max P						(3a) HOOP	4844	16700
	GR + Max P	22	395		1.95	1.63	(18) SUST	5153	16700
	Cold to T1	188	141	50	1.95	1.63	(17) DISP	1626	25050
	SRSS	417	254		1.95	1.63	(18) OCC	4381	22211
A05 F-	Max P						(3a) HOOP	4844	16700
	GR + Max P	3	17		1.95	1.63	(18) SUST	2205	16700
	Cold to T1	90	51	130	1.95	1.63	(17) DISP	873	25050
	SRSS	377	117		1.95	1.63	(18) OCC	3650	22211

Point name	Load combination	ASME B31.3c (1992) CODE COMPLIANCE (Moments in ft-lb)			S.I.F		(Stress in psi)		Code Allow.
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no.	Code type	
A05 F+	Max P						(3a) HOOP	4844	16700
	GR + Max P	17	3		1.00	1.00	(18) SUST	2151	16700
	Cold to T1	51	90	130	1.00	1.00	(17) DISP	620	25050
	SRSS	117	377		1.00	1.00	(18) OCC	1895	22211
A06 N-	Max P						(3a) HOOP	4844	16700
	GR + Max P	56	54		1.00	1.00	(18) SUST	2436	16700
	Cold to T1	14	200	130	1.00	1.00	(17) DISP	891	25050
	SRSS	98	354		1.00	1.00	(18) OCC	1763	22211
A06 N+	Max P						(3a) HOOP	4844	16700
	GR + Max P	54	56		1.95	1.63	(18) SUST	2729	16700
	Cold to T1	200	14	130	1.95	1.63	(17) DISP	1537	25050
	SRSS	354	98		1.95	1.63	(18) OCC	3405	22211
A06 M	Max P						(3a) HOOP	4844	16700
	GR + Max P	72	498		1.95	1.63	(18) SUST	6011	16700
	Cold to T1	297	89	87	1.95	1.63	(17) DISP	2258	25050
	SRSS	392	300		1.95	1.63	(18) OCC	4355	22211
A06 F-	Max P						(3a) HOOP	4844	16700
	GR + Max P	77	646		1.95	1.63	(18) SUST	7161	16700
	Cold to T1	328	111	4	1.95	1.63	(17) DISP	2488	25050
	SRSS	414	393		1.95	1.63	(18) OCC	4946	22211
A06 F+	Max P						(3a) HOOP	4844	16700
	GR + Max P	646	77		1.00	1.00	(18) SUST	5187	16700
	Cold to T1	111	328	4	1.00	1.00	(17) DISP	1295	25050
	SRSS	393	414		1.00	1.00	(18) OCC	2740	22211
A07	Max P						(3a) HOOP	4844	16700
	GR + Max P	691	67		1.00	1.00	(18) SUST	5400	16700
	Cold to T1	70	292	4	1.00	1.00	(17) DISP	1120	25050
	SRSS	420	419		1.00	1.00	(18) OCC	2850	22211
A08	Max P						(3a) HOOP	4844	16700
	GR + Max P	691	59		1.00	1.00	(18) SUST	5303	16700
	Cold to T1	36	262	4	1.00	1.00	(17) DISP	987	25050
	SRSS	420	407		1.00	1.00	(18) OCC	2807	22211
A09	Max P						(3a) HOOP	4844	16700
	GR + Max P	578	50		1.00	1.00	(18) SUST	4852	16700
	Cold to T1	2	232	4	1.00	1.00	(17) DISP	867	25050
	SRSS	347	339		1.00	1.00	(18) OCC	2327	22211

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			S.L.F		(Stress in psi)		Code Stress	Code Allow.
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no.	Code type		
A10	Max P							(3a) HOOP	4844	16700
	GR + Max P	124	26		1.00	1.00	(18)	SUST	2676	16700
	Cold to T1	96	146		4	1.00	1.00	(17)	DISP	654
	SRSS	88	92		1.00	1.00	(18)	OCC	609	22211
A11 N-	Max P							(3a) HOOP	4844	16700
	GR + Max P	86	20		1.00	1.00	(18)	SUST	2488	16700
	Cold to T1	123	123		4	1.00	1.00	(17)	DISP	649
	SRSS	78	30		1.00	1.00	(18)	OCC	403	22211
A11 N+	Max P							(3a) HOOP	4844	16700
	GR + Max P	86	20		2.56	2.13	(18)	SUST	3138	16700
	Cold to T1	123	123		4	2.56	2.13	(17)	DISP	1527
	SRSS	78	30		2.56	2.13	(18)	OCC	1010	22211
A11 M	Max P							(3a) HOOP	4844	16700
	GR + Max P	163	72		2.56	2.13	(18)	SUST	4203	16700
	Cold to T1	103	82		81	2.56	2.13	(17)	DISP	1220
	SRSS	99	69		2.56	2.13	(18)	OCC	1402	22211
A11 F-	Max P							(3a) HOOP	4844	16700
	GR + Max P	207	82		2.56	2.13	(18)	SUST	4742	16700
	Cold to T1	38	7		112	2.56	2.13	(17)	DISP	558
	SRSS	115	93		2.56	2.13	(18)	OCC	1699	22211
A11 F+	Max P							(3a) HOOP	4844	16700
	GR + Max P	82	207		1.00	1.00	(18)	SUST	3134	16700
	Cold to T1	7	38		112	1.00	1.00	(17)	DISP	444
	SRSS	93	115		1.00	1.00	(18)	OCC	708	22211
A12	Max P							(3a) HOOP	4844	16700
	GR + Max P	82	211		1.00	1.00	(18)	SUST	3150	16700
	Cold to T1	9	19		112	1.00	1.00	(17)	DISP	426
	SRSS	92	117		1.00	1.00	(18)	OCC	714	22211
A13	Max P							(3a) HOOP	4844	16700
	GR + Max P	81	213		1.00	1.00	(18)	SUST	3160	16700
	Cold to T1	10	7		112	1.00	1.00	(17)	DISP	422
	SRSS	92	119		1.00	1.00	(18)	OCC	721	22211
A14 N-	Max P							(3a) HOOP	4844	16700
	GR + Max P	80	219		1.00	1.00	(18)	SUST	3184	16700
	Cold to T1	13	22		112	1.00	1.00	(17)	DISP	430
	SRSS	95	124		1.00	1.00	(18)	OCC	748	22211

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			S.L.F		(Stress in psi)		Code Stress	Code Allow.
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no.	Code type		
A14 N+	Max P							(3a) HOOP	4844	16700
	GR + Max P	80	219		1.95	1.63	(18)	SUST	3932	16700
	Cold to T1	13	22		112	1.95	1.63	(17)	DISP	450
	SRSS	95	124		1.95	1.63	(18)	OCC	1314	22211
A14 M	Max P							(3a) HOOP	4844	16700
	GR + Max P	145	163		1.95	1.63	(18)	SUST	3927	16700
	Cold to T1	30	38		137	1.95	1.63	(17)	DISP	604
	SRSS	143	93		1.95	1.63	(18)	OCC	1521	22211
A14 F-	Max P							(3a) HOOP	4844	16700
	GR + Max P	310	12		1.95	1.63	(18)	SUST	4972	16700
	Cold to T1	48	33		167	1.95	1.63	(17)	DISP	740
	SRSS	204	23		1.95	1.63	(18)	OCC	1919	22211
A14 F+	Max P							(3a) HOOP	4279	16700
	GR + Max P	310	12		1.00	1.00	(18)	SUST	4661	16700
	Cold to T1	48	33		167	1.00	1.00	(17)	DISP	1228
	SRSS	204	23		1.00	1.00	(18)	OCC	1876	22211
A14A	Max P							(3a) HOOP	4279	16700
	GR + Max P	386	21		1.00	1.00	(18)	SUST	5358	16700
	Cold to T1	54	79		167	1.00	1.00	(17)	DISP	1337
	SRSS	226	29		1.00	1.00	(18)	OCC	2087	22211
A15	Max P							(3a) HOOP	4279	16700
	GR + Max P	37	57		1.00	1.00	(18)	SUST	2444	16700
	Cold to T1	55	265		167	1.00	1.00	(17)	DISP	2216
	SRSS	326	83		1.00	1.00	(18)	OCC	3078	22211
*** Segment A end ***										
*** Segment B begin ***										
B02	Max P							(3a) HOOP	4279	16700
	GR + Max P	13	5		1.00	1.00	(18)	SUST	1952	16700
	Cold to T1	189	124		213	1.00	1.00	(17)	DISP	2166
	SRSS	45	259		1.00	1.00	(18)	OCC	2407	22211
B03 N-	Max P							(3a) HOOP	4279	16700
	GR + Max P	1	0		1.00	1.00	(18)	SUST	1828	16700
	Cold to T1	7	117		213	1.00	1.00	(17)	DISP	1692
	SRSS	28	113		1.00	1.00	(18)	OCC	1063	22211

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			S.I.F		(Stress in psi)		Code	Code
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no.	type		
B03 N+	Max P						(3a)	HOOP	4279	16700
	GR + Max P	0	1		1.78	1.48	(18)	SUST	1831	16700
	Cold to T1	117	7	213	1.78	1.48	(17)	DISP	2071	25050
	SRSS	113	28		1.78	1.48	(18)	OCC	1871	22211
B03 M	Max P						(3a)	HOOP	4279	16700
	GR + Max P	1	4		1.78	1.48	(18)	SUST	1881	16700
	Cold to T1	103	104	167	1.78	1.48	(17)	DISP	2031	25050
	SRSS	154	51		1.78	1.48	(18)	OCC	2594	22211
B03 F-	Max P						(3a)	HOOP	4279	16700
	GR + Max P	1	5		1.78	1.48	(18)	SUST	1899	16700
	Cold to T1	75	139	66	1.78	1.48	(17)	DISP	1770	25050
	SRSS	142	59		1.78	1.48	(18)	OCC	2450	22211
B03 F+	Max P						(3a)	HOOP	4844	16700
	GR + Max P	5	1		1.00	1.00	(18)	SUST	2092	16700
	Cold to T1	139	75	66	1.00	1.00	(17)	DISP	640	25050
	SRSS	59	142		1.00	1.00	(18)	OCC	740	22211
B04 N-	Max P						(3a)	HOOP	4844	16700
	GR + Max P	3	0		1.00	1.00	(18)	SUST	2081	16700
	Cold to T1	23	14	66	1.00	1.00	(17)	DISP	267	25050
	SRSS	93	52		1.00	1.00	(18)	OCC	511	22211
B04 N+	Max P						(3a)	HOOP	4844	16700
	GR + Max P	3	0		2.56	2.13	(18)	SUST	2106	16700
	Cold to T1	23	14	66	2.56	2.13	(17)	DISP	349	25050
	SRSS	93	52		2.56	2.13	(18)	OCC	1259	22211
B04 M	Max P						(3a)	HOOP	4844	16700
	GR + Max P	0	1		2.56	2.13	(18)	SUST	2077	16700
	Cold to T1	24	32	47	2.56	2.13	(17)	DISP	367	25050
	SRSS	76	31		2.56	2.13	(18)	OCC	980	22211
B04 F-	Max P						(3a)	HOOP	4844	16700
	GR + Max P	2	1		2.56	2.13	(18)	SUST	2098	16700
	Cold to T1	45	32	21	2.56	2.13	(17)	DISP	507	25050
	SRSS	7	8		2.56	2.13	(18)	OCC	123	22211
B04 F+	Max P						(3a)	HOOP	4844	16700
	GR + Max P	1	2		1.00	1.00	(18)	SUST	2079	16700
	Cold to T1	32	45	21	1.00	1.00	(17)	DISP	221	25050
	SRSS	8	7		1.00	1.00	(18)	OCC	53	22211

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			S.I.F		(Stress in psi)		Code	Code
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no.	type		
B05	Max P						(3a)	HOOP	4844	16700
	GR + Max P	1	3		1.00	1.00	(18)	SUST	2080	16700
	Cold to T1	25	46	21	1.00	1.00	(17)	DISP	210	25050
	SRSS	18	16		1.00	1.00	(18)	OCC	117	22211
B06	Max P						(3a)	HOOP	4844	16700
	GR + Max P	1	3		1.00	1.00	(18)	SUST	2081	16700
	Cold to T1	21	46	21	1.00	1.00	(17)	DISP	204	25050
	SRSS	25	29		1.00	1.00	(18)	OCC	181	22211
B07	Max P						(3a)	HOOP	4844	16700
	GR + Max P	1	4		1.00	1.00	(18)	SUST	2085	16700
	Cold to T1	1	48	21	1.00	1.00	(17)	DISP	194	25050
	SRSS	56	90		1.00	1.00	(18)	OCC	508	22211
*** Segment B end ***										
*** Segment C begin ***										
5	Max P						(3a)	HOOP	6905	16700
	GR + Max P	29	3		1.00	1.00	(18)	SUST	3140	16700
	Cold to T1	2	35	0	1.00	1.00	(17)	DISP	175	25050
	SRSS	133	35		1.00	1.00	(18)	OCC	895	22211
C01	Max P						(3a)	HOOP	6905	16700
	GR + Max P	29	3		1.00	1.00	(18)	SUST	3140	16700
	Cold to T1	2	35	0	1.00	1.00	(17)	DISP	175	25050
	SRSS	133	35		1.00	1.00	(18)	OCC	895	22211
*** Segment C end ***										
*** Segment D begin ***										
3	Max P						(3a)	HOOP	6905	16700
	GR + Max P	19	10		1.00	1.00	(18)	SUST	3096	16700
	Cold to T1	33	70	0	1.00	1.00	(17)	DISP	388	25050
	SRSS	148	70		1.00	1.00	(18)	OCC	1065	22211
D01	Max P						(3a)	HOOP	6905	16700
	GR + Max P	19	10		1.00	1.00	(18)	SUST	3095	16700
	Cold to T1	33	70	0	1.00	1.00	(17)	DISP	388	25050
	SRSS	148	70		1.00	1.00	(18)	OCC	1065	22211
*** Segment D end ***										

SYSTEM SUMMARY

Maximum displacements (in)

Maximum X :	0.173	Point : A14 N	Load Comb.: TOTAL
Maximum Y :	0.380	Point : A06 M	Load Comb.: TOTAL
Maximum Z :	0.158	Point : A05 N	Load Comb.: TOTAL
Max. total:	0.411	Point : A06 F	Load Comb.: TOTAL

Maximum rotations (deg)

Maximum X :	0.373	Point : A04 N	Load Comb.: TOTAL
Maximum Y :	0.464	Point : A14 F	Load Comb.: TOTAL
Maximum Z :	0.335	Point : A10	Load Comb.: TOTAL
Max. total:	0.506	Point : A04 N	Load Comb.: TOTAL

Maximum restraint forces(lb)

Maximum X :	1037	Point : D01	Load Comb.: TOTAL
Maximum Y :	1596	Point : D01	Load Comb.: TOTAL
Maximum Z :	24766	Point : D01	Load Comb.: TOTAL
Max. total:	24839	Point : D01	Load Comb.: TOTAL

Maximum restraint moments(ft-lb)

Maximum X :	321	Point : 3	Load Comb.: TOTAL
Maximum Y :	395	Point : 3	Load Comb.: TOTAL
Maximum Z :	1011	Point : 3	Load Comb.: TOTAL
Max. total:	1132	Point : 3	Load Comb.: TOTAL

SYSTEM SUMMARY

Maximum pipe forces (lb)

Maximum X :	1037	Point : 3	Load Comb.: TOTAL
Maximum Y :	1598	Point : 3	Load Comb.: TOTAL
Maximum Z :	24766	Point : 3	Load Comb.: TOTAL
Max. total:	24839	Point : 3	Load Comb.: TOTAL

Maximum pipe moments (ft-lb)

Maximum X :	666	Point : A14A	Load Comb.: TOTAL
Maximum Y :	665	Point : A06 F	Load Comb.: TOTAL
Maximum Z :	1181	Point : A07	Load Comb.: TOTAL
Max. total:	1358	Point : A07	Load Comb.: TOTAL

SYSTEM SUMMARY

Maximum sustained stress

Point : A06 F
Stress psi : 7161
Allowable psi : 16700
Ratio : 0.43
Load combination : GR + Max P

Maximum displacement stress

Point : A06 F
Stress psi : 2488
Allowable psi : 25050
Ratio : 0.10
Load combination : Cold to T1

Maximum occasional stress

Point : A06 F
Stress psi : 4946
Allowable psi : 22211
Ratio : 0.22
Load combination : SRSS

Maximum hoop stress

Point : 5
Stress psi : 6905
Allowable psi : 16700
Ratio : 0.41
Load combination : Max P

Maximum sustained stress ratio

Point : A06 F
Stress psi : 7161
Allowable psi : 16700
Ratio : 0.43
Load combination : GR + Max P

Maximum displacement stress ratio

Point : A06 F
Stress psi : 2488
Allowable psi : 25050
Ratio : 0.10
Load combination : Cold to T1

SYSTEM SUMMARY

Maximum occasional stress ratio

Point : A06 F
Stress psi : 4946
Allowable psi : 22211
Ratio : 0.22
Load combination : SRSS

Maximum hoop stress ratio

Point : 5
Stress psi : 6905
Allowable psi : 16700
Ratio : 0.41
Load combination : Max P

*** The system satisfies ASME B31.3 code requirements ***
*** for the selected options ***

APPDX - B

POINT DATA LISTING

POINT NAME	TYPE	-----OFFSETS (ft)----- X Y Z			PIPE ID	DESCRIPTION
*** SEGMENT A						
A00	Run	0	0	0	4PIPE	
A01	Flex	0.22	0	-0.11		Wt= 144 lb, Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
A02	Run	0.04	0	-0.02		
A03	Bend	0.30	0	-0.15		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
A04	Bend	0	-1.49	0		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A05	Bend	5.46	0	0		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A06	Bend	0	0	2.00		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A07	Run	1.63	0	0		
A08	Run	0.92	0	0		
A09	Run	0.92	0	0		
A10	Run	2.66	0	0		
A11	Bend	1.04	0	0		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
A12	Flex	0	-0.40	0		Wt= 144 lb, Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
A13	Run	0	-0.04	0		
A14	Bend	0	-0.60	0		Long Elbow, Radius = 6.00 inch

POINT DATA LISTING

POINT NAME	TYPE	-----OFFSETS (ft)----- X Y Z			PIPE ID	DESCRIPTION
*** SEGMENT B						
A14 F	Far				3PIPE	
A14A	Run	0	0	-0.66		Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A15	Run	0	0	-0.64		
B02	Run	13.19	-2.84	-1.78	3PIPE	
B03	Bend	0	0	-1.30		Long Elbow, Radius = 4.50 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.78, Out = 1.48 Flex = 4.577
B03 F	Far				4PIPE	
B04	Bend	-1.30	0	0		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
B05	Flex	0	-0.40	0		Wt= 144 lb, Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
B06	Run	0	-0.04	0		
B07	Run	0	-0.19	0		
*** SEGMENT C						
5	Run	13.19	3.50	-1.78	3.5PIPE	
C01	Run	0	0	-0.25		
*** SEGMENT D						
3	Run	13.19	3.50	0.42	3.5PIPE	
D01	Run	0	0	0.25		

Total weight of empty pipes : 254 lb

COMPONENT DATA LISTING

POINT NAME	---COORDINATE(ft) X Y Z	--- DATA TYPE DESCRIPTION
*** SEGMENT A		
A00	0.00 0.00 0.00	ANCHOR Rigid Thermal movements : None
A01	0.22 0.00 -0.11	
A02	0.26 0.00 -0.13	
A03 N	0.26 0.00 -0.13	
A03	0.56 0.00 -0.28 TI	
A03 M	0.47 -0.10 -0.24	
A03 F	0.56 -0.33 -0.28	
A04 N	0.56 -0.99 -0.28	
A04	0.56 -1.49 -0.28 TI	
A04 M	0.71 -1.34 -0.28	
A04 F	1.06 -1.49 -0.28	
A05 N	5.52 -1.49 -0.28	
A05	6.02 -1.49 -0.28 TI	
A05 M	5.87 -1.49 -0.13	
A05 F	6.02 -1.49 0.22	
A06 N	6.02 -1.49 1.22	
A06	6.02 -1.49 1.72 TI	
A06 M	6.17 -1.49 1.57	
A06 F	6.52 -1.49 1.72	
A07	7.65 -1.49 1.72	WEIGHT 23 lb , No offsets
A08	8.57 -1.49 1.72	WEIGHT 35 lb , No offsets FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb
A09	9.49 -1.49 1.72	WEIGHT 17 lb , No offsets
A10	12.15 -1.49 1.72	WEIGHT 27 lb , No offsets FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb FLANGE Weld neck SIF = 1.00 Rating = 400, Weight = 35 lb
A11 N	12.86 -1.49 1.72	
A11	13.19 -1.49 1.72 TI	
A11 M	13.09 -1.59 1.72	
A11 F	13.19 -1.82 1.72	
A12	13.19 -1.89 1.72	
A13	13.19 -1.93 1.72	
A14 N	13.19 -2.03 1.72	
A14	13.19 -2.53 1.72 TI	
A14 M	13.19 -2.39 1.57	
A14 F	13.19 -2.53 1.22	
A14A	13.19 -2.53 1.06	Y-STOP ID : A14A 1, Connected to Ground Gap-Below = 0.00, Above = 1.00 inch Friction = 0.00 Gaps set Weightless
A15	13.19 -2.53 0.42	

COMPONENT DATA LISTING

POINT NAME	---COORDINATE(ft) X Y Z	--- DATA TYPE DESCRIPTION
*** SEGMENT B		
B02	13.19 -2.84 -1.78	
B03 N	13.19 -2.84 -2.70	
B03	13.19 -2.84 -3.08 TI	
B03 H	13.08 -2.84 -2.97	
B03 F	12.81 -2.84 -3.08	
B04 N	12.22 -2.84 -3.08	
B04	11.89 -2.84 -3.08 TI	
B04 M	11.99 -2.94 -3.08	
B04 F	11.89 -3.17 -3.08	
B05	11.89 -3.24 -3.08	
B06	11.89 -3.28 -3.08	
B07	11.89 -3.47 -3.08	ANCHOR Rigid Thermal movements : None
*** SEGMENT C		
5	13.19 3.50 -1.78	ANCHOR Translational stiffness lb/in X= FREE Y= FREE Z= FREE Rotational stiffness ft-lb/deg X= RIGID Y= RIGID Z= RIGID
C01	13.19 3.50 -2.03	ANCHOR Rigid Thermal movements : None
*** SEGMENT D		
3	13.19 3.50 0.42	ANCHOR Translational stiffness lb/in X= FREE Y= FREE Z= FREE Rotational stiffness ft-lb/deg X= RIGID Y= RIGID Z= RIGID
D01	13.19 3.50 0.67	ANCHOR Rigid Thermal movements : None

Number of points in the system : 51

PIPE DATA LISTING

Pipe ID/ Material	Nom/ Sch	O.D. inch	-----Thickness(inch)-----				Spec Grav	Weight(Lb/ft)		
			W.Th.	Corr Mill	Insu	Ling		Pipe	Other Total	
4PIPE AU	4 STD	4.500	0.237	0.06	0.03	0	0	1.20	11.04	0 17.66
3.5PIPE AU	3.500 STD	4.000	0.226	0.06	0.03	0	0	0	9.32	0 9.32
3PIPE AU	3.000 STD	3.500	0.216	0.06	0.03	0	0	1.20	7.75	0 11.60

MATERIAL DATA LISTING

Material Name	Pipe ID	Density lb/cu.ft	Pois. Ratio	Temper. deg F	Modulus E6 psi	Expans. in/100ft	Allow.
							psi
AU	4PIPE	501.0	0.30	40.0 70.0	28.42	0.3088	16700.0 16700.0
AU	3PIPE	501.0	0.30	40.0 70.0	28.42	0.3088	16700.0 16700.0
AU	3.5PIPE	501.0	0.30	40.0 70.0	28.42	0.3088	16700.0 16700.0

FRAME POINT DATA LISTING

POINT NAME	---COORDINATE(ft)---			DATA TYPE	DESCRIPTION
	X	Y	Z		
A15	13.19	-2.53	0.42	Y-STOP	ID : 1 1, Connected to Ground Gap-Below = 0.00, Above = 1.00 inch Friction = 0.00 Gaps set Weightless
1	13.19	-2.84	-0.68		
B02	13.19	-2.84	-1.78	ANCHOR	Translational stiffness lb/in X= FREE Y= FREE Z= FREE Rotational stiffness ft-lb/deg X= RIGID Y= RIGID Z= RIGID Thermal movements : None
5	13.19	3.50	-1.78		
4	13.19	3.50	-0.68	WEIGHT 1000 lb , No offsets	
3	13.19	3.50	0.42		
6	13.19	-2.04	-0.68		
2	13.19	-2.84	0.42		
7	13.19	2.08	-0.68		
WEIGHT 2250 lb , Offsets in ft DX = 0.00, DY = 0.00, DZ = 0.03					

BEAM DATA LISTING

BEAM ID	POINT NAME	LENGTH (ft)	SECTION ID/ MATERIAL ID	BETA ANGLE	RIGID END(ft)	RELEASE		
						AX	Y-Y	Z-Z
M2	From 2	1.10	W4X13 A36	0.00	0.00	N	N	N
	To 1					N	N	N
M9	From 2	0.31	W4X13 A36	0.00	0.00	N	N	N
	To A15					N	N	N
M1	From 1	1.10	W4X13 A36	0.00	0.00	N	N	N
	To B02					N	N	N
M6	From 5	6.34	W4X13 A36	0.00	0.00	N	N	N
	To B02					N	N	N
M3	From A15	6.03	W4X13 A36	0.00	0.00	N	N	N
	To 3					N	N	N
M4	From 3	1.10	W4X13 A36	0.00	0.00	N	N	N
	To 4					N	N	N
M5	From 4	1.10	W4X13 A36	0.00	0.00	N	N	N
	To 5					N	N	N
M7	From 1	0.80	XS12 A36	0.00	0.00	N	N	N
	To 6					N	N	N
M8	From 6	4.12	XS12 A36	0.00	0.00	N	N	N
	To 7					N	N	N
M10	From 4	1.42	XS12 A36	0.00	0.00	N	N	N
	To 7					N	N	N

CROSS SECTION LISTING

Section ID	Area(sq.in)			Inertia (in**4)		
	Axial	Y-Shear	Z-Shear	Torsion	Y-Y Bend	Z-Z Bend
W4X13	3.83	1.16	1.86	0.2	3.9	11.3
XS12	19.20	9.60	9.60	724.0	362.0	362.0

BEAM MATERIAL LISTING

MATERIAL ID	Elastic modulus E6 psi	Poissons ratio	Yield Stress psi	Density lb/cu.ft	Expansion E-6 /F	Ultimate stress psi
A36	29.000	0.250	36000	490.00	6.50000	58000

TEMPERATURE AND PRESSURE DATA

-----C A S E 1-----				-----C A S E 2-----				-----C A S E 3-----			
POINT NAME	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft		

*** SEGMENT A
 A00 490 70.00 0.309
 A14 F 490 70.00 0.309
 A15 490 70.00 0.309

*** SEGMENT B
 B02 490 70.00 0.309
 B03 F 490 70.00 0.309
 B07 490 70.00 0.309

*** SEGMENT C
 5 490 70.00 0.309
 C01 490 70.00 0.309

*** SEGMENT D
 3 490 70.00 0.309
 D01 490 70.00 0.309

ANALYSIS SUMMARY

Current model revision number : 3

Static - Date and Time of analysis Jun 28, 1996 11:05 AM
 Model Revision Number 3
 Number of load cases 3
 Load cases analyzed GR T1 P1
 Gaps/Friction/Yielding considered No
 Hanger design run No
 Cut short included No
 Weight of contents included Yes
 Pressure stiffening case 0
 Water elevation for buoyancy loads Not considered

CODE COMPLIANCE COMBINATIONS

Combination	Category	Method	Load	Factor	Allowable	Remarks
GR + Max P	Sustain	Sum	Gravity Max Long	1.00 1.00	Automatic	Default
Cold to T1	Expansion	Sum	Thermal 1	1.00	Automatic	Default
Max P	Hoop		Max Hoop	1.00	Automatic	Default

OTHER USER COMBINATIONS

Combination	Method	Load	Factor	Remarks
GR	Sum	Gravity	1.00	Default
T1	Sum	Thermal 1	1.00	Default
P1	Sum	Press 1	1.00	Default

CODE COMPLIANCE

Y - Factor 0.40
 Weld efficiency factor 1.00
 Range reduction factor 1.00
 Design Pressure Factor 1.00
 Minimum stress ratio used in reports... 0.00
 Include corrosion in stress calcs. Y
 Include torsion in code stress N
 Include axial force in code stress N
 Longitudinal pressure calculation PD/4t
 Include rigorous pressure Y

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
*** Segment A begin ***							
A00	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
A01	GR	0.000	0.000	0.000	0.000	0.000	-0.002
	T1	0.001	0.000	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
A02	GR	0.000	0.000	0.000	0.153	-0.015	-0.124
	T1	0.001	0.000	0.000	0.012	0.000	-0.015
	P1	0.000	0.000	0.000	0.001	0.000	-0.002
A03 N	GR	0.000	0.000	0.000	0.153	-0.015	-0.124
	T1	0.001	0.000	0.000	0.012	0.000	-0.015
	P1	0.000	0.000	0.000	0.001	0.000	-0.002
A03 M	GR	-0.002	-0.003	-0.002	0.151	-0.015	-0.128
	T1	0.001	-0.001	-0.001	0.011	0.000	-0.016
	P1	0.000	0.000	0.000	0.001	0.000	-0.002
A03 F	GR	-0.009	-0.004	-0.010	0.154	-0.014	-0.132
	T1	0.000	-0.002	-0.002	0.010	-0.001	-0.017
	P1	0.000	0.000	0.000	0.001	0.000	-0.002
A04 N	GR	-0.027	-0.004	-0.031	0.156	-0.015	-0.134
	T1	-0.002	-0.004	-0.003	0.010	-0.001	-0.017
	P1	0.000	0.000	0.000	0.001	0.000	-0.002
A04 M	GR	-0.037	-0.008	-0.042	0.159	-0.014	-0.142
	T1	-0.003	-0.005	-0.004	0.009	-0.002	-0.013
	P1	0.000	-0.001	0.000	0.001	0.000	-0.001
A04 F	GR	-0.041	-0.019	-0.046	0.161	-0.014	-0.147
	T1	-0.002	-0.006	-0.004	0.008	-0.003	-0.007
	P1	0.000	-0.001	0.000	0.001	0.000	-0.001
A05 N	GR	-0.041	-0.139	-0.032	0.175	-0.013	-0.093
	T1	0.012	-0.009	0.001	0.005	-0.008	0.001
	P1	0.001	-0.001	0.000	0.000	-0.001	0.000
A05 M	GR	-0.042	-0.150	-0.032	0.165	-0.011	-0.052
	T1	0.013	-0.009	0.002	0.004	-0.012	0.003
	P1	0.001	-0.001	0.000	0.000	-0.001	0.000

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A05 F	GR	-0.042	-0.163	-0.031	0.155	-0.010	-0.033
	T1	0.012	-0.009	0.004	0.003	-0.014	0.004
	P1	0.001	-0.001	0.000	0.000	-0.001	0.000
A06 N	GR	-0.045	-0.196	-0.031	0.154	-0.011	-0.001
	T1	0.009	-0.010	0.007	0.002	-0.014	0.005
	P1	0.001	-0.001	0.001	0.000	-0.001	0.001
A06 M	GR	-0.046	-0.206	-0.031	0.137	-0.016	0.020
	T1	0.009	-0.010	0.008	0.002	-0.009	0.006
	P1	0.001	-0.010	0.001	0.000	-0.001	0.001
A06 F	GR	-0.046	-0.207	-0.029	0.122	-0.023	0.068
	T1	0.010	-0.009	0.009	0.001	-0.004	0.008
	P1	0.001	-0.001	0.001	0.000	0.000	0.001
A07	GR	-0.046	-0.187	-0.024	0.117	-0.026	0.098
	T1	0.013	-0.007	0.010	0.001	-0.001	0.009
	P1	0.001	-0.001	0.001	0.000	0.000	0.001
A08	GR	-0.046	-0.166	-0.018	0.113	-0.028	0.124
	T1	0.016	-0.006	0.010	0.001	0.001	0.009
	P1	0.002	-0.001	0.001	0.000	0.000	0.001
A09	GR	-0.046	-0.140	-0.013	0.109	-0.030	0.147
	T1	0.019	-0.004	0.009	0.001	0.003	0.009
	P1	0.002	0.000	0.000	0.000	0.000	0.001
A10	GR	-0.046	-0.044	0.005	0.097	-0.034	0.186
	T1	0.027	0.001	0.007	0.002	0.007	0.008
	P1	0.003	0.000	0.001	0.000	0.001	0.001
A11 N	GR	-0.046	-0.017	0.010	0.094	-0.035	0.186
	T1	0.029	0.003	0.006	0.002	0.007	0.008
	P1	0.003	0.000	0.001	0.000	0.001	0.001
A11 M	GR	-0.042	-0.008	0.010	0.091	-0.038	0.176
	T1	0.030	0.003	0.005	0.002	0.009	0.006
	P1	0.003	0.000	0.001	0.000	0.001	0.001
A11 F	GR	-0.034	-0.004	0.007	0.085	-0.040	0.160
	T1	0.031	0.002	0.005	0.003	0.010	0.005
	P1	0.003	0.000	0.000	0.000	0.001	0.001
A12	GR	-0.032	-0.004	0.006	0.085	-0.040	0.160
	T1	0.031	0.002	0.005	0.003	0.010	0.005
	P1	0.003	0.000	0.000	0.000	0.001	0.001

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A13	GR	-0.031	-0.004	0.005	0.046	-0.096	0.058
	T1	0.031	0.002	0.005	0.002	0.084	0.003
	P1	0.003	0.000	0.000	0.000	0.009	0.000
A14 N	GR	-0.030	-0.004	0.004	0.045	-0.096	0.057
	T1	0.031	0.001	0.005	0.002	0.084	0.003
	P1	0.003	0.000	0.000	0.000	0.009	0.000
A14 M	GR	-0.023	-0.003	0.001	0.037	-0.092	0.041
	T1	0.028	0.000	0.004	0.001	0.084	0.004
	P1	0.003	0.000	0.000	0.000	0.009	0.000
A14 F	GR	-0.015	-0.001	0.000	0.018	-0.087	0.033
	T1	0.022	0.000	0.003	0.001	0.084	0.005
	P1	0.002	0.000	0.000	0.000	0.009	0.000
A14A	GR	-0.013	0.000	0.000	0.013	-0.086	0.028
	T1	0.019	0.000	0.003	0.001	0.084	0.005
	P1	0.002	0.000	0.000	0.000	0.009	0.001
A15	GR	-0.001	0.000	0.000	0.000	-0.084	0.008
	T1	0.008	0.000	0.001	0.000	0.081	0.008
	P1	0.001	0.000	0.000	0.000	0.008	0.001
*** Segment A end ***							
*** Segment B begin ***							
B02	GR	0.000	0.000	0.000	-0.001	0.000	0.000
	T1	0.007	0.000	0.001	0.001	0.006	0.006
	P1	0.001	0.000	0.000	0.000	0.001	0.001
B03 N	GR	0.000	0.000	0.000	-0.001	0.000	0.000
	T1	0.005	0.000	-0.002	0.002	0.008	0.001
	P1	0.001	0.000	0.000	0.000	0.001	0.000
B03 M	GR	0.000	0.000	0.000	-0.001	0.000	-0.001
	T1	0.004	0.001	-0.003	0.003	0.011	-0.001
	P1	0.000	0.000	0.000	0.000	0.001	0.000
B03 F	GR	0.000	0.000	0.000	-0.001	0.000	-0.001
	T1	0.003	0.001	-0.002	0.003	0.013	-0.005
	P1	0.000	0.000	0.000	0.000	0.001	-0.001
B04 N	GR	0.000	0.000	0.000	-0.001	0.000	-0.001
	T1	0.001	0.001	-0.001	0.003	0.013	-0.005
	P1	0.000	0.000	0.000	0.000	0.001	-0.001

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
B04 M	GR	0.000	0.000	0.000	-0.001	0.000	-0.002
	T1	0.001	0.001	0.000	0.003	0.014	-0.005
	P1	0.000	0.000	0.000	0.000	0.001	-0.001
B04 F	GR	0.000	0.000	0.000	0.000	0.000	-0.001
	T1	0.000	0.001	0.000	0.002	0.014	-0.005
	P1	0.000	0.000	0.000	0.000	0.001	-0.001
B05	GR	0.000	0.000	0.000	0.000	0.000	-0.001
	T1	0.000	0.001	0.000	0.002	0.014	-0.004
	P1	0.000	0.000	0.000	0.000	0.001	-0.001
B06	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.001	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
B07	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment B end ***							
*** Segment C begin ***							
5	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.001	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
C01	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment C end ***							
*** Segment D begin ***							
3	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	-0.001	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
D01	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment D end ***							
A15	GR	-0.001	0.000	0.000	0.000	-0.084	0.008
	T1	0.008	0.000	0.001	0.000	0.081	0.008
	P1	0.001	0.000	0.000	0.000	0.008	0.001

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
1	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.008	0.000	0.001	0.000	0.003	0.006
	P1	0.001	0.000	0.000	0.000	0.000	0.001
B02	GR	0.000	0.000	0.000	-0.001	0.000	0.000
	T1	0.007	0.000	0.001	0.001	0.006	0.006
	P1	0.001	0.000	0.000	0.000	0.001	0.001
5	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.001	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
4	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.000	0.000	0.000	0.000	0.002	0.006
	P1	0.000	0.000	0.000	0.000	0.000	0.001
3	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	-0.001	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
6	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.007	0.000	0.001	-0.001	0.003	0.006
	P1	0.001	0.000	0.000	0.000	0.000	0.001
2	GR	-0.001	0.000	0.000	-0.001	-0.005	0.008
	T1	0.009	0.000	0.001	0.000	0.008	0.008
	P1	0.001	0.000	0.000	0.000	0.001	0.001
7	GR	0.000	0.000	0.000	0.000	-0.001	0.000
	T1	0.002	0.000	0.000	-0.001	0.002	0.006
	P1	0.000	0.000	0.000	0.000	0.000	0.001

SUPPORT FORCES

Point/ Supp. ID	Connect/ Type	Load Combination	Dirn	LOCAL		GLOBAL		
				Force	Deform	Dirn	Force	Deform
A14A A14A 1 Stiff	Y - Stop :RIGID	GR	down	1024	0.000	X		-0.013
						Y	-1024	0.000
						Z		0.000
	T1	down	7	0.000	X		0.019	
					Y	-7	0.000	
					Z		0.003	
	P1	down	1	0.000	X		0.002	
					Y	-1	0.000	
					Z		0.000	
1 1 Stiff	Y - Stop :RIGID	GR	down	3293	0.000	X		0.000
						Y	-3293	0.000
						Z		0.000
	T1	down	11	0.000	X		0.008	
					Y	-11	0.000	
					Z		0.001	
	P1	down	1	0.000	X		0.001	
					Y	-1	0.000	
					Z		0.000	

RESTRAINT REACTIONS

Point name	Load combination	FORCES (lb)			MOMENTS (ft-lb)					
		X	Y	Z	Result	X	Y	Z	Result	
A00	Anchor	GR								
		T1	57	-358	9	362	-28	-39	-186	193
		P1	-58	7	-6	59	-2	8	-16	18
A14A	Y - Stop	GR								
		T1	0	-1024	0	1024	0	0	0	0
		P1	0	-7	0	7	0	0	0	0
B07	Anchor	GR								
		T1	5	-167	2	167	-1	0	-4	4
		P1	2	-39	-21	44	0	4	-10	10
5	Anchor	GR								
		T1	0	0	0	0	50	4	2	50
		P1	0	0	0	0	-1	-14	57	59
C01	Anchor	GR								
		T1	22	-231	-6	232	29	3	0	29
		P1	-56	-3	-4864	4864	0	-7	0	7
3	Anchor	GR								
		T1	0	0	0	0	-80	13	-134	156
		P1	0	0	0	0	-8	-33	150	153
D01	Anchor	GR								
		T1	-84	154	-5	175	19	10	0	22
		P1	112	52	4891	4892	7	-14	0	15
1	Y - Stop	GR								
		T1	0	-3293	0	3293	0	0	0	0
		P1	0	-11	0	11	0	0	0	0
5	Anchor	GR								
		T1	0	0	0	0	50	4	2	50
		P1	0	0	0	0	-1	-14	57	59

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)			Result	MOMENTS (ft-lb)			Result
		X	Y	Z		X	Y	Z	
*** Segment A begin ***									
A00	GR	-57	358	-9	362	28	39	186	193
	T1	58	-7	6	59	2	-8	16	18
	P1	6	-1	1	6	0	-1	2	2
A01	GR	-57	209	-9	217	-11	31	108	113
	T1	58	-7	6	59	3	0	18	18
	P1	6	-1	1	6	0	0	2	2
A02	GR	-57	209	-9	217	-16	29	97	103
	T1	58	-7	6	59	3	0	18	18
	P1	6	-1	1	6	0	0	2	2
A03 N	GR	-57	209	-9	217	-17	29	97	102
	T1	58	-7	6	59	3	1	18	18
	P1	6	-1	1	6	0	0	2	2
A03 M	GR	-57	205	-9	212	-39	21	58	74
	T1	58	-7	6	59	5	9	14	17
	P1	6	-1	1	6	0	1	1	2
A03 F	GR	-57	200	-9	208	-50	18	54	76
	T1	58	-7	6	59	6	12	1	13
	P1	6	-1	1	6	1	1	0	1
A04 N	GR	-57	188	-9	197	-56	18	91	109
	T1	58	-7	6	59	11	12	-37	40
	P1	6	-1	1	6	1	1	-4	4
A04 M	GR	-57	181	-9	190	-60	17	84	105
	T1	58	-7	6	59	13	13	-57	59
	P1	6	-1	1	6	1	1	-6	6
A04 F	GR	-57	174	-9	184	-61	14	30	69
	T1	58	-7	6	59	14	15	-63	66
	P1	6	-1	1	6	1	2	-7	7
A05 N	GR	-57	96	-9	112	-61	-27	-573	577
	T1	58	-7	6	59	14	44	-30	55
	P1	6	-1	1	6	1	5	-3	6
A05 M	GR	-57	89	-9	106	-48	-22	-605	608
	T1	58	-7	6	59	13	38	-27	48
	P1	6	-1	1	6	1	4	-3	5

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)			Result	MOMENTS (ft-lb)			Result
		X	Y	Z		X	Y	Z	
A05 F	GR	-57	82	-9	100	-17	-3	-618	618
	T1	58	-7	6	59	10	18	-26	33
	P1	6	-1	1	6	1	2	-3	3
A06 N	GR	-57	64	-9	86	56	54	-618	623
	T1	58	-7	6	59	3	-40	-26	48
	P1	6	-1	1	6	0	-4	-3	5
A06 M	GR	-57	57	-9	81	77	72	-627	636
	T1	58	-7	6	59	0	-59	-25	64
	P1	6	-1	1	6	0	-6	-3	7
A06 F	GR	-57	50	-9	76	85	77	-646	656
	T1	58	-7	6	59	-1	-66	-22	69
	P1	6	-1	1	6	0	-7	-2	7
A07 -	GR	-57	30	-9	65	85	67	-691	700
	T1	58	-7	6	59	-1	-58	-14	60
	P1	6	-1	1	6	0	-6	-1	6
A07 +	GR	-57	7	-9	58	85	67	-691	700
	T1	58	-7	6	59	-1	-58	-14	60
	P1	6	-1	1	6	0	-6	-1	6
A08 -	GR	-57	-9	-9	58	85	59	-691	698
	T1	58	-7	6	59	-1	-52	-7	53
	P1	6	-1	1	6	0	-5	-1	5
A08 +	GR	-57	-114	-9	128	85	59	-691	698
	T1	58	-7	6	59	-1	-52	-7	53
	P1	6	-1	1	6	0	-5	-1	5
A09 -	GR	-57	-130	-9	142	85	50	-578	587
	T1	58	-7	6	59	-1	-46	0	46
	P1	6	-1	1	6	0	-5	0	5
A09 +	GR	-57	-147	-9	158	85	50	-578	587
	T1	58	-7	6	59	-1	-46	0	46
	P1	6	-1	1	6	0	-5	0	5
A10 -	GR	-57	-194	-9	202	85	26	-124	153
	T1	58	-7	6	59	-1	-29	19	35
	P1	6	-1	1	6	0	-3	2	4
A10 +	GR	-57	-291	-9	297	85	26	-124	153
	T1	58	-7	6	59	-1	-29	19	35
	P1	6	-1	1	6	0	-3	2	4

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A11 N	GR	-57	-304	-9	309	85	20	86	122
	T1	58	-7	6	59	-1	-25	25	35
	P1	6	-1	1	6	0	-3	3	4
A11 H	GR	-57	-308	-9	314	84	18	163	185
	T1	58	-7	6	59	0	-23	21	31
	P1	6	-1	1	6	0	-2	2	3
A11 F	GR	-57	-313	-9	318	82	17	207	223
	T1	58	-7	6	59	1	-22	8	24
	P1	6	-1	1	6	0	-2	1	2
A12	GR	-57	-458	-9	462	82	17	211	227
	T1	58	-7	6	59	2	-22	4	23
	P1	6	-1	1	6	0	-2	0	2
A13	GR	-57	-458	-9	462	81	17	213	229
	T1	58	-7	6	59	2	-22	1	23
	P1	6	-1	1	6	0	-2	0	2
A14 N	GR	-57	-460	-9	463	80	17	219	234
	T1	58	-7	6	59	3	-22	-4	23
	P1	6	-1	1	6	0	-2	0	2
A14 M	GR	-57	-467	-9	470	145	9	239	279
	T1	58	-7	6	59	6	-14	-25	29
	P1	6	-1	1	6	1	-1	-3	3
A14 F	GR	-57	-474	-9	477	310	-12	247	397
	T1	58	-7	6	59	10	7	-33	35
	P1	6	-1	1	6	1	1	-3	4
A14A -	GR	-57	-476	-9	479	386	-21	247	459
	T1	58	-7	6	59	11	16	-33	38
	P1	6	-1	1	6	1	2	-3	4
A14A +	GR	-57	549	-9	552	386	-21	247	459
	T1	58	0	6	58	11	16	-33	38
	P1	6	0	1	6	1	2	-3	4
A15	GR	-57	541	-9	544	37	-57	247	256
	T1	58	0	6	58	11	53	-33	63
	P1	6	0	1	6	1	6	-3	7

*** Segment A end ***

*** Segment B begin ***

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
B02	GR	5	19	2	20	13	-5	4	15
	T1	2	-39	-21	44	-38	-25	43	62
	P1	0	-5	-2	5	-5	-2	5	7
B03 N	GR	5	8	2	10	1	0	4	4
	T1	2	-39	-21	44	-1	-23	43	49
	P1	0	-5	-2	5	0	-2	5	6
B03 H	GR	5	5	2	7	-1	1	5	5
	T1	2	-39	-21	44	9	-21	38	44
	P1	0	-5	-2	5	1	-2	5	5
B03 F	GR	5	2	2	5	-2	1	5	6
	T1	2	-39	-21	44	13	-15	28	34
	P1	0	-5	-2	5	1	-1	3	4
B04 N	GR	5	-9	2	10	-2	0	3	4
	T1	2	-39	-21	44	13	-3	5	14
	P1	0	-5	-2	5	1	0	0	2
B04 H	GR	5	-13	2	14	-1	0	0	1
	T1	2	-39	-21	44	11	2	-5	12
	P1	0	-5	-2	5	1	0	-1	2
B04 F	GR	5	-18	2	19	-1	0	-2	3
	T1	2	-39	-21	44	6	4	-9	12
	P1	0	-5	-2	5	1	0	-1	2
B05	GR	5	-163	2	163	-1	0	-3	3
	T1	2	-39	-21	44	5	4	-9	11
	P1	0	-5	-2	5	1	0	-1	2
B06	GR	5	-163	2	163	-1	0	-3	3
	T1	2	-39	-21	44	4	4	-9	11
	P1	0	-5	-2	5	1	0	-1	2
B07	GR	5	-167	2	167	-1	0	-4	4
	T1	2	-39	-21	44	0	4	-10	10
	P1	0	-5	-2	5	0	0	-1	1

*** Segment B end ***

*** Segment C begin ***

5	GR	22	-229	-6	230	-29	-3	0	29
	T1	-56	-3	-4864	4864	0	7	0	7
	P1	-5	2	-435	435	0	1	0	1

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
001	GR	22	-231	-6	232	29	3	0	29
	T1	-56	-3	-4864	4864	0	-7	0	7
	P1	-5	2	-435	435	0	-1	0	1
*** Segment C end ***									
*** Segment D begin ***									
3	GR	-84	156	-5	177	-19	-10	0	22
	T1	112	52	4891	4892	-7	14	0	15
	P1	11	5	437	438	-1	1	0	2
001	GR	-84	154	-5	175	19	10	0	22
	T1	112	52	4891	4892	7	-14	0	15
	P1	11	5	437	438	1	-1	0	2

*** Segment D end ***

ASME B31.3c (1992) CODE COMPLIANCE

Point name	Load combination	(Moments in ft-lb)			S.I.F		(Stress in psi)		Code	Code
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	In	Out	Eq. Load no. type	Stress		
*** Segment A begin ***										
A00	Max P						(3a) HOOP	7304	16700	
	GR + Max P	181	38		1.00	1.00	(18) SUST	4000	16700	
	Cold to T1	15	8		5	1.00	(17) DISP	68	25050	
A01	Max P						(3a) HOOP	7304	16700	
	GR + Max P	94	31		1.00	1.00	(18) SUST	3586	16700	
	Cold to T1	17	0		5	1.00	(17) DISP	67	25050	
A02	Max P						(3a) HOOP	7304	16700	
	GR + Max P	81	29		1.00	1.00	(18) SUST	3527	16700	
	Cold to T1	17	1		5	1.00	(17) DISP	68	25050	
A03 N-	Max P						(3a) HOOP	7304	16700	
	GR + Max P	81	29		1.00	1.00	(18) SUST	3525	16700	
	Cold to T1	17	1		5	1.00	(17) DISP	68	25050	
A03 N+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	81	29		2.56	2.13	(18) SUST	4148	16700	
	Cold to T1	17	1		5	2.56	(17) DISP	168	25050	
A03 M	Max P						(3a) HOOP	7304	16700	
	GR + Max P	36	28		2.56	2.13	(18) SUST	3639	16700	
	Cold to T1	14	5		8	2.56	(17) DISP	145	25050	
A03 F-	Max P						(3a) HOOP	7304	16700	
	GR + Max P	26	69		2.56	2.13	(18) SUST	3887	16700	
	Cold to T1	4	5		12	2.56	(17) DISP	70	25050	
A03 F+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	50	54		1.00	1.00	(18) SUST	3467	16700	
	Cold to T1	6	1		12	1.00	(17) DISP	50	25050	
A04 N-	Max P						(3a) HOOP	7304	16700	
	GR + Max P	55	87		1.00	1.00	(18) SUST	3610	16700	
	Cold to T1	11	37		12	1.00	(17) DISP	151	25050	
A04 N+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	87	55		1.95	1.63	(18) SUST	4039	16700	
	Cold to T1	37	11		12	1.95	(17) DISP	283	25050	
A04 M	Max P						(3a) HOOP	7304	16700	
	GR + Max P	78	28		1.95	1.63	(18) SUST	3880	16700	
	Cold to T1	57	18		0	1.95	(17) DISP	427	25050	

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			CODE COMPLIANCE			(Stress in psi)		
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F. In	S.I.F. Out	Eq. no.	Load type	Code Stress	Code Allow.
A04 F-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	23	15		1.95	1.63	(18)	SUST	3360	16700
	Cold to T1	63	15	14	1.95	1.63	(17)	DISP	468	25050
A04 F+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	23	15		1.00	1.00	(18)	SUST	3245	16700
	Cold to T1	63	15	14	1.00	1.00	(17)	DISP	246	25050
A05 N-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	576	22		1.00	1.00	(18)	SUST	5878	16700
	Cold to T1	30	44	14	1.00	1.00	(17)	DISP	204	25050
A05 N+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	22	576		1.95	1.63	(18)	SUST	7614	16700
	Cold to T1	44	30	14	1.95	1.63	(17)	DISP	370	25050
A05 M	Max P						(3a)	HOOP	7304	16700
	GR + Max P	18	397		1.95	1.63	(18)	SUST	6221	16700
	Cold to T1	38	28	10	1.95	1.63	(17)	DISP	325	25050
A05 F-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	1	16		1.95	1.63	(18)	SUST	3242	16700
	Cold to T1	18	10	26	1.95	1.63	(17)	DISP	175	25050
A05 F+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	16	1		1.00	1.00	(18)	SUST	3193	16700
	Cold to T1	10	18	26	1.00	1.00	(17)	DISP	124	25050
A06 N-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	56	49		1.00	1.00	(18)	SUST	3472	16700
	Cold to T1	3	40	26	1.00	1.00	(17)	DISP	178	25050
A06 N+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	49	56		1.95	1.63	(18)	SUST	3750	16700
	Cold to T1	40	3	26	1.95	1.63	(17)	DISP	307	25050
A06 M	Max P						(3a)	HOOP	7304	16700
	GR + Max P	66	500		1.95	1.63	(18)	SUST	7063	16700
	Cold to T1	59	18	17	1.95	1.63	(17)	DISP	451	25050
A06 F-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	70	648		1.95	1.63	(18)	SUST	8216	16700
	Cold to T1	66	22	1	1.95	1.63	(17)	DISP	497	25050
A06 F+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	648	70		1.00	1.00	(18)	SUST	6241	16700
	Cold to T1	22	66	1	1.00	1.00	(17)	DISP	259	25050

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			CODE COMPLIANCE			(Stress in psi)		
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F. In	S.I.F. Out	Eq. no.	Load type	Code Stress	Code Allow.
A07	Max P						(3a)	HOOP	7304	16700
	GR + Max P	693	61		1.00	1.00	(18)	SUST	6450	16700
	Cold to T1	14	58	1	1.00	1.00	(17)	DISP	224	25050
A08	Max P						(3a)	HOOP	7304	16700
	GR + Max P	691	53		1.00	1.00	(18)	SUST	6440	16700
	Cold to T1	7	52	1	1.00	1.00	(17)	DISP	197	25050
A09	Max P						(3a)	HOOP	7304	16700
	GR + Max P	578	46		1.00	1.00	(18)	SUST	5896	16700
	Cold to T1	0	46	1	1.00	1.00	(17)	DISP	173	25050
A10	Max P						(3a)	HOOP	7304	16700
	GR + Max P	122	23		1.00	1.00	(18)	SUST	5896	16700
	Cold to T1	19	29	1	1.00	1.00	(17)	DISP	371	25050
A11 N-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	88	17		1.00	1.00	(18)	SUST	3544	16700
	Cold to T1	25	25	1	1.00	1.00	(17)	DISP	130	25050
A11 N+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	88	17		2.56	2.13	(18)	SUST	4211	16700
	Cold to T1	25	25	1	2.56	2.13	(17)	DISP	305	25050
A11 M	Max P						(3a)	HOOP	7304	16700
	GR + Max P	166	70		2.56	2.13	(18)	SUST	5269	16700
	Cold to T1	21	16	16	2.56	2.13	(17)	DISP	244	25050
A11 F-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	208	82		2.56	2.13	(18)	SUST	5801	16700
	Cold to T1	8	1	22	2.56	2.13	(17)	DISP	112	25050
A11 F+	Max P						(3a)	HOOP	7304	16700
	GR + Max P	82	208		1.00	1.00	(18)	SUST	4187	16700
	Cold to T1	1	8	22	1.00	1.00	(17)	DISP	89	25050
A12	Max P						(3a)	HOOP	7304	16700
	GR + Max P	82	211		1.00	1.00	(18)	SUST	4201	16700
	Cold to T1	2	4	22	1.00	1.00	(17)	DISP	85	25050
A13	Max P						(3a)	HOOP	7304	16700
	GR + Max P	81	213		1.00	1.00	(18)	SUST	4209	16700
	Cold to T1	2	1	22	1.00	1.00	(17)	DISP	84	25050
A14 N-	Max P						(3a)	HOOP	7304	16700
	GR + Max P	80	218		1.00	1.00	(18)	SUST	4231	16700
	Cold to T1	3	4	22	1.00	1.00	(17)	DISP	86	25050

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			CODE COMPLIANCE		(Stress in psi)		Code Stress	Code Allow.
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F In	S.I.F Out	Eq. Load no. type	Eq. Load no. type		
A14 N+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	80	218		1.95	1.63	(18) SUST	4978	16700	
	Cold to T1	3	4	22	1.95	1.63	(17) DISP	90	25050	
A14 M	Max P						(3a) HOOP	7304	16700	
	GR + Max P	146	162		1.95	1.63	(18) SUST	4975	16700	
	Cold to T1	6	8	27	1.95	1.63	(17) DISP	121	25050	
A14 F-	Max P						(3a) HOOP	7304	16700	
	GR + Max P	311	11		1.95	1.63	(18) SUST	6030	16700	
	Cold to T1	10	7	33	1.95	1.63	(17) DISP	148	25050	
A14 F+	Max P						(3a) HOOP	6451	16700	
	GR + Max P	311	11		1.00	1.00	(18) SUST	5595	16700	
	Cold to T1	10	7	33	1.00	1.00	(17) DISP	246	25050	
A14A	Max P						(3a) HOOP	6451	16700	
	GR + Max P	387	19		1.00	1.00	(18) SUST	6294	16700	
	Cold to T1	11	16	33	1.00	1.00	(17) DISP	267	25050	
A15	Max P						(3a) HOOP	6451	16700	
	GR + Max P	38	51		1.00	1.00	(18) SUST	3334	16700	
	Cold to T1	11	53	33	1.00	1.00	(17) DISP	442	25050	
*** Segment A end ***										
*** Segment B begin ***										
B02	Max P						(3a) HOOP	6451	16700	
	GR + Max P	8	7		1.00	1.00	(18) SUST	2844	16700	
	Cold to T1	38	25	43	1.00	1.00	(17) DISP	433	25050	
B03 N-	Max P						(3a) HOOP	6451	16700	
	GR + Max P	0	2		1.00	1.00	(18) SUST	2765	16700	
	Cold to T1	1	23	43	1.00	1.00	(17) DISP	338	25050	
B03 N+	Max P						(3a) HOOP	6451	16700	
	GR + Max P	2	0		1.78	1.48	(18) SUST	2779	16700	
	Cold to T1	23	1	43	1.78	1.48	(17) DISP	414	25050	
B03 M	Max P						(3a) HOOP	6451	16700	
	GR + Max P	1	7		1.78	1.48	(18) SUST	2839	16700	
	Cold to T1	21	21	33	1.78	1.48	(17) DISP	406	25050	
B03 F-	Max P						(3a) HOOP	6451	16700	
	GR + Max P	0	9		1.78	1.48	(18) SUST	2866	16700	
	Cold to T1	15	28	13	1.78	1.48	(17) DISP	354	25050	

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			CODE COMPLIANCE		(Stress in psi)		Code Stress	Code Allow.
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F In	S.I.F Out	Eq. Load no. type	Eq. Load no. type		
B03 F+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	9	0		1.00	1.00	(18) SUST	4007	16700	
	Cold to T1	28	15	13	1.00	1.00	(17) DISP	128	25050	
B04 N-	Max P						(3a) HOOP	7304	16700	
	GR + Max P	4	0		1.00	1.00	(18) SUST	3132	16700	
	Cold to T1	5	3	13	1.00	1.00	(17) DISP	53	25050	
B04 N+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	4	0		2.56	2.13	(18) SUST	3159	16700	
	Cold to T1	5	3	13	2.56	2.13	(17) DISP	70	25050	
B04 M	Max P						(3a) HOOP	7304	16700	
	GR + Max P	1	0		2.56	2.13	(18) SUST	3122	16700	
	Cold to T1	5	6	9	2.56	2.13	(17) DISP	77	25050	
B04 F-	Max P						(3a) HOOP	7304	16700	
	GR + Max P	4	0		2.56	2.13	(18) SUST	3159	16700	
	Cold to T1	9	6	4	2.56	2.13	(17) DISP	101	25050	
B04 F+	Max P						(3a) HOOP	7304	16700	
	GR + Max P	0	4		1.00	1.00	(18) SUST	3131	16700	
	Cold to T1	6	9	4	1.00	1.00	(17) DISP	44	25050	
B05	Max P						(3a) HOOP	7304	16700	
	GR + Max P	0	4		1.00	1.00	(18) SUST	3132	16700	
	Cold to T1	5	9	4	1.00	1.00	(17) DISP	42	25050	
B06	Max P						(3a) HOOP	7304	16700	
	GR + Max P	0	4		1.00	1.00	(18) SUST	3133	16700	
	Cold to T1	4	9	4	1.00	1.00	(17) DISP	41	25050	
B07	Max P						(3a) HOOP	7304	16700	
	GR + Max P	0	5		1.00	1.00	(18) SUST	3138	16700	
	Cold to T1	0	10	4	1.00	1.00	(17) DISP	39	25050	
*** Segment B end ***										
*** Segment C begin ***										
5	Max P						(3a) HOOP	6905	16700	
	GR + Max P	28	2		1.00	1.00	(18) SUST	2920	16700	
	Cold to T1	0	7	0	1.00	1.00	(17) DISP	35	25050	
C01	Max P						(3a) HOOP	6905	16700	
	GR + Max P	29	2		1.00	1.00	(18) SUST	2921	16700	
	Cold to T1	0	7	0	1.00	1.00	(17) DISP	35	25050	
*** Segment C end ***										

ASME B31.3c (1992) CODE COMPLIANCE

Point name	Load combination	Moments in ft-lb)			(Stress in psi)			
		In-Pl. Moment	Out-Pl. Moment	Torsion	S.I.F. In	S.I.F. Out	Eq. Load no.	Code type

*** Segment D begin ***

3	Max P							(3a) HOOP	6905	16700
	GR + Max P	20	9		1.00	1.00		(18) SUST	2876	16700
	Cold to T1	7	14	0	1.00	1.00		(17) DISP	78	25050
D01	Max P							(3a) HOOP	6905	16700
	GR + Max P	20	9		1.00	1.00		(18) SUST	2876	16700
	Cold to T1	7	14	0	1.00	1.00		(17) DISP	78	25050

*** Segment D end ***

SYSTEM SUMMARY

Maximum displacements (in)

Maximum X :	-0.046	Point :	A06 F	Load Comb.:	GR
Maximum Y :	-0.207	Point :	A06 F	Load Comb.:	GR
Maximum Z :	-0.046	Point :	A04 F	Load Comb.:	GR
Max. total:	0.214	Point :	A06 F	Load Comb.:	GR

Maximum rotations (deg)

Maximum X :	0.175	Point :	A05 N	Load Comb.:	GR
Maximum Y :	-0.096	Point :	A14 N	Load Comb.:	GR
Maximum Z :	0.186	Point :	A11 N	Load Comb.:	GR
Max. total:	0.218	Point :	A04 F	Load Comb.:	GR

Maximum restraint forces(lb)

Maximum X :	112	Point :	D01	Load Comb.:	T1
Maximum Y :	-3293	Point :	1	Load Comb.:	GR
Maximum Z :	4891	Point :	D01	Load Comb.:	T1
Max. total:	4892	Point :	D01	Load Comb.:	T1

Maximum restraint moments(ft-lb)

Maximum X :	-80	Point :	3	Load Comb.:	GR
Maximum Y :	-39	Point :	A00	Load Comb.:	GR
Maximum Z :	-186	Point :	A00	Load Comb.:	GR
Max. total:	193	Point :	A00	Load Comb.:	GR

SYSTEM SUMMARY

Maximum pipe forces (lb)

Maximum X :	112	Point :	3	Load Comb.:	T1
Maximum Y :	549	Point :	A14A	Load Comb.:	GR
Maximum Z :	4891	Point :	3	Load Comb.:	T1
Max. total:	4892	Point :	3	Load Comb.:	T1

Maximum pipe moments (ft-lb)

Maximum X :	386	Point :	A14A	Load Comb.:	GR
Maximum Y :	77	Point :	A06 F	Load Comb.:	GR
Maximum Z :	-691	Point :	A07	Load Comb.:	GR
Max. total:	700	Point :	A07	Load Comb.:	GR

SYSTEM SUMMARY

Maximum sustained stress

Point	:	A06 F
Stress psi	:	8216
Allowable psi	:	16700
Ratio	:	0.49
Load combination	:	GR + Max P

Maximum displacement stress

Point	:	A06 F
Stress psi	:	497
Allowable psi	:	25050
Ratio	:	0.02
Load combination	:	Cold to T1

Maximum hoop stress

Point	:	A00
Stress psi	:	7304
Allowable psi	:	16700
Ratio	:	0.44
Load combination	:	Max P

Maximum sustained stress ratio

Point	:	A06 F
Stress psi	:	8216
Allowable psi	:	16700
Ratio	:	0.49
Load combination	:	GR + Max P

Maximum displacement stress ratio

Point	:	A06 F
Stress psi	:	497
Allowable psi	:	25050
Ratio	:	0.02
Load combination	:	Cold to T1

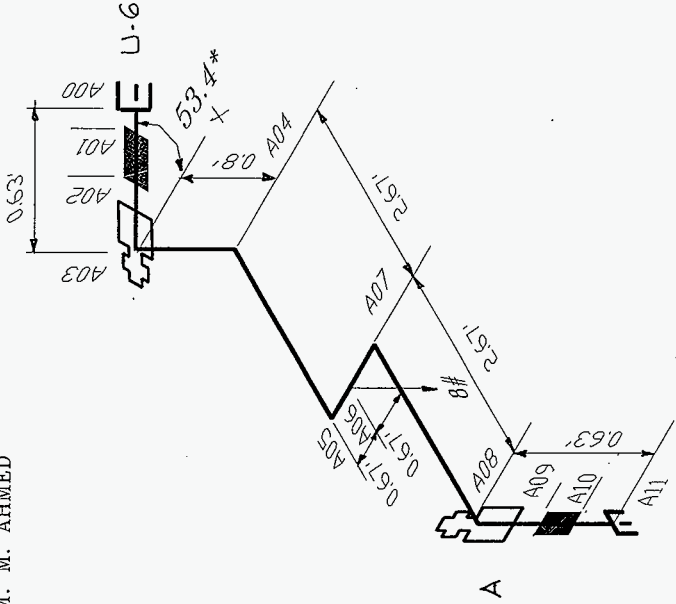
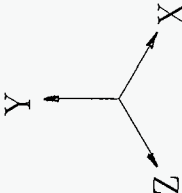
Maximum hoop stress ratio

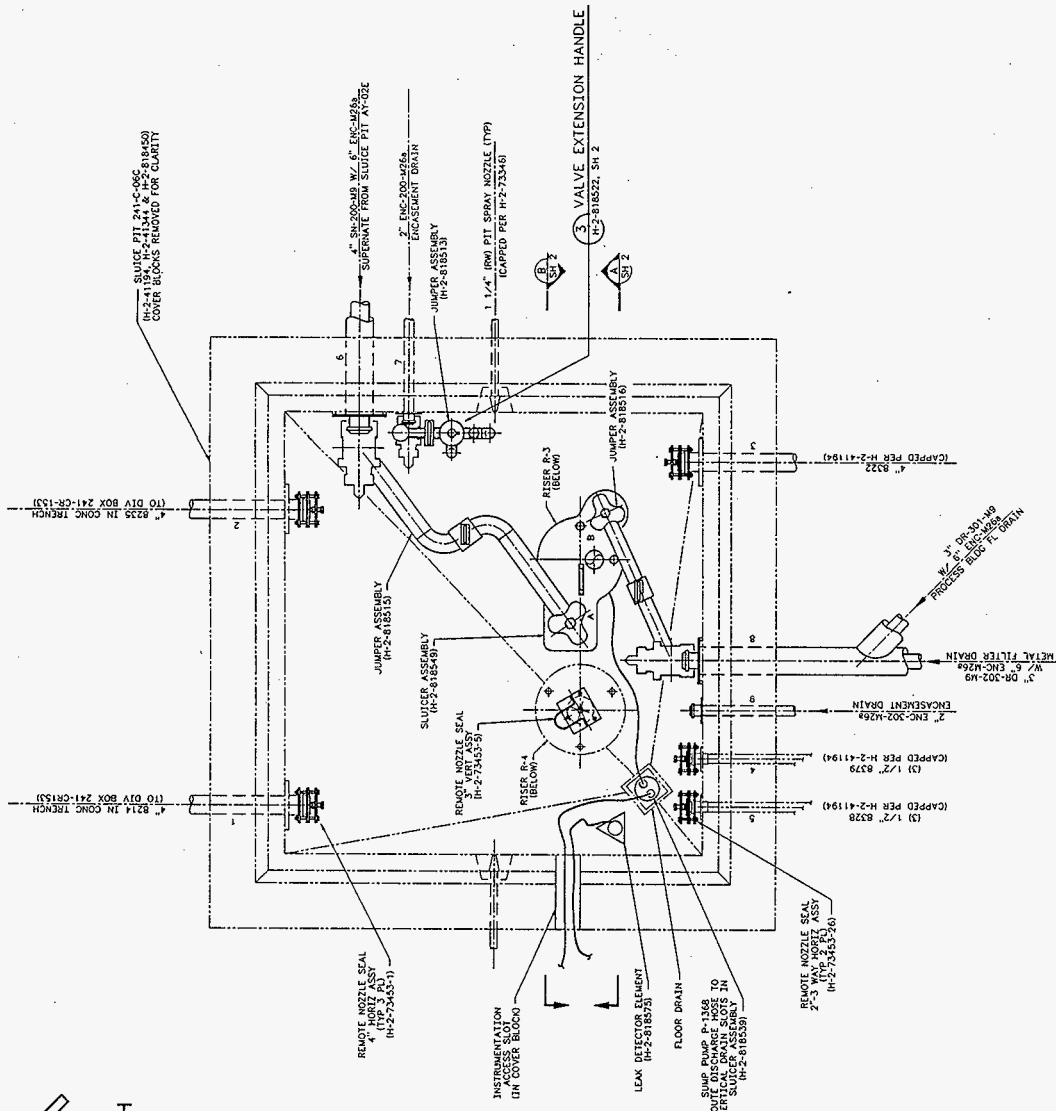
Point	:	A00
Stress psi	:	7304
Allowable psi	:	16700
Ratio	:	0.44
Load combination	:	Max P

APPDX - C

JUMPER ASSEMBLY U6-A / SLUICE PIT C-06C

DRAWING: H-2-818515 FILE NAME: J6A-S AP FILE NAME: MA-J6A
PREPARED BY: M. M. AHMED





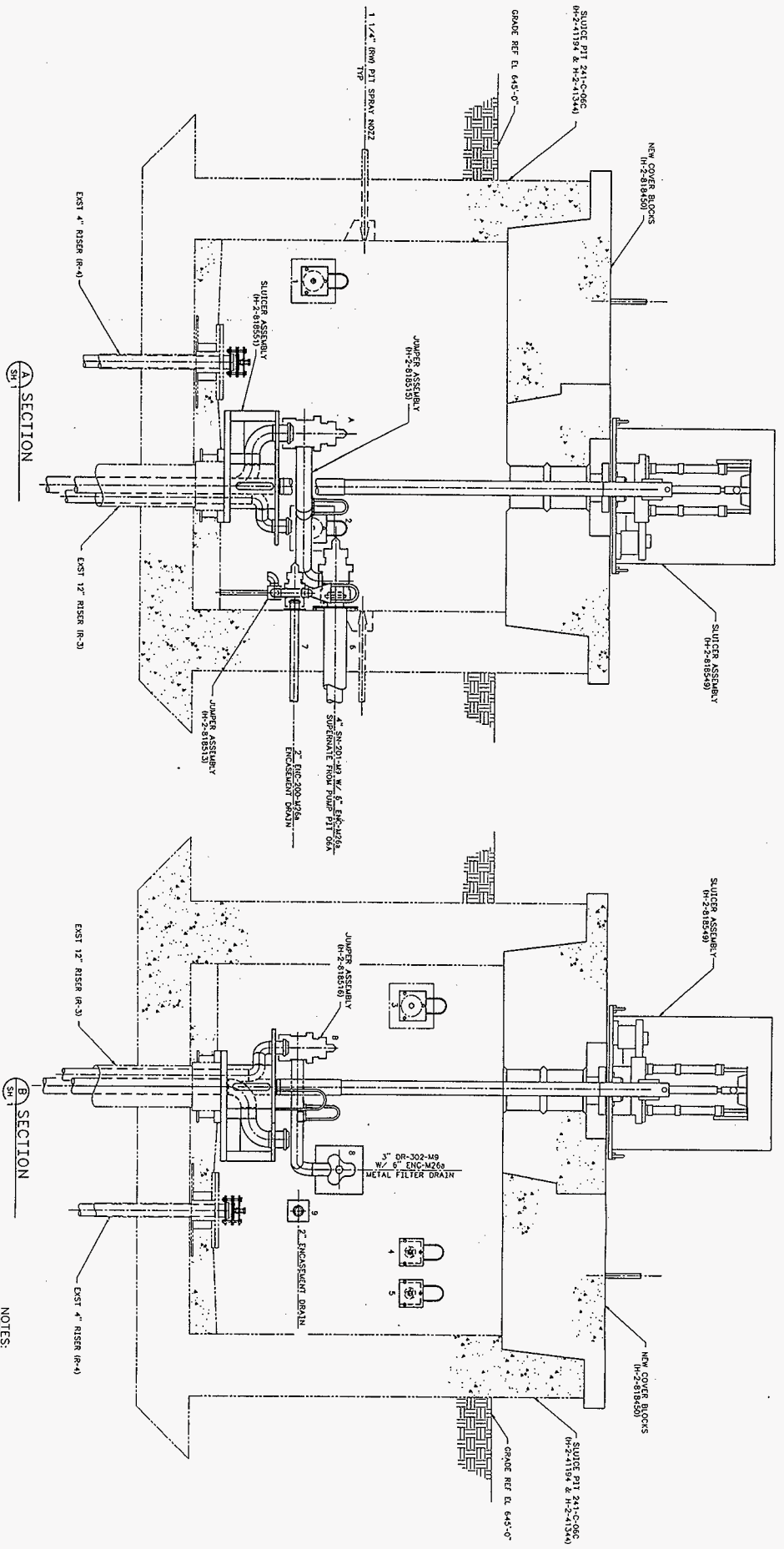
PLAN
SLUICE PIT
241-C-06C

NOTES:
1. FOR GENERAL NOTES, LEGEND AND KEY PLAN SEE H-2-81852-8.



CONTRACT NO.	H-2-81852	DATE	11/85
DESIGNER	U.S. DEPARTMENT OF ENERGY		
CONTRACTOR	U.S. DEPARTMENT OF ENERGY		
PROJECT	SLUICE PIT 241-C-06C		
DRAWING NO.	H-2-81852-0	REV.	0
DATE	11/85	BY	ER4379
CHECKED	ER4379	DATE	11/85
APPROVED	ER4379	DATE	11/85

NO.	DESCRIPTION	DATE	BY	CHKD.
1	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379
2	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379
3	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379
4	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379
5	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379
6	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379
7	ISSUED FOR CONSTRUCTION	11/85	ER4379	ER4379

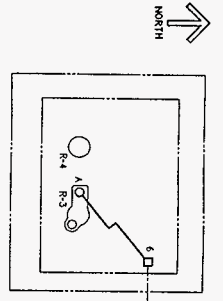
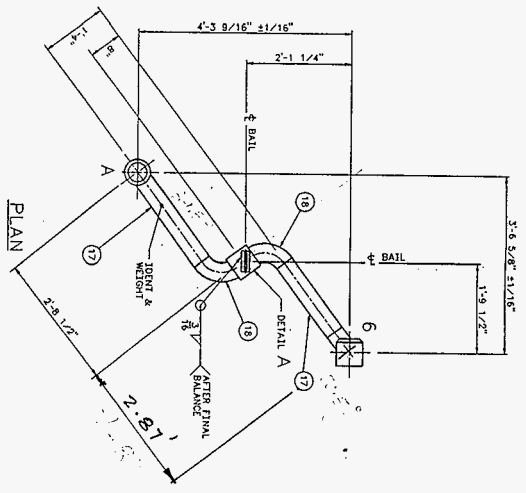


NOTES:
1. FOR NOTES, SEE SHEET 1.

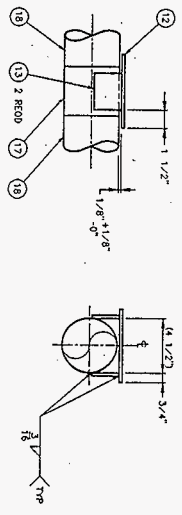


U.S. DEPARTMENT OF ENERGY KATONAH HANDED COMPANY JUNIPER PILING SUICER ARRANGEMENT SUICER PI 241-C-06C W-300 TANK ELECTRICAL STUDIES H-2-818526		SHEET NO. 10 PROJECT NO. H-2-818526 DATE 11-15-74 DRAWN BY H-2-818526 CHECKED BY 0 APPROVED BY 0
DESIGNED BY DRAWN BY CHECKED BY APPROVED BY	DATE SCALE SHEET NO. PROJECT NO.	SHEET NO. 10 PROJECT NO. H-2-818526 DATE 11-15-74 DRAWN BY H-2-818526 CHECKED BY 0 APPROVED BY 0

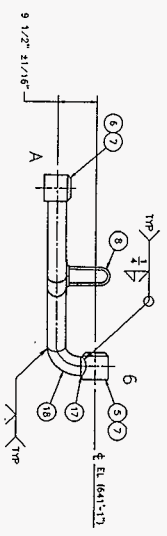
2. PLOT SCALE: 1/4" = 1'-0"
KENDCO



241-C-06C
SLUICE PIT
KEY PLAN
SCALE NONE



DETAIL A
SCALE 3\"/>



ELEVATION

1 ASSEMBLY
SCALE 1\"/>

QTY	PART NUMBER	PARTS/MATERIAL LIST	MATERIAL/REFERENCE	SHEET NO.
1	ASSEMBLY			2
1	H-2-32440-1	CONN. NOZZLE 4"		3
1	H-2-32440-2	CONN. VERT. 4"		4
2	H-2-38977-8	GASKET 4" TYPE 111		6
1	H-2-90151-3	LIFTING BAIL		7
1				8
1				9
1				10
1				11
1		PLATE 1/4" THK x 6" x 6"	ASTM A 36	12
2		BAR 1/4" THK x 2 1/4" x 3"	ASTM A 240 304L	13
1				14
1				15
1				16
AR		PIPE 4" SCHED 40S	ASTM A 312 OR 12 304L	17
3		ELBOW 90° LR 4" SCHED 40S BW	ASTM A 403 MF-5 304L	18
				19
				20

NOTE:

1. FABRICATION AND TESTING SHALL BE IN ACCORDANCE WITH HANDBOOK SPECIFICATION HS-B5-0064 REV B.
2. HYDROSTATIC TEST AT 480 PSIG.
3. JUMPER WEIGHT IS 525 LB APPROX.
4. USE 101 COAT OF AMERCOCK 400 IN PLACE OF AMERCOCK 400.



NO.	REV.	DESCRIPTION	DATE	BY	CHKD.
1		ASSEMBLY			
2		PILOT SCALE 1:12			
3					
4					
5					
6					
7					

NO.	REV.	DESCRIPTION	DATE	BY	CHKD.
1		ASSEMBLY			
2		PILOT SCALE 1:12			
3					
4					
5					
6					
7					

NO.	REV.	DESCRIPTION	DATE	BY	CHKD.
1		ASSEMBLY			
2		PILOT SCALE 1:12			
3					
4					
5					
6					
7					

NO.	REV.	DESCRIPTION	DATE	BY	CHKD.
1		ASSEMBLY			
2		PILOT SCALE 1:12			
3					
4					
5					
6					
7					

NO.	REV.	DESCRIPTION	DATE	BY	CHKD.
1		ASSEMBLY			
2		PILOT SCALE 1:12			
3					
4					
5					
6					
7					

POINT DATA LISTING

POINT NAME	TYPE	----OFFSETS (ft)----			PIPE ID	DESCRIPTION
		X	Y	Z		
*** SEGMENT A						
A00	Run	0	0	0	4PIPE	
A01	Flex	-0.11	0	0.15		Wt= 144 lb , Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
A02	Run	-0.02	0	0.03		
A03	Bend	-0.22	0	0.33		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
A04	Bend	0	-0.84	0		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A05	Bend	0	0	2.67		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A06	Run	0.67	0	0		
A07	Bend	0.67	0	0		Long Elbow, Radius = 6.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 1.95, Out = 1.63 Flex = 5.272
A08	Bend	0	0	2.67		Short Elbow, Radius = 4.00 inch Bend angle change = 90.00 deg Mid point at 50.00 percent SIF - In 2.56, Out = 2.13 Flex = 7.908
A09	Flex	0	-0.40	0		Wt= 144 lb , Ar= 3.2 sq.in Axial= 23300000 Tors= 304.0 Y-Shear= RIGID Bend= 2080.0 Z-Shear= RIGID Bend= 2080.0
A10	Run	0	-0.04	0		
A11	Run	0	-0.19	0		

Total weight of empty pipes : 104 lb

COMPONENT DATA LISTING

POINT NAME	---COORDINATE(ft)---			DATA TYPE	DESCRIPTION
	X	Y	Z		
*** SEGMENT A					
A00	0.00	0.00	0.00	ANCHOR	Rigid Thermal movements : None
A01	-0.11	0.00	0.15		
A02	-0.13	0.00	0.18		
A03	N -0.17	0.00	0.23		
A03	-0.35	0.00	0.51	TI	
A03	M -0.30	-0.10	0.43		
A03	F -0.35	-0.33	0.31		
A04	N -0.35	-0.34	0.51		
A04	-0.35	-0.84	0.51	TI	
A04	M -0.35	-0.69	0.66		
A04	F -0.35	-0.84	1.01		
A05	N -0.35	-0.84	2.68		
A05	-0.35	-0.84	3.18	TI	
A05	M -0.21	-0.84	3.04		
A05	F 0.15	-0.84	3.18		
A06	0.32	-0.84	3.18	WEIGHT 8 lb	, No offsets
A07	N 0.49	-0.84	3.18		
A07	0.99	-0.84	3.18	TI	
A07	M 0.84	-0.84	3.33		
A07	F 0.99	-0.84	3.68		
A08	N 0.99	-0.84	5.52		
A08	0.99	-0.84	5.85	TI	
A08	M 0.99	-0.94	5.75		
A08	F 0.99	-1.17	5.85		
A09	0.99	-1.24	5.85		
A10	0.99	-1.28	5.85		
A11	0.99	-1.47	5.85	ANCHOR	Rigid Thermal movements : None

Number of points in the system : 27

PIPE DATA LISTING

Pipe ID/ Material	Nom/ Sch	O.D. inch	-----Thickness(inch)-----				Spec Grav	Weight(lb/ft)			
			W.Th.	Corr	Mill	Insu		Ling	Pipe	Other	Total
4PIPE	4	4.500	0.237	0.06	0.03	0	0	1.12	11.04	0	17.22
AU	STD										

MATERIAL DATA LISTING

Material Name	Pipe ID	Density lb/cu.ft	Pois. Ratio	Temper. deg F	Modulus E6 psi	Expans. in/100ft	Allow. psi

TEMPERATURE AND PRESSURE DATA

POINT NAME	CASE 1			CASE 2			CASE 3		
	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft

*** SEGMENT A

A00	325	180	1.544
A11	325	180	1.544

ANALYSIS SUMMARY

Current model revision number : 1

Static -	Date and Time of analysis	Sep 11, 1995	12:54 PM
	Model Revision Number	1	
	Number of load cases	2	
	Load cases analyzed	GR T1	
	Gaps/Friction/Yielding considered	No	
	Hanger design run	No	
	Cut short included	No	
	Weight of contents included	Yes	
	Pressure stiffening case	0	
	Water elevation for buoyancy loads	Not considered	
Modal -	Date and Time of analysis	Sep 11, 1995	12:54 PM
	Model Revision Number	1	
	Number of modes	1	
	Cutoff frequency (Hz)	33.0	
	Weight of contents included	Yes	
	Pressure stiffening case	0	
	Water elevation for buoyancy loads	Not considered	
Response -	Date and Time of analysis	Sep 11, 1995	12:55 PM
	Model Revision Number	1	
	Number of load cases	3	
	Load cases analyzed	R1 R2 R3	
	Date and time of modal analysis	Sep 11, 1995	12:54 PM
	Number of modes	1	
	Cutoff frequency (Hz)	33.0	
	Model revision of modal analysis	1	
	Weight of contents included	Yes	
	Pressure stiffening case	0	
	Water elevation for buoyancy loads	Not considered	

CODE COMPLIANCE COMBINATIONS

Combination	Category	Method	Load	Factor	Allowable	Remarks
GR + Max P	Sustain	Sum	Gravity Max Long	1.00 1.00	Automatic	Default
Cold to T1	Expansion	Sum	Thermal 1	1.00	Automatic	Default
Sus. + R1	Occasion	Abs sum	Response 1 Max Sus	1.00 1.00	Automatic	Default
Sus. + R2	Occasion	Abs sum	Response 2 Max Sus	1.00 1.00	Automatic	Default
Sus. + R3	Occasion	Abs sum	Response 3 Max Sus	1.00 1.00	Automatic	Default
Max P	Hoop		Max Hoop	1.00	Automatic	Default

OTHER USER COMBINATIONS

Combination	Method	Load	Factor	Remarks
GR	Sum	Gravity	1.00	Default
T1	Sum	Thermal 1	1.00	Default
R1	Sum	Response 1	1.00	Default
R2	Sum	Response 2	1.00	Default
R3	Sum	Response 3	1.00	Default
GR+T1	Sum	Gravity Thermal 1	1.00 1.00	User
SEIS	Abs sum	Response 1 Response 2 Response 3	1.00 1.00 1.00	User
TOTAL	Abs sum	GR+T1 SEIS	1.00 1.00	User

Design Pressure Factor 1.00
 Minimum stress ratio used in reports... 0.00
 Include corrosion in stress calcs. Y
 Include torsion in code stress N
 Include axial force in code stress N
 Longitudinal pressure calculation PD/4t
 Include rigorous pressure Not analyzed

CODE COMPLIANCE

Y - Factor 0.40
 Weld efficiency factor 1.00
 Range reduction factor 1.00

RESPONSE SPECTRUM LOAD CASES :

Number of load cases analysed : 3

Load case 1 - R1

Missing mass : No
 ZPA : No

Combination method : SRSS

X- Spectrum : SC1
 Multiplier : 1.00

SC1

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.03	0.250	0.06
0.400	0.10	0.600	0.16	1.100	0.28
1.640	0.42	8.000	0.42	12.000	0.34
20.000	0.26	33.000	0.20	100.000	0.20

Load case 2 - R2

Missing mass : No
 ZPA : No

Combination method : SRSS

Y- Spectrum : SC1
 Multiplier : 1.00

SC1

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.03	0.250	0.06
0.400	0.10	0.600	0.16	1.100	0.28
1.640	0.42	8.000	0.42	12.000	0.34
20.000	0.26	33.000	0.20	100.000	0.20

Load case 3 - R3

Missing mass : No
 ZPA : No

Combination method : SRSS

Z- Spectrum : SC1
 Multiplier : 1.00

SC1

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.03	0.250	0.06
0.400	0.10	0.600	0.16	1.100	0.28
1.640	0.42	8.000	0.42	12.000	0.34
20.000	0.26	33.000	0.20	100.000	0.20

F R E Q U E N C I E S

Mode Number	Frequency (Rads/sec)	Frequency (Hertz)	Period (Sec)	Participation factors		
				X	Y	Z
1	207.2037	32.9775	0.030	0.497	-0.054	0.000

MODE SHAPES

Point name	Mode	Frequency	TRANSLATIONS			ROTATIONS (deg)		
			X	Y	Z	X	Y	Z
*** Segment A begin ***								
A00	1	32.9775	0.000	0.000	0.000	0.000	0.000	0.000
A01	1	32.9775	0.001	-0.001	0.001	0.030	0.040	0.030
A02	1	32.9775	0.009	-0.011	0.005	0.601	2.387	3.813
A03 N	1	32.9775	0.036	-0.046	0.023	0.610	2.399	3.823
A03 M	1	32.9775	0.224	-0.186	0.076	0.882	2.703	4.130
A03 F	1	32.9775	0.482	-0.250	0.059	1.009	2.796	4.305
A04 N	1	32.9775	0.488	-0.250	0.057	1.009	2.797	4.305
A04 M	1	32.9775	0.895	-0.281	-0.019	0.955	2.835	4.238
A04 F	1	32.9775	1.234	-0.344	-0.047	0.711	2.763	4.118
A05 N	1	32.9775	2.147	-0.555	-0.049	0.503	2.289	3.703
A05 M	1	32.9775	2.280	-0.475	-0.098	0.416	1.164	3.597
A05 F	1	32.9775	2.303	-0.233	-0.135	0.451	-0.177	3.259
A06	1	32.9775	2.303	-0.117	-0.127	0.433	-0.303	3.220
A07 N	1	32.9775	2.302	-0.003	-0.115	0.415	-0.437	3.178
A07 M	1	32.9775	2.253	0.202	-0.023	0.489	-2.227	2.738
A07 F	1	32.9775	2.015	0.245	0.065	0.533	-4.064	2.539
A08 N	1	32.9775	0.227	0.044	0.063	0.550	-4.927	1.838
A08 M	1	32.9775	0.013	0.014	0.050	0.668	-5.135	1.651
A08 F	1	32.9775	-0.022	0.000	0.014	0.689	-5.226	1.328
A09	1	32.9775	-0.004	0.000	0.004	0.686	-5.220	1.319
A10	1	32.9775	0.001	0.000	0.001	0.018	-0.016	0.011
A11	1	32.9775	0.000	0.000	0.000	0.000	0.000	0.000
*** Segment A end ***								

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
*** Segment A begin ***							
A00	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	R1	0.000	0.000	0.000	0.000	0.000	0.000
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
A01	SEIS	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.000	0.000	0.000	0.000	0.000
	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	-0.002	0.000	0.002	0.001	-0.003	0.000
	R1	0.000	0.000	0.000	0.000	0.000	0.000
	R2	0.000	0.000	0.000	0.000	0.000	0.000
A02	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.002	0.000	0.002	0.001	-0.003	0.000
	SEIS	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.002	0.000	0.002	0.001	0.003	0.000
	GR	0.000	0.000	0.000	0.011	0.001	0.004
	T1	-0.002	0.000	0.002	0.209	-0.128	-0.211
A03 N	R1	0.000	0.000	0.000	0.001	0.002	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.002	0.000	0.002	0.220	-0.127	-0.206
	SEIS	0.000	0.000	0.000	0.001	0.002	0.004
	TOTAL	0.002	0.000	0.002	0.221	0.129	0.210
A03 M	GR	0.000	0.000	0.000	0.011	0.001	0.004
	T1	-0.004	-0.001	0.002	0.210	-0.129	-0.211
	R1	0.000	0.000	0.000	0.001	0.002	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.004	-0.001	0.002	0.220	-0.127	-0.206
A03 F	SEIS	0.000	0.000	0.000	0.001	0.002	0.004
	TOTAL	0.004	0.001	0.002	0.221	0.130	0.210
	GR	0.000	-0.001	0.000	0.011	0.001	0.005
	T1	-0.016	-0.006	-0.004	0.220	-0.126	-0.202
	R1	0.000	0.000	0.000	0.001	0.002	0.004
	R2	0.000	0.000	0.000	0.000	0.000	0.000
A05 M	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.016	-0.006	-0.004	0.231	-0.125	-0.198
	SEIS	0.000	0.000	0.000	0.001	0.003	0.004
	TOTAL	0.016	0.007	0.004	0.232	0.128	0.202

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DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A03 F	GR	0.000	-0.001	-0.001	0.011	0.001	0.004
	T1	-0.029	-0.011	-0.015	0.205	-0.115	-0.183
	R1	0.000	0.000	0.000	0.001	0.003	0.004
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.028	-0.012	-0.015	0.216	-0.114	-0.179
	SEIS	0.000	0.000	0.000	0.001	0.003	0.004
	TOTAL	0.029	0.012	0.015	0.217	0.116	0.184
A04 N	GR	0.000	-0.001	-0.001	0.011	0.001	0.004
	T1	-0.029	-0.011	-0.015	0.205	-0.114	-0.183
	R1	0.000	0.000	0.000	0.001	0.003	0.004
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.028	-0.012	-0.016	0.216	-0.113	-0.179
	SEIS	0.000	0.000	0.000	0.001	0.003	0.004
	TOTAL	0.029	0.012	0.016	0.217	0.116	0.184
A04 M	GR	0.001	-0.001	-0.002	0.012	0.001	0.003
	T1	-0.045	-0.022	-0.026	0.141	-0.095	-0.153
	R1	0.001	0.000	0.000	0.001	0.003	0.004
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.044	-0.023	-0.028	0.152	-0.094	-0.149
	SEIS	0.001	0.000	0.000	0.001	0.003	0.004
	TOTAL	0.045	0.023	0.028	0.153	0.097	0.153
A04 F	GR	0.001	-0.002	-0.002	0.012	0.000	0.003
	T1	-0.055	-0.030	-0.024	0.041	-0.049	-0.129
	R1	0.001	0.000	0.000	0.001	0.002	0.004
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.054	-0.033	-0.026	0.053	-0.049	-0.126
	SEIS	0.001	0.000	0.000	0.001	0.003	0.004
	TOTAL	0.055	0.033	0.026	0.053	0.052	0.130
A05 N	GR	0.001	-0.006	-0.002	0.009	-0.001	0.002
	T1	-0.063	-0.033	0.001	-0.017	0.015	-0.088
	R1	0.002	0.000	0.000	0.000	0.002	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.062	-0.039	-0.001	-0.009	0.014	-0.086
	SEIS	0.002	0.001	0.000	0.000	0.002	0.004
	TOTAL	0.064	0.039	0.001	0.009	0.017	0.090

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A05 M	GR	0.001	-0.006	-0.002	0.003	-0.003	0.003
	T1	-0.055	-0.032	0.004	-0.057	0.125	-0.036
	R1	0.002	0.000	0.000	0.000	0.001	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.055	-0.039	0.002	-0.054	0.122	-0.063
	SEIS	0.002	0.000	0.000	0.000	0.001	0.004
	TOTAL	0.057	0.039	0.002	0.054	0.124	0.067
A05 F	GR	0.000	-0.006	-0.002	0.001	-0.004	0.004
	T1	-0.045	-0.034	-0.007	-0.075	0.199	-0.036
	R1	0.002	0.000	0.000	0.000	0.000	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.044	-0.040	-0.009	-0.074	0.196	-0.032
	SEIS	0.002	0.000	0.000	0.000	0.000	0.003
	TOTAL	0.047	0.040	0.009	0.074	0.196	0.035
A06	GR	0.000	-0.006	-0.001	0.000	-0.004	0.004
	T1	-0.042	-0.035	-0.015	-0.077	0.202	-0.035
	R1	0.002	0.000	0.000	0.000	0.000	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.042	-0.041	-0.016	-0.077	0.198	-0.030
	SEIS	0.002	0.000	0.000	0.000	0.000	0.003
	TOTAL	0.044	0.041	0.016	0.077	0.198	0.034
A07 N	GR	0.000	-0.006	-0.001	0.000	-0.004	0.004
	T1	-0.040	-0.036	-0.022	-0.080	0.202	-0.034
	R1	0.002	0.000	0.000	0.000	0.000	0.003
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.039	-0.042	-0.023	-0.080	0.198	-0.029
	SEIS	0.002	0.000	0.000	0.000	0.000	0.003
	TOTAL	0.041	0.042	0.023	0.081	0.198	0.033
A07 M	GR	0.000	-0.005	-0.001	-0.003	-0.003	0.006
	T1	-0.029	-0.036	-0.034	-0.089	0.158	-0.026
	R1	0.002	0.000	0.000	0.000	0.002	0.002
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.028	-0.041	-0.035	-0.092	0.154	-0.020
	SEIS	0.002	0.000	0.000	0.000	0.002	0.003
	TOTAL	0.030	0.041	0.035	0.092	0.157	0.023

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A07 F	GR	0.000	-0.005	-0.001	-0.008	-0.002	0.007
	T1	-0.018	-0.029	-0.033	-0.099	0.078	-0.025
	R1	0.002	0.009	0.000	0.000	0.004	0.002
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	-0.018	-0.034	-0.034	-0.107	0.076	-0.018
	SEIS	0.002	0.000	0.000	0.000	0.004	0.003
	TOTAL	0.020	0.034	0.034	0.107	0.080	0.020
A08 N	GR	0.000	-0.001	-0.001	-0.011	-0.001	0.006
	T1	0.002	0.007	-0.005	-0.076	0.036	-0.035
	R1	0.000	0.000	0.000	0.000	0.004	0.002
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.001	0.006	-0.006	-0.087	0.035	-0.029
	SEIS	0.000	0.000	0.000	0.000	0.005	0.002
	TOTAL	0.002	0.006	0.006	0.088	0.040	0.031
A08 M	GR	0.000	0.000	-0.001	-0.012	-0.001	0.006
	T1	0.002	0.008	0.000	-0.019	0.027	-0.039
	R1	0.000	0.000	0.000	0.001	0.005	0.001
	R2	0.000	0.000	0.000	0.000	0.001	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.002	0.007	-0.001	-0.031	0.025	-0.033
	SEIS	0.000	0.000	0.000	0.001	0.005	0.002
	TOTAL	0.002	0.007	0.001	0.031	0.030	0.035
A08 F	GR	0.000	0.000	0.000	-0.010	-0.002	0.005
	T1	0.001	0.004	0.001	0.014	0.025	-0.040
	R1	0.000	0.000	0.000	0.001	0.005	0.001
	R2	0.000	0.000	0.000	0.000	0.001	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.001	0.004	0.000	0.003	0.024	-0.035
	SEIS	0.000	0.000	0.000	0.001	0.005	0.001
	TOTAL	0.001	0.004	0.000	0.004	0.029	0.036
A09	GR	0.000	0.000	0.000	-0.010	-0.002	0.005
	T1	0.000	0.003	0.000	0.014	0.025	-0.040
	R1	0.000	0.000	0.000	0.000	0.005	0.001
	R2	0.000	0.000	0.000	0.000	0.001	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.003	0.000	0.004	0.024	-0.035
	SEIS	0.000	0.000	0.000	0.001	0.005	0.001
	TOTAL	0.000	0.003	0.000	0.004	0.029	0.036

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A10	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.003	0.000	0.002	0.000	-0.001
	R1	0.000	0.000	0.000	0.000	0.000	0.000
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	-0.000	0.000
	GR+T1	0.000	0.003	0.000	0.002	0.000	-0.001
A11	SEIS	0.000	0.000	0.000	0.000	0.000	0.000
	TOTAL	0.000	0.003	0.000	0.002	0.000	0.001
	GR	0.000	0.000	0.000	0.000	0.000	0.000
A11	T1	0.000	0.000	0.000	0.000	0.000	0.000
	R1	0.000	0.000	0.000	0.000	0.000	0.000
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
	SEIS	0.000	0.000	0.000	0.000	0.000	0.000
TOTAL	0.000	0.000	0.000	0.000	0.000	0.000	

*** Segment A end ***

SUPPORT FORCES

Point/ Supp. ID	Connect/ Type	Load Combination	LOCAL			GLOBAL		
			Dirn	Force	Deform	Dirn	Force	Deform

*** End of system , no supports encountered. ***

RESTRAINT REACTIONS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result

A00	Anchor												
	GR	6	-222	29	224	55	7	38	67				
	T1	-447	427	-2062	2153	76	-588	-60	596				
	R1	9	1	7	12	4	5	3	7				
	R2	1	0	1	1	0	1	0	1				
	R3	0	0	0	0	0	0	0	0				
	GR+T1	-441	205	-2034	2091	131	-581	-22	596				
	SEIS	10	1	8	13	4	6	4	8				
	TOTAL	451	206	2042	2101	135	586	26	602				
A11	Anchor												
	GR	-6	-208	-29	210	-27	0	12	30				
	T1	447	-427	2062	2153	458	8	-175	491				
	R1	10	1	7	12	3	1	0	3				
	R2	1	0	1	1	0	0	0	0				
	R3	0	0	0	0	0	0	0	0				
	GR+T1	-441	-635	2034	2175	431	7	-163	461				
	SEIS	11	1	8	14	3	2	0	3				
	TOTAL	452	636	2042	2186	434	9	164	464				

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)				
		X	Y	Z	Result	X	Y	Z	Result	
*** Segment A begin ***										
A00	GR	-6	222	-29	224	-55	-7	-38	67	
	T1	447	-427	2062	2153	-76	588	60	596	
	R1	9	1	7	12	4	5	3	7	
	R2	1	0	1	1	0	0	0	1	
	R3	0	0	0	0	0	0	0	0	
	GR+T1	441	-205	2034	2091	-131	581	22	596	
	SEIS	10	1	8	13	4	6	4	8	
	TOTAL	451	206	2042	2101	135	586	26	602	
	A01	GR	-6	75	-29	80	-21	-3	-13	25
		T1	447	-427	2062	2153	-141	287	12	320
R1		9	1	7	12	4	4	3	7	
R2		1	0	1	1	0	0	0	1	
R3		0	0	0	0	0	0	0	0	
GR+T1		441	-352	2034	2110	-162	284	-1	327	
SEIS		10	1	8	13	4	5	4	7	
TOTAL		451	353	2042	2120	166	289	5	333	
A02		GR	-6	75	-29	80	-17	-2	-10	20
		T1	447	-427	2062	2153	-154	232	4	279
	R1	9	1	7	12	4	4	3	6	
	R2	1	0	1	1	0	0	0	1	
	R3	0	0	0	0	0	0	0	0	
	GR+T1	441	-352	2034	2110	-171	230	-7	286	
	SEIS	10	1	8	13	4	5	4	7	
	TOTAL	451	353	2042	2120	175	235	10	293	
	A03 N	GR	-6	73	-29	79	-13	-1	-8	15
		T1	447	-427	2062	2153	-176	136	-11	223
R1		9	1	7	12	4	4	3	6	
R2		1	0	1	1	0	0	0	1	
R3		0	0	0	0	0	0	0	0	
GR+T1		441	-353	2034	2111	-189	135	-19	233	
SEIS		10	1	8	13	4	5	4	7	
TOTAL		451	354	2042	2121	193	140	23	239	
A03 M		GR	-6	69	-29	75	-2	4	2	5
		T1	447	-427	2062	2153	-59	-221	-111	254
	R1	9	1	7	12	3	3	2	5	
	R2	1	0	1	1	0	0	0	1	
	R3	0	0	0	0	0	0	0	0	
	GR+T1	441	-358	2034	2111	-60	-211	-109	250	
	SEIS	10	1	8	13	3	4	3	5	
	TOTAL	451	359	2042	2121	63	221	111	255	

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)				
		X	Y	Z	Result	X	Y	Z	Result	
A03 F	GR	-6	64	-29	71	-3	6	7	10	
	T1	447	-427	2062	2153	393	-369	-239	590	
	R1	9	1	7	12	1	3	0	3	
	R2	1	0	1	1	0	0	0	0	
	R3	0	0	0	0	0	0	0	0	
	GR+T1	441	-362	2034	2112	390	-363	-232	581	
	SEIS	10	1	8	13	1	3	0	3	
	TOTAL	451	363	2042	2122	391	366	232	584	
	A04 N	GR	-6	64	-29	71	-3	6	7	10
		T1	447	-427	2062	2153	407	-369	-242	600
R1		9	0	7	12	1	3	0	3	
R2		1	0	1	1	0	0	0	0	
R3		0	0	0	0	0	0	0	0	
GR+T1		441	-362	2034	2112	403	-363	-235	591	
SEIS		10	1	8	13	1	3	0	3	
TOTAL		451	363	2042	2122	404	366	235	594	
A04 M		GR	-6	58	-29	65	-5	7	9	12
		T1	447	-427	2062	2153	1073	-434	-400	1225
	R1	8	0	7	11	2	2	3	4	
	R2	1	0	1	1	0	0	0	0	
	R3	0	0	0	0	0	0	0	0	
	GR+T1	441	-369	2034	2113	1068	-428	-391	1215	
	SEIS	9	0	8	12	2	2	3	4	
	TOTAL	450	370	2042	2123	1071	429	394	1219	
	A04 F	GR	-6	51	-29	59	10	9	10	17
		T1	447	-427	2062	2153	1224	-592	-466	1438
R1		6	0	7	9	3	1	4	5	
R2		1	0	1	1	0	0	0	1	
R3		0	0	0	0	0	0	0	0	
GR+T1		441	-376	2034	2114	1234	-583	-455	1439	
SEIS		7	0	8	11	3	1	5	6	
TOTAL		448	376	2042	2124	1238	585	460	1444	
A05 N		GR	-6	22	-29	37	71	19	74	74
		T1	447	-427	2062	2153	512	-1339	-466	1507
	R1	2	1	7	8	5	11	4	12	
	R2	0	0	1	1	0	1	0	1	
	R3	0	0	0	0	0	0	0	0	
	GR+T1	441	-405	2034	2120	583	-1320	-455	1513	
	SEIS	2	1	8	9	3	13	5	14	
	TOTAL	443	406	2042	2128	585	1332	460	1526	

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GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A05 M	GR	-6	15	-29	35	78	17	8	80
	T1	447	-427	2062	2153	361	-1198	-407	1312
	R1	1	2	7	7	2	13	6	14
	R2	0	0	1	1	0	1	0	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-412	2034	2121	438	-1178	-396	1318
	SEIS	1	2	8	8	2	15	5	16
	TOTAL	442	413	2041	2129	441	1193	400	1333
A05 F	GR	-6	8	-29	31	80	8	4	80
	T1	447	-427	2062	2153	298	-531	-252	659
	R1	0	2	7	7	2	16	5	17
	R2	0	0	1	1	0	2	1	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-418	2034	2122	378	-524	-249	692
	SEIS	0	2	8	8	2	18	6	19
	TOTAL	441	420	2041	2130	380	541	254	708
A06	GR	-6	6	-29	30	80	3	2	80
	T1	447	-427	2062	2153	298	-181	-180	392
	R1	0	2	7	7	2	17	5	18
	R2	0	0	1	1	0	2	1	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-42	2034	2123	378	-178	-177	454
	SEIS	0	2	8	8	2	19	6	20
	TOTAL	441	423	2041	2131	380	197	183	465
A06 +	GR	-6	-2	-29	30	80	3	2	80
	T1	447	-427	2062	2153	298	-181	-180	392
	R1	3	2	7	8	2	17	5	18
	R2	0	0	1	1	0	2	1	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-429	2034	2125	378	-178	-177	454
	SEIS	3	2	8	9	2	19	6	20
	TOTAL	444	431	2041	2133	380	197	183	465
A07 N	GR	-6	-5	-29	30	80	-2	3	80
	T1	447	-427	2062	2153	298	170	-107	360
	R1	4	2	7	8	2	18	6	19
	R2	0	0	1	1	0	2	1	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-432	2034	2125	378	168	-104	426
	SEIS	4	2	8	9	2	20	6	21
	TOTAL	445	434	2041	2134	380	188	110	438

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A07 M	GR	-6	-12	-29	32	78	-12	6	79
	T1	447	-427	2062	2153	336	834	44	867
	R1	6	2	7	9	2	20	6	21
	R2	1	0	1	1	0	2	1	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-439	2034	2127	314	822	50	881
	SEIS	6	2	8	10	2	22	7	23
	TOTAL	447	441	2041	2136	316	844	57	903
A07 F	GR	-6	-19	-29	35	73	-14	8	74
	T1	447	-427	2062	2153	85	978	106	987
	R1	9	1	7	12	1	19	7	20
	R2	1	0	1	1	0	2	1	2
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-446	2034	2128	157	964	114	983
	SEIS	10	1	8	13	1	21	7	22
	TOTAL	451	447	2041	2138	158	985	122	1005
A08 N	GR	-6	-51	-29	58	9	-2	8	12
	T1	447	-427	2062	2153	-699	157	106	724
	R1	10	1	7	12	1	2	7	7
	R2	1	0	1	1	0	0	1	1
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-477	2034	2135	-690	154	114	716
	SEIS	11	1	8	14	2	2	7	8
	TOTAL	452	479	2041	2145	692	156	122	719
A08 M	GR	-6	-55	-29	62	-6	-1	9	11
	T1	447	-427	2062	2153	-598	51	63	604
	R1	10	1	7	12	1	0	6	6
	R2	1	0	1	1	0	0	1	1
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-482	2034	2136	-605	50	71	611
	SEIS	11	1	8	14	1	1	6	6
	TOTAL	452	483	2041	2146	606	51	77	613
A08 F	GR	-6	-60	-29	66	-19	0	10	21
	T1	447	-427	2062	2153	-154	8	-43	160
	R1	10	1	7	12	1	1	3	4
	R2	1	0	1	1	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-486	2034	2137	-173	7	-33	176
	SEIS	11	1	8	14	1	2	4	4
	TOTAL	452	488	2041	2147	173	9	36	177

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A09	GR	-6	-205	-29	207	-21	0	11	23
	T1	447	-427	2062	2153	-16	8	-73	75
	R1	10	1	7	12	1	1	3	3
	R2	1	0	1	1	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-631	2034	2174	-37	7	-62	73
	SEIS	11	1	8	14	1	2	3	4
	TOTAL	452	633	2042	2185	38	9	65	76
A10	GR	-6	-205	-29	207	-22	0	11	24
	T1	447	-427	2062	2153	66	8	-91	112
	R1	10	1	7	12	1	1	2	3
	R2	1	0	1	1	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-631	2034	2174	44	7	-80	91
	SEIS	11	1	8	14	2	2	3	3
	TOTAL	452	633	2042	2185	46	9	82	95
A11	GR	-6	-208	-29	210	-27	0	12	30
	T1	447	-427	2062	2153	458	8	-175	491
	R1	10	1	7	12	3	1	0	3
	R2	1	0	1	1	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	441	-635	2034	2175	431	7	-163	461
	SEIS	11	1	8	14	3	2	0	3
	TOTAL	452	636	2042	2186	434	9	164	464
*** Segment A end ***									

ASME B31.3c (1992) CODE COMPLIANCE

Point name	Load combination	(Moments in ft-lb)			S.I.F		(Stress in psi)		Code	Allow.	
		In-Pl.	Out-Pl.	Torsion	In	Out	Eq. Load no.	Stress			
*** Segment A begin ***											
A00	Max P						(3a)	HOOP	4844	16700	
	GR + Max P	67	7		1.00	1.00	(18)	SUST	2588	16700	
	Cold to T1	25	588		94	1.00	1.00	(17)	DISP	2225	25050
	Sus. + R1	72	12		1.00	1.00	(18)	OCC	2422	22211	
	Sus. + R2	67	8		1.00	1.00	(18)	OCC	2391	22211	
	Sus. + R3	67	7		1.00	1.00	(18)	OCC	2388	22211	
A01	Max P						(3a)	HOOP	4844	16700	
	GR + Max P	25	3		1.00	1.00	(18)	SUST	2187	16700	
	Cold to T1	106	287		94	1.00	1.00	(17)	DISP	1195	25050
	Sus. + R1	30	8		1.00	1.00	(18)	OCC	2218	22211	
	Sus. + R2	25	4		1.00	1.00	(18)	OCC	2190	22211	
	Sus. + R3	25	3		1.00	1.00	(18)	OCC	2187	22211	
A02	Max P						(3a)	HOOP	4844	16700	
	GR + Max P	20	2		1.00	1.00	(18)	SUST	2161	16700	
	Cold to T1	126	232		88	1.00	1.00	(17)	DISP	1040	25050
	Sus. + R1	25	7		1.00	1.00	(18)	OCC	2192	22211	
	Sus. + R2	20	3		1.00	1.00	(18)	OCC	2165	22211	
	Sus. + R3	20	2		1.00	1.00	(18)	OCC	2161	22211	
A03 N-	Max P						(3a)	HOOP	4844	16700	
	GR + Max P	15	1		1.00	1.00	(18)	SUST	2138	16700	
	Cold to T1	153	136		88	1.00	1.00	(17)	DISP	833	25050
	Sus. + R1	20	5		1.00	1.00	(18)	OCC	2168	22211	
	Sus. + R2	16	2		1.00	1.00	(18)	OCC	2141	22211	
	Sus. + R3	15	1		1.00	1.00	(18)	OCC	2138	22211	
A03 N+	Max P						(3a)	HOOP	4844	16700	
	GR + Max P	15	1		2.56	2.13	(18)	SUST	2251	16700	
	Cold to T1	153	136		88	2.56	2.13	(17)	DISP	1850	25050
	Sus. + R1	20	5		2.56	2.13	(18)	OCC	2323	22211	
	Sus. + R2	16	2		2.56	2.13	(18)	OCC	2259	22211	
	Sus. + R3	15	1		2.56	2.13	(18)	OCC	2251	22211	
A03 M	Max P						(3a)	HOOP	4844	16700	
	GR + Max P	0	5		2.56	2.13	(18)	SUST	2113	16700	
	Cold to T1	110	198		114	2.56	2.13	(17)	DISP	1945	25050
	Sus. + R1	4	7		2.56	2.13	(18)	OCC	2163	22211	
	Sus. + R2	1	5		2.56	2.13	(18)	OCC	2119	22211	
	Sus. + R3	0	5		2.56	2.13	(18)	OCC	2113	22211	

Point name	Load combination	ASME B31.3c (1992) CODE COMPLIANCE						(Stress in psi)			
		(Moments in ft-lb)			S.I.F		Eq. no.	Load type	Code Stress	Code Allow.	
		In-Pl. Moment	Out-Pl. Moment	Torsion	In	Out					
A03 F-	Max P						(3a) HOOP	4844	16700		
	GR + Max P						SUST	2146	16700		
	Cold to T1	194	417	369	2.56	2.13	(17) DISP	4043	25050		
	Sus. + R1	2	8		2.56	2.13	(18) OCC	2157	22211		
	Sus. + R2	2	8		2.56	2.13	(18) OCC	2148	22211		
	Sus. + R3	1	8		2.56	2.13	(18) OCC	2146	22211		
A03 F+	Max P						(3a) HOOP	4844	16700		
	GR + Max P	3	7		1.00	1.00	(18) SUST	2103	16700		
	Cold to T1	393	239	369	1.00	1.00	(17) DISP	2201	25050		
	Sus. + R1	4	7		1.00	1.00	(18) OCC	2107	22211		
	Sus. + R2	3	7		1.00	1.00	(18) OCC	2104	22211		
	Sus. + R3	3	7		1.00	1.00	(18) OCC	2103	22211		
A04 N-	Max P						(3a) HOOP	4844	16700		
	GR + Max P	3	7		1.00	1.00	(18) SUST	2104	16700		
	Cold to T1	407	242	369	1.00	1.00	(17) DISP	2240	25050		
	Sus. + R1	4	7		1.00	1.00	(18) OCC	2108	22211		
	Sus. + R2	3	7		1.00	1.00	(18) OCC	2104	22211		
	Sus. + R3	3	7		1.00	1.00	(18) OCC	2104	22211		
A04 N+	Max P						(3a) HOOP	4844	16700		
	GR + Max P	3	7		1.95	1.63	(18) SUST	2130	16700		
	Cold to T1	407	242	369	1.95	1.63	(17) DISP	3583	25050		
	Sus. + R1	4	7		1.95	1.63	(18) OCC	2138	22211		
	Sus. + R2	3	7		1.95	1.63	(18) OCC	2131	22211		
	Sus. + R3	3	7		1.95	1.63	(18) OCC	2130	22211		
A04 M	Max P						(3a) HOOP	4844	16700		
	GR + Max P	5	11		1.95	1.63	(18) SUST	2165	16700		
	Cold to T1	1073	590	24	1.95	1.63	(17) DISP	8604	25050		
	Sus. + R1	7	12		1.95	1.63	(18) OCC	2184	22211		
	Sus. + R2	5	12		1.95	1.63	(18) OCC	2167	22211		
	Sus. + R3	5	11		1.95	1.63	(18) OCC	2165	22211		
A04 F-	Max P						(3a) HOOP	4844	16700		
	GR + Max P	10	9		1.95	1.63	(18) SUST	2183	16700		
	Cold to T1	1224	592	466	1.95	1.63	(17) DISP	9777	25050		
	Sus. + R1	13	10		1.95	1.63	(18) OCC	2214	22211		
	Sus. + R2	10	9		1.95	1.63	(18) OCC	2187	22211		
	Sus. + R3	10	9		1.95	1.63	(18) OCC	2183	22211		
A04 F+	Max P						(3a) HOOP	4844	16700		
	GR + Max P	10	9		1.00	1.00	(18) SUST	2130	16700		
	Cold to T1	1224	592	466	1.00	1.00	(17) DISP	5367	25050		
	Sus. + R1	13	10		1.00	1.00	(18) OCC	2146	22211		
	Sus. + R2	10	9		1.00	1.00	(18) OCC	2132	22211		
	Sus. + R3	10	9		1.00	1.00	(18) OCC	2130	22211		

Point name	Load combination	ASME B31.3c (1992) CODE COMPLIANCE						(Stress in psi)			
		(Moments in ft-lb)			S.I.F		Eq. no.	Load type	Code Stress	Code Allow.	
		In-Pl. Moment	Out-Pl. Moment	Torsion	In	Out					
A05 N-	Max P						(3a) HOOP	4844	16700		
	GR + Max P	71	19		1.00	1.00	(18) SUST	2418	16700		
	Cold to T1	512	1339	466	1.00	1.00	(17) DISP	5626	25050		
	Sus. + R1	73	30		1.00	1.00	(18) OCC	2474	22211		
	Sus. + R2	71	20		1.00	1.00	(18) OCC	2424	22211		
	Sus. + R3	71	19		1.00	1.00	(18) OCC	2418	22211		
A05 N+	Max P						(3a) HOOP	4844	16700		
	GR + Max P	19	71		1.95	1.63	(18) SUST	2648	16700		
	Cold to T1	1339	512	466	1.95	1.63	(17) DISP	10587	25050		
	Sus. + R1	30	73		1.95	1.63	(18) OCC	2607	22211		
	Sus. + R2	20	71		1.95	1.63	(18) OCC	2659	22211		
	Sus. + R3	19	71		1.95	1.63	(18) OCC	2648	22211		
A05 M	Max P						(3a) HOOP	4844	16700		
	GR + Max P	17	49		1.95	1.63	(18) SUST	2483	16700		
	Cold to T1	1195	540	30	1.95	1.63	(17) DISP	9507	25050		
	Sus. + R1	30	51		1.95	1.63	(18) OCC	2483	16700		
	Sus. + R2	18	50		1.95	1.63	(18) OCC	2497	22211		
	Sus. + R3	17	49		1.95	1.63	(18) OCC	2483	22211		
A05 F-	Max P						(3a) HOOP	4844	16700		
	GR + Max P	8	4		1.95	1.63	(18) SUST	2142	16700		
	Cold to T1	531	252	298	1.95	1.63	(17) DISP	4310	25050		
	Sus. + R1	23	9		1.95	1.63	(18) OCC	2296	22211		
	Sus. + R2	9	4		1.95	1.63	(18) OCC	2159	22211		
	Sus. + R3	8	4		1.95	1.63	(18) OCC	2142	22211		
A05 F+	Max P						(3a) HOOP	4844	16700		
	GR + Max P	4	8		1.00	1.00	(18) SUST	2106	16700		
	Cold to T1	252	531	298	1.00	1.00	(17) DISP	2462	25050		
	Sus. + R1	9	23		1.00	1.00	(18) OCC	2186	22211		
	Sus. + R2	4	9		1.00	1.00	(18) OCC	2115	22211		
	Sus. + R3	4	8		1.00	1.00	(18) OCC	2106	22211		
A06	Max P						(3a) HOOP	4844	16700		
	GR + Max P	2	3		1.00	1.00	(18) SUST	2083	16700		
	Cold to T1	180	181	298	1.00	1.00	(17) DISP	1464	25050		
	Sus. + R1	8	20		1.00	1.00	(18) OCC	2169	22211		
	Sus. + R2	3	5		1.00	1.00	(18) OCC	2092	22211		
	Sus. + R3	2	3		1.00	1.00	(18) OCC	2083	22211		
A07 N-	Max P						(3a) HOOP	4844	16700		
	GR + Max P	3	2		1.00	1.00	(18) SUST	2084	16700		
	Cold to T1	107	170	298	1.00	1.00	(17) DISP	1343	25050		
	Sus. + R1	9	20		1.00	1.00	(18) OCC	2175	22211		
	Sus. + R2	4	4		1.00	1.00	(18) OCC	2094	22211		
	Sus. + R3	3	2		1.00	1.00	(18) OCC	2084	22211		

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			CODE COMPLIANCE (Stress in psi)					
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F.		Eq. Load no.	Code type	Code Stress	Code Allow.
		In	Out		In	Out				
A07 N+	Max P							(3a) HOOP	4844	16700
	GR + Max P	2	3		1.95	1.63	(18)	SUST	2420	16700
	Cold to T1	170	107	298	1.95	1.63	(17)	DISP	2097	16700
	Sus. + R1	20	9		1.95	1.63	(18)	OCC	2274	22211
	Sus. + R2	4	4		1.95	1.63	(18)	OCC	2116	22211
	Sus. + R3	2	3		1.95	1.63	(18)	OCC	2097	22211
A07 M	Max P							(3a) HOOP	4844	16700
	GR + Max P	12	51		1.95	1.63	(18)	SUST	2478	16700
	Cold to T1	834	136	198	1.95	1.63	(17)	DISP	6176	25050
	Sus. + R1	32	54		1.95	1.63	(18)	OCC	2668	22211
	Sus. + R2	14	51		1.95	1.63	(18)	OCC	2498	22211
	Sus. + R3	12	51		1.95	1.63	(18)	OCC	2478	22211
A07 F-	Max P							(3a) HOOP	4844	16700
	GR + Max P	14	73		1.95	1.63	(18)	SUST	2646	16700
	Cold to T1	978	85	106	1.95	1.63	(17)	DISP	7155	25050
	Sus. + R1	33	73		1.95	1.63	(18)	OCC	2825	22211
	Sus. + R2	16	73		1.95	1.63	(18)	OCC	2666	22211
	Sus. + R3	14	73		1.95	1.63	(18)	OCC	2646	22211
A07 F+	Max P							(3a) HOOP	4844	16700
	GR + Max P	73	14		1.00	1.00	(18)	SUST	2179	16700
	Cold to T1	85	978	106	1.00	1.00	(17)	DISP	3685	25050
	Sus. + R1	73	33		1.00	1.00	(18)	OCC	2512	22211
	Sus. + R2	73	16		1.00	1.00	(18)	OCC	2430	22211
	Sus. + R3	73	14		1.00	1.00	(18)	OCC	2420	22211
A08 N-	Max P							(3a) HOOP	4844	16700
	GR + Max P	9	2		1.00	1.00	(18)	SUST	2179	16700
	Cold to T1	699	157	106	1.00	1.00	(17)	DISP	2703	25050
	Sus. + R1	10	4		1.00	1.00	(18)	OCC	2121	22211
	Sus. + R2	9	3		1.00	1.00	(18)	OCC	2111	22211
	Sus. + R3	9	2		1.00	1.00	(18)	OCC	2109	22211
A08 N+	Max P							(3a) HOOP	4844	16700
	GR + Max P	9	2		2.56	2.13	(18)	SUST	2179	16700
	Cold to T1	699	157	106	2.56	2.13	(17)	DISP	6803	25050
	Sus. + R1	10	4		2.56	2.13	(18)	OCC	2202	22211
	Sus. + R2	9	3		2.56	2.13	(18)	OCC	2179	22211
	Sus. + R3	9	2		2.56	2.13	(18)	OCC	2176	22211
A08 M	Max P							(3a) HOOP	4844	16700
	GR + Max P	6	5		2.56	2.13	(18)	SUST	2162	16700
	Cold to T1	598	80	8	2.56	2.13	(17)	DISP	5749	25050
	Sus. + R1	7	9		2.56	2.13	(18)	OCC	2201	22211
	Sus. + R2	7	6		2.56	2.13	(18)	OCC	2167	22211
	Sus. + R3	6	5		2.56	2.13	(18)	OCC	2162	22211

Point name	Load combination	ASME B31.3c (1992) (Moments in ft-lb)			CODE COMPLIANCE (Stress in psi)					
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F.		Eq. Load no.	Code type	Code Stress	Code Allow.
		In	Out		In	Out				
A08 F-	Max P							(3a) HOOP	4844	16700
	GR + Max P	19	10		2.56	2.13	(18)	SUST	2319	16700
	Cold to T1	154	43	8	2.56	2.13	(17)	DISP	1508	25050
	Sus. + R1	19	13		2.56	2.13	(18)	OCC	2353	22211
	Sus. + R2	19	11		2.56	2.13	(18)	OCC	2322	22211
	Sus. + R3	19	10		2.56	2.13	(18)	OCC	2319	22211
A08 F+	Max P							(3a) HOOP	4844	16700
	GR + Max P	19	10		1.00	1.00	(18)	SUST	2168	16700
	Cold to T1	154	43	8	1.00	1.00	(17)	DISP	597	25050
	Sus. + R1	19	13		1.00	1.00	(18)	OCC	2184	22211
	Sus. + R2	19	11		1.00	1.00	(18)	OCC	2170	22211
	Sus. + R3	19	10		1.00	1.00	(18)	OCC	2168	22211
A09	Max P							(3a) HOOP	4844	16700
	GR + Max P	21	11		1.00	1.00	(18)	SUST	2177	16700
	Cold to T1	16	73	8	1.00	1.00	(17)	DISP	279	25050
	Sus. + R1	22	13		1.00	1.00	(18)	OCC	2191	22211
	Sus. + R2	21	11		1.00	1.00	(18)	OCC	2179	22211
	Sus. + R3	21	11		1.00	1.00	(18)	OCC	2177	22211
A10	Max P							(3a) HOOP	4844	16700
	GR + Max P	22	11		1.00	1.00	(18)	SUST	2183	16700
	Cold to T1	66	91	8	1.00	1.00	(17)	DISP	420	25050
	Sus. + R1	23	13		1.00	1.00	(18)	OCC	2195	22211
	Sus. + R2	22	11		1.00	1.00	(18)	OCC	2184	22211
	Sus. + R3	22	11		1.00	1.00	(18)	OCC	2183	22211
A11	Max P							(3a) HOOP	4844	16700
	GR + Max P	27	12		1.00	1.00	(18)	SUST	2209	16700
	Cold to T1	45	175	8	1.00	1.00	(17)	DISP	1831	25050
	Sus. + R1	30	12		1.00	1.00	(18)	OCC	2222	22211
	Sus. + R2	28	12		1.00	1.00	(18)	OCC	2210	22211
	Sus. + R3	27	12		1.00	1.00	(18)	OCC	2209	22211

*** Segment A end ***

SYSTEM SUMMARY

Maximum displacements (in)

Maximum X :	0.064	Point : A05 N	Load Comb.: TOTAL
Maximum Y :	0.042	Point : A07 N	Load Comb.: TOTAL
Maximum Z :	0.035	Point : A07 M	Load Comb.: TOTAL
Max. total:	0.075	Point : A05 N	Load Comb.: TOTAL

Maximum rotations (deg)

Maximum X :	0.232	Point : A03 M	Load Comb.: TOTAL
Maximum Y :	0.202	Point : A07 N	Load Comb.: T1
Maximum Z :	-0.211	Point : A03 N	Load Comb.: T1
Max. total:	0.333	Point : A03 M	Load Comb.: TOTAL

Maximum restraint forces(lb)

Maximum X :	452	Point : A11	Load Comb.: TOTAL
Maximum Y :	636	Point : A11	Load Comb.: TOTAL
Maximum Z :	-2062	Point : A00	Load Comb.: T1
Max. total:	2186	Point : A11	Load Comb.: TOTAL

Maximum restraint moments(ft-lb)

Maximum X :	458	Point : A11	Load Comb.: T1
Maximum Y :	-588	Point : A00	Load Comb.: T1
Maximum Z :	-175	Point : A11	Load Comb.: T1
Max. total:	602	Point : A00	Load Comb.: TOTAL

SYSTEM SUMMARY

Maximum pipe forces (lb)

Maximum X :	452	Point : A08 M	Load Comb.: TOTAL
Maximum Y :	636	Point : A11	Load Comb.: TOTAL
Maximum Z :	2062	Point : A00	Load Comb.: T1
Max. total:	2186	Point : A11	Load Comb.: TOTAL

Maximum pipe moments (ft-lb)

Maximum X :	1238	Point : A04 F	Load Comb.: TOTAL
Maximum Y :	-1339	Point : A05 N	Load Comb.: T1
Maximum Z :	-466	Point : A04 F	Load Comb.: T1
Max. total:	1526	Point : A05 N	Load Comb.: TOTAL

SYSTEM SUMMARY

Maximum sustained stress

Point : A05 N
Stress psi : 2648
Allowable psi : 16700
Ratio : 0.16
Load combination : GR + Max P

Maximum displacement stress

Point : A05 N
Stress psi : 10387
Allowable psi : 25050
Ratio : 0.41
Load combination : Cold to T1

Maximum occasional stress

Point : A07 F
Stress psi : 2825
Allowable psi : 22211
Ratio : 0.13
Load combination : Sus. + R1

Maximum hoop stress

Point : A00
Stress psi : 4844
Allowable psi : 16700
Ratio : 0.29
Load combination : Max P

Maximum sustained stress ratio

Point : A05 N
Stress psi : 2648
Allowable psi : 16700
Ratio : 0.16
Load combination : GR + Max P

Maximum displacement stress ratio

Point : A05 N
Stress psi : 10387
Allowable psi : 25050
Ratio : 0.41
Load combination : Cold to T1

SYSTEM SUMMARY

Maximum occasional stress ratio

Point : A07 F
Stress psi : 2825
Allowable psi : 22211
Ratio : 0.13
Load combination : Sus. + R1

Maximum hoop stress ratio

Point : A00
Stress psi : 4844
Allowable psi : 16700
Ratio : 0.29
Load combination : Max P

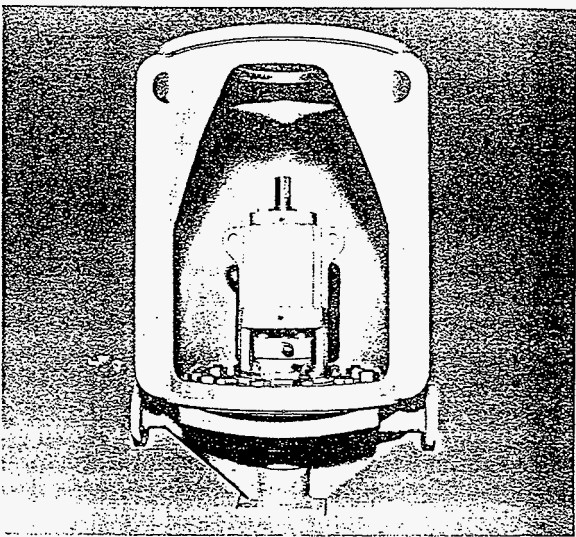
*** The system satisfies ASME B31.3 code requirements ***
*** for the selected options ***

(NEW Pump IN Sump) as of 4/6/85
PIT

Series 1400

In-Line Bearing Frame True Line Pumps

- Partial emissions hydraulics for optimum efficiency
 - Heads to 700 FT (213m) at 3550 RPM with a single stage
- 700 lb. casing
- Back Pullout feature
- Mag Drive option
- High speed designs available
- Variable speed options available



Series 1400 2' x 2' - 13' Vertical In-Line Pump

Figure 1

Optimized Low Flow, High Head Performance

True Line pumps from Lawrence Pumps Inc. are designed to deliver long term mechanical and hydraulic reliability for your low flow, high head applications.

- Designed to API-610 Standards
- Mechanical seal life is extended because the robust shaft design, combined with low radial loads minimizes shaft deflection
- Alignment between the pump and motor is guaranteed by a precision machined fit
- Liberal internal clearances and reduced radial thrust virtually eliminate the potential for any internal metal to metal contact
- Extreme temperature designs available (-50°F/-45°C to 700°F/370°C)
- Back Pullout feature allows ease of maintenance without disturbing the motor or connecting piping
- Seal chamber design accommodates Single, Double and Tandem mechanical seals
- Complete seal flush piping arrangements available

There are no close clearance wear rings or stuffing box bushing, and the axial clearances on both the front and back of the impeller are typically 0.035". Ultimately, the True Line design will withstand prolonged process upsets, or loss of suction, if seal lubrication is maintained.

Improved Process Pump Performance. Reduced Total Cost of Ownership.

- High efficiency at low flows
- Stable operations at all design points
- Minimum shaft deflection at off-design flows
- Reliable performance through process upsets
- Low maintenance from high seal reliability
- True In-Line with suction and discharge flanges on same plane
- Low NPSH characteristics



NEMA FRAMES 182HP — 449HP — (See drawings below.) — Dimensions in inches/millimeters

FRAME	P	T	CAST IRON TERMINAL BOX										AG		BY		SHAFT AND KEY				Wt. (LBS./KG.)					
			AA	AB	AC	AD	AJ	AX(1)	BB	BO	BE	BF	STD	LG(1)	AM(4)	BT(1)	LG(1)	Y	UN(1)	EPW(1)		LGTH	SO KEY	STD.	LG(1)	
182HP	9.50	1.44	75	77.5	6.31	1.94	9.12	8.250	19	10.00	.62	44	17.55	19.25	2.750	9.38	10.12	2.75	1.1250	.875	1.16	1.28	250	120	145	
	241	36.5	19	197	150	49.3	232	210	4.8	254	15.9	11.1	446	484	69.85	738	57	69.3	28.575	22.22	29.5	32.5	6.35	54	85	
L184HP	9.50	1.44	75	77.5	6.31	1.94	9.12	8.250	19	10.00	.62	44	18.56	20.66	2.750	9.88	10.62	2.75	1.1250	.875	1.16	1.28	250	130	155	
	241	36.5	19	197	150	49.3	232	210	4.8	254	15.9	11.1	471	509	69.85	751	270	69.3	28.575	22.22	29.5	32.5	6.35	59	70	
213HP	10.50	1.44	1.00	9.75	7.75	2.50	9.12	8.250	19	10.00	.62	44	19.50	21.25	2.750	9.75	10.62	2.75	1.1250	.875	1.38	1.28	250	175	190	
	254	36.5	25.4	248	197	63.5	232	210	4.8	254	15.9	11.1	495	540	69.85	748	270	69.3	28.575	22.22	35.0	32.5	6.35	79	86	
L215HP	10.50	1.44	1.00	9.75	7.75	2.50	9.12	8.250	19	10.00	.62	44	21.00	21.83	2.750	10.50	10.94	2.75	1.1250	.875	1.38	1.28	250	200	215	
	257	36.5	25.4	243	197	63.5	232	210	4.8	254	15.9	11.1	533	535	69.3	751	273	69.3	28.575	22.22	35.0	32.5	6.35	91	97	
254HP	13.88	2.25	1.18	10.19	8.22	2.50	9.12	8.250	19	10.00	.62	44	22.94	—	2.750	10.81	—	2.75	1.1250	.875	1.75	1.28	250	215	—	
	352	57.2	31.8	259	209	63.5	232	210	4.8	254	15.9	11.1	532	—	69.85	275	—	69.3	28.575	22.22	44.4	32.5	6.35	98	—	
258HP	13.88	2.25	1.18	10.19	8.22	2.50	9.12	8.250	19	10.00	.62	44	24.69	—	2.750	11.69	—	2.75	1.1250	.875	1.75	1.28	250	245	—	
	350	57.2	31.8	259	209	63.5	232	210	4.8	254	15.9	11.1	527	—	69.85	297	—	69.3	28.575	22.22	44.4	32.5	6.35	110	—	
284HP	15.50	2.25	1.18	12.00	9.56	3.00	9.12	8.250	25	16.00	.69	44	25.58	—	2.750	12.25	—	2.75	1.1250	.875	1.75	1.28	250	270	—	
	384	57.2	38	305	243	75.2	232	210	6.4	304	17.2	11.1	621	—	69.85	328	—	69.3	28.575	22.22	44.4	32.5	6.35	122	—	
285HP	15.50	2.25	1.18	12.00	9.56	3.00	9.12	8.250	25	16.00	.69	44	27.38	—	2.750	13.00	—	2.75	1.1250	.875	1.75	1.28	250	300	—	
	384	57.2	38	305	243	75.2	232	210	6.4	304	17.2	11.1	695	—	69.85	328	—	69.3	28.575	22.22	44.4	32.5	6.35	145	—	
284PH1	16.50	2.25	1.5	12.00	9.56	3.00	14.75	13.500	25	16.00	.88	69	25.58	—	4.500	12.25	—	4.50	1.6250	1.250	1.75	1.28	375	300	—	
	429	57.2	38	305	243	76.2	317	315	6.4	492	22.2	17.5	657	—	114	311	—	114	41.3	31.8	44.4	77	9.5	136	—	
286PH1	15.50	2.25	1.5	12.00	9.56	3.00	14.75	13.500	25	16.00	.88	69	27.38	—	4.500	13.00	—	4.50	1.6250	1.250	1.75	1.28	375	350	—	
	429	57.2	38	305	243	76.2	317	315	6.4	492	22.2	17.5	635	—	114	311	—	114	41.3	31.8	44.4	77	9.5	159	—	
324HP	17.50	2.25	1.5	14.69	10.94	3.60	14.75	13.500	25	16.00	.88	69	29.06	—	4.500	13.50	—	4.50	1.6250	1.250	2.12	3.03	375	480	—	
	475	57.2	50.8	373	278	92.1	375	343	6.4	419	22.2	17.5	738	—	114	343	—	114	41.4	27.5	53.8	57.2	77.0	9.52	118	—
324PH	17.50	2.25	2	14.69	10.94	3.60	14.75	13.500	25	16.00	.88	69	30.56	—	4.500	14.25	—	4.50	1.6250	1.250	2.12	3.03	375	530	—	
	451	57.2	50.8	373	278	92.1	375	343	6.4	419	22.2	17.5	776	—	114	362	—	114	41.4	27.5	53.8	77.0	9.52	240	—	
364HP	20.50	2.25	3	17.94	13.75	4.10	14.75	13.500	25	16.00	.88	69	33.28	—	4.500	15.38	—	4.50	1.6250	1.250	2.25	3.03	375	650	—	
	521	57.2	76	456	349	105	375	343	6.4	419	22.2	17.5	847	—	114	390	—	114	41.4	27.5	57.2	77.0	9.52	299	—	
365HP	20.50	2.25	3	17.94	13.75	4.10	14.75	13.500	25	16.00	.88	69	33.38	—	4.500	15.38	—	4.50	1.6250	1.250	2.25	3.03	375	710	—	
	521	57.2	76	456	349	105	375	343	6.4	419	22.2	17.5	917	—	114	390	—	114	41.4	27.5	57.2	77.0	9.52	322	—	
404HP	22.62	2.50	3	19.19	15.00	4.10	14.75	13.500	25	16.00	.94	69	35.19	—	4.500	15.12	—	4.50	1.6250	1.250	2.25	3.03	375	950	—	
	575	63.5	75	487	381	105	375	343	6.4	419	23.8	17.5	893	—	114	384	—	114	41.4	27.5	57.2	77.0	9.52	431	—	
405HP	22.62	2.50	3	19.19	15.00	4.10	14.75	13.500	25	16.00	.94	69	35.19	—	4.500	15.12	—	4.50	1.6250	1.250	2.25	3.03	375	1075	—	
	575	63.5	75	487	381	105	375	343	6.4	419	23.8	17.5	893	—	114	384	—	114	41.4	27.5	57.2	77.0	9.52	488	—	
444HP	25.50	2.50	3	22.13	17.37	4.60	14.75	13.500	25	16.00	1.06	69	40.88	—	4.500	18.00	—	4.50	2.125	1.750	2.25	3.03	500	1405	—	
	648	63.5	76	565	441	152	375	343	6.4	419	27.0	17.5	1038	—	114	457	—	114	53.975	14.45	57.2	77.0	12.70	637	—	
445HP	25.50	2.50	3	22.13	17.37	4.60	14.75	13.500	25	16.00	1.06	69	40.88	—	4.500	18.00	—	4.50	2.125	1.750	2.25	3.03	500	1555	—	
	648	63.5	76	562	441	152	375	343	6.4	419	27.0	17.5	1038	—	114	457	—	114	53.975	14.45	57.2	77.0	12.70	705	—	
447HP	25.50	2.50	5	23.37	18.12	7.00	14.75	13.500	25	16.00	1.06	69	49.38	—	4.500	22.25	—	4.50	2.125	1.750	2.25	3.03	500	2900	—	
	641	83.5	(5)	594	460	178	375	343	6.4	419	27.0	17.5	1254	—	114	565	—	114	53.975	14.45	57.2	77.0	12.70	1314	—	
449HP	25.50	2.50	(5)	23.37	18.12	7.00	14.75	13.500	25	16.00	1.06	69	49.38	—	4.500	22.25	—	4.50	2.125	1.750	2.25	3.03	500	2900	—	
	641	83.5	(5)	594	460	178	375	343	6.4	419	27.0	17.5	1254	—	114	565	—	114	53.975	14.45	57.2	77.0	12.70	1314	—	

- (1) "U" Varies — up to 2.125 ± .0005 — .0005 53.975 ± .000 — .013 (3) "EU" Varies — +.000 — .005 — +.00 — .13
 — 2.125 and larger — .000 — .001 — +.05 — .00 (4) "AH" Varies — +.030 — .030 — +.76 — .76
 (2) "AK" Varies — 180HP — 250HP — +.003 — .000 — +.05 — .00 (5) "AA" Varies with full load amps: 4" (120 mm) × 201 amps, 5" (127 mm) × 335 amps, 6" (152 mm) × 600 amps
 Maximum permissible shaft runout when measured at end of shaft extension is .002 inches (0.051 mm) Max. T.I.R. (0.03 inches, .076 mm) on 447HP — 449HP.
 If mounting clearance is required, consult factory.
 "Face runout and eccentricity — 182HP — 445HP, .004 Max. T.I.R. — .10
 — 447HP — 449HP, .007 Max. T.I.R. — .13

NORMAL THRUST, TEFC Above-NEMA, 350 — 400 HP

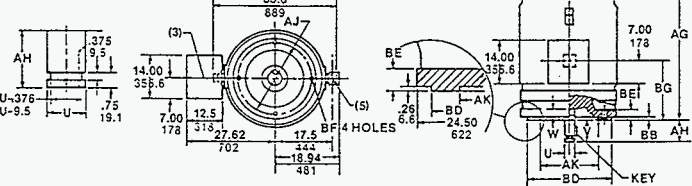
5010P Frame, 460 Volts

HP	350	400
Full Load Speed	3560	2558
Frame Size	5010P	5010P
Up or Down Thrust	720	720

5010P Frame, 2300, 4000 Volts

HP	200	250	300	350	400
Full Load Speed	1189	1783	3571	3570	3579
Frame Size	5010P	5010P	5010P	5010P	5010P
Up or Down Thrust	560	720	720	720	720

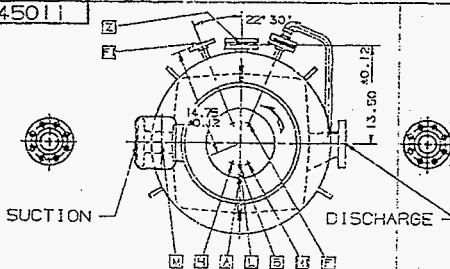
- 5010P Frames
- 460, 2300, 4000 Volts
- 3-Phase, 60 Hz
- 1.0 S.F., Continuous Duty
- 40°C. Ambient
- Class B Insulation
- Ball Bearing
- Solid Shaft



ABOVE NEMA 3010P — Dimensions in inches/millimeters

FRAME	RPM	V	AG	AJ	AX(1)	BB	BO	BE	BG	W	BF	KEY	WT.			
	(MAX.)	(230V)										SO, LGTH.	(LBS./KG.)			
5010P	3600	2.375	6.75	62.5	6.75	14.75	13.500	.25	20.00	1.75	17.94	4.44	168	.825	4.00	2200
5010P	3600	60.3	171	1587	171	374	343	6.4	508							

C 45011



CONNECTIONS:

- A 1/4-18 NPT - ALTERNATE LUBRICANT INLET TO THRUST BEARING (PLUGGED)
- B 1/4-18 NPT - OIL OUTLET FROM BEARING HOUSING
- F 1/2-14 NPT - INLET TO SEAL THRU API PLAN 11 FROM PUMP DISCHARGE
- E 1/2-300 LB. R.F. S.W. FLANGE - CASING VENT
- H 1/2-14 NPT - SEAL VENT
- K 1/2-14 NPT - SEAL DRAIN
- L 1/4-18 NPT - OIL MIST INLET TO BEARING HOUSING
- M MOTOR CONNECTION FOR POWER LEADS
- Q 3/4-300 LB. R.F. S.W. FLANGE - CASING DRAIN (COVERED WITH BLIND FLANGE)

WEIGHTS:

- PUMP 800 LBS.
- MOTOR 2700 ~~4500~~ LBS.
- TOTAL 3500 ~~5200~~ LBS., APPROXIMATE

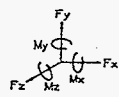
DRIVER DATA:

- 250 HP - 3500 RPM, T.E.F.C., VERTICAL ELECTRIC MOTOR, RELIANCE FRAME NO. 326TC

NOZZLE LOADS:

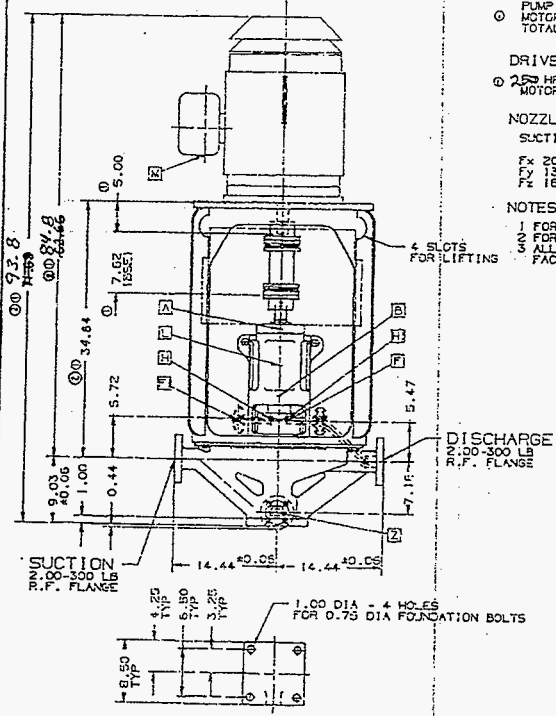
SUCTION & DISCHARGE

- Fx 200 LBS. Mx 340 FT-LBS.
- Fy 150 LBS. My 250 FT-LBS.
- Fz 160 LBS. Mz 170 FT-LBS.



NOTES:

- 1 FOR ASSEMBLY SEE DRAWING NO. C45010.
- 2 FOR PARTS LIST SEE "LIST OF COMPONENTS".
- 3 ALL FLANGE CONNECTIONS TO CONFORM TO A FACE FINISH OF 125 RMS.



2	ADP	RAM	QMS DECREASED D.5D
1	MO	SR	MOTOR WAS 325 HP FRAME
NO.	BY	DATE	CHK. BY
REVISIONS			

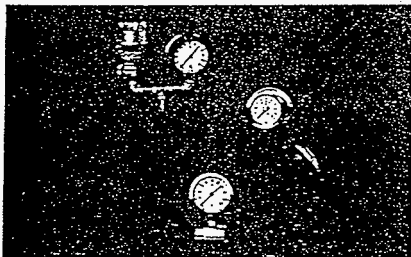
LAWRENCE PUMPS INC.		TRUE-LINE SERIES	
PUMP SIZE	1400	CUSTOMER:	
SERIES	1400	PURCHASE ORDER NO.:	
MODEL	14-4A	PUMP SERIAL NO.:	
DR. BY:	JT	PUMP ITRD NO.:	
CHK. BY:	RAM 11/08/86	CERTIFIED BY:	
APPR. BY:		DATE:	
DATE:	21. NOV 86		

PUMP ASSEMBLY:	
PUMP PARTS LIST:	
PUMP ELEVATION	
DRAWING NO.: C 45011	

Pressure Sensors

F-03-23

- ▶ Protects and isolates instrumentation
- ▶ Full 360° pressure reading
- ▶ Self cleaning, flexing action
- ▶ Won't clog like traditional diaphragm seals
- ▶ Excellent pump protection
- ▶ Accuracy, $\pm 2\%$ of installed instrument



Red Valve's complete line of Pressure Sensors isolate and protect instrumentation from the process line, providing accurate, repeatable instrument measurement.

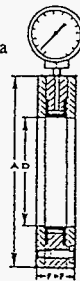
Reliable, accurate instrument readings are often difficult or even impossible to obtain on pipelines carrying slurries, solids, or chemical process slurries.

With Series 40 and 42 Pressure Sensors, line pressure is sensed 360° through the flexible rubber sleeve. The captive fluid is displaced through the pressure sensor body to the instrument's bourdon tube. Instruments are isolated and protected from the process assuring positive and accurate readings.

The thru-bolted Series 40 can be mounted in any flow direction, submerged in a tank, or mounted with a blind flange as a dead end to monitor tank levels.

The threaded end model Series 42 is available in sizes 1/2"-2".

Two Pressure Sensors with a differential pressure switch can be used to monitor pressure drop across a filter or pump. They can also be used to send a signal to shut a pump or open a by-pass valve.



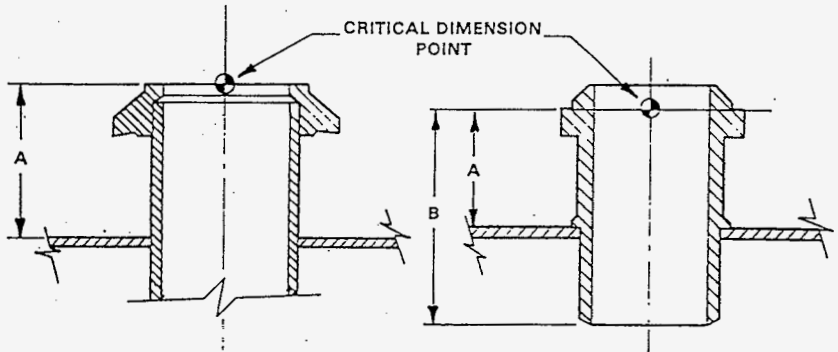
Materials of Construction

- ▶ Carbon Steel, Stainless Steel, or PVC body
- ▶ Sleeves available in Pure Gum Rubber, Neoprene, Hypalon, Chlorobutyl, Polyurethane, Buna-N, Viton and EPDM
- ▶ ANSI Class 150, 300, 600

Series 40

VALVE SIZE D	LENGTH F to F	BODY O.D. A	WORKING PRESSURE (psl)	VALVE WEIGHT (lbs)
1"	1-7/8"	4-1/2"	200	6
1-1/2"	1-7/8"	5"	200	8
2"	1-7/8"	6"	200	12
2-1/2"	1-7/8"	7"	200	16
3"	1-7/8"	7-1/2"	200	18
4"	2-1/8"	9"	200	27
5"	2-1/4"	10"	200	32
6"	2-1/4"	11"	200	37
8"	2-1/2"	13-1/2"	200	58
10"	2-3/4"	16"	200	79
12"	3"	19"	200	125
14"	3"	21"	200	157
16"	3"	23-1/2"	200	190
18"	3"	25"	200	250
20"	3"	27-1/2"	200	185
24"	3"	32"	200	220

8.3 PUREX and Hanford Nozzle



HANFORD NOZZLE

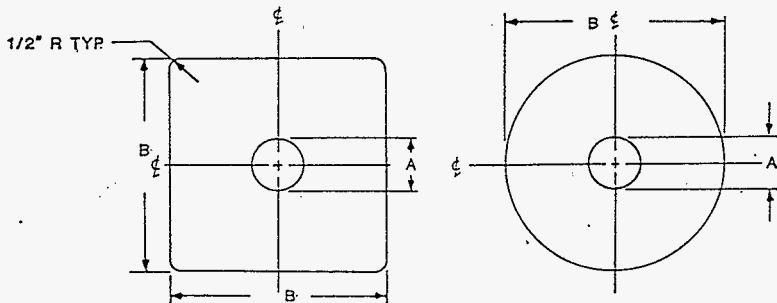
PUREX NOZZLE

Size	Type	A	B	Weight (lbs.)	Drawing No.
2"	Hanford	3-5/8		5	H-2-32705-9
3"	Hanford	3-5/8		3.5	H-2-32705-8
4-Way	Hanford	3-5/8		6	H-2-32705*
1"	Purex	1-7/8		1.5	H-2-90184-2
2"	Purex	2-1/4	3-29/32	2.5	H-2-90185-2
3-Way	Purex	2-9/32		5	H-2-32447-1
3"	Purex	3	4-13/16	6.5	H-2-90186-2
4"	Purex	3-3/8	5-5/32	11	H-2-90187-2

*See Drawing for Dash Number.

Handwritten notes:
1 13/16
4 1/16
3 9/16
3 13/16
11 7/16

8.4 Kick-Off Plate



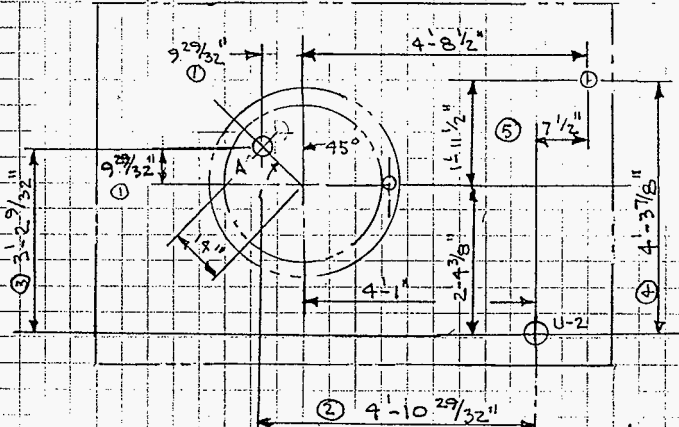
TYPE	SHAPE	SIZE	WT#	A	B	THK.	C.S. P.N.	SST. P.N.
PUREX	SQ.	1	4	1-25/32	6-1/4	3/8	H-2-30600-2	H-2-30600-1
PUREX	SQ.	2	3.5	2-21/32	6-1/4	3/8	H-2-30600-4	H-2-30600-3
PUREX	SQ.	3	12	3-25/32	11	3/8	H-2-30600-8	H-2-30600-5
PUREX	SQ.	4	16.5	4-27/32	13	3/8	H-2-30600-8	H-2-30600-7
PUREX	RD.	1	3	1-25/32	6	3/8	H-2-3194-1 TYPE II	H-2-3194-1 TYPE I
PUREX	RD.	2	2.5	2-21/32	6-1/4	3/8	H-2-3194-2 TYPE II	H-2-3194-2 TYPE I
PUREX	RD.	3	9	3-25/32	11	3/8	H-2-3194-3 TYPE II	H-2-3194-3 TYPE I
PUREX	RD.	4	12.5	4-27/32	13	3/8	H-2-3194-4 TYPE II	H-2-3194-4 TYPE I
HANFORD	SQ.	2	12.5	2-7/16	11	3/8	H-2-74755-11	H-2-74755-1
HANFORD	SQ.	3	12	3-9/16	11	3/8	H-2-74755-12	H-2-74755-2
HANFORD	SQ.	4-WAY	13	?	11	3/8	H-2-74755-?	H-2-74755-?
HANFORD	RD.	2	9.5	2-7/16	11	3/8	H-2-74755-16	H-2-74755-6
HANFORD	RD.	3	9	3-9/16	11	3/8	H-2-74755-17	H-2-74755-7
HANFORD	RD.	4-WAY	10	?	11	3/8	H-2-74755-?	H-2-74755-?

Client <u>WHC</u>	WO/Job No. <u>W-320/</u>
Subject <u>JUMPER CRITICALS FOR</u>	Date <u>6-20-95</u> By <u>D. PECK</u>
<u>H-2-818503</u>	Checked _____ By _____
Location <u>SLUCE PIT 241-AY-OZE</u>	Revised _____ By _____

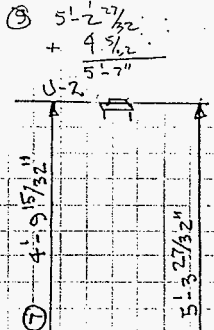


REF: H-2-818543, SH1 (REVIEW 14)
 H-2-818494, SH1 (REVIEW 14)
 H-2-818498, SH1 (REVIEW 15)
 H-2-818499, SH2 (REVIEW 8)

$$\begin{array}{r} 5'-11\frac{5}{8}" \\ - 5'-3\frac{27}{32}" \\ \hline 7\frac{25}{32}" \\ - 4\frac{5}{32}" \\ \hline 3\frac{5}{8}" \end{array}$$



$$\begin{array}{r} 5'-2\frac{27}{32}" \\ + 4\frac{5}{32}" \\ \hline 5'-7" \\ - 4'-9\frac{15}{32}" \\ \hline 5'-3\frac{27}{32}" \end{array}$$



① $\sin 45^\circ \times 14" = 9.90" = 9\frac{29}{32}"$

② $4'-1" + 9\frac{29}{32}" = 4'-10\frac{29}{32}"$

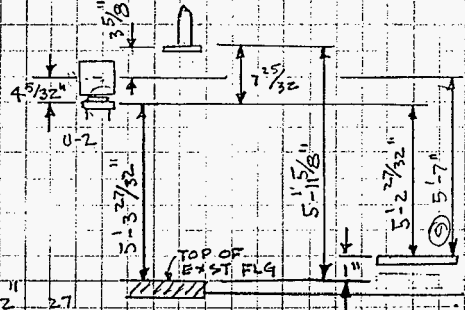
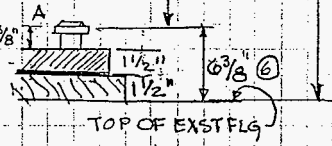
③ $2'-4\frac{3}{8}" + 9\frac{29}{32}" = 3'-2\frac{9}{32}"$

④ $2'-4\frac{3}{8}" + 1'-11\frac{1}{2}" = 4'-3\frac{7}{8}"$

⑤ $4'-8\frac{1}{2}" - 4'-1" = 7\frac{1}{2}"$

⑥ $3\frac{3}{8}" - 1\frac{1}{2}" = 6\frac{3}{8}"$

⑦ $5'-15\frac{27}{32}" - 6\frac{3}{8}" = 4'-9\frac{15}{32}"$





KAISER ENGINEERS HANFORD

JUMPER CALCULATION

JOB NO. ER4319	DWG. NO. H-2-818503	DATE 6-13-95	PAGE 4/4
BLDG. NO. 241-AY		BY BAM	CHKD.

B -
PN 8L

PART	DESCRIPTION	WEIGHT LBS.	X - X AXIS		Y - Y AXIS	
			MOM. ARM IN.	MOMENT IN. LBS.	MOM. ARM IN.	MOMENT IN. LBS.
A	2" BLK CONN	86.98	2.187	190.22	19.62	1706.55
B	2 bars	2.03	48	97.44	12.87	26.12
C						
D						
E						
F						
G		4297		127796		62380
H	SH 3/4	42762		127341.33		61056.82
J						
K	8x8 x 1/2	9.09	3715	286.33	12.87	116.99
L						
M						
N						
P						
Q						
R						
S						
T						
U		4380				
V		4380		128238		65373
		$\Sigma W = 4375.72$	$\Sigma Mx = 127915.32$		$\Sigma My = 62906.48$	
		$\bar{x} = \frac{\Sigma Mx}{\Sigma W} = 29.239$	29.27 $29\ 1/4$		$\bar{y} = \frac{\Sigma My}{\Sigma W} = 14.376$ $14\ 3/8$	

	COEFF. OF EXP. / OF	Δt OF	ΔX		ΔY		ΔZ	
			EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION
JUMPER								
VESSEL								
DUNNAGE								
TOLERANCE								
			$\Delta X =$		$\Delta Y =$		$\Delta Z =$	

KAISER ENGINEERS HANFORD

JUMPER CALCULATION

JOB NO. ER 4319	DWG. NO. 14-2-818503	DATE 6-9-95	PAGE 1/4
BLDG. NO. 241-AY	BY BAM	CHKD. D. PECK	

PART	DESCRIPTION	WEIGHT LBS.	X - X AXIS		Y - Y AXIS	
			MOM. ARM IN.	MOMENT IN. LBS.	MOM. ARM IN.	MOMENT IN. LBS.
U-2 A	4' HORIZ COUW	120	∅	∅	∅	∅
A	4' HORIZ COUW	120	38.25	4590.0	58.87	7064.4
X B	2" MALE COUW	2.5	∅	∅	18.37	45.92
	2" KP (Round)	2.5	∅	∅	18.37	45.92
X ↑	4" PIPE (14 3/16)	12.87	48.62	625.74	58.87	757.65
Y →	4" PIPE 2-7 3/8"	28.2	55.37	1561.43	43.19	1217.96
→	4" x 2" ROCR	3	55.37	166.11	25.5	76.5
	2" FLY					
	2" FLY	(30)	55.37	1716.47	19.69	1610.39
	PRES IND	36		1993		709
	PRES GAGE					
	2" PIPE 3"	.91	54.25	49.37	14.37	13.07
	2" PIPE 6 1/16	2.1	52.31	109.85	12.87	27.03
	2" FLY 9"	(10)	31.5	567	12.87	231.66
	2" FLY 9"	23	2	725		296
	2 1/2" 4" ROCR	3	12.19	36.57	12.87	38.61
	4" PIPE 10 1/4"	9.21	5.12	47.15	12.87	118.53
	4" PIPE 6"	5.39	∅	∅	12.87	69.37
	(2) 4" FLY	(50) - 60	∅	∅	12.87	643.577
	PRES IND	27	∅	∅	12.87	347.49

$\Sigma W = 435.68$ $\bar{x} = \frac{\Sigma Mx}{\Sigma W} = 455.68$	$\Sigma Mx = 9469.69$ 9929	$\Sigma My = 11308.0$ 11601 $\bar{y} = \frac{\Sigma My}{\Sigma W} =$
--	---------------------------------	--

	COEFF. OF EXP. / °F	Δt OF	ΔX		ΔY		ΔZ	
			EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION
JUMPER								
VESSEL								
DUNNAGE								
TOLERANCE								
			Δ X =		Δ Y =		Δ Z =	

2" CONN. SEE 54 4/4

ADD 9 FOR STUDS 6 NUTS

ADD 10 FOR STUDS NUTS

KAISER ENGINEERS HANFORD

JUMPER CALCULATION



JOB NO. ER4319	DWG. NO. A-2-818503	DATE 6-9-95	PAGE 2/4
BLDG. NO. Z41-A4		BY BAM	CHKD.

X
↓
3500
⊙
⊙
⊙
⊙
→
102 ←
⊙

PART	DESCRIPTION	WEIGHT LBS.	X - X AXIS		Y - Y AXIS	
			MOM. ARM IN.	MOMENT IN. LBS.	MOM. ARM IN.	MOMENT IN. LBS.
A	PRES GAGE	1	∅	∅	12.87	12.87
B	4" PIPE 14 1/8"	12.7	∅	∅	12.87	163.45
C	FLOW METER	35	∅	∅	12.87	450.45
D	4" FLG (2)	(50)60	∅	∅	12.87	643.75
E	4" PIPE 14 1/4"	12.8	∅	∅	12.87	164.73
F	4" PIPE 9 1/16"	8.71	∅	∅	8.0	69.68
G	2" PIPE 3 1/4"	.99	∅	∅	10.75	10.58
H	2" PIPE 3"	.91	∅	∅	10.37	16.71
J	PUMP W/MTR	3500	31.5	110250.0	12.87	45045.0
K		455.68		9924		11601
L	SH 1	(435.68)		(9469.69)		(4308.0)
M		455.68				
N						
P						
Q						
R						
S						
T						
U						
V						

$\Sigma W = 4057.179$ $\bar{x} = \frac{\Sigma Mx}{\Sigma W} = \frac{4077.79}{2950} = 2.950$	$\Sigma Mx = 119719.69$ 120174	$\Sigma My = 57890.9$ 58312 $\bar{y} = \frac{\Sigma My}{\Sigma W} = 14.266$
--	-------------------------------------	---

	COEFF. OF EXP. / °F	Δt OF	ΔX		ΔY		ΔZ	
			EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION
JUMPER								
VESSEL								
DUNNAGE								
TOLERANCE								
			$\Delta X =$		$\Delta Y =$		$\Delta Z =$	

JUMPER CALCULATION

JOB NO. ER4319	DWG. NO. H-2-818503	DATE 6-11-95	PAGE 3/4
BLDG. NO. 241-AY		BY BAM	CHKD.

PART	DESCRIPTION	WEIGHT LBS.	X - X AXIS		Y - Y AXIS	
			MOM. ARM IN.	MOMENT IN. LBS.	MOM. ARM IN.	MOMENT IN. LBS.
L A	W4x13	60.7	50.75	3080.5	12.87	781.21
R B	W4x13	60.7	12.25	743.57	12.87	781.21
L C	1 1/2 φ BAR	2.4	50.75	121.8	12.87	30.89
R D	1 1/2 φ BAR	2.4	12.25	29.4	12.87	30.89
E	MTR RING	30.20	31.5	951.3	12.87	405.40 ^{30.89}
F	C3x4.1	3.03	47.69	144.5	-0.25	-75.75 ¹⁶
G	C3x4.1	3.03	44.62	135.2	-2.62	-7.94
H	C3x4.1	23.2	48	1113.6	(25) ³⁶	580.08 ³⁵
J	SUPPORT	5.5	46.87	257.78	58.87	323.78
L K	2 BARS 2 1/2 x 3/8	1.79	47.94	37.87	12.87	10.17
R L	" "	.79	15.06	11.9	12.87	10.17
M	PLATE 3/8 " THICK	1.52	51.87	78.84	-7.5	-11.4
N	PLATE 3/8 " THICK	1.2	49.75	59.7	-5.37	-6.4
L P	2" FLG BKT	6.04	47.06	284.24	12.87	77.73
R Q	2" FLG BKT	6.04	15.87	95.85	12.87	77.73
R	2 BARS	2.03	(15.19) ^{30.89}	30.83	12.87	26.12
S	Mounting PL	10.26	31.5	323.19	12.87	132.04
T		<219.83>		<7621.64>	4060	<3165.85>
U	SH 2 OF 3 4077	4057.79	120174	119778.69	58320	57890.97
V						

$\Sigma W = 4277.62$
 $\Sigma Mx = 127341.33$
 $\Sigma My = 61056.82$

$\bar{x} = \frac{\Sigma Mx}{\Sigma W} = 29.769 \text{ CA } 29 \frac{3}{4}''$
 $\bar{y} = \frac{\Sigma My}{\Sigma W} = 14.273$
 $1' 2 \frac{1}{4}'' \text{ } \curvearrowright$

	COEFF. OF EXP. / °F	Δt °F	ΔX		ΔY		ΔZ	
			EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION	EXPANDING LENGTH	EXPANSION
JUMPER								
VESSEL								
DUNNAGE								
TOLERANCE								
			ΔX =		ΔY =		ΔZ =	

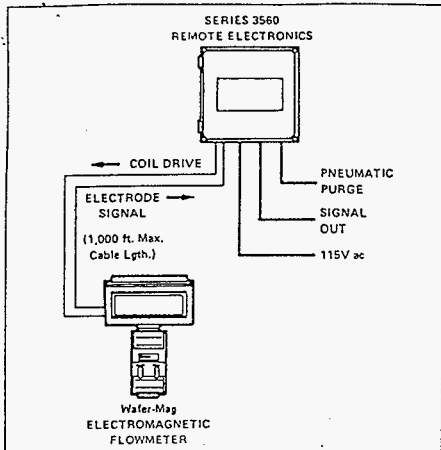


Figure 4 Typical Installation — Series 3560 Remote Electronics With Wafer-Mag Magnetic Flowmeter

- For accurate operation, it is mandatory that the flowmeter be installed so that the flow tube will be completely full of process liquid under all operating conditions. When the meter is only partially filled, even though the electrodes are covered, an inaccurate measurement will result. Refer to Figure 5.

It is desirable that the meter be installed in a vertical pipe run with flow upward, refer to Figure 6. In slurry applications, the vertical position insures a more even distribution of solids under all flow conditions and minimizes the chances of suspended solids adhering to the flow tube.

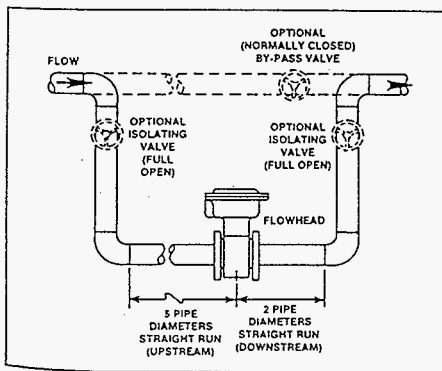


Figure 5 Horizontal Installation Piping (Meter shown in Optional Line Drop)

- The position of the flow tube in relation to other devices in the system is also important in assuring system accuracy. Any upstream tee, elbows, valves, etc., should be placed at least five pipe diameters from the flowhead to minimize any obstructions or flow disturbances.

Liquid Flow Characteristics

The characteristics of the liquid to be metered, the liquid flow parameters and the environment of the meter are determining factors in the accuracy of a particular meter.

- Conductivity** - electrical conductivity is simply a way of expressing the ability of a liquid to conduct electricity. Just as copper wire is a better conductor than steel wire, some liquids are better conductors than others. Of even greater importance, however, is the fact that some liquids have little or no conductivity.

The conductivity of a metered liquid must be at least 3 micromhos/cm for use with the flowmeter. The conductivity of the liquid can change throughout the process operation as long as it is homogenous and remains above the 3 micromhos/cm threshold.

- Acids/Caustics** - the chemical composition of the product to be measured must be compatible with the meter liner and electrodes to assure maximum service life from the meter.

Refer to a publication such as the Corrosion Survey, National Association of Corrosion Engineers, etc., to determine the compatibility of the liner with the process liquid. Many process slurries or liquids are designated by a generic name but may contain other

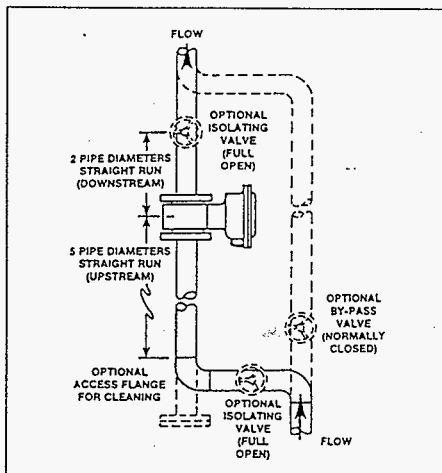


Figure 6 Vertical Installation

ORDERING INFORMATION

1. Meter Size
2. Liner Material
3. Electrode Material
 1. Pressure and Temperature Requirements (Ambient and Process)
5. Flow Rate (Normal, Maximum, Minimum)
6. Electrical Requirements
7. Output(s) Required
8. Liquid to be metered (name, conductivity)
9. Meter Orientation (horizontal or vertical)
10. Accessories Required

See Wafer Mag Model Code Table 6.

Purchase Specification

The magnetic flowmeter shall be of the wafer body construction utilizing the pulsed direct current field coil excitation to eliminate manual zero verification.

The flowmeter transmitter shall produce an output signal of 0-10 kHz proportional to flow rate and provide a factored and/or scaled pulse output. Factoring, to convert to engineering unit output (gpm, liters per minute, m³/hr, etc.), is accomplished by dividing the pulse output frequency by 1,000 to 15,000 using direct reading detent switches (1,000 to F,000). Scaling is accomplished by setting a field selectable DIP switch for divide by 1, 10, 100, 1,000 or 10,000.

Table 5 Dimensions (Meter Sizes 0.15" through 8") with Remote Electronics

Meter Size		Flange Size		A		B		C		D	
Inch	mm	Inch	mm	Inch	mm	Inch	mm	Inch	mm	Inch	mm
0.15	4	1/2	15	4-15/16	125	6-3/8	162	3-9/16	90	2-3/16	56
0.30	8										
1/2	15										
1	25	1	25	9	132	6-15/16	176	4-1/2	114	2-3/16	56
1-1/2	40	1-1/2	40	9	133	6-7/8	175	3-1/4	83	2-3/4	70
2	50	2	50	9-5/16	141	7-1/2	191	3-7/8	98	3-1/4	83
3	76	3	76	9-15/16	157	8-3/4	222	5-1/8	130	4-11/16	119
4	102	4	102	10-5/8	175	10-1/8	257	6-1/2	165	5-7/8	149
6	152	6	152	11-11/16	202	12-1/4	311	8-9/16	218	6-7/8	175
8	203	8	203	12-3/4	229	14-3/8	365	10-11/16	272	8-7/8	225

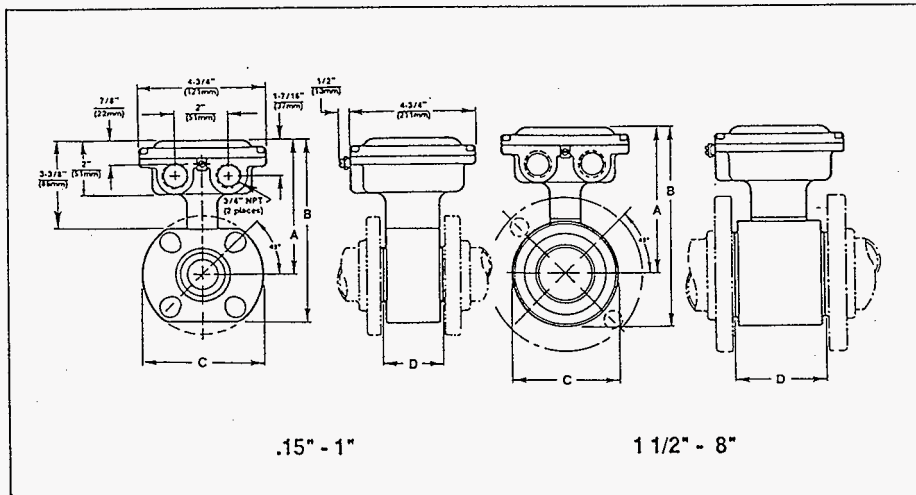


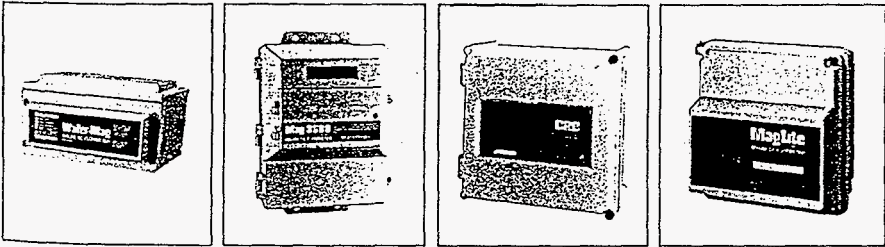
Figure 8 Dimensions (Meter Sizes 0.15" through 8") with Remote Electronics

Table 6 Wafer-Mag Electronics — Outputs, Performance Options

Outputs, Performance Options		Electronics Model			
		Integral Series III	Remote 3560 CRE Series III	3570 Integra/Remote (MagLite)	3580 SMART
Accuracy:	±0.5% Rate	Yes	Yes	Optional	Yes
	±1% Rate 0.5% Optional	N/A	N/A	Yes	N/A
Output Range: (For above stated accuracy)	1-10 m/Sec.	Yes	Yes	Yes	
	1-10 k/Sec.	Yes	Yes	N/A	N/A
	1-30 f/Sec.	N/A	N/A	N/A	Yes
Electrical Outputs:	4-20 mA Analog	Yes	Yes	Yes	Yes
	0-20 mA Analog	Yes	Yes	Yes	N/A
	0-10 kHz Unfactored	Note (3)	Note (3)	Yes	N/A
	0-10 kHz Scaled	Yes	Yes	N/A	N/A
	0-1 kHz Scaled	Note (2)	Note (2)	N/A	Yes
Repeatability:	0.25% Rate	Yes	Yes	Yes	0.2%
Power Consumption:	less than 15 Watts	Yes	Yes	Yes	
	less than 20 Watts				Yes
Input Voltage: (Note 1)	10-30 Vdc	Yes	Yes	Yes	N/A
	115 Vac 50-60 Hz	Yes	Yes	Yes	Yes
	220 Vac 50-60 Hz	Yes	Yes	Yes	N/A
Communications:	Hart Protocol	N/A	N/A	N/A	Yes
Bi-Directional Flow Option:		N/A	Yes	N/A	Standard
Integral Mounting:		Yes	No	N/A	Note 4
Remote Mounting:		No	Yes	N/A	Yes
Universal:	Integral/Remote	No	No	Yes	No
Class I, Division 2, NEMA 4X (FM Approved)		Yes	Yes	Yes	Yes
Features:		Most Flexible	Most Flexible	Best Value	All Flowheads
		Integral	Remote	All Flowheads	Digital Keypad
					Micro Processor

NOTES:

- (1) Series III Integral and 3560/3570 have jumper-selected power inputs 10-30 Vdc 115 Vac and 230 Vac.
- (2) Can use divide-by 10 scaler function to reduce 10 kHz to 1,000 Hz.
- (3) 0-10- kHz unfactored output when factor option board is removed (two-board module).
- (4) Integral mounting with adaptor



The Series III Integral Electronics represents the most versatile unit in the Brooks family of Mag electronics. Multitude of user-selectable output signals.

The newest member of the Brooks group is the Model 3580 "Smart" Electronics. This micro-processor-based magnetic flow transmitter offers keypad entry and display of selectable information, aids in flow-tube and transmitter testing and provides diagnostic helps in operation and maintenance of the system and communications.

The Model 3560 Complete Remote Electronics "CRE" eliminates the need for supplying line voltage at the flowmeter elements and allows for convenient, accessible location. It can be mounted up to 1,000 feet from the flowmeter and offers many selectable features. (Models 3561, 3562, 3563 and 3564.)

The Model 3570, possibly Brooks' best value in electronic instrumentation, is used on all flowheads 0.15" - 8" and exclusively on the MagLite. Mounting may be integral or remote.

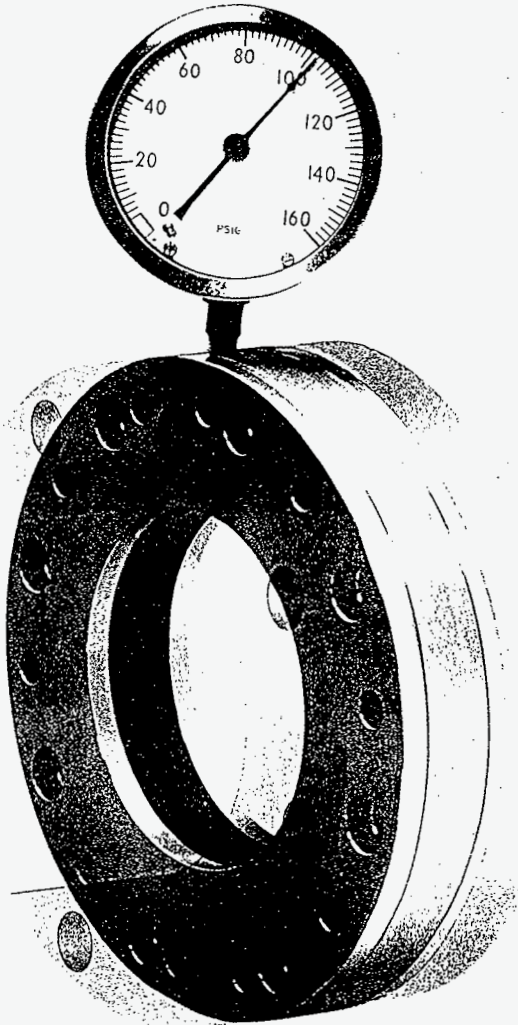
Figure 9 Wafer-Mag Electronics

Electromagnetic Flowmeters

Meters & Electronics

RED VALVE

Pressure Sensors



SERIES 40

RED VALVE PRESSURE SENSORS provide a full 360° circumferential reading of line fluids, while isolating them from the gauge. The flexible rubber sleeve provides accurate pressure readings of slurries and other hard to gauge corrosive fluids. Plugging or chemical attack of the gauge is eliminated.

A full circumferential reading eliminates pressure reading changes from trapped air or sediment deposits in the line. Readings taken at one point with a pipe tap are prone to bridge or clog from sediment.

Accuracy is dependent on the gauge. Red Valve's standard gauge has an accuracy of 2%. Gauges with greater accuracy are also available.

Pressure sensors can be mounted in any flow direction. Flow pressure is transferred from the flexible rubber sleeve to the gauge through a captive fluid. The sensor cannot clog because there are no line obstructions inducing buildup. Build up is fully eliminated by the flexing action of the sleeve which can break off adhering substances.

Series 40 FLANGED SENSORS conform to ANSI #125 flange drilling specifications. Class #300, #600, #900, #1200 flanges are also available. The small liquid chamber eliminates inaccurate readings due to thermal expansion of the fluid. Large chamber volumes are available on request.

The Series 40 is bolted directly in the line during installation and removal, eliminating costly welding associated with diaphragm seals.

The Series 40 can be mounted in any flow direction or it can be mounted with a blind flange as a dead end. End flange mounting can monitor tank levels.

FEATURES The Standard Red Valve Series "40" Sensor is manufactured with a very small chamber for the sensing liquid (less than ¼ of cubic inch). The volume of liquid in the Sensor Body is kept at a minimum because the pressure reading of the Sensor is effected by the thermal expansion or contraction of the sensing liquid. If a larger volume is required for bellows type instrumentation, it can be furnished on request.

In many applications the process is brought from atmospheric to over 100 degrees. A large volume of sensing liquid would expand and cause an increase of pressure because of the thermal expansion of the sensing liquid.

Red Valve Series "40" Sensors have less than ¼ of a cubic inch of liquid.

Instrumentation _____
GAUGE, TRANSMITTER, PRESSURE SWITCH

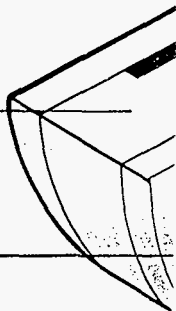
Epoxy Seal _____

Fluid Chamber _____

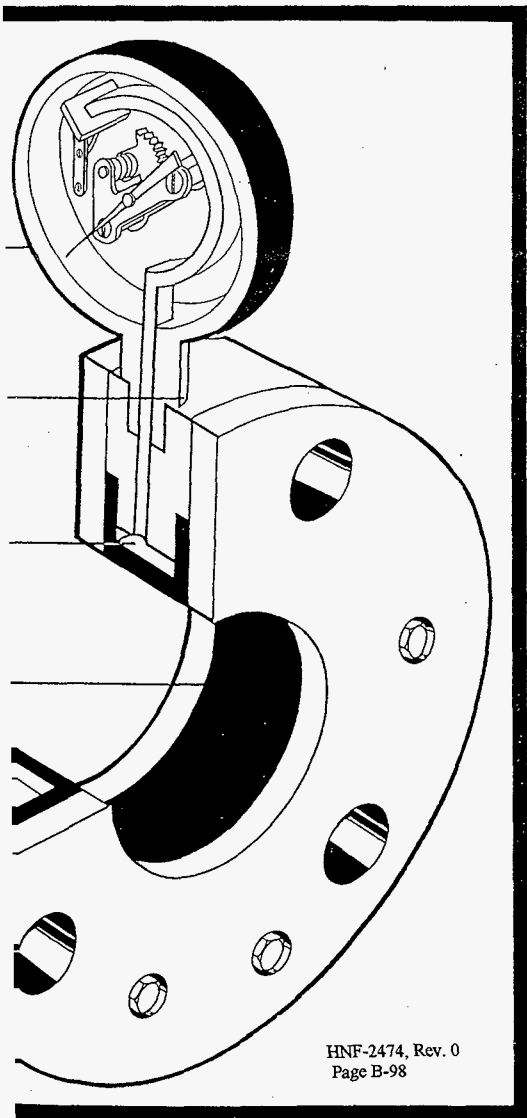
360° Sensing Sleeve _____

Solid Center Flange _____

Thru Flange Bolting _____



HNF-2474, Rev. 0
Page B-97



HNF-2474, Rev. 0
Page B-98

EXCEPTIONS: WHEN USING INSTRUMENTATION THAT REQUIRES A LARGE VOLUME OF SENSING LIQUID AS NEEDED BY CERTAIN MAKES OF PRESSURE SWITCHES, DIFFERENTIAL PRESSURE METERS, ETC., USER SHOULD SPECIFY THIS ON HIS PURCHASE ORDER. (We will then furnish a Sensor with a larger volume.)

EPOXY SEAL-FILL CONNECTION Red Valve Pressure Sensors are now being furnished with the **Cap Fill Connection** covered with an epoxy plastic seal. Precision workmanship and caution is exercised in filling the Pressure Sensors at the factory. All air pockets are removed with a vacuum pump after which the fill connection is sealed with a fill plug. The epoxy plastic over the hole makes the fill plug tamper-proof. The epoxy plastic can be chipped out very easily with a nail setting tool or drilled out if necessary in the future.

SOLID CONSTRUCTION Pressure Sensors are now being furnished with double the "holding screws" to hold the unit firmly together. This prevents overcompressing in installation which could change the pressure reading.

Full flange to the O.D. of the pipe guarantees that the Series 40 Sensor is centered when installed and makes installation easier, particularly for large diameter sensors.

THE CENTER FLANGE IS FURNISHED SOLID AT THE CIRCUMFERENCE TO PREVENT COMPRESSION.

Filling Instructions Red Valve Pressure Sensors should be refilled with a vacuum pump after the instrument is assembled to assure that all air is removed from the cavity of the sensor and instruments. If the sensor can not be removed from the line or a portable vacuum is not available, the alternate method is to gravity fill the sensor.

Following are simple instructions:

Gravity fill the sensor and lightly tap it to eliminate air bubbles. The gauge can also be gravity filled or filled separately under vacuum in the instrument shop.

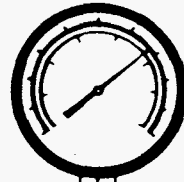
After the gauge is filled, a piece of Saran Wrap should be placed over the threads of the pressure gauge to keep the liquid in the gauge from spilling when the gauge is inverted to thread into the sensor. The Saran Wrap should be left on when inverting the sensor because the Saran Wrap breaks very easily when the gauge is threaded into the sensor. This will prevent fill liquid from dripping out.

Gravity fill is not as good as vacuum fill but if care is exercised, it can work.

Diaphragm Seals



Pressure Sensors



**Clogged and
Inoperable**

**Accurate
Measurement**

Red Valve Pressure Sensors will not clog!
Assuring accurate measurement and dependable systems control.

 **Red Valve
Company, Inc.**

P.O. Box 548 600 N. Bell Ave.
CARNEGIE, PA 15106
(412) 279-0044 Telex 86-6138
FAX (412) 279-7878

GPS-501 3/85

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SPECIFICATIONS

SIZES Series 40 Pressure Sensors are available in sizes 1" to 48". Pressure Sensors have an inside diameter equal to Schedule 40 Pipe.

PRESSURES Standard operating pressures are equivalent to the pressure rating of the flange. Standard Series 40 Sensors are manufactured to A.N.S.I. Class 125/150 pound flange and bolting dimensions.

Sensors can be supplied with Class 300, 600, 900 and 1200 pound flanges with comparable pressure ratings of 740, 1480, 2220, and 2960 PSI. Higher pressure ratings up to 20,000 PSI with raised faces or ring-grooves are also available.

MATERIALS Standard body and end flanges are Carbon Steel. Optional Materials include Stainless Steel, P.V.C. and Teflon.

ACCURACY The accuracy is equal to the gauge installed on the sensor. A $\pm 2\%$ of the gauge reading is common with off-the-shelf gauges. Higher accuracy and sensitivity is obtainable with more precise instrumentation.

GAUGES All standard ranges are available. A 0-100 psi gauge is furnished as standard unless otherwise specified. 0-60 psi and 0-200 psi gauges are also furnished at no extra charge.

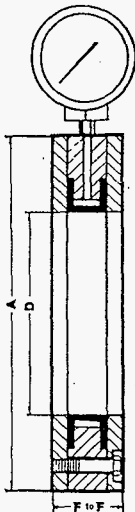
LIQUIDS Standard sensing liquid is a 50% mixture of ethylene glycol (anti-freeze) and water. Special fluids can be supplied on request.

Mineral Oil	220°F	Vegetable Oil	230°F
Distilled	200°F	Silicone Oil	450°F

(FDA Approved)

ELASTOMERS The standard Sensing Sleeve material is Buna-N. Other materials available include Pure Gum Rubber, Neoprene, Butyl, Hypalon, EPT, Food Grade Rubber and TEFLON. Other elastomers are available on request.

SPECIFICATIONS FOR FLANGED PRESSURE SENSORS This specification covers; Pressure Sensors and Gauge Isolator devices designed for isolating pipeline flow media from a gauge or instrument. The Sensor shall be flanged and bolted directly into ANSI flanged pipelines. Face-to-face shall be no greater than a wafer style of a butterfly valve. The flanges shall have thru bolt



SIZE "D"	LENGTH F TO F*	O.D. "A"	STEEL (LBS.)
1"	1 7/8"	4 1/2"	6
1 1/2"	1 7/8"	5"	8
2"	1 7/8"	6"	12
2 1/2"	1 7/8"	7"	16
3"	1 7/8"	7 1/2"	18
4"	2 1/8"	9"	27
5"	2 1/4"	10"	32
6"	2 1/4"	11"	37
8"	2 1/2"	13 1/2"	58
10"	2 3/4"	16"	79
12"	3"	19"	125
14"	3"	21"	157
16"	3"	23 1/2"	190
18"	3"	25"	250
20"	3"	27 1/2"	185
24"	3"	32"	220
30"	3"	38 3/4"	295
36"	4"	46"	380
42"	4"	53"	475
48"	4"	59 1/2"	610

* F to F of Teflon* Sensors is different.

holes to enable positive alignment in the pipeline. Inside diameter of the sensor shall be the same as the mating pipe with a full thru uninterrupted flow. There shall be no dead ends or crevices and flow passage shall make the sensor self-cleaning.

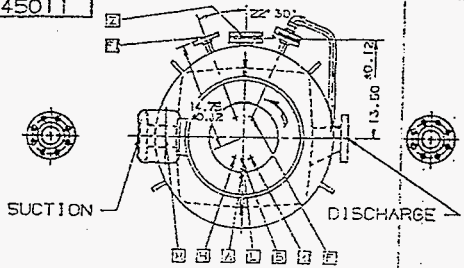
The Pressure Sensing Ring shall measure pressure for 360° around the full inside circumference of the pipeline. The sensing ring shall also be clamped into the body for the full radial width of the sensor. Pressure shall be transmitted to the gauge by a locked in and sealed fluid such as ethylene glycol or silicone oil. The Sensor shall have an auxiliary tapped and plugged port to allow filling connection of other equipment.

Red Valve Pressure Sensors are also available with capillary tubing for remote locations of instrumentation. Capillary tubing is available in various materials and sizes per your specifications. Red Valve can also fill and mount your instrumentation on our Pressure Sensors.

All Sensors shall be as the Series 40, as manufactured by the Red Valve Co., of Carnegie, PA, 600 N. Bell Ave.

* Teflon is Du Pont trademark for TFE fluorocarbon resin

C 45011



CONNECTIONS:

- A 1/4-18 NPT - ALTERNATE LUBRICANT INLET TO THRUST BEARING (PLUGGED)
- B 1/4-18 NPT - OIL OUTLET FROM BEARING HOUSING
- F 1/2-14 NPT - INLET TO SEAL THRU API PLAN 11 FROM FLAP DISCHARGE
- G 1/2-300 LB. R.F. S.W. FLANGE - CASING VENT
- H 1/2-14 NPT - SEAL VENT
- I 1/2-14 NPT - SEAL DRAIN
- J 1/4-18 NPT - DIL MIST INLET TO BEARING HOUSING
- K MOTOR CONNECTION FOR POWER LEADS
- L 3/4-300 LB. R.F. S.W. FLANGE - CASING DRAIN (COVERED WITH BLIND FLANGE)

WEIGHTS:

- PUMP 800 LBS.
- MOTOR 2700 450 LBS.
- TOTAL 3500 1250 LBS. APPROXIMATE

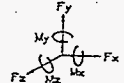
DRIVER DATA:

- 250 HP - 3500 RPM, T.E.F.C., VERTICAL ELECTRIC MOTOR, RELUCTANCE FRAME NO. 326TC

NOZZLE LOADS:

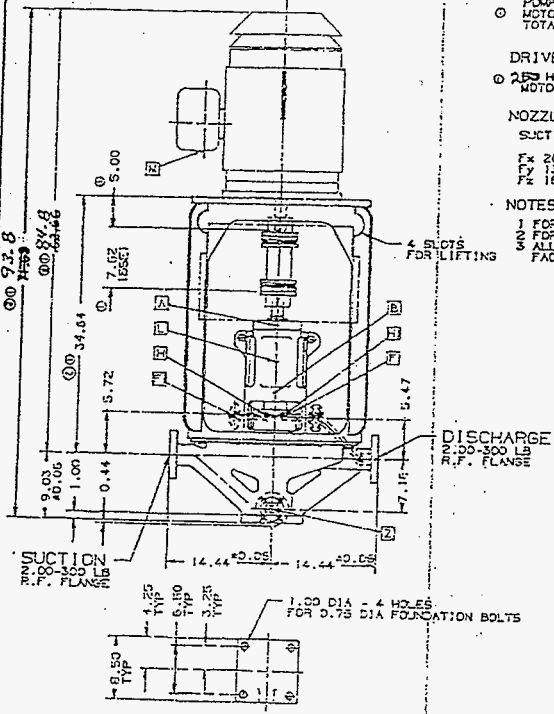
SUCTION & DISCHARGE

- Fx 200 LBS. Mx 340 FT-LBS.
- Fy 150 LBS. My 250 FT-LBS.
- Fz 160 LBS. Mz 170 FT-LBS.



NOTES:

- 1 FOR ASSEMBLY SEE DRAWING NO. C45010.
- 2 FOR PARTS LIST SEE "LIST OF COMPONENTS".
- 3 ALL FLANGE CONNECTIONS TO CONFORM TO A FACE FINISH OF 125 RMS.



2	REV	BAR DIMS DECREASED DSD
1	MOD	RAW MOTOR WAS 326 HP FRAME
NO	BY	DATE
NO	BY	DATE

LAWRENCE PUMPS INC.

TRUE-LINE SERIES

PUMP SIZE	300-300 LB
SERIAL	1402
MODEL	14-A
NO	REV
CHK BY	RAW 11/26/98
APPV	RYL
DATE	21 NOV 98

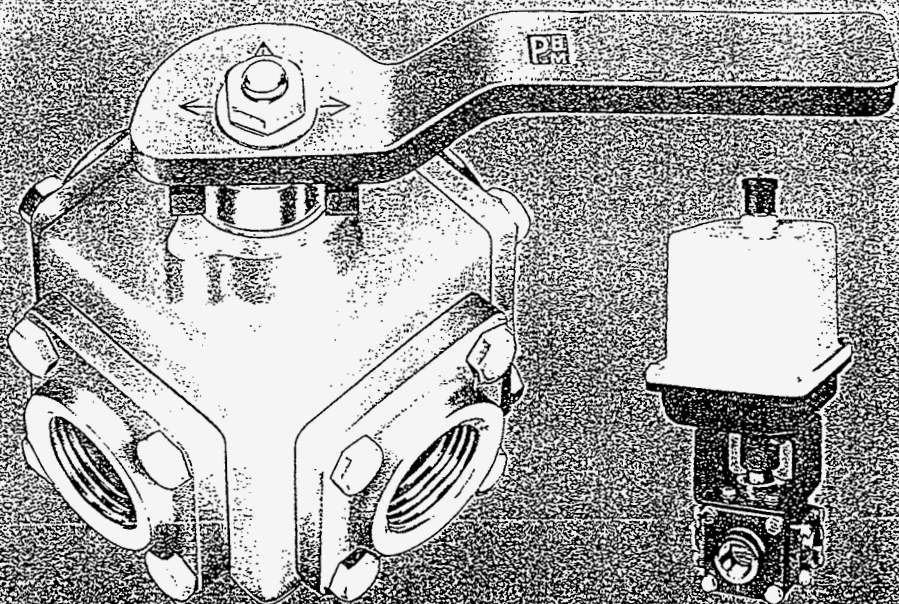
CUSTOMER	
PURCHASE ORDER NO.	
PUMP SERIAL NO.	
PUMP ITEM NO.	
CERTIFIED BY	

PUMP ASSEMBLY	
PUMP PARTS LIST	
PUMP ELEVATION	
DRAWING NO.	C 45011



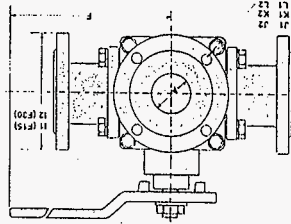
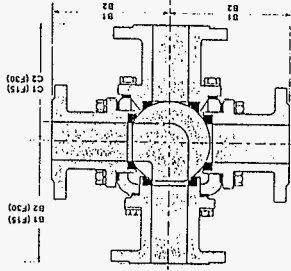
Multi-Port Ball Valves

Three, Four, and Five-Way



Dimensions for 3, 4 & 6-WAY W/P

DRAWINGS ARE FOR ILLUSTRATION PURPOSES ONLY. PLEASE CONSULT FACTORY PRIOR TO ANY FABRICATION OR INSTALLATION WORK.



PIPE SIZE (inch)	PORT CL TO FACE (inch)	PORT CL TO CENTER (inch)	PORT CL TO BOTTOM (inch)	HANDLE TO TOP (inch)	HANDLE TO CENTER (inch)	HANDLE TO BOTTOM (inch)	NO. BOLTS	BOLT DIA.	BOLT HOLE DIA.	W/1. S/S & C/S	3-WAY W/P
1/2"	1.50	5.28	2.88	5.28	5.17	5.00	4	3/8	4	50	46
3/4"	1.66	5.50	3.00	5.50	5.47	5.00	4	3/8	4	50	46
1"	1.81	5.72	3.09	5.97	5.61	5.00	4	7/8	4	62	64
1 1/4"	2.00	6.00	3.30	6.28	5.97	5.00	4	1	4	62	105
2"	2.25	7.28	3.75	7.28	7.50	5.00	4	1 1/2	4	62	150
3"	2.50	8.50	4.00	8.50	8.00	5.00	4	1 3/4	4	62	220

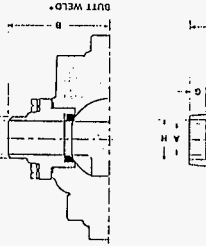
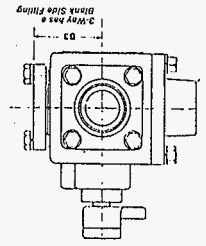
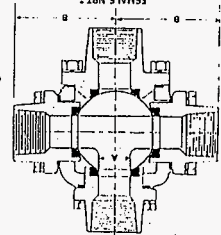
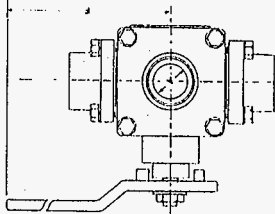
150# RF Flange (F15)

PIPE SIZE (inch)	PORT CL TO FACE (inch)	PORT CL TO CENTER (inch)	PORT CL TO BOTTOM (inch)	HANDLE TO TOP (inch)	HANDLE TO CENTER (inch)	HANDLE TO BOTTOM (inch)	NO. BOLTS	BOLT DIA.	BOLT HOLE DIA.	W/1. S/S & C/S	3-WAY W/P
1/2"	1.50	5.41	2.88	5.41	5.17	5.00	4	3/8	4	75	57
3/4"	1.69	5.69	3.00	6.09	5.61	5.00	4	3/8	4	75	57
1"	1.81	5.97	3.09	6.28	5.97	5.00	4	7/8	4	75	57
1 1/4"	2.00	6.00	3.30	6.09	5.61	5.00	4	1	4	75	57
2"	2.25	7.47	3.75	7.47	7.50	5.00	4	1 1/2	4	75	117
3"	2.50	8.50	4.00	8.50	8.00	5.00	4	1 3/4	4	75	117

300# RF Flange (F30)

Note: Consult factory for 6" MF valve.

Female NPT • Socket • Butt Weld

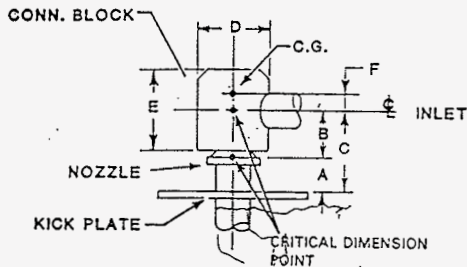


PIPE SIZE (inch)	PORT CL TO FACE (inch)	PORT CL TO CENTER (inch)	PORT CL TO BOTTOM (inch)	HANDLE TO TOP (inch)	HANDLE TO CENTER (inch)	HANDLE TO BOTTOM (inch)	NO. BOLTS	BOLT DIA.	BOLT HOLE DIA.	W/1. S/S & C/S	3-WAY W/P
1/2"	1.47	2.00	1.47	2.00	2.00	2.00	4	3/8	4	50	46
3/4"	1.62	2.26	1.62	2.26	2.26	2.26	4	3/8	4	50	46
1"	1.77	2.51	1.77	2.51	2.51	2.51	4	7/8	4	62	64
1 1/4"	1.92	2.76	1.92	2.76	2.76	2.76	4	1	4	62	105
2"	2.17	3.01	2.17	3.01	3.01	3.01	4	1 1/2	4	62	150
3"	2.42	3.26	2.42	3.26	3.26	3.26	4	1 3/4	4	62	220

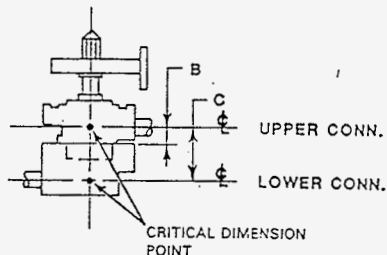
* Port shown. See Flow Charts, pages 5 & 6, for other positions.
 † Weights for other metals and for 4 Way supplied upon request.
 ‡ NOT STANDARD BONE DE-FILL.

8.0 DESIGN DATA

8.1 PUREX Connector



PROCESS CONNECTOR



ELECTRICAL CONNECTOR

Connector Block		Dimensions (inches)						Horizontal		Vertical	
Size	Type	A	B	C	D	E	F	Weight (lbs.)	Drawing No.	Weight (lbs.)	Drawing No.
1	1 x 1	1-7/8	1-23/32	3-19/32	2-1/2	2-1/2		15.5	H-2-32410-1	15.5	H-2-32410-2
1	Blank	1-7/8	1-23/32	3-19/32	2-1/2	2-1/2		16	H-2-32410-3	16.5	H-2-32410-4
2	2 x 2	2-1/4	2-7/16	4-11/16	3-3/8	3-5/8	1-1/8				
2	2 x 1 1/2 Dip tube	2-1/4	2-7/16	4-11/16	3-3/8	3-5/8	1-1/8	29	H-2-32420-1	29	H-2-32420-2
2	3 Way 3 1/2	2-9/32	2-13/32	4-11/16	3-3/8	3-5/8	1-1/8	31.5	H-2-32427-1	32	H-2-32427-2
2	Recirculation									31.5	H-2-32427-7
2	Thermowell	2-1/4	2-7/8	5-1/8	3-3/8					*	H-2-90154
2	Blank	2-1/4	2-7/8	4-11/16	3-3/8	3-5/8	1-1/8	32	H-2-32420-3	34.5	H-2-32420-4
3	3 x 3	3	3-9/13	6-9/16	5	5-9/16	1-7/16	76	H-2-32430-1	77	H-2-32430-2
3	3 x 2 Dip Tube	3	3-9/13	6-9/16	5	5-9/16				80	H-2-32430-5
3	2 x 3	3	3-9/13	6-9/16	5	5-9/16	1-7/16	76	H-2-32430-3	77	H-2-32430-4
3	2 x 2 Dip Tube	3	3-9/13	6-9/16	5	5-9/16				83	H-2-32430-6
3	3-Way 3 1/2	3	3-9/13	6-9/16	5	5-9/16				87.5	H-2-32437-1
3	Thermowell	3	3-9/13	6-9/16	5	5-9/16				*	H-2-90156
3	Blank	3	3-9/13	6-9/16	5	5-9/16	1-7/16	87.5	H-2-32430-7	83	H-2-32430-8
4	4 x 4	3-3/8	4-5/32	7-17/32	6-3/8	6-1/2	1-11/16	120	H-2-32440-1	122	H-2-32440-2
4	4 x 3 Dip Tube	3-3/8	4-5/32	7-17/32	6-3/8	6-1/2				126	H-2-32440-3
4	3 x 3 Dip Tube	3-3/8	4-5/32	7-17/32	6-3/8	6-1/2				131	H-2-32440-4
4	2 x 3 Dip Tube	3-3/8	4-5/32	7-17/32	6-3/8	6-1/2				135	H-2-32440-5
4	Blank	3-3/8	4-5/32	7-17/32	6-3/8	6-1/2	1-11/16	150	H-2-32440-6	153	H-2-32440-7
	Elec. Upper		1-11/16					56	H-2-32400-2	7-Point	
	Eqpt. Conn. Lower			5-9/32				29	H-2-32401-2	7-Point	
	Wall Conn. Lower			5-27/32					H-2-32408-1	7-Point	

*Establish Weight from Drawing

Author: Dianna L Stone at ~KEH10
Date: 06/25/96 03:18 PM
Priority: Normal
TO: Mohammed M Ahmed at ~WHC4
Subject: Datum

----- Message Contents -----

Hi Mo,

The top side of the plate to which the seismic restraint is welded is located 6' - 0 9/32" above the centerline of the Pump Discharge (shown as Detail 2, Lifting Frame Sub-Assy, on Dwg H-2-818508, Sht 2).

Hope that works for you.
Dianna

W320-27-017

Distributor Stress Analysis

This sheet shows the status and description of the attached Design Analysis sheets.

Discipline 27-Piping WO/Job No. ER4319/W320 Calculation No. W320-27-017
 Project No. & Name W320, 241-6166 Sluicing
 Calculation Item Distributor ~~Trace~~ Analysis

These calculations apply to:

Dwg. No. 4-2-918507 Rev. No. 0
 Dwg. No. _____ Rev. No. _____
 Other (Study, CDR) _____ Rev. No. _____

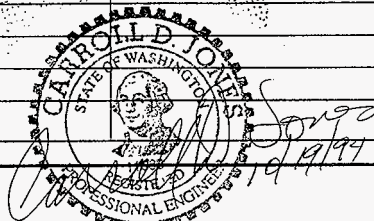
The status of these calculations is:

- Preliminary Calculations
- Final Calculations
- Check Calculations (On Calculation Dated _____)
- Void Calculation (Reason Voided _____)

Incorporated in Final Drawings? Yes No
 This calculation verified by independent "check" calculations? Yes No

Original and Revised Calculation Approvals:

	Rev. 0 Signature/Date	Rev. 1 Signature/Date	Rev. 2 Signature/Date
Originator	<i>[Signature]</i> 12.2		
Checked by	<i>[Signature]</i> 8.26-94		
Approved by	<i>CD Jones</i> 8/31/94		
Checked Against Approved Vendor Data	<i>[Signature]</i> 4.10.98		



INDEX

Design Analysis Page No.	Description
<u>i</u>	<u>Calculation Identification and Index</u>
<u>ii</u>	<u>Calculation Cross Index</u>
<u>iii</u>	<u>Design Verification Criteria Criteria</u>
<u>1.</u>	<u>Objectives, Criteria, Given Data, Assumptions, Methods, References, Calculation</u>
<u>2.</u>	<u>Conclusions</u>
<u>Appendix A</u>	<u>Autopipe Analysis</u>

EXPIRES: 5/3/98

CALCULATION CROSS INDEX (Typical)

Subject Calculation No. D22D-27-017

Subject Calculation Revision No.	Superceded by Calculation No.	These interfacing calculation/documents provide input to the subject calculation, and if revised may require revision of the subject calculation.	Results and conclusions of the subject calculation are used in these interfacing calculations and/or documents.		Does the output interface calculation/documents require revision?		Has the output interface calculation/documents been revised?		Discipline manager's signature and date indicating evaluation complete.
			Calculation/Document No.	Revision No.	Yes	No	Yes	No	
D		H-2-9185.37		01					<p><i>J.P. Brown</i> 7/16/98</p>

DESIGN VERIFICATION SCREENING CRITERIA

Pg. iii of iiiProject/Document No. W320/W320-27-016

When the design or design change affects hardware, formal design verification must be performed if one or more of the following questions must be answered affirmatively (YES).

- | YES | NO | |
|-------|---|--|
| _____ | _____ <input checked="" type="checkbox"/> | 1. Does the design or design change involved meet the established criteria to be considered Safety Class 1? |
| _____ | _____ <input checked="" type="checkbox"/> | 2. Does this design or design change cause or permit changes to Safety Class 1 instrument or alarm setpoints outside of previously approved operational limits? |
| _____ | _____ <input checked="" type="checkbox"/> | 3. Does this design or design change significantly affect the nuclear or environmental safety consequences of a malfunction or failure of the structure, system, or component? |
| _____ | _____ <input checked="" type="checkbox"/> | 4. Does this design or design change involve or change design that has previously undergone formal design verification? |

Carroll D. Jones 8/31/94
Assigned Lead Engineer Date

Danny L. Evans 4/28/98
Responsible Discipline Manager Date

Original Design Package Distribution:

Project Manager

Design Verification Officer

Engineering Document Control

Design Change Distribution:

Attach to Engineering Change Notice

DESIGN ANALYSIS

Client WHC
Subject Distributor Stress Analysis

WO/Job No. ER4319/W320
Date 7-19-94 By Rodney Campbell
Checked 8-24-94 By *R. Ahmad*
Revised By

Location: 241-AY-02

OBJECTIVE:

The objective of this calculation is to verify that the design of the distributor meets ASME B31.3 code stress requirements.

CRITERIA:

1. Functional Design Criteria WHC-SD-W320-FDC-001, Rev. 2.
2. ASME B31.3-1993, Chemical Plant and Petroleum Refinery Piping, With Addenda.

GIVEN DATA:

1. Specific Gravity of fluid: 1.25
2. Ambient temperature: 40°F.
3. Maximum temperature: 180°F.

ASSUMPTIONS:

1. The same volume of material flows throughs the four nozzles at the same velocity, causing static equilibrium.
2. There is no damping from the fluid in the tank, on the distributor during an earthquake.

METHODS:

1. Autopipe Version 4.50
2. Hand calculations

REFERENCES:

1. Drawing H-2-818537, Rev. 0.
2. ASME B31.3-1993, Chemical Plant and Petroleum Refinery Piping, With Addenda.

CALCULATION:

1. Calculate the Stress Intensification Factors of the tee required for the analysis. (Ref. 2, Appendix D, Unreinforced Fabricated Tee)

$T := .337 \text{ in}$ Average wall thickness of pipe being welded to.

$r_2 := 2.25 \text{ in}$ Radius of pipe being welded to.

$$h := \frac{T}{r_2}$$

$$h = 0.15$$

$$i_o := \frac{0.9}{\frac{2}{h^3}} \quad \text{Out of plane Stress Intensification Factor.}$$

$$i_o = 3.19$$

$$i_1 := \frac{3}{4} i_o + \frac{1}{4} \quad \text{In plane Stress Intensification Factor.}$$

$$i_1 = 2.64$$

According to the AutoPipe calculation (Appendix A), the piping meets ASME B31.3-1993. The piping system is built from 4", schedule 80 pipe and 2", schedule 80 nozzles.

DESIGN ANALYSIS

Client WHC
Subject Distributor Stress Analysis

Location: 241-AY

WO/Job No. ER4319/W320
Date 7-19-94 By Rodney Campbell
Checked 8-24-94 By *Al. Ahmed*
Revised By

CONCLUSIONS:

The distributor as designed meets ASME B31.3.

APPENDIX A

P O I N T D A T A L I S T I N G

POINT -----OFFSETS (ft)-----
NAME TYPE X Y Z PIPE ID DESCRIPTION

*** SEGMENT A

A00	Run	0	0	0	4SCHED80
A01	Run	0	-2.54	0	
A02	Tee	0	-35.15	0	
A03	Run	0	-0.42	0	

*** SEGMENT B

A02	Tee	0	-37.69	0	2SCHED80
B01	Run	0.50	0	0	

*** SEGMENT C

A02	Tee	0	-37.69	0	2SCHED80
C01	Run	-0.50	0	0	

*** SEGMENT D

A02	Tee	0	-37.69	0	2SCHED80
D01	Run	0	0	0.50	

* SEGMENT E

A02	Tee	0	-37.69	0	2SCHED80
E01	Run	0	0	-0.50	

Total weight of empty pipes : 595 lb

COMPONENT DATA LISTING

POINT NAME	---COORDINATE(ft)		DATA Z	DATA TYPE	DESCRIPTION
	X	Y			
*** SEGMENT A					
A00	0.00	0.00	0.00	ANCHOR	Rigid Thermal movements : None
A01	0.00	-2.54	0.00		
A02	0.00	-37.69	0.00	TEE	Other SIF - In = 2.60, Out = 3.14
A03	0.00	-38.10	0.00		
*** SEGMENT B					
B02	0.00	-37.69	0.00	TEE	Other SIF - In = 2.60, Out = 3.14
B01	0.50	-37.69	0.00		
*** SEGMENT C					
C02	0.00	-37.69	0.00	TEE	Other SIF - In = 2.60, Out = 3.14
C01	-0.50	-37.69	0.00		
* SEGMENT D					
D02	0.00	-37.69	0.00	TEE	Other SIF - In = 2.60, Out = 3.14
D01	0.00	-37.69	0.50		
*** SEGMENT E					
E02	0.00	-37.69	0.00	TEE	Other SIF - In = 2.60, Out = 3.14
E01	0.00	-37.69	-0.50		

Number of points in the system : 12

PIPE DATA LISTING

Pipe ID/ Material	Nom/ Sch	O.D. inch	-----Thickness(inch)-----				Spec Grav	Weight(lb/ft)			
			W.Th.	Corr	Mill	Insu		Ling	Pipe	Other	Total
4SCHED80 A312-TP304L	4 80S	4.500	0.337	0.06	0.04	0	0	1.25	15.33	0	21.56
2SCHED80 A312-TP304L	2.000 80S	2.375	0.218	0.06	0.03	0	0	1.25	5.14	0	6.74

M A T E R I A L D A T A L I S T I N G

Material Name	Pipe ID	Density lb/cu.ft	Pois. Ratio	Temper. deg F	Modulus E6 psi	Expans. in/100ft	Allow. psi
A312-TP304L	4SCHED80	501.0	0.30	40.0 180.0	28.42	1.5442	16700.0 16700.0
A312-TP304L	2SCHED80	501.0	0.30	40.0 180.0	28.42	1.5442	16700.0 16700.0

TEMPERATURE AND PRESSURE DATA

POINT NAME	-----C A S E 1-----			-----C A S E 2-----			-----C A S E 3-----		
	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft	PRESS. psi	TEMPER deg F	EXPAN. in/100ft

*** SEGMENT A

A00	320	180	1.544						
A03	320	180	1.544						

*** SEGMENT B

A02	320	180	1.544						
B01	320	180	1.544						

*** SEGMENT C

A02	320	180	1.544						
C01	320	180	1.544						

*** SEGMENT D

A02	320	180	1.544						
D01	320	180	1.544						

*** SEGMENT E

2	320	180	1.544						
E01	320	180	1.544						

A N A L Y S I S S U M M A R Y

Current model revision number : 32

Static - Date and Time of analysis Aug 11, 1994 7:03 AM
 Model Revision Number 32
 Number of load cases 3
 Load cases analyzed GR T1 P1
 Gaps/Friction/Yielding considered No
 Hanger design run No
 Cut short included No
 Weight of contents included Yes
 Pressure stiffening case 0
 Water elevation for buoyancy loads Not considered

Modal - Date and Time of analysis Aug 11, 1994 7:03 AM
 Model Revision Number 32
 Number of modes 2
 Cutoff frequency (Hz) 33.0
 Weight of contents included Yes
 Pressure stiffening case 0
 Water elevation for buoyancy loads Not considered

Response - Date and Time of analysis Aug 11, 1994 7:04 AM
 Model Revision Number 32
 Number of load cases 3
 Load cases analyzed R1 R2 R3
 Date and time of modal analysis Aug 11, 1994 7:03 AM
 Number of modes 2
 Cutoff frequency (Hz) 33.0
 Model revision of modal analysis 32
 Weight of contents included Yes
 Pressure stiffening case 0
 Water elevation for buoyancy loads Not considered

CODE COMPLIANCE COMBINATIONS

Combination	Category	Method	Load	Factor	Allowable	Remarks
GR + Max P	Sustain	Sum	Gravity Max Long	1.00 1.00	Automatic	Default
Cold to T1	Expansion	Sum	Thermal 1	1.00	Automatic	Default
Sus. + R1	Occasion	Abs sum	Response 1 Max Sus	1.00 1.00	Automatic	Default
Sus. + R2	Occasion	Abs sum	Response 2 Max Sus	1.00 1.00	Automatic	Default
Sus. + R3	Occasion	Abs sum	Response 3 Max Sus	1.00 1.00	Automatic	Default
Max P	Hoop		Max Hoop	1.00	Automatic	Default
RTOTAL	Occasion	SRSS	Response 1 Response 2 Response 3	1.00 1.00 1.00	Automatic	User

OTHER USER COMBINATIONS

Combination	Method	Load	Factor	Remarks
GR	Sum	Gravity	1.00	Default
T1	Sum	Thermal 1	1.00	Default
P1	Sum	Press 1	1.00	Default
R1	Sum	Response 1	1.00	Default
R2	Sum	Response 2	1.00	Default
R3	Sum	Response 3	1.00	Default
GR+T1	Sum	Gravity Thermal 1	1.00 1.00	User
RT	Abs sum	Response 1 Response 2 Response 3	1.00 1.00 1.00	User
GR+T1+RT	Abs sum	GR+T1 RT	1.00 1.00	User

CODE COMPLIANCE

Y - Factor 0.40
Weld efficiency factor 1.00
Range reduction factor 1.00
Design Pressure Factor 1.00
Minimum stress ratio used in reports... 0.00
Include corrosion in stress calcs. Y
Include torsion in code stress N
Include axial force in code stress N
Longitudinal pressure calculation PD/4t
Include rigorous pressure N

RESPONSE SPECTRUM LOAD CASES :

Number of load cases analysed : 3

Load case 1 - R1

Missing mass : No
ZPA : No

Combination method : SRSS

X- Spectrum : SC2&3
Multiplier : 1.00

SC2&3

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.02	0.250	0.04
0.400	0.06	0.600	0.09	1.100	0.17
1.640	0.25	8.000	0.25	12.000	0.20
20.000	0.16	33.000	0.12	100.000	0.12

Load case 2 - R2

Missing mass : No
ZPA : No

Combination method : SRSS

Y- Spectrum : SC2&3
Multiplier : 1.00

SC2&3

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.02	0.250	0.04
0.400	0.06	0.600	0.09	1.100	0.17
1.640	0.25	8.000	0.25	12.000	0.20
20.000	0.16	33.000	0.12	100.000	0.12

Load case 3 - R3

Missing mass : No
ZPA : No

Combination method : SRSS

Z- Spectrum : SC2&3
Multiplier : 1.00

SC2&3

Freq(Hz)	Grav()	Freq(Hz)	Grav()	Freq(Hz)	Grav()
0.100	0.01	0.160	0.02	0.250	0.04
0.400	0.06	0.600	0.09	1.100	0.17
1.640	0.25	8.000	0.25	12.000	0.20
20.000	0.16	33.000	0.12	100.000	0.12

D I S P L A C E M E N T S

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
*** Segment A begin ***							
A00	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	0.000	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
	R1	0.000	0.000	0.000	0.000	0.000	0.000
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1	0.000	0.000	0.000	0.000	0.000	0.000
	RT	0.000	0.000	0.000	0.000	0.000	0.000
	GR+T1+RT	0.000	0.000	0.000	0.000	0.000	0.000
A01	GR	0.000	0.000	0.000	0.000	0.000	0.000
	T1	0.000	-0.039	0.000	0.000	0.000	0.000
	P1	0.000	0.000	0.000	0.000	0.000	0.000
	R1	0.022	0.000	0.000	0.000	0.000	0.081
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	0.022	0.081	0.000	0.000
	GR+T1	0.000	-0.039	0.000	0.000	0.000	0.000
	RT	0.022	0.000	0.022	0.081	0.000	0.081
	GR+T1+RT	0.022	0.039	0.022	0.081	0.000	0.081
A02	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.000	3.286	0.624	0.000	0.624
	GR+T1+RT	3.286	0.584	3.286	0.624	0.000	0.624
A03	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.588	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.341	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.341	0.624	0.000	0.000
	GR+T1	0.000	-0.590	0.000	0.000	0.000	0.000
	RT	3.341	0.000	3.341	0.624	0.000	0.624
	GR+T1+RT	3.341	0.590	3.341	0.624	0.000	0.624

*** Segment A end ***

* Segment B begin ***

D I S P L A C E M E N T S

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A02	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.000	3.286	0.624	0.000	0.624
	GR+T1+RT	3.286	0.584	3.286	0.624	0.000	0.624
B01	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.008	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.065	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	0.008	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.065	3.286	0.624	0.000	0.624
	GR+T1+RT	3.294	0.649	3.286	0.624	0.000	0.624
*** Segment B end ***							
*** Segment C begin ***							
A02	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.000	3.286	0.624	0.000	0.624
	GR+T1+RT	3.286	0.584	3.286	0.624	0.000	0.624
C01	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	-0.008	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.065	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	-0.008	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.065	3.286	0.624	0.000	0.624
	GR+T1+RT	3.294	0.649	3.286	0.624	0.000	0.624

* Segment C end ***

*** Segment D begin ***

DISPLACEMENTS

Point name	Load combination	TRANSLATIONS (in)			ROTATIONS (deg)		
		X	Y	Z	X	Y	Z
A02	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.000	3.286	0.624	0.000	0.624
GR+T1+RT	3.286	0.584	3.286	0.624	0.000	0.624	
D01	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	0.008	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.065	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	0.008	0.000	0.000	0.000
	RT	3.286	0.065	3.286	0.624	0.000	0.624
GR+T1+RT	3.286	0.649	3.294	0.624	0.000	0.624	
*** Segment D end ***							
*** Segment E begin ***							
A02	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	0.000	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.000	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	0.000	0.000	0.000	0.000
	RT	3.286	0.000	3.286	0.624	0.000	0.624
GR+T1+RT	3.286	0.584	3.286	0.624	0.000	0.624	
E01	GR	0.000	-0.002	0.000	0.000	0.000	0.000
	T1	0.000	-0.582	-0.008	0.000	0.000	0.000
	P1	0.000	-0.005	0.000	0.000	0.000	0.000
	R1	3.286	0.000	0.000	0.000	0.000	0.624
	R2	0.000	0.000	0.000	0.000	0.000	0.000
	R3	0.000	0.065	3.286	0.624	0.000	0.000
	GR+T1	0.000	-0.584	-0.008	0.000	0.000	0.000
	RT	3.286	0.065	3.286	0.624	0.000	0.624
GR+T1+RT	3.286	0.649	3.294	0.624	0.000	0.624	

* Segment E end ***

S U P P O R T F O R C E S

Point/ Supp. ID	Connect/ Type	Load Combination	L O C A L			G L O B A L		
			Dirn	Force	Deform	Dirn	Force	Deform

*** End of system , no supports encountered. ***

RESTRAINT REACTIONS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A00	Anchor								
	GR	0	-835	0	835	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	29	0	0	29	0	0	1097	1097
	R2	0	0	0	0	0	0	0	0
	R3	0	0	29	29	1097	0	0	1097
	GR+T1	0	-835	0	835	0	0	0	0
	RT	29	0	29	41	1097	0	1097	1552
	GR+T1+RT	29	835	29	836	1097	0	1097	1552

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
*** Segment A begin ***									
A00	GR	0	835	0	835	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	29	0	0	29	0	0	1097	1097
	R2	0	0	0	0	0	0	0	0
	R3	0	0	29	29	1097	0	0	1097
	GR+T1	0	835	0	835	0	0	0	0
	RT	29	0	29	41	1097	0	1097	1552
	GR+T1+RT	29	835	29	836	1097	0	1097	1552
A01	GR	0	780	0	780	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	29	0	0	29	0	0	1023	1023
	R2	0	0	0	0	0	0	0	0
	R3	0	0	29	29	1023	0	0	1023
	GR+T1	0	780	0	780	0	0	0	0
	RT	29	0	29	41	1023	0	1023	1447
	GR+T1+RT	29	780	29	781	1023	0	1023	1447
A02 -	GR	0	22	0	22	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	29	0	0	29	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	29	29	0	0	0	0
	GR+T1	0	22	0	22	0	0	0	0
	RT	29	0	29	41	0	0	0	0
	GR+T1+RT	29	22	29	47	0	0	0	0
A02 +	GR	0	9	0	9	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	9	0	9	0	0	0	0
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	9	0	9	0	0	0	0

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
A03	GR	0	0	0	0	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	0	0	0	0	0	0	0
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	0	0	0	0	0	0	0

*** Segment A end ***

*** Segment B begin ***

A02	GR	0	3	0	3	0	0	1	1
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	3	0	3	0	0	1	1
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	3	0	3	0	0	1	1

B01	GR	0	0	0	0	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	0	0	0	0	0	0	0
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	0	0	0	0	0	0	0

*** Segment B end ***

*** Segment C begin ***

A02	GR	0	3	0	3	0	0	-1	1
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	3	0	3	0	0	-1	1
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	3	0	3	0	0	1	1

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
C01	GR	0	0	0	0	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	0	0	0	0	0	0	0
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	0	0	0	0	0	0	0

*** Segment C end ***

*** Segment D begin ***

A02	GR	0	3	0	3	-1	0	0	1
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	3	0	3	-1	0	0	1
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	3	0	3	1	0	0	1

D01	GR	0	0	0	0	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	0	0	0	0	0	0	0
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	0	0	0	0	0	0	0

*** Segment D end ***

*** Segment E begin ***

A02	GR	0	3	0	3	1	0	0	1
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	3	0	3	1	0	0	1
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	3	0	3	1	0	0	1

GLOBAL FORCES & MOMENTS

Point name	Load combination	FORCES (lb)				MOMENTS (ft-lb)			
		X	Y	Z	Result	X	Y	Z	Result
E01	GR	0	0	0	0	0	0	0	0
	T1	0	0	0	0	0	0	0	0
	P1	0	0	0	0	0	0	0	0
	R1	0	0	0	0	0	0	0	0
	R2	0	0	0	0	0	0	0	0
	R3	0	0	0	0	0	0	0	0
	GR+T1	0	0	0	0	0	0	0	0
	RT	0	0	0	0	0	0	0	0
	GR+T1+RT	0	0	0	0	0	0	0	0

*** Segment E end ***

GENERAL PIPE STRESS REPORT

(Stress in psi)

Point name	Load combination	Hoop Stress	Longitudinal Max	Longitudinal Min	Tors. Shear	Principal Max	Principal Min	Max Shear	Loc	Oct Shear
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*** Segment A begin ***

A00	SIFI= 1.00	SIFO= 1.00								
GR		0	227	227	0	227	0	114	NA	227
T1		0	0	0	0	0	0	0	NA	0
P1	2936	1001	1001	1001	0	2936	1001	1468	NA	2585
R1		0	3601	-3601	0	3601	-3601	1801	MT	3601
R2		0	0	0	0	0	0	0	NA	0
R3		0	3601	-3601	0	3601	-3601	1801	MT	3601
GR+T1		0	227	227	0	227	0	114	NA	227
RT		0	5093	-5093	0	5093	-5093	2547	MT	5093
GR+T1+RT		0	4866	-5320	0	4866	-5320	2660	MC	5320

A01	SIFI= 1.00	SIFO= 1.00								
GR		0	212	212	0	212	0	106	NA	212
T1		0	0	0	0	0	0	0	MT	0
P1	2936	1001	1001	1001	0	2936	1001	1468	NA	2585
R1		0	3357	-3357	0	3357	-3357	1678	MT	3357
R2		0	0	0	0	0	0	0	NA	0
R3		0	3357	-3357	0	3357	-3357	1678	MT	3357
GR+T1		0	212	212	0	212	0	106	NA	212
RT		0	4748	-4748	0	4748	-4748	2374	MT	4748
GR+T1+RT		0	4535	-4960	0	4535	-4960	2480	MC	4960

A02	- SIFI= 2.60	SIFO= 3.14								
GR		0	6	6	0	6	0	3	NA	6
T1		0	0	0	0	0	0	0	MT	0
P1	2936	1001	1001	1001	0	2936	1001	1468	NA	2585
R1		0	1	-1	0	1	-1	1	MT	1
R2		0	0	0	0	0	0	0	NA	0
R3		0	1	-1	0	1	-1	1	MT	1
GR+T1		0	6	6	0	6	0	3	NA	6
RT		0	2	-2	0	2	-2	1	MT	2
GR+T1+RT		0	-4	-8	0	0	-8	4	MC	8

A02	+ SIFI= 2.60	SIFO= 3.14								
GR		0	2	2	0	2	0	1	NA	2
T1		0	0	0	0	0	0	0	MC	0
P1	2936	1001	1001	1001	0	2936	1001	1468	NA	2585
R1		0	1	-1	0	1	-1	1	MT	1
R2		0	0	0	0	0	0	0	NA	0
R3		0	1	-1	0	1	-1	1	MT	1
GR+T1		0	2	2	0	2	0	1	NA	2
RT		0	2	-2	0	2	-2	1	MT	2
GR+T1+RT		0	-1	-4	0	0	-4	2	MC	4

GENERAL PIPE STRESS REPORT

(Stress in psi)

Point name	Load combination	Hoop Stress	Longitudinal		Tors. Shear	Principal		Max Shear	Loc	Oct Shear
			Max	Min		Max	Min			
A03	SIFI= 1.00	SIFO= 1.00								
GR		0	0	0	0	0	0	0	NA	0
T1		0	0	0	0	0	0	0	MC	0
P1		2936	1001	1001	0	2936	1001	1468	NA	2585
R1		0	0	0	0	0	0	0	NA	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	NA	0
GR+T1		0	0	0	0	0	0	0	MC	0
RT		0	0	0	0	0	0	0	NA	0
GR+T1+RT		0	0	0	0	0	0	0	MC	0

*** Segment A end ***

*** Segment B begin ***

A02	SIFI= 2.60	SIFO= 3.14								
GR		0	46	-46	0	46	-46	23	MT	46
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	0	0	0	0	0	0	MC	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	4	-4	0	4	-4	2	MT	4
GR+T1		0	46	-46	0	46	-46	23	MT	46
RT		0	4	-4	0	4	-4	2	MC	4
GR+T1+RT		0	46	-46	0	46	-46	23	MC	46

B01	SIFI= 1.00	SIFO= 1.00								
GR		0	0	0	0	0	0	0	MT	0
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	0	0	0	0	0	0	NA	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	NA	0
GR+T1		0	0	0	0	0	0	0	MT	0
RT		0	0	0	0	0	0	0	NA	0
GR+T1+RT		0	0	0	0	0	0	0	MC	0

*** Segment B end ***

*** Segment C begin ***

GENERAL PIPE STRESS REPORT

(Stress in psi)

Point name	Load combination	Hoop Stress	Longitudinal		Tors. Shear	Principal		Max Shear	Loc	Oct Shear
			Max	Min		Max	Min			
A02	SIFI= 2.60	SIFO= 3.14								
GR		0	46	-46	0	46	-46	23	MT	46
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	0	0	0	0	0	0	MC	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	4	-4	0	4	-4	2	MT	4
GR+T1		0	46	-46	0	46	-46	23	MT	46
RT		0	4	-4	0	4	-4	2	MC	4
GR+T1+RT		0	46	-46	0	46	-46	23	MC	46
C01	SIFI= 1.00	SIFO= 1.00								
GR		0	0	0	0	0	0	0	MT	0
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	0	0	0	0	0	0	NA	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	NA	0
GR+T1		0	0	0	0	0	0	0	MT	0
RT		0	0	0	0	0	0	0	NA	0
GR+T1+RT		0	0	0	0	0	0	0	MC	0
*** Segment C end ***										
*** Segment D begin ***										
A02	SIFI= 2.60	SIFO= 3.14								
GR		0	46	-46	0	46	-46	23	MT	46
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	4	-4	0	4	-4	2	MT	4
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	MC	0
GR+T1		0	46	-46	0	46	-46	23	MT	46
RT		0	4	-4	0	4	-4	2	MC	4
GR+T1+RT		0	46	-46	0	46	-46	23	MC	46
D01	SIFI= 1.00	SIFO= 1.00								
GR		0	0	0	0	0	0	0	MT	0
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	0	0	0	0	0	0	NA	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	NA	0
GR+T1		0	0	0	0	0	0	0	MT	0
RT		0	0	0	0	0	0	0	NA	0
GR+T1+RT		0	0	0	0	0	0	0	MC	0

*** Segment D end ***

GENERAL PIPE STRESS REPORT

(Stress in psi)

Point name	Load combination	Hoop Stress	Longitudinal Max	Longitudinal Min	Tors. Shear	Principal Max	Principal Min	Max Shear	Loc	Oct Shear
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*** Segment E begin ***

A02	SIFI= 2.60	SIFO= 3.14								
GR		0	46	-46	0	46	-46	23	MT	46
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	4	-4	0	4	-4	2	MT	4
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	MC	0
GR+T1		0	46	-46	0	46	-46	23	MT	46
RT		0	4	-4	0	4	-4	2	MC	4
GR+T1+RT		0	46	-46	0	46	-46	23	MC	46

E01	SIFI= 1.00	SIFO= 1.00								
GR		0	0	0	0	0	0	0	MT	0
T1		0	0	0	0	0	0	0	MT	0
P1		2773	859	859	0	2773	859	1386	NA	2459
R1		0	0	0	0	0	0	0	NA	0
R2		0	0	0	0	0	0	0	NA	0
R3		0	0	0	0	0	0	0	NA	0
GR+T1		0	0	0	0	0	0	0	MT	0
RT		0	0	0	0	0	0	0	NA	0
GR+T1+RT		0	0	0	0	0	0	0	MC	0

*** Segment E end ***

		ASME B31.3c (1992) CODE COMPLIANCE							
		(Moments in ft-lb)			(Stress in psi)				
Point	Load	In-Pl.	Out-Pl.	Torsion	S.I.F	Eq. Load	Code	Code	
name	combination	Moment	Moment	Moment	In	Out	no.	type	Stress Allow.
*** Segment A begin ***									
A00	Max P						(3a)	HOOP	2936 16700
	GR + Max P	0	0		1.00	1.00	(18)	SUST	1300 16700
	Cold to T1	0	0	0	1.00	1.00	(17)	DISP	0 25050
	Sus. + R1	0	1097		1.00	1.00	(18)	OCC	4901 22211
	Sus. + R2	0	0		1.00	1.00	(18)	OCC	1300 22211
	Sus. + R3	1097	0		1.00	1.00	(18)	OCC	4901 22211
	RTOTAL	1097	1097		1.00	1.00	(18)	OCC	5093 22211
A01	Max P						(3a)	HOOP	2936 16700
	GR + Max P	0	0		1.00	1.00	(18)	SUST	1300 16700
	Cold to T1	0	0	0	1.00	1.00	(17)	DISP	0 25050
	Sus. + R1	0	1023		1.00	1.00	(18)	OCC	4657 22211
	Sus. + R2	0	0		1.00	1.00	(18)	OCC	1300 22211
	Sus. + R3	1023	0		1.00	1.00	(18)	OCC	4657 22211
	RTOTAL	1023	1023		1.00	1.00	(18)	OCC	4748 22211
2	Max P						(3a)	HOOP	2936 16700
	GR + Max P	0	0		2.60	3.14	(18)	SUST	1300 16700
	Cold to T1	0	0	0	2.60	3.14	(17)	DISP	0 25050
	Sus. + R1	0	0		2.60	3.14	(18)	OCC	1301 22211
	Sus. + R2	0	0		2.60	3.14	(18)	OCC	1300 22211
	Sus. + R3	0	0		2.60	3.14	(18)	OCC	1301 22211
	RTOTAL	0	0		2.60	3.14	(18)	OCC	2 22211
A02	+ Max P						(3a)	HOOP	2936 16700
	GR + Max P	0	0		2.60	3.14	(18)	SUST	1300 16700
	Cold to T1	0	0	0	2.60	3.14	(17)	DISP	0 25050
	Sus. + R1	0	0		2.60	3.14	(18)	OCC	1301 22211
	Sus. + R2	0	0		2.60	3.14	(18)	OCC	1300 22211
	Sus. + R3	0	0		2.60	3.14	(18)	OCC	1301 22211
	RTOTAL	0	0		2.60	3.14	(18)	OCC	2 22211
A03	Max P						(3a)	HOOP	2936 16700
	GR + Max P	0	0		1.00	1.00	(18)	SUST	1300 16700
	Cold to T1	0	0	0	1.00	1.00	(17)	DISP	0 25050
	Sus. + R1	0	0		1.00	1.00	(18)	OCC	1300 22211
	Sus. + R2	0	0		1.00	1.00	(18)	OCC	1300 22211
	Sus. + R3	0	0		1.00	1.00	(18)	OCC	1300 22211
	RTOTAL	0	0		1.00	1.00	(18)	OCC	0 22211

*** Segment A end ***

** Segment B begin ***

ASME B31.3c (1992) CODE COMPLIANCE										
(Moments in ft-lb)										
Point name	Load combination	In-Pl. Out-Pl.		Torsion Moment	S.I.F		(Stress in psi)			
		Moment	Moment		In	Out	Eq. no.	Load type	Code Stress	Code Allow.
A02	Max P						(3a)	HOOP	2773	16700
	GR + Max P			2.60	3.14		(18)	SUST	1227	16700
	Cold to T1	0	0	0	2.60	3.14	(17)	DISP	0	25050
	Sus. + R1	1	0		2.60	3.14	(18)	OCC	1227	22211
	Sus. + R2	1	0		2.60	3.14	(18)	OCC	1227	22211
	Sus. + R3	1	0		2.60	3.14	(18)	OCC	1229	22211
	RTOTAL	0	0		2.60	3.14	(18)	OCC	2	22211
B01	Max P						(3a)	HOOP	2773	16700
	GR + Max P	0	0		1.00	1.00	(18)	SUST	1203	16700
	Cold to T1	0	0	0	1.00	1.00	(17)	DISP	0	25050
	Sus. + R1	0	0		1.00	1.00	(18)	OCC	1203	22211
	Sus. + R2	0	0		1.00	1.00	(18)	OCC	1203	22211
	Sus. + R3	0	0		1.00	1.00	(18)	OCC	1203	22211
	RTOTAL	0	0		1.00	1.00	(18)	OCC	0	22211
*** Segment B end ***										
*** Segment C begin ***										
A02	Max P						(3a)	HOOP	2773	16700
	GR + Max P	1	0		2.60	3.14	(18)	SUST	1248	16700
	Cold to T1	0	0	0	2.60	3.14	(17)	DISP	0	25050
	Sus. + R1	1	0		2.60	3.14	(18)	OCC	1249	22211
	Sus. + R2	1	0		2.60	3.14	(18)	OCC	1248	22211
	Sus. + R3	1	0		2.60	3.14	(18)	OCC	1252	22211
	RTOTAL	0	0		2.60	3.14	(18)	OCC	4	22211
C01	Max P						(3a)	HOOP	2773	16700
	GR + Max P	0	0		1.00	1.00	(18)	SUST	1203	16700
	Cold to T1	0	0	0	1.00	1.00	(17)	DISP	0	25050
	Sus. + R1	0	0		1.00	1.00	(18)	OCC	1203	22211
	Sus. + R2	0	0		1.00	1.00	(18)	OCC	1203	22211
	Sus. + R3	0	0		1.00	1.00	(18)	OCC	1203	22211
	RTOTAL	0	0		1.00	1.00	(18)	OCC	0	22211
*** Segment C end ***										
*** Segment D begin ***										
A02	Max P						(3a)	HOOP	2773	16700
	GR + Max P	0	1		2.60	3.14	(18)	SUST	1255	16700
	Cold to T1	0	0	0	2.60	3.14	(17)	DISP	0	25050
	Sus. + R1	0	1		2.60	3.14	(18)	OCC	1258	22211
	Sus. + R2	0	1		2.60	3.14	(18)	OCC	1255	22211
	Sus. + R3	0	1		2.60	3.14	(18)	OCC	1255	22211
	RTOTAL	0	0		2.60	3.14	(18)	OCC	3	22211

Point name	Load combination	ASME B31.3c (1992) CODE COMPLIANCE (Moments in ft-lb)			(Stress in psi)				
		In-Pl. Moment	Out-Pl. Moment	Torsion Moment	S.I.F In	S.I.F Out	Eq. Load no. type	Code Stress	Code Allow.
D01	Max P						(3a) HOOP	2773	16700
	GR + Max P	0	0		1.00	1.00	(18) SUST	1203	16700
	Cold to T1	0	0	0	1.00	1.00	(17) DISP	0	25050
	Sus. + R1	0	0		1.00	1.00	(18) OCC	1203	22211
	Sus. + R2	0	0		1.00	1.00	(18) OCC	1203	22211
	Sus. + R3	0	0		1.00	1.00	(18) OCC	1203	22211
	RTOTAL	0	0		1.00	1.00	(18) OCC	0	22211

*** Segment D end ***

*** Segment E begin ***

A02	Max P						(3a) HOOP	2773	16700
	GR + Max P	0	1		2.60	3.14	(18) SUST	1258	16700
	Cold to T1	0	0	0	2.60	3.14	(17) DISP	0	25050
	Sus. + R1	0	1		2.60	3.14	(18) OCC	1261	22211
	Sus. + R2	0	1		2.60	3.14	(18) OCC	1258	22211
	Sus. + R3	0	1		2.60	3.14	(18) OCC	1258	22211
	RTOTAL	0	0		2.60	3.14	(18) OCC	3	22211

E01	Max P						(3a) HOOP	2773	16700
	GR + Max P	0	0		1.00	1.00	(18) SUST	1203	16700
	Cold to T1	0	0	0	1.00	1.00	(17) DISP	0	25050
	Sus. + R1	0	0		1.00	1.00	(18) OCC	1203	22211
	Sus. + R2	0	0		1.00	1.00	(18) OCC	1203	22211
	Sus. + R3	0	0		1.00	1.00	(18) OCC	1203	22211
	RTOTAL	0	0		1.00	1.00	(18) OCC	0	22211

*** Segment E end ***

S Y S T E M S U M M A R Y

Maximum displacements (in)

Maximum X :	3.341	Point : A03	Load Comb.: R1
Maximum Y :	0.649	Point : B01	Load Comb.: GR+T1+RT
Maximum Z :	3.341	Point : A03	Load Comb.: R3
Max. total:	4.761	Point : A03	Load Comb.: GR+T1+RT

Maximum rotations (deg)

Maximum X :	0.624	Point : D01	Load Comb.: GR+T1+RT
Maximum Z :	0.624	Point : B01	Load Comb.: GR+T1+RT
Max. total:	0.883	Point : D01	Load Comb.: GR+T1+RT

Maximum restraint forces(ib)

Maximum X :	29	Point : A00	Load Comb.: R1
Maximum Y :	-835	Point : A00	Load Comb.: GR
Maximum Z :	29	Point : A00	Load Comb.: R3
Max. total:	836	Point : A00	Load Comb.: GR+T1+RT

Maximum restraint moments(ft-lb)

Maximum X :	1097	Point : A00	Load Comb.: R3
Maximum Z :	1097	Point : A00	Load Comb.: R1
Max. total:	1552	Point : A00	Load Comb.: RT

S Y S T E M S U M M A R Y

Maximum pipe forces (lb)

Maximum X :	29	Point :	A00	Load Comb.:	R1
Maximum Y :	835	Point :	A00	Load Comb.:	GR
Maximum Z :	29	Point :	A00	Load Comb.:	R3
Max. total:	836	Point :	A00	Load Comb.:	GR+TI+RT

Maximum pipe moments (ft-lb)

Maximum X :	1097	Point :	A00	Load Comb.:	R3
Maximum Y :	0	Point :	A02	Load Comb.:	R3
Maximum Z :	1097	Point :	A00	Load Comb.:	R1
Max. total:	1552	Point :	A00	Load Comb.:	RT

S Y S T E M S U M M A R Y

Maximum sustained stress

Point : A00
Stress psi : 1300
Allowable psi : 16700
Ratio : 0.08
Load combination : GR + Max P

Maximum displacement stress

Point : A02
Stress psi : 0
Allowable psi : 25050
Ratio : 0.00
Load combination : Cold to T1

Maximum occasional stress

Point : A00
Stress psi : 5093
Allowable psi : 22211
Ratio : 0.23
Load combination : RTOTAL

Maximum hoop stress

Point : A00
Stress psi : 2936
Allowable psi : 16700
Ratio : 0.18
Load combination : Max P

Maximum sustained stress ratio

Point : A00
Stress psi : 1300
Allowable psi : 16700
Ratio : 0.08
Load combination : GR + Max P

Maximum displacement stress ratio

Point : A02
Stress psi : 0
Allowable psi : 25050
Ratio : 0.00
Load combination : Cold to T1

SYSTEM SUMMARY

Maximum occasional stress ratio

Point : A00
Stress psi : 5093
Allowable psi : 22211
Ratio : 0.23
Load combination : RTOTAL

Maximum hoop stress ratio

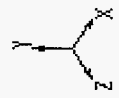
Point : A00
Stress psi : 2936
Allowable psi : 16700
Ratio : 0.18
Load combination : Max P

*** The system satisfies ASME B31.3 code requirements ***
*** for the selected options ***

Point A01 A

A00

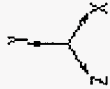
A01



AY FARM DISTRIBUTOR ANALYSIS
W320 TANK 241-C-106 SLUICING ER4319

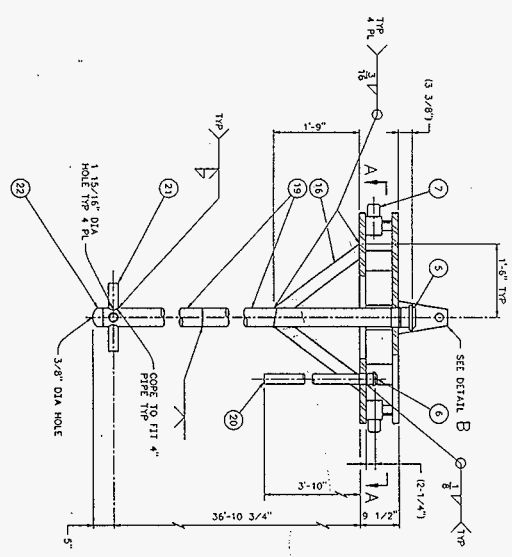
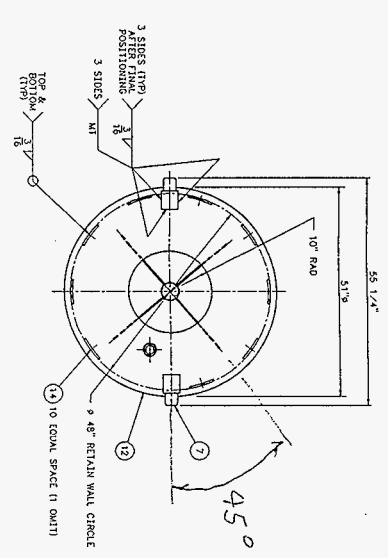
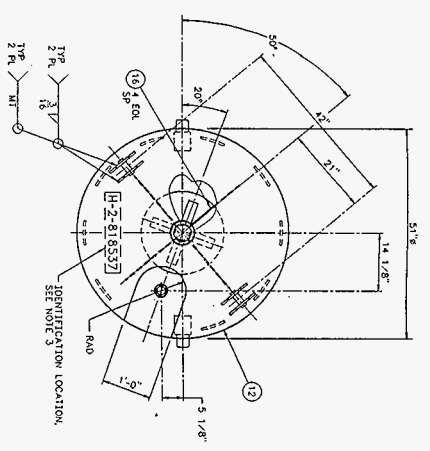
Point A02 A

CELL E01
A02
~~D01 B01~~
~~F03~~

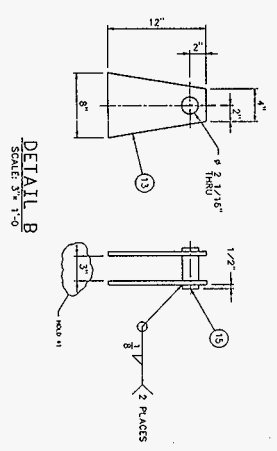


AY FARM DISTRIBUTOR ANALYSIS
W320 TANK 241-C-106 SLUICING ER4319

11250 27100



1 FLANGE ASSEMBLY
SCALE: 1" = 1'-0"
FOR TRUE ORIENTATION SEE PLAN



DETAIL B
SCALE: 3" = 1'-0"

REV	PART/COMB NUMBER	DESCRIPTION	QUANTITY	UNIT	REVISION
010	ASSEMBLY		1		
1	M-2-S010D-3	NOZZLE WELD CORR 4" SCHED 80	3		
1	M-2-S010B-2	NOZZLE WELD CORR 2"	5		
2	M-2-S010A-1	TRONNUN	6		
1	M-2-S010I-5	LIFTING BALL	8		
			9		
			10		
			11		
AR	PL 1 1/2"	ASTM A 36	12		
AR	PL 1/2" x 6' x 6 1/2"	ASTM A 36	13		
9	PL 1/2" x 6' x 6 1/2"	ASTM A 36	14		
2	BAR 82" x 5" LG	ASTM A 36	15		
AR	BAR 2" x 3/8"	ASTM A 276 304L	16		
			17		
			18		
AR	PIPE, 4" SCHED 80S	ASTM A 312 OR TP 304L	19		
AR	PIPE, 2" SCHED 40S	ASTM A 312 OR TP 304L	20		
4	PIPE, 2" SCHED 80S x 6' LG	ASTM A 312 OR TP 304L	21		
1	Cap, 4" SCHED 80 BW	ASTM A 312 OR TP 304L	22		

- NOTES:
- SEE DIMENSIONS, NOTES, LEGEND AND KEY PLAN.
 - SEE CODE M-2-818537.
 - ASSEMBLY WELDING PROTECTIVE COATING AND PAINTING SHALL BE CONFORMANCE WITH SPEC W-20-C-3.
 - PAINT 2" HIGH BLACK LETTERS ON YELLOW BACKGROUND.
 - DIMENSIONS ARE IN INCHES. TOLERANCES ARE AS FOLLOWS UNLESS OTHERWISE NOTED.
 - FRACCTIONS 1/8" DIMENSIONS 100" ANGLES 25°

C.D. JONES, PE

U.S. DEPARTMENT OF ENERGY
 FEDERAL BUREAU OF SURVEYING
 U.S. GEOLOGICAL SURVEY
 DISTRICT OFFICE
 107 KATHERINE HANCOCK COMPANY
 DISTRIBUTOR
 02A-RC-(A)-(B)
 W-20-TANK 241-C-100 SURVEILING
 11-2-818537
 10

NO.	DESCRIPTION	DATE	BY	CHKD BY
1	ISSUED FOR CONSTRUCTION	11/25/80	C.D. JONES	
2	REVISION			
3	REVISION			
4	REVISION			
5	REVISION			
6	REVISION			
7	REVISION			
8	REVISION			

W320-27-019

Winch Analysis -Slurry/Sluice Pump

This sheet shows the status and description of the attached Design Analysis sheets.

Discipline 27, Piping and Vessels WO/Job No. ER4319 Calculation No. W320-27-019
 Project No. & Name Project W-320 Waste Retrieval for Tank 241-C-106
 Calculation Item WINCH ANALYSIS - Slurry/Sluice Pump

These calculations apply to:

Dwg. No. H-2-818442, -443, -444, -494, -495, -496, Rev. No. 01
 Dwg. No. -497, -498, -499 Rev. No. 01
 Other (Study, CDR) _____ Rev. No. _____

The status of these calculations is:

- Preliminary Calculations
- Final Calculations
- Check Calculations (On Calculation Dated _____)
- Void Calculation (Reason Voided _____)

Incorporated in Final Drawings? Yes No
 This calculation verified by independent "check" calculation? Yes No

Original and Revised Calculation Approvals:

	Rev. 0 Signature / Date	Rev. 1 Signature / Date	Rev. 2 Signature / Date
Originator	<u>D.L. Stone</u> <u>9/8/95</u>		
Checked by	<u>Michael R. Curtis</u> <u>9/8/95</u>		
Approved by	<u>CP Jones</u> <u>9/8/95</u>		
Checked Against Approved Vendor Data	<u>M. Johnson</u> <u>4.10.98</u>		

INDEX

Design Analysis Page No.	Description
<u>i</u>	<u>Calculation Identification and Index</u>
<u>ii</u>	<u>Calculation Cross Index</u>
<u>1</u>	<u>Objective, Design Criteria, Design Inputs and Assumptions</u>
<u>2-3</u>	<u>References</u>
<u>4- 78</u>	<u>Calculations and Conclusion</u>
<u>Appendix</u>	<u>Supporting Documentation</u>

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-01.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	<i>D.L. Stribo</i>
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. COSTER MRC.
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

OBJECTIVES:

The objective of this calculation is to provide a detailed design for a winch, to raise and lower submersible pumps inside tanks 241-C-106 and 241-AY-102. The pumps will transfer the contents of Tank 241-C-106 to Tank 241-AY-102, and provide supernate to the sluicing process.

DESIGN CRITERIA:

1. Project W320, Tank 241-C-106 Sluicing, Functional Design Criteria (WSC document No. WHC-SD-W320-FDC-001, Rev. 0).
2. SDC 4.1, Rev. 11, Design Loads For Facilities.
3. DOE-RL-92-36, *Hanford Site Hoisting and Rigging Manual*, Section 9.2.3.2, January 15, 1993
4. Project 93L-EWW-320, Tank 241-C-106, Letter of Instruction, Addendum 14

DESIGN INPUTS:

Design Methodology :

- Seismic analysis was performed according to SDC 4.1, Rev. 11, Design Loads For Facilities. (Ref. 2)
- Standard engineering methods were used to determine Shear, Tensile, Bearing and Bending stresses. (Ref. 9)
- Weld analysis uses Blodgetts's method for analysis of a weld as a line (Ref. 9)
- Moment Distribution Method for Indeterminant Beams is used to determine reaction forces on Axle 12. (Ref. 13)
- Shaft design methods include torsion analysis for shafts of varying diameters, and minimum shaft diameter equation. (Ref. 14)
- Bearing loads were determined using Equivalent Load equations provided in bearing vendor catalogs. (Ref. 8)
- Bearing loads were determined using Equivalent Load equations provided in bearing vendor catalogs. (Ref. 26 & 27)
- Stress on Reel Hub is determined by standard beam analysis and correction for a curved beam. (Ref. 8)
- Gear Strength is determined using the AGMA Strength Formula and Lewis Beam Strength Formula. (Ref. 12)

Data:

Safety factor will be 3 to 1, ($SF_y = 3$), against Yield Strength ,
or 5 to 1, ($SF_u = 5$) against Ultimate Strength. Ref. 3

Assumptions:

1. Pit and winch are both rigid structures and are rigidly connected.
2. Hose/Pipe connection at plate is assumed as a hinged joint and that the pipe will be allowed to deflect with movement of the pump and assemblies.
3. Pump and motor weight = 3000 lbs. (Ref. 23 & 24)

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-02.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>M.R.C.</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

References :

1. Project W320, Tank 241-C-106 Sluicing, Functional Design Criteria (WSC document No. WHC-SD-W320-FDC-001, Rev. 2).
2. SDC 4.1, Rev. 12, Design Loads For Facilities.
3. NUREG-0800, U.S. Nuclear Regulatory Commission Standard Review Plan, Section 3.7.2, Rev.2 (8/89)
4. DOE-RL-92-36, *Hanford Site Hoisting and Rigging Manual*, Section 9.2.3.2, January 15, 1993
5. ANSI B16.9
6. ANSI B16.5
7. Steel Construction Manual, Ninth edition.
8. Machinery's Handbook, 24th edition
9. Design of Welded Structures, Homer Blodgett.
10. Formulas for Stress and Strain, R.J. Roark and W.C. Young, Edition 5
11. Machine Design, Shigley & Mische, Edition 5, 1989
12. Machine Design, Shigley & Mische, Edition 1
13. Structural Analysis, Jack. C McCormac, 4th Edition, 1984
14. Mechanics and Materials, I.J. Levinson, 2nd Edition.
15. Crane Technical Paper No. 410, *Flow of Fluids Through Pipes, Fittings and Valves, 25th Printing*
16. Joseph T. Ryerson, Inc., *Stock List and Data Book*, 1994
17. T-M Swivel Joint Catalog.
18. Gates Hose Catalog, No.39496-000, Dated 3/30/92
19. Catalog sheets by Fax from American Hose & Fitting Co., 7/6/95
20. Boston Gear Catalog, 1995.
21. Associated Spring Catalog
22. Mc White Wire Rope Catalog, 4th Edition, 1991
23. Lawrence Pump information sheets, dated June 6, 1995.
24. Reliance Motor information sheets, dated June 6, 1995.
25. Crucible Materials Corporation, *ARMCO Product Data Bulletin No. S-45*
26. Timkin Bearing Selection Handbook for Tapered Roller Bearings, Revised 1986
27. Torrington Bearing Catalog Form # 100-889-50M, 1st Edition, Second Printing
28. Journal Entry, June 6, 1995. Teleconference with Torrington Bearing Co.
29. *Mathcad*, Version 4.0, by MathSoft, Inc.
30. *Excel*, Version 4.0, by Microsoft.
31. *Autocad*, Release 12, Autodesk, inc, 1993, Inc.
32. Calculation #W320-27-016 Stress Analysis for W-320 C-Farm Jumpers

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-02.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

- 33.0. Project W320 Drawing No. H-2-818442, Rev. 0
- 33.1. Project W320 Drawing No. H-2-818443, Rev. 0
- 33.2. Project W320 Drawing No. H-2-818444, Rev. 0
- 33.3. Project W320 Drawing No. H-2-818494, Rev. 0
- 33.4. Project W320 Drawing No. H-2-818495, Rev. 0
- 33.5. Project W320 Drawing No. H-2-818496, Rev. 0
- 33.6. Project W320 Drawing No. H-2-818497, Rev. 0
- 33.7. Project W320 Drawing No. H-2-818498, Rev. 0
- 33.8. Project W320 Drawing No. H-2-818499, Rev. 0

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-01.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	J. T. [Signature]
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	H. R. WOSTER, MRC
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

CALCULATIONS:

Page:

- 5 - 8 Seismic Evaluation
- 9 - 20 Analysis of Pump and Sheave Assembly Components
- 21 - 25 Analysis of Retrieval Cable Assembly
- 26 - 40 Axle 12 - Component Design
- 41 - 46 Design of Reel Hub
- 47 - 50 Gear Analysis
- 51 - 54 Axle 11 - Component Design
- 55 - 60 Axle 15 - Component Design
- 61 - 62 Analysis of Winch Housing Stresses
- 63 - 67 Upper Driveshaft Analysis
- 68 - 70 Guide Idler Component Analysis
- 71 - 75 Plate Stress and Bolt Analysis
- 76 - 78 Analysis of Lifting Brackets

CONCLUSIONS:

The results of this analysis indicate that the components of of the Winch Assembly meet, exceed or fall within acceptable limits of the Safety Factor requirements set forth in DOE-RL-92-36, *Hanford Site Hoisting and Rigging Manual*, Section 9.2.3.2, January 15, 1993.

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-03.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Seismic Analysis of Winch and Pump Assembly to Safety Class 1 Criteria

The winch / pump assembly is classified as Safety Class 3 per the Functional Design Criteria for the project. Based on the SDC 4.1, Rev. 11, the design for the winch / pump assembly is required to be analyzed to Safety Class 1 criteria, due to the potential structural failure and impact of the winch/ pump assembly on the Safety Class 1 structure (the tank structure). Safety Class 1 design requires a dynamic analysis to be performed using the response spectra per SDC 4.1, Rev.11 (Figure 3) as the seismic input for the design. Conservatively, an equivalent static load method is used in the evaluation since the winch may be represented by a single model configuration. A factor of 1.5 is applied to the peak acceleration of the appropriate response spectrum per SDC 4.1, Rev.11.

The upper, winch structure is rigidly attached to the top of the base slab of the pump pit in both horizontal directions. The pump pit is considered a rigid structure, therefore, the ground acceleration is used as the seismic input.

The lower winch structure consists of a cable and sheave assembly suspending the process pump and includes the associated hose and connections. The configuration of the lower structure, therefore can be represented as a single mass suspended by a flexible cable, subjected to seismic loading.

The primary concern for the tank structure is the potential impact of the pump with the tank wall. The maximum displacement of the pump will be evaluated based on the seismic input and a comparison made as to the proximity of the pump relative to the tank wall.

The remaining concern is the structural adequacy of the winch plate, which supports the winch/ pump assembly, spanning the penetration opening in the tank. The magnitude of the vertical, seismic response required to be used is defined in SDC 4.1, Rev. 11, as 2/3 of the peak, horizontal response.

Seismic Components: (Assume 2 % damping, conservative) (Ref. 4, Section 3.7.2, 11 - 1.b)

$g := 386.4 \cdot \frac{\text{in}}{\text{sec}^2}$ Acceleration of gravity (Ref. 2, Table 7)

LF := 1.5 Load Factor

$a_g := 0.2 \cdot g$ Peak Horizontal Free-field Ground Acceleration (Ref. 4, Section 3.7.2, 11 - 1.b.iii)
 (Ref. 2, Fig.3)

A := 2.74 Acceleration Amplification at 2% damping (Ref. 2, Table 7, 2 % Damping)

$a_{\text{maxh}} := a_g \cdot A \cdot 1.5$ $a_{\text{maxh}} = 317.621 \cdot \frac{\text{in}}{\text{sec}^2} = 0.822 \cdot g$ Horizontal seismic component

$a_{\text{maxv}} := \left(\frac{2}{3}\right) \cdot a_{\text{maxh}}$ $a_{\text{maxv}} = 211.747 \cdot \frac{\text{in}}{\text{sec}^2} = 0.548 \cdot g$ Vertical seismic component (Ref. 2, B.3.3)

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-03.mcd
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Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Potential Impact of the Pump with the Tank Wall

The primary concern to be addressed in this evaluation is the potential interaction of the pump with the tank wall, due to pendulum action of the pump/ cable system.

Determine the maximum, horizontal displacement of the pump relative to the tank wall:

- | | | | |
|--|--------------------------------|---|-----------------|
| $L := 26\text{-ft}$ | $L = 312\text{-in}$ | (Maximum "L" USED, conservative) | |
| $f := \left(\frac{1}{2 \cdot \pi}\right) \cdot \sqrt{\frac{g}{L}}$ | $f = 0.177\text{-time}^{-1}$ | Natural Frequency | |
| $T := \frac{1}{f}$ | $T = 5.646\text{-sec}$ | Natural Period | |
| $d := 13\text{-in}$ | | Displacement | Ref. 2, Fig.3 |
| $D := 1.63$ | | Spectrum Amplification Factor at 2% damping | Ref. 2, Table 7 |
| $d_{SDC4.1} := D \cdot d$ | $d_{SDC4.1} = 21.19\text{-in}$ | Based on engineering judgement, displacement of 21" is considered to be negligible. | |

Hose / Pipe Assembly

- | | | | |
|---|---|---|-----------------|
| $SG := 1.2$ | | Specific Gravity of Process Fluid | Ref. 1 |
| $\rho_w := 62.41 \cdot \frac{\text{lb}}{\text{ft}^3}$ | | Density of water at Minimum Operating Temperature | Ref. 15, Pg. A6 |
| $\rho := SG \cdot \rho_w$ | $\rho = 74.892 \cdot \frac{\text{lb}}{\text{ft}^3}$ | Density of Process Fluid at Minimum Operating Temperature | |
| $W_f := 7.5 \cdot \frac{\text{lb}}{\text{ft}}$ | | Weight per foot of hose. | Ref. F, Pg. 69 |
| $L_{\text{hose}} := (9.8 + 10.8) \cdot \text{ft}$ | | Combined length of hoses | |
| $R_h := 2\text{-in}$ | | Inner Radius of hose. | Ref. 18, Pg. 69 |
| $W_{\text{hose}} := W_f \cdot L_{\text{hose}}$ | $W_{\text{hose}} = 154.5 \cdot \text{lb}$ | Weight of 4" hose. | Ref. 33.2, Sh.2 |
| $V_h := \pi \cdot R_h^2 \cdot L_{\text{hose}}$ | $V_h = 1.798 \cdot \text{ft}^3$ | Volume of hose. | |
| $\text{Hose contents} := V_h \cdot \rho$ | $\text{Hose contents} = 134.633 \cdot \text{lb}$ | Weight of Process Fluid fluid in combined length of hose. | |

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Calc. No. W-320-27-019
Revision No.: 0
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DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-03.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Hose / Pipe Assembly (continued....)

$W_p := 10.79 \frac{\text{lb}}{\text{ft}}$ Weight per foot of pipe. Ref. 15

pipe L := 114-in Pipe Length

$W_{\text{pipe}} := W_p \cdot \text{pipe L}$ $W_{\text{pipe}} = 102.505 \cdot \text{lb}$ Weight of 93.8" of 4" pipe.

$R_p := 2.013\text{-in}$ Inner Radius of pipe.

$V_p := \pi R_p^2 \cdot \text{pipe L}$ $V_p = 0.84 \cdot \text{ft}^3$ Volume of pipe.

Pipe contents := $V_p \cdot \rho$ $\text{Pipe contents} = 62.898 \cdot \text{lb}$ Weight of Process Fluid fluid in pipe.

Coupling := 4 (27-lbf) Est. Weight of 4 Hose Couplings Ref. 19, Pg. 399

Swivel joint := 36-lbf + 2(72-lbf) Weight of 1, 3" and 2, 4" swivel joints Ref. 17, Pg. 11

pipe CG := $\frac{\text{pipe L}}{2}$ $\text{pipe CG} = 57 \cdot \text{in}$ Center of Gravity of Pipe

Mass pipe := $\frac{W_{\text{pipe}} + \text{Pipe contents}}{g}$
 $\text{Mass pipe} = 165.269 \cdot \text{lb}$ Mass of the Pipe

Mass hose := $\frac{W_{\text{hose}} + \text{Hose contents} + \text{Coupling} + \text{Swivel joint}}{g}$
 $\text{Mass hose} = 576.668 \cdot \text{lb}$ Mass of the Hose

$F_{\text{pipeH}} := \text{Mass pipe} \cdot a_{\text{maxH}}$ $F_{\text{pipeH}} = 135.961 \cdot \text{lb}$ Horizontal Seismic Force due to Pipe

$F_{\text{hoseH}} := \text{Mass hose} \cdot a_{\text{maxH}}$ $F_{\text{hoseH}} = 474.403 \cdot \text{lb}$ Horizontal Seismic Force due to Hose Assembly

$F_V := (\text{Mass pipe} + 2 \cdot \text{Mass hose}) \cdot (g + a_{\text{maxV}})$ $F_V = 2.043 \cdot 10^3 \cdot \text{lb}$ Vertical Seismic Force due to Pipe/Hose Assembly

$M := (F_{\text{pipeH}}) \cdot (\text{pipe CG}) + (F_{\text{hoseH}}) \cdot (\text{pipe L})$ $M = 6.183 \cdot 10^4 \cdot \text{lb-in}$ Moment due to Pipe/Hose Assembly

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-03.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Stress at Pipe Connection to Winch Plate

Shear Stress due to Horizontal Seismic Force

$A := 3.174 \cdot \text{in}^2$		Area of Shear plane (Metal Area of pipe)	
$S_y := 30000 \cdot \frac{\text{lb}}{\text{in}^2}$		Yield strength of Schedule 40S, ASTM A-312 Gr Tp 304 Pipe	
$S_{\text{all}} := (0.40) \cdot S_y$	$S_{\text{all}} = 1.2 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$	Allowable Shear Stress	Ref. 7, Part 5, Chap. D.3.1
$S_{\text{act}} := \frac{F_{\text{pipeH}} + F_{\text{hoseH}}}{A}$	$S_{\text{act}} = 192.301 \cdot \frac{\text{lb}}{\text{in}^2}$	Calculated stress on pipe.	
$SF := \frac{S_{\text{all}}}{S_{\text{act}}}$	$SF = 62.402$	Safety factor > 3	<u>OK</u>

Tensile Stress due to Vertical Seismic Force

$T_{\text{all}} := (0.45) \cdot S_y$	$T_{\text{all}} = 1.35 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$	Allowable Stress in Tension	Ref. 11, Page 12
$T_{\text{act}} := \frac{F \cdot V}{A}$	$T_{\text{act}} = 643.619 \cdot \text{psi}$	Calculated Tensile Stress on Pipe resulting from Bending Moment at Plate	
$\frac{T_{\text{all}}}{T_{\text{act}}} = 20.975$		Safety factor > 3	<u>OK</u> See Assumptions, #2

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-04.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/18/95	By	M.R. CUSTER MKC
Location	C TANK FARM -- HANFORD 200 EAST AREA	Revised		By	

WEIGHT of PUMP ASSEMBLY (Supported by the Sheave Assembly)

(Most conservative configuraiton considered as the pump in the fully elevated position.)

Weight of Pump

W_{pump} := 3000-lbf

Weight of pump.

Ref. 33.4
 Ref. 33.6, Sh. 1&2

Ref. 23

Weight of Hose and Contents

W_{hose} := 81-lbf

Weight of 10.8' of 4" hose.

Ref. 33.2

W_{contents} := 70.584-lbf

Weight of Process Fluid fluid in 8 feet of hose.

Weight of pipe fittings (on pump discharge).

W_C := 2·(27-lbf)

Weight of Hose Couplings

Ref. 33.0, Sh. 1, Item 1

Ref. 19

Flange₃ := 13-lbf

3", 300 lb, slip on flange

Ref. 33.4, Sh. 1, Item 27

Ref. 6

E₉₀ := 4.58-lbf

3" elbow (90)

Ref. 33.4, Sh. 1, Item 24

Ref. 5

SJ₃ := 37-lbf

3" swivel joint

Ref. 33.4, Sh. 1, Item 11

Ref. 17, Pg. 11

SJ₄ := 1.5·(72-lbf)

1-1/2 4" swivel joints

Ref. 33.4, Sh. 1, Item 12

Ref. 17, Pg. 11

Reducer := 3.5-lbf

3 x 4 Standard Reducer

Ref. 33.4, Sh. 1, Item 26

Ref. 5

Pipe₃ := 7.58 $\frac{\text{lbf}}{\text{ft}}$

Attached 3 inch pipe

Ref. 33.4, Sh. 1, item 30

Ref. 15, Pg. B-16

Pipe₄ := 10.79 $\frac{\text{lbf}}{\text{ft}}$

Attached 4 inch pipe

Ref. 33.4, Sh. 1, Item 29

Ref. 15, Pg. B-17

L_{pipe} := 2.25-in

Length of attached pipes (typ)

RE_{3x4} := L_{pipe} (Pipe₃ + Pipe₄) + Reducer

RE_{3x4} = 6.944 ·lbf

W_{fj} := Flange₃ + E₉₀ + SJ₃ + SJ₄ + RE_{3x4}

W_{fj} = 169.524 ·lbf Combined Weight of fittings and joints

Total Weight of Pump Assembly :

W_P := W_{pump} + W_{hose} + W_{contents} + W_{fj} + W_C W_P = 3.375·10³ ·lbf

Pump Assembly

W_{PA} := 1.2·W_P

W_{PA} = 4.05·10³ ·lbf

Impact Loading Considered

Ref. 7, Pg. 5-29, Section A4.2

DESIGN ANALYSIS

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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

CHECK OF PUMP / WINCH SUPPORT SYSTEM COMPONENTS

3/4" Diam. Pump Support BOLTS

(Assume A307 structural bolts, worst case.)

Ref. 33.4, Pg. 1, Item 33

$n_b := 8$	Number of bolts used
$A_{blt} := 0.442 \cdot \text{in}^2$	Cross sectional area of bolt
$F_v := 10000 \cdot \frac{\text{lb}}{\text{in}^2}$	Allowable shear strength of bolts

Ref. 7, Pg 4-5, Table I-D

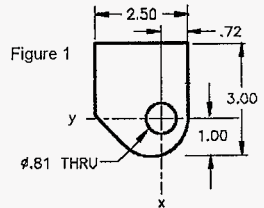
$S_{act} := \frac{(W_{PA} - W_{pump}) \cdot n_b}{A_{blt}}$	$S_{act} = 296.982 \cdot \frac{\text{lb}}{\text{in}^2}$	Calculated shear stress
--	---	-------------------------

$SF := \frac{F_v}{S_{act}}$	$SF = 33.672$	Safety factor > 5	<u>OK</u>
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Pump Support Brackets welded to the 3"x3" square tube plates.

Ref. 33.6, Sh. 2, Detail A.

$w := \left[\left(1 - \frac{0.81}{2} \right) \cdot \text{in} \right]$	$w = 0.595 \cdot \text{in}$	Width of Shear plane.
$th := \left(\frac{3}{8} \right) \cdot \text{in}$	$th = 0.375 \cdot \text{in}$	Plate thickness
$AR_b := 4 \cdot w \cdot th$	$AR_b = 0.892 \cdot \text{in}^2$	Area of brackets loaded by the bolt.
$S_y := 25000 \cdot \frac{\text{lb}}{\text{in}^2}$		Yield strength of ASTM A240 Type 304L



$S_{all} := (0.45) \cdot S_y$	$S_{all} = 1.125 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$	Allowable Stress	Ref. 7, Pg 5, Chap. D.3.1
$S_{act} := \frac{W_{PA}}{AR_b}$	$S_{act} = 4.538 \cdot 10^3 \cdot \frac{\text{lb}}{\text{in}^2}$	Calculated stress on brackets.	
$SF := \frac{S_y}{S_{act}}$	$SF = 5.509$	Safety factor > 3	<u>OK</u>

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-04.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP** Date **9/8/95** By **D.L. STONE**
 PROJECT **W-320 WASTE RETRIEVAL for TANK 241-C-106** Checked **9/8/95** By **M.R. CUSTER M.R.C.**
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

Sizing of Bolts connecting Pump to Sheave Assembly Brackets

Ref. 33.6, Pg 2

Shear stress

$$D := \left(0.81 - \frac{1}{16}\right) \cdot \text{in}$$

Bolt Diameter

$$A_a := \frac{\pi D^2}{4}$$

$$A_a = 0.439 \cdot \text{in}^2$$

Cross Sectional Area

$$F_{\text{bolt}} := \left(\frac{W_{PA}}{4}\right)$$

$$F_{\text{bolt}} = 1.013 \cdot 10^3 \cdot \text{lb}$$

Force of Pump Assembly felt by each shear plane.

$$S_{\text{act}} := \frac{F_{\text{bolt}}}{A_a}$$

$$S_{\text{act}} = 2.307 \cdot 10^3 \cdot \frac{\text{lb}}{\text{in}^2}$$

Calculated Shear Stress per Bolt.

$$S_y := 100000 \cdot \frac{\text{lb}}{\text{in}^2}$$

Yield strength for ASTM A 193, Class 2, B8T

$$S_{\text{all}} := 0.22 \cdot S_y$$

$$S_{\text{all}} = 2.2 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$$

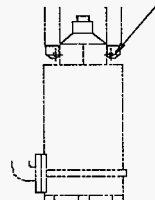
Allowable Shear Stress

Ref. 7, Pg. 4.5

$$SF_S := \frac{S_{\text{all}}}{S_{\text{act}}}$$

$$SF_S = 9.535$$

Safety factor > 3 OK



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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/15/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

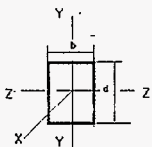
Welds connecting pump support brackets to the 3" x 3" plate.

Ref. 33.6, Sh. 2, Detail A.

$F_x := \frac{W \cdot PA}{4}$ Weld Tensile Load
("x" direction on subroutine sketch, below)

$M_{arm} := \left(\frac{2.50}{2} - 0.72 \right) \cdot in$ Moment arm (Moment about Z)
(distance from weld centerline to load application point)

WELD PROPERTIES:



$b := 0.375 \cdot in$	Weld Outline Dimension - short side	Ref. 9, Pg. 7.4 6 & 7
$d := 2.5 \cdot in$	Weld Outline Dimension - long side	
$C_y := \frac{b}{2}$	$C_y = 0.188 \cdot in$	Location of center of gravity with respect to y axis.
$C_z := \frac{d}{2}$	$C_z = 1.25 \cdot in$	Location of center of gravity with respect to z axis.
$A_w := 2 \cdot (b + d)$	$A_w = 5.75 \cdot in$	Weld length (1 Bracket)
$S_{wy} := b \cdot d + \frac{b^2}{3}$	$S_{wy} = 0.984 \cdot in^2$	Section modulus about y axis
$S_{wz} := b \cdot d + \frac{d^2}{3}$	$S_{wz} = 3.021 \cdot in^2$	Section modulus about z axis
$J_w := \frac{(b + d)^3}{6}$	$J_w = 3.961 \cdot in^3$	Polar Moment of Inertia

JOINT LOADS:

Tensile	$F_x := 1.013 \cdot 10^3 \cdot lbf$	$F_y := 0 \cdot lbf$	$F_z := 0 \cdot lbf$
Moment	$M_x := 0 \cdot in \cdot lbf$	$M_y := 0 \cdot in \cdot lbf$	$M_z := (F_x \cdot M_{arm})$

WELD STRESS:

$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5}$ $f_w = 353.74 \cdot \frac{lbf}{in}$

FILLET WELD SIZE REQUIRED:

$S = 15800 \cdot psi$ Allowable Weld Stress per AWS & AISC Ref. 9, Pg. 7.4-8

$w := \frac{f_w}{0.707 \cdot S}$ $w = 0.032 \cdot in$ Minimum Allowable Fillet Weld Required by Design

$w_{Code} := \left(\frac{3}{16} \cdot in \right)$ Minimum Size Fillet Weld Required by Code for material thickness of 3/8". Ref. 7, Part 5, Section J.2.2, Table J2.4

$w_{act} := \left(\frac{1}{4} \cdot in \right)$ Weld size specified in design drawings. Ref. 33.6, Pg. 2

$SF_{f_w} := \frac{w_{act}}{w}$ $SF_{f_w} = 7.895$ Safety factor based on a 1/4" weld > 3 OK

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-04.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

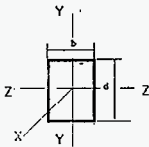
Weld on the two 3" x 3" x 3/8" plates attached to the 3" square tubes.

$F_x := \frac{W_{PA}}{2}$ Weld Tensile Load Ref. 33.6, Sh. 1&2, Item 17
(In "x" direction on subroutine sketch, below)

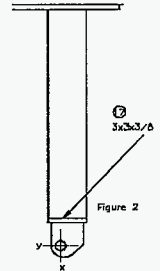
$M_{arm} := \left(\frac{2.50}{2} - 0.72 \right) \cdot in$ Moment arm (Moment about Z imposed by eccentric bracket loading.)

Ref. 9, Pg. 7.4 6 & 7

WELD PROPERTIES:



$b := 3 \cdot in$	Weld Outline Dimension - short side
$d := 3 \cdot in$	Weld Outline Dimension - long side
$C_y := \frac{b}{2}$	$C_y = 1.5 \cdot in$ Location of center of gravity with respect to y axis.
$C_z := \frac{d}{2}$	$C_z = 1.5 \cdot in$ Location of center of gravity with respect to z axis.
$A_w := 2 \cdot (b + d)$	$A_w = 12 \cdot in$ Weld length (1 Tube)
$S_{wy} := b \cdot d + \frac{b^2}{3}$	$S_{wy} = 12 \cdot in^2$ Section modulus about y axis
$S_{wz} := b \cdot d + \frac{d^2}{3}$	$S_{wz} = 12 \cdot in^2$ Section modulus about z axis
$J_w := \frac{(b + d)^3}{6}$	$J_w = 36 \cdot in^3$ Polar Moment of Inertia



JOINT LOADS:

Tensile	$F_x = 2.025 \cdot 10^3 \cdot lbf$	$F_y := 0 \cdot lbf$	$F_z := 0 \cdot lbf$
Moment	$M_x := 0 \cdot in \cdot lbf$	$M_y := 0 \cdot in \cdot lbf$	$M_z := (F_x \cdot M_{arm})$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5} \quad f_w = 258.196 \cdot \frac{lbf}{in}$$

FILLET WELD SIZE REQUIRED:

$S := 15800 \cdot psi$	Allowable Weld Stress per AWS & AISC	Ref. 9, Pg. 7.4-8
$w := \frac{f_w}{0.707 \cdot S}$	$w = 0.023 \cdot in$	Minimum Allowable Fillet Weld Required by Design
$w_{Code} := \left(\frac{3}{16} \cdot in \right)$	Minimum Size Fillet Weld Required by Code for material thickness of 3/8".	Ref. 7, Part 5, Section J.2.2, Table J2.4
$w_{act} := \left(\frac{5}{16} \cdot in \right)$	Weld size specified in design drawings.	Ref. 33.6, Pg. 2
$SF_{f_w} := \frac{w_{act}}{w}$	$SF_{f_w} = 13.52$	Safety factor based on a 5/16" weld > 3 <u>OK</u>

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DESIGN ANALYSIS

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

WEIGHT of SHEAVE ASSEMBLY (Supported by Axles)

$$W_{\text{steel}} = 490.8 \frac{\text{lb}}{\text{ft}^3} \quad (\text{Ref.7, Page 6-8})$$

Weight of 2 Sheaves

$$V_{\text{sheavs}} = 4 \left[(0.25 \cdot \text{in}) \left[\frac{\pi (9.5 \cdot \text{in})^2}{4} - \frac{\pi (1.508 \cdot \text{in})^2}{4} \right] \right] + 2 \left[(2 \cdot \text{in}) \left[\frac{\pi (7.25 \cdot \text{in})^2}{4} - \frac{\pi (1.508 \cdot \text{in})^2}{4} \right] \right] \cdot \text{Volume}$$

(Ref. 33.6, Item 3)

$$W_{\text{sheavs}} = (V_{\text{sheavs}} \cdot W_{\text{steel}}) \quad W_{\text{sheavs}} = 64.498 \cdot \text{lb} \quad \text{Weight}$$

Weight of 2 Square Tubing Support Arms

$$w_{\text{tube}} = 8.81 \frac{\text{lb}}{\text{ft}} \quad \text{Weight of } 3 \times 3 \times 1/4 \text{ Square Carbon Steel Tubing :} \quad (\text{Ref. 16})$$

$$L_{\text{tube}} = 2 \cdot (17.875 \cdot \text{in}) \quad \text{Total Length of 2 Support Arms} \quad (\text{Ref. 33.6, Item 2})$$

$$W_{\text{tube}} = L_{\text{tube}} \cdot w_{\text{tube}} \quad W_{\text{tube}} = 26.246 \cdot \text{lb} \quad \text{Weight}$$

$$w_{\text{psf}} = 15.32 \frac{\text{lb}}{\text{ft}^2} \quad \text{Weight of 3/8 in. Hot Rolled Carbon Steel plate per square foot:} \quad (\text{Ref. 16})$$

Weight of 4 Support Brackets

$$A_{\text{plate}} = 4 \left[(2.5) \cdot (3) - 1 - \frac{\pi (0.81)^2}{4} \right] \cdot \text{in}^2 \quad A_{\text{plate}} = 23.939 \cdot \text{in}^2 \quad \text{Area of 4 Brackets} \quad (\text{Ref. 33.6, Item A})$$

$$W_{\text{brackets}} = A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{brackets}} = 2.547 \cdot \text{lb} \quad \text{Weight}$$

Weight of 2, 3 x 3 x 3/8 End Plates

(Ref. 33.6, Item 2)

$$A_{\text{plate}} = 2 \cdot (3 \cdot 3 \cdot \text{in}^2) \quad A_{\text{plate}} = 18 \cdot \text{in}^2 \quad \text{Area of 2 Plates}$$

$$W_{\text{ends}} = A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{ends}} = 1.915 \cdot \text{lb} \quad \text{Weight}$$

Sheave housing walls

(Ref. 33.6, Detail 2)

$$A_{\text{plate}} = 2 \cdot \left[((21) \cdot (11)) - (2 \cdot 2) - 2 \cdot \left[\frac{\pi (1.505)^2}{4} \right] \right] \cdot \text{in}^2 \quad A_{\text{plate}} = 446.884 \cdot \text{in}^2 \quad \text{Area of Wall Plate}$$

$$W_{\text{housing}} = A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{housing}} = 47.544 \cdot \text{lb} \quad \text{Weight}$$

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9 / 8 / 95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Sheave housing cross brace

(Ref. 33.6, Item 2)

$$A_{\text{plate}} := 2 \cdot (((8.625) \cdot (11)) - (3 \cdot 3) - (4.5 \cdot 4.5)) \cdot \text{in}^2 \quad A_{\text{plate}} = 131.25 \cdot \text{in}^2 \quad \text{Area of Cross Brace Plate}$$

$$W_{\text{brace}} := A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{brace}} = 13.964 \cdot \text{lb} \cdot \text{ft} \quad \text{Weight}$$

Sheave housing Stabilizer Braces

(Ref. 33H.6, Item 2)

$$A_{\text{plate}} := 4 \cdot ((1.5) \cdot (7)) \cdot \text{in}^2 \quad A_{\text{plate}} = 42 \cdot \text{in}^2 \quad \text{Area of Bracket Plates}$$

$$W_{\text{braces}} := A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{braces}} = 4.468 \cdot \text{lb} \cdot \text{ft} \quad \text{Weight}$$

Sheave Assembly Support Bracket

$$A_{\text{plate}} := \left[\left[\frac{\pi}{2} (1.5^2) + (3 \cdot 3) \right] - \frac{\pi}{4} (1.5^2) \right] \cdot \text{in}^2 \quad A_{\text{plate}} = 10.767 \cdot \text{in}^2 \quad \text{Area of Bracket Plates}$$

(Ref. 33.6, Item 2)

$$W_{\text{sab}} := A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{sab}} = 1.146 \cdot \text{lb} \cdot \text{ft} \quad \text{Weight}$$

Retrieval Cable Bracket

(Ref. 33H.6, Item 6)

$$V_{\text{cb}} := \left[2 \cdot (0.47) \cdot \left[\left[\frac{\pi (1.25^2)}{2} + (1.25 \cdot 3) \right] - \frac{\pi (0.75^2)}{4} \right] + (0.25 \cdot 1.5 \cdot 3) + 2 \cdot (2 \cdot 3 \cdot 0.54) \right] \cdot \text{in}^3$$

$$V_{\text{cb}} = 13.022 \cdot \text{in}^3 \quad \text{Metal Volume of Bracket}$$

$$w_{240} := 0.284 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^3} \quad \text{Weight of ASTM A-240 Tp 304 SST}$$

$$W_{\text{cb}} := V_{\text{cb}} \cdot w_{240} \quad W_{\text{cb}} = 3.698 \cdot \text{lb} \cdot \text{ft} \quad \text{Weight}$$

Retrieval Cable

$$w_{\text{rope}} := 24.3 \cdot \frac{\text{lb} \cdot \text{ft}}{100 \cdot \text{ft}} \quad \text{Weight per 100 ft of 3/8", 7 x 19 in Strand Core Wire Rope} \quad \text{Ref. 33.4, Item 20}$$

$$L_{\text{rope}} := (638.33 - 608) \cdot \text{ft} \quad L_{\text{rope}} = 30.33 \cdot \text{ft} \quad \text{Total Length of Wire Rope}$$

$$W_{\text{rope}} := w_{\text{rope}} \cdot L_{\text{rope}} \quad W_{\text{rope}} = 7.37 \cdot \text{lb} \cdot \text{ft} \quad \text{Weight of 75 ft of Wire Rope} \quad \text{Ref. 22}$$

Sheave Assembly Base Plate

$$A_{\text{plate}} := ((20.5) \cdot (3.5)) \cdot \text{in}^2 \quad A_{\text{plate}} = 71.75 \cdot \text{in}^2 \quad \text{Area of Plate} \quad \text{(Ref. 33.6, Item 2)}$$

$$w_{\text{psf}} := 20.42 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{ft}^2} \quad \text{Weight of 1/2 in. Hot Rolled Carbon Steel plate per square foot:} \quad \text{(Ref. 16)}$$

$$W_{\text{base}} := A_{\text{plate}} \cdot w_{\text{psf}} \quad W_{\text{base}} = 10.175 \cdot \text{lb} \cdot \text{ft} \quad \text{Weight}$$

DESIGN ANALYSIS

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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Weight of Axles

$w_{N60} := \left(6.01 \frac{\text{lb}}{\text{ft}} \right)$	Weight per ft of 1-1/2 Inch Diameter Nitronic 60 Round Stock	(Ref. 16)
$L_{axle} := 2 \cdot (5.0 \text{ in})$	Total Length of 2 Axles	(Ref. 33.6, Item 2)
$W_{axle} := L_{axle} \cdot w_{N60}$	$W_{axle} = 5.008 \cdot \text{lb}$	Weight

Total Weight of Sheave Assembly

$W_{SA} := W_{sheavs} + W_{tube} + W_{brackets} + W_{ends} + W_{housing} + W_{brace} +$ $+ W_{braces} + W_{sab} + W_{cb} + W_{rope} + W_{base} + W_{axle}$	$W_{SA} = 188.578 \cdot \text{lb}$
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DESIGN ANALYSIS

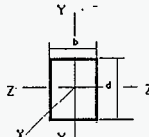
Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-04.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9 / 8 / 95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9 / 8 / 95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Weld Strength of the welds attaching the 3" square tube to the Sheave Assembly Baseplate.

$F_x := \frac{W_{PA} + W_{tube} + W_{brackets} + W_{ends}}{2}$ Weight of Sheave Assembly components below baseplate. Ref. 33.6, Sh. 2.

$M_{arm} := \left(\frac{2.50}{2} - 0.72 \right) \cdot in$ Moment arm (Moment about Z imposed by eccentric bracket loading).

WELD PROPERTIES:

	$b := 3 \cdot in$	Weld Outline Dimension - short side	Ref. 9, Pg. 7.4 6 & 7
	$d := 3 \cdot in$	Weld Outline Dimension - long side	
	$C_y := \frac{b}{2}$	$C_y = 1.5 \cdot in$	Location of center of gravity with respect to y axis.
	$C_z := \frac{d}{2}$	$C_z = 1.5 \cdot in$	Location of center of gravity with respect to z axis.
	$A_w := 2 \cdot (b + d)$	$A_w = 12 \cdot in$	Weld length (1 Tube)
	$S_{wy} := b \cdot d + \frac{b^2}{3}$	$S_{wy} = 12 \cdot in^2$	Section modulus about y axis
	$S_{wz} := b \cdot d + \frac{d^2}{3}$	$S_{wz} = 12 \cdot in^2$	Section modulus about z axis
	$J_w := \frac{(b + d)^3}{6}$	$J_w = 36 \cdot in^3$	Polar Moment of Inertia

JOINT LOADS:

Tensile	$F_x = 2.04 \cdot 10^3 \cdot lbf$	$F_y := 0 \cdot lbf$	$F_z := 0 \cdot lbf$
Moment	$M_x := 0 \cdot in \cdot lbf$	$M_y := 0 \cdot in \cdot lbf$	$M_z := (F_x \cdot M_{arm})$

WELD STRESS:

$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5}$ $f_w = 260.153 \cdot \frac{lbf}{in}$

FILLET WELD SIZE REQUIRED:

$S := 15800 \cdot psi$	Allowable Weld Stress per AWS & AISC	Ref. 9, Pg. 7.4-8
$w := \frac{f_w}{0.707 \cdot S}$	$w = 0.023 \cdot in$	Minimum Allowable Fillet Weld Required by Design
$w_{Code} := \left(\frac{3}{16} \right) \cdot in$	Minimum Size Fillet Weld Required by Code for material thickness of 1/2".	Ref. 7, Part 5, Section J.2.2, Table J2.4
$SF_{f_w} := \frac{w_{Code}}{w}$	$SF_{f_w} = 8.051$	Safety factor based on a 3/16" weld > 3 <u>OK</u>

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY**
Subject **WINCH ANALYSIS - SLURRY/SUICE PUMP**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
Location **C TANK FARM - HANFORD 200 EAST AREA**

WO/Job No. **ER4319**
Date **9/8/95**
Checked **9/8/95**
Revised

Filename **WIN-04.mcd**
By **D.L. STONE**
By **M.R. CUSTER** *MRC*
By

Stress on sheave axles

Ref. 33.6, Pg 2, Item 44

Shear stress

$D := 1.5\text{-in}$

Axle Diameter

$A_a := \frac{\pi D^2}{4}$

$A_a = 1.767 \cdot \text{in}^2$

Cross Sectional Area

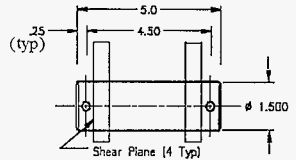


Figure 3

$F_{axle} := \left[\frac{(W_{SA} - W_{axle}) + W_{PA}}{8} \right]$

$F_{axle} = 529.212 \cdot \text{lbF}$

Force of sheave assembly felt by each shear plane.

$S_{actS} := \frac{F_{axle}}{A_a}$

$S_{actS} = 299.473 \frac{\text{lbF}}{\text{in}^2}$

Calculated Shear Stress at each shear plane.

$S_y := 44400 \frac{\text{lbF}}{\text{in}^2}$

Yield Strength of ASTM A-276, *ARMCO Nitronic 60* Stainless Steel
(1.5" not listed. Stress used is conservative for 1")

Ref. 25, Table 36

$S_{allS} := 0.4 \cdot S_y$

$S_{allS} = 1.776 \cdot 10^4 \frac{\text{lbF}}{\text{in}^2}$

Allowable Shear Stress

Ref. 11, Sec.1.8

$SF_S := \frac{S_{allS}}{S_{actS}}$

$SF_S = 59.304$

Safety factor > 3 OK

Bending stress

$D_m := 0.18\text{-in}$

Thrust Washer Thickness
(Moment arm on axle between sheave and housing.)

Ref. 33H.6, Pg. 2, Item 5

$M := F_{axle} \cdot D_m$

Moment applied to axle

$y := \frac{1.5}{2}\text{-in}$

Distance from centroidal axis to outside fiber.

$I := \frac{\pi D^4}{64}$

Axle Moment of Inertia

$S_{actB} := \frac{M \cdot y}{I}$

$S_{actB} = 287.494 \frac{\text{lbF}}{\text{in}^2}$

Calculated Maximum Bending Stress per axle.

Ref. 8, Pg. 177
(Table of Simple Stresses)

$S_{allB} := 0.75 \cdot S_y$

$S_{allB} = 3.33 \cdot 10^4 \frac{\text{lbF}}{\text{in}^2}$

Allowable Bending Stress

Ref. 7, Pg 5-48
(F2-1)

$SF_B := \frac{S_{allB}}{S_{actB}}$

$SF_B = 115.828$

Safety factor > 3 OK

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY WO/Job No. ER4319
 Subject WINCH ANALYSIS - SLURRY/SLUICE PUMP Date 9/8/95
 PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked 9/8/95
 Location C TANK FARM - HANFORD 200 EAST AREA Revised

Filename WIN-04.mcd
 By D.L. STONE
 By M.R. CUSTER *MRC*
 By

Shear Stress of Sheave Housing at Axles

Calculation considers two plates in double shear at each axle.

D := 5-in Depth of Housing Plate section Ref. 33H.6, Pg. 2, Item 2

Th := $\frac{3}{8}$ -in Thickness of Sheave Housing Plate

A_a := D · Th · 4 A_a = 7.5 · in² Area of metal supported by axles

F_{SH} := W_{PA} + W_{tube} + W_{brackets} + W_{ends} + W_{housing} + W_{brace} + W_{braces} + W_{sub} + W_{cb} + W_{base}

F_{SH} = 4.162 · 10³ · lbf Force on sheave Housing

S_y := 25000 $\frac{\text{lbf}}{\text{in}^2}$ Yield strength of ASTM A240 Type 304L

S_{all} := (0.45) · S_y S_{all} = 1.125 · 10⁴ $\frac{\text{lbf}}{\text{in}^2}$ Allowable Stress Ref.7, Chap. D.3.1

S_{act} := $\frac{F_{SH}}{A_a}$ S_{act} = 554.911 $\frac{\text{lbf}}{\text{in}^2}$ Calculated Shear stress on Housing.

SF := $\frac{S_{all}}{S_{act}}$ SF = 20.274 Safety factor > 3 OK

DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY WO/Job No. ER4319 Filename WIN-04.mcd
Subject WINCH ANALYSIS - SLURRY/SLUIICE PUMP Date 9/8/95 By D.L. STONE
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked 9/8/95 By M.R. CUSTER MRC
Location C TANK FARM - HANFORD 200 EAST AREA Revised By

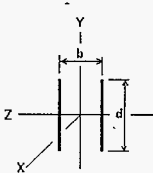
Strength of Weld between housing walls and 1/2" base plate.

$$F_x = W_{PA} + W_{tube} + W_{brackets} + W_{ends} + W_{base}$$

$$F_x = 4.091 \cdot 10^3 \cdot \text{lb}f \quad \text{Weld Tensile Load (In "x" direction on subroutine sketch, below)}$$

WELD PROPERTIES:

b = 3.5-in Weld Outline Dimension - short side Ref. 9, Pg. 7.4 6 &7
d = 20.5-in Weld Outline Dimension - long side



$C_y := \frac{b}{2}$ $C_y = 1.75 \cdot \text{in}$ Location of center of gravity with respect to y axis.

$C_z := \frac{d}{2}$ $C_z = 10.25 \cdot \text{in}$ Location of center of gravity with respect to z axis.

$A_w := 2 \cdot d$ $A_w = 41 \cdot \text{in}$ Weld length

$S_{wy} := b \cdot d$ $S_{wy} = 71.75 \cdot \text{in}^2$ Section modulus about y axis

$S_{wz} := \frac{d^2}{3}$ $S_{wz} = 140.083 \cdot \text{in}^2$ Section modulus about z axis

$J_w := \frac{d \cdot (3 \cdot b^2 + d^2)}{6}$ $J_w = 1.561 \cdot 10^3 \cdot \text{in}^3$ Polar Moment of Inertia

JOINT LOADS:

Tensile $F_x = 4.091 \cdot 10^3 \cdot \text{lb}f$ $F_y := 0 \cdot \text{lb}f$ $F_z := 0 \cdot \text{lb}f$
Moment $M_x := 0 \cdot \text{in} \cdot \text{lb}f$ $M_y := 0 \cdot \text{in} \cdot \text{lb}f$ $M_z := 0 \cdot \text{in} \cdot \text{lb}f$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5} \quad f_w = 99.781 \cdot \frac{\text{lb}f}{\text{in}}$$

FILLET WELD SIZE REQUIRED:

S := 15800-psi Allowable Weld Stress per AWS & AISC Ref. 9, Pg. 7.4-8

$w := \frac{f_w}{0.707 \cdot S}$ $w = 0.009 \cdot \text{in}$ Minimum Allowable Fillet Weld Required by Design

$w_{Code} := \left(\frac{3}{16} \right) \cdot \text{in}$ Minimum Size Fillet Weld Required by Code for material thickness of 1/2". Ref. 7, Part 5, Section J.2.2, Table J2.4

$SF_{f_w} := \frac{w_{Code}}{w}$ $SF_{f_w} = 20.991$ Safety factor based on a 3/16" weld > 3 **OK**

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-05.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER MRC
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check of Retrieval Cable and Brackets

Check of Retrieval Cable

$W_{PA} := 3375 \cdot \text{lb}$ Total Weight of Pump Assembly, impact loading not considered.

$W_{SA} := 188.578 \cdot \text{lb}$ Total Weight of Sheave Assembly

$W_{PA} + W_{SA} = 3.564 \cdot 10^3 \cdot \text{lb}$

$S_{\text{cable}} := 22800 \cdot \text{lb}$ Breaking Strength of Type 304 Stainless Steel, 1/2", 6 x 19 IWRC Cable

$SF := \frac{S_{\text{cable}}}{(W_{PA} + W_{SA})}$ $SF = 6.398$ Safety factor > 5 OK

Cable Support Brackets welded to the Winch Plate

Ref. 33.7, Sh. 2, Detail 3

Shear

$$w := \left(1.25 - \frac{0.75}{2} \right) \cdot \text{in}$$

$$w = 0.875 \cdot \text{in}$$

Width of Shear plane.

$$th := (.4687) \cdot \text{in}$$

$$th = 0.469 \cdot \text{in}$$

Plate thickness

$$A := 2 \cdot (w \cdot th)$$

$$A = 0.82 \cdot \text{in}^2$$

Area of bracket loaded by the clevis pin.

$$S_y := 30000 \cdot \frac{\text{lb}}{\text{in}^2}$$

Yield strength of ASTM A240 Type 304

$$S_{\text{all}} := (0.45) \cdot S_y$$

$$S_{\text{all}} = 1.35 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$$

Allowable Stress

Ref.7, Part 5, Chap. D.3.1

$$S_{\text{act}} := \frac{(W_{PA} + W_{SA})}{A}$$

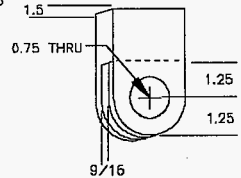
$$S_{\text{act}} = 4.345 \cdot 10^3 \cdot \frac{\text{lb}}{\text{in}^2}$$

Calculated stress on brackets.

$$SF := \frac{S_{\text{all}}}{S_{\text{act}}}$$

$$SF = 3.107$$

Safety factor > 3 OK



DESIGN ANALYSIS

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9 / 8 / 95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Cable Support Brackets (continued...)

Bearing Stress

$d := 0.75 \cdot \text{in}$

Hole Diameter

$A := \left(\frac{\pi \cdot d}{2} \cdot \text{th} \right)$

$A = 0.552 \cdot \text{in}^2$

Bearing Surface Area.

$S_y := 30000 \frac{\text{lb}}{\text{in}^2}$

Yield strength of ASTM A240 Type 304

$S_{\text{all}} := (0.9) \cdot S_y$

$S_{\text{all}} = 2.7 \cdot 10^4 \frac{\text{lb}}{\text{in}^2}$

Allowable Stress

Ref.11, Pg 12

$S_{\text{act}} := \frac{W_{PA} + W_{SA}}{A}$

$S_{\text{act}} = 6.454 \cdot 10^3 \frac{\text{lb}}{\text{in}^2}$

Calculated stress on brackets.

$SF := \frac{S_{\text{all}}}{S_{\text{act}}}$

$SF = 4.184$

Safety factor > 3 OK

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HANFORD COMPANY

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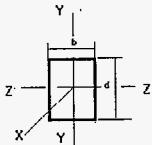
DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-05.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9/8/95	By	D.L. STONE
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Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Weld connecting Support Bracket to the Winch Plate.

Ref. 33.7, Sh. 2, Detail 3

WELD PROPERTIES:



$b := 1.5 \text{ in}$	Weld Outline Dimension - short side	Ref. 9, Pg. 7.4 6 & 7
$d := 3 \text{ in}$	Weld Outline Dimension - long side	
$Cy := \frac{b}{2}$	$Cy = 0.75 \text{ in}$	Location of center of gravity with respect to y axis.
$Cz := \frac{d}{2}$	$Cz = 1.5 \text{ in}$	Location of center of gravity with respect to z axis.
$Aw := 2 \cdot (b + d)$	$Aw = 9 \text{ in}$	Weld length (1 Bracket)
$Swy := b \cdot d + \frac{b^2}{3}$	$Swy = 5.25 \cdot \text{in}^2$	Section modulus about y axis
$Swz := b \cdot d + \frac{d^2}{3}$	$Swz = 7.5 \cdot \text{in}^2$	Section modulus about z axis
$Jw := \frac{(b + d)^3}{6}$	$Jw = 15.187 \cdot \text{in}^3$	Polar Moment of Inertia

JOINT LOADS:

Tensile	$Fx := W_{PA} + W_{SA}$	$Fy := 0 \text{ lbf}$	$Fz := 0 \text{ lbf}$
Moment	$Mx := 0 \text{ in-lbf}$	$My := 0 \text{ in-lbf}$	$Mz := 0 \text{ in-lbf}$

WELD STRESS:

$$f_w := \left[\left(\frac{Fx}{Aw} + \frac{My}{Swy} + \frac{Mz}{Swz} \right)^2 + \left(\frac{Fy}{Aw} + \frac{Mx \cdot Cy}{Jw} \right)^2 + \left(\frac{Fz}{Aw} + \frac{Mx \cdot Cz}{Jw} \right)^2 \right]^{0.5} \quad f_w = 395.953 \frac{\text{lbf}}{\text{in}}$$

FILLET WELD SIZE REQUIRED:

$S := 15800 \text{ psi}$	Allowable Weld Stress per AWS & AISI	Ref. 9, Pg. 7.4-8
$w := \frac{f_w}{0.707 \cdot S}$	$w = 0.035 \text{ in}$	Minimum Allowable Fillet Weld Required by Design
$w_{Code} := \left(\frac{5}{16} \text{ in} \right)$	Minimum Size Fillet Weld Required by Code for material thickness of over 3/4".	Ref. 7, Part 5, Section J.2.2, Table J2.4
$SF_{f_w} := \frac{w_{Code}}{w}$	$SF_{f_w} = 8.816$	Safety factor based on a 5/16" weld > 3 <u>OK</u>

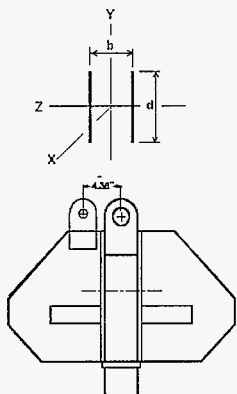
DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-05.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Weld connecting Eye Tabs to Sheave Housing

Ref. 33.6, Sh. 2, Detail 3

WELD PROPERTIES:



$b := 3 \cdot \text{in}$	Weld Outline Dimension - short side	Ref. 9, Pg. 7.4 6 & 7
$d := 2 \cdot \text{in}$	Weld Outline Dimension - long side	
$Cy := \frac{b}{2}$	$Cy = 1.5 \cdot \text{in}$	Location of center of gravity with respect to y axis.
$Cz := \frac{d}{2}$	$Cz = 1 \cdot \text{in}$	Location of center of gravity with respect to z axis.
$Aw := 2 \cdot d$	$Aw = 4 \cdot \text{in}$	Weld length
$Swy := b \cdot d$	$Swy = 6 \cdot \text{in}^2$	Section modulus about y axis
$Swz := \frac{d^2}{3}$	$Swz = 1.333 \cdot \text{in}^2$	Section modulus about z axis
$Jw := \frac{d \cdot (3 \cdot b^2 + d^2)}{6}$	$Jw = 10.333 \cdot \text{in}^3$	Polar Moment of Inertia

JOINT LOADS:

Tensile	$Fx := 0 \cdot \text{lbf}$	$Fy := \frac{W_{PA} + W_{SA}}{2}$	$Fz := 0 \cdot \text{lbf}$
Moment	$Mx := 0 \cdot \text{in} \cdot \text{lbf}$	$My := 0 \cdot \text{in} \cdot \text{lbf}$	$Mz := 0 \cdot \text{in} \cdot \text{lbf}$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{0.5} \quad f_w = 445.447 \cdot \frac{\text{lbf}}{\text{in}}$$

FILLET WELD SIZE REQUIRED:

$S := 15800 \cdot \text{psi}$	Allowable Weld Stress per AWS & AISC	Ref. 9, Pg. 7.4-8
$w := \frac{f_w}{0.707 \cdot S} \quad w = 0.04 \cdot \text{in}$	Minimum Allowable Fillet Weld Required by Design	
$w_{\text{Code}} := \left(\frac{3}{16} \cdot \text{in} \right)$	Minimum Size Fillet Weld Required by Code for material thickness of 3/8".	Ref. 7, Part 5, Section J.2.2, Table J2.4
$w_{\text{act}} := \left(\frac{1}{4} \cdot \text{in} \right)$	Weld size specified in design drawings.	Ref. H.6, Pg. 2
$SF_{f_w} := \frac{w_{\text{act}}}{w} \quad SF_{f_w} = 6.269$	Safety factor based on a 3/16" weld	> 3 OK

DESIGN ANALYSIS

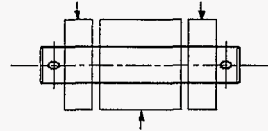
Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-05.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD	200 EAST AREA	Revised		By

Shear on Eye Bolt

Ref. 33.6, Pg 2

$D := \left(\frac{45}{64}\right) \cdot \text{in}$ $D = 0.703 \cdot \text{in}$

Axle Diameter



$A_a := \frac{\pi D^2}{4}$ $A_a = 0.388 \cdot \text{in}^2$

Cross Sectional Area

$F_{\text{axle}} := \left(\frac{W_{SA} + W_{PA}}{2} \right)$

$F_{\text{axle}} = 1.782 \cdot 10^3 \cdot \text{lbF}$ Force of sheave assembly felt by each shear plane.

$S_{\text{actS}} := \frac{F_{\text{axle}}}{A_a}$

$S_{\text{actS}} = 4.589 \cdot 10^3 \cdot \frac{\text{lbF}}{\text{in}^2}$ Calculated Shear Stress per axle.

$S_y := 100000 \cdot \frac{\text{lbF}}{\text{in}^2}$

Yield Strength of ASTM 193, Class 2, B8T

$S_{\text{allS}} = 0.4 \cdot S_y$

$S_{\text{allS}} = 4 \cdot 10^4 \cdot \frac{\text{lbF}}{\text{in}^2}$

Allowable Shear Stress

Ref. 11, Sec.1.8

$SF_S := \frac{S_{\text{allS}}}{S_{\text{actS}}}$

$SF_S = 8.717$

Safety factor > 3 OK

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-06.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUCE PUMP	Date	9 / 8 / 95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Weight of Reel Axle (12) Supported Components

$W_{PA} := 4050 \cdot \text{lbf}$ Weight of Pump Assembly (Impact Considered)

$W_{SA} := 188.578 \cdot \text{lbf}$ Weight of Sheave Assembly

Weight of Reel

(Ref. 33.5, Sh. 2&4)

Weight of 2 Flanges

$w_{psf} := 15.32 \cdot \frac{\text{lbf}}{\text{ft}^2}$ Weight of 3/8 in. Hot Rolled Carbon Steel plate per square foot: (Ref. 16)

$A_{plate} := 2 \cdot \left[\frac{\pi}{4} (28^2 - 3.25^2) \right] \cdot \text{in}^2$ $A_{plate} = 1.215 \cdot 10^3 \cdot \text{in}^2$ Area of 2 Flanges

$W_{flanges} := A_{plate} \cdot w_{psf}$ $W_{flanges} = 129.253 \cdot \text{lbf}$ Flange Weight

Weight of Inner (Axle) Hub

$w_{r3} := 24.03 \cdot \frac{\text{lbf}}{\text{ft}}$ Weight per ft of 3 Inch Diameter 304L Round Stock Ref.16

$w_{r1.938} := 10.02 \cdot \frac{\text{lbf}}{\text{ft}}$ Weight per ft of 1-15/16 Inch Diameter 304L Round Stock Ref. 16

$L_{hub} := (3.75 \cdot \text{in})$ Total Length of Hub

$W_{hubA} := L_{hub} (w_{r3} - w_{r1.938})$ $W_{hubA} = 4.378 \cdot \text{lbf}$ Inner Hub Weight

Weight of Outer (Reel) Hub

(Ref. 33.5, Sh. 6)

$W_{steel} := 490.8 \cdot \frac{\text{lbf}}{\text{ft}^3}$ (Ref. 7, Page 6-8)

$r_{solid} := \left(\frac{11.75}{2} \right) \cdot \text{in}$ $r_{solid} = 5.875 \cdot \text{in}$

$A_{solid} := \pi \cdot r_{solid}^2$ $r_{cutout} := 3.62 \cdot \text{in}$

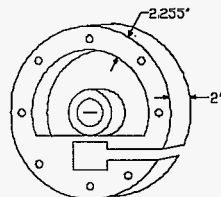
$A_{cutout} := \left(\frac{\pi \cdot r_{cutout}^2}{2} \right) + 2 \cdot \left[(1.5 \cdot \text{in}) \cdot r_{cutout} \right] + \left[(1.5 \cdot 2.25) \cdot \text{in}^2 \right] + \left[\left(\frac{3}{8} \cdot 5 \right) \cdot \text{in}^2 \right]$

$A_{cutout} = 36.694 \cdot \text{in}^2$ Approximation of Semicircular Cutout Area.

$w := 2 \cdot \text{in}$ Hub Length

$V_{hub} := (w) \cdot (A_{solid} - A_{cutout})$ $V_{hub} = 143.479 \cdot \text{in}^3$ Metal Volume of Hub

$W_{hubB} := (V_{hub} \cdot W_{steel})$ $W_{hubB} = 40.752 \cdot \text{lbf}$ Approximate weight



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Calc. No. **W-320-27-019**
 Revision No.: **0**
 Page No. **27** of **7**

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-06.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

TOTAL Weight of Reel

$$W_{reel} := W_{flanges} + W_{hubA} + W_{hubB} \quad W_{reel} = 174.383 \cdot \text{lb}$$

Weight of Gear

$$W_{gear25} := 105 \cdot \text{lb} \quad \text{Weight of 54 tooth 18 PD, 3 DP, 3" Spur Gear} \quad \text{Ref. 20}$$

obtained by telecon from Boston Gear, 8/15/95.

Weight of Wire Rope

$$w_{rope} := 0.69 \frac{\text{lb}}{\text{ft}} \quad \text{Weight per ft of 1/4 x 1-1/2 in Flat Wire Rope} \quad \text{Ref. 8, Pg. 332, Tbl 7}$$

$$L_{rope} := 75 \cdot \text{ft} \quad \text{Total Length of Wire Rope} \quad \text{Ref. 33.1}$$

$$W_{rope} := w_{rope} \cdot L_{rope} \quad W_{rope} = 51.75 \cdot \text{lb} \quad \text{Weight of 75 ft of Wire Rope}$$

Weight of Reel / Spur Gear Shaft (Axle)

$$w_2 := 10.68 \frac{\text{lb}}{\text{ft}} \quad \text{Weight per ft of 2 Inch Diameter 4140 Round Stock} \quad \text{Ref. 16}$$

$$w_{1.938} := 10.02 \frac{\text{lb}}{\text{ft}} \quad \text{Weight per ft of 1-15/16 Inch Diameter 4140 Round Stock} \quad \text{Ref. 16}$$

$$w_{1.25} := 4.17 \frac{\text{lb}}{\text{ft}} \quad \text{Weight per ft of 1-1/4 Inch Diameter 4140 Round Stock} \quad \text{Ref. 16}$$

$$W_{shaft} := [(10.55 \cdot \text{in}) \cdot w_{1.938}] + [(5.44 \cdot \text{in}) \cdot w_2] + [(5.5 \cdot \text{in}) \cdot w_{1.25}] \quad W_{shaft} = 15.562 \cdot \text{lb} \quad \text{Axle Weight}$$

Weight of Pump, Sheave and Rope Assemblies on Reel Hub

$$W_{Hub} := W_{PA} + W_{SA} + W_{rope} \quad W_{Hub} = 4.29 \cdot 10^3 \cdot \text{lb}$$

Weight of Loaded Reel on Axle (12)

$$W_{Reel} := W_{PA} + W_{SA} + W_{reel} + W_{rope} \quad W_{Reel} = 4.465 \cdot 10^3 \cdot \text{lb}$$

Total Weight of Reel / Spur Gear Assembly on Bearings

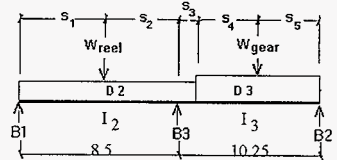
$$W_{RA} := W_{PA} + W_{SA} + W_{reel} + W_{gear25} + W_{rope} + W_{shaft} \quad W_{RA} = 4.585 \cdot 10^3 \cdot \text{lb}$$

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-07.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP** Date **9/8/95** By **D.L. STONE**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked **9/8/95** By **M.R. CUSTER MRC**
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

Shear on Reel Axle (12) Bearings

$D2 := \left(10.032 \cdot \frac{\text{lbft}}{\text{ft}}\right)$ $D3 := \left(10.69 \cdot \frac{\text{lbft}}{\text{ft}}\right)$ Weight per foot of Shaft
 Ref. C, Page 1-108



- $s_1 := 3.89 \text{ in}$ Span 1 = Axle Length between Bearing 1 and Reel. Ref. 33.5, Pages 1 & 5
 $s_2 := 4.61 \text{ in}$ Span 2 = Axle Length between Reel and Center Bearing (Bearing 3).
 $s_3 := 3.435 \text{ in}$ Span 3 = Axle Length between Bearing 3 and Diameter change.
 $s_4 := 1.625 \text{ in}$ Span 4 = Axle Length between Diameter Change and Gear 25.
 $s_5 := 5.19 \text{ in}$ Span 5 = Axle Length between Gear 25 and Bearing 2.

$d_2 := 1.9375 \text{ in}$ $I_2 := \frac{\pi d_2^4}{64}$ $I_2 = 0.6917 \cdot \text{in}^4$ Moment of Inertia for shaft diameter = 1.9375 in
 Ref. 8, Page 191

$d_3 := 2 \text{ in}$ $I_3 := \frac{\pi d_3^4}{64}$ $I_3 = 0.7854 \cdot \text{in}^4$ Moment of Inertia for shaft diameter = 2 in

$K_1 := \frac{I_2}{(s_1 + s_2)} \cdot \left(\frac{3}{4}\right)$ $K_1 = 0.061 \cdot \text{in}^3$ Stiffness Factor adjusted for simple supports. Ref. 13, Page 530

$K_2 := \frac{I_3}{(s_3 + s_4 + s_5)} \cdot \left(\frac{3}{4}\right)$ $K_2 = 0.057 \cdot \text{in}^3$ Stiffness Factor adjusted for simple supports.

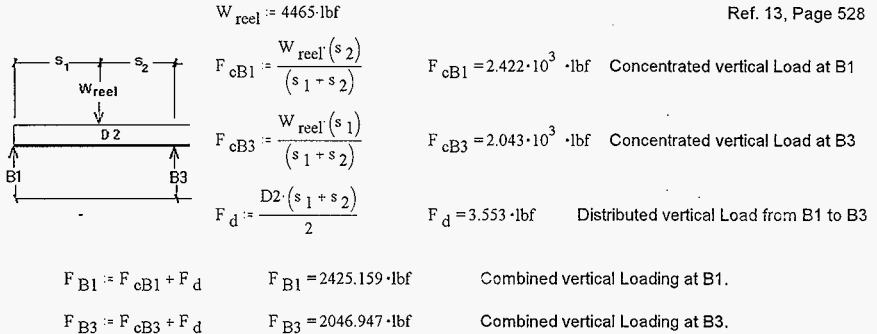
$DF_{K1} := \frac{K_1}{K_1 + K_2}$ $DF_{K1} = 0.515$ Distribution Factor Ref. 13, Page 528

$DF_{K2} := \frac{K_2}{K_1 + K_2}$ $DF_{K2} = 0.485$ Distribution Factor

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-07.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>M.R.C.</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Loading on Axle from Bearings B1 to B3



Fixed End Moments (Span B1 - B3)

Ref. 13, Page 528
Ref. 33.5, Pages 1 & 5

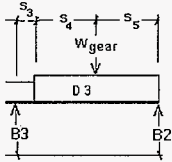
$P := W_{reel}$ Concentrated Load
 $l := s_1 + s_2$ Total Spacing
 $w := D2$ Distributed Load (Conservatively assumed as spanning full length of beam.)
 $FEM_{cB1} := \frac{P \cdot s_1 \cdot s_2^2}{l^2}$ $FEM_{cB1} = 5.109 \cdot 10^3 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$ Left Fixed End Moment resulting from Concentrated Load
 $FEM_{cB3} := \frac{P \cdot s_1^2 \cdot s_2}{l^2}$ $FEM_{cB3} = 4.311 \cdot 10^3 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$ Right Fixed End Moment resulting from Concentrated Load
 $M_d := \frac{w \cdot l^2}{12}$ $M_d = 5.033 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$ Fixed End Moment resulting from Uniformly Distributed Load
 $FEM_L := FEM_{cB1} + M_d$ $FEM_L = 5114.024 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$
 $FEM_R := FEM_{cB3} + M_d$ $FEM_R = 4316.09 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-07.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SUICE PUMP** Date **9/8/95** By **D.L. STONE**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked **9/8/95** By **M.R. CUSTER**
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

Loading on Axle from Bearings B3 to B2

Ref. 13, Page 528



$$W_{gear} := 105 \cdot \text{lb} \cdot \text{f}$$

$$F_{cB3} := \frac{W_{gear} \cdot (s_5)}{(s_3 + s_4 + s_5)} \quad F_{cB3} = 53.166 \cdot \text{lb} \cdot \text{f} \quad \text{Concentrated vertical Load felt at B3}$$

$$F_{cB2} := \frac{W_{gear} \cdot (s_3 + s_4)}{(s_3 + s_4 + s_5)} \quad F_{cB2} = 51.834 \cdot \text{lb} \cdot \text{f} \quad \text{Concentrated vertical Load felt at B2}$$

$$F_d := \frac{(D2 \cdot s_3) + [D3 \cdot (s_4 + s_5)]}{2}$$

$$F_d = 4.471 \cdot \text{lb} \cdot \text{f} \quad \text{Distributed vertical Load from B1 and B2}$$

$$F_{B2} := F_{cB2} + F_d \quad F_{B2} = 56.305 \cdot \text{lb} \cdot \text{f} \quad \text{Combined vertical Loading at B2.}$$

$$F_{B3} := F_{cB3} + F_d \quad F_{B3} = 57.637 \cdot \text{lb} \cdot \text{f} \quad \text{Combined vertical Loading at B3.}$$

Fixed End Moments (Span B3 - B2)

Ref. 13, Page 528

$P := W_{gear}$ Concentrated Load

$l := (s_3 + s_4 + s_5)$ Total Spacing

$w := D3$ Distributed Load (Conservatively assumed as spanning full length of beam.)

$$FEM_{cB3} := \frac{P \cdot (s_3 + s_4) \cdot (s_5)^2}{l^2} \quad FEM_{cB3} = 136.216 \cdot \text{lb} \cdot \text{f} \cdot \text{in} \quad \text{Left Fixed End Moment resulting from Concentrated Load}$$

$$FEM_{cB2} := \frac{P \cdot (s_3 + s_4)^2 \cdot (s_5)}{l^2} \quad FEM_{cB2} = 132.804 \cdot \text{lb} \cdot \text{f} \cdot \text{in} \quad \text{Right Fixed End Moment resulting from Concentrated Load}$$

$$M_d := \frac{w \cdot l^2}{12} \quad M_d = 7.799 \cdot \text{lb} \cdot \text{f} \cdot \text{in} \quad \text{Moment resulting from Distributed Load}$$

$$FEM_R := FEM_{cB2} + M_d \quad FEM_R = 140.603 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$$

$$FEM_L := FEM_{cB3} + M_d \quad FEM_L = 144.015 \cdot \text{lb} \cdot \text{f} \cdot \text{in}$$

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Revision No. 0

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DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD CO.
 Subject WINCH ANALYSIS - SLURRY /SLUICE PUMP
 PROJECT W-320 WASTE RETRIEVAL for Tank 241-C-106
 Location C TANK FARM - HANFORD 200 EAST AREA

WO/Job No. ER4319
 Date 9/8/95
 Checked 9/8/95
 Revised

Filename WIN-8.XLS
 By D. L. STONE
 By M. R. CUSTER *MRC*
 By

REEL SHAFT FORCES AND MOMENTS

Determined by Moment Distribution Method for Beams

	0.515	0.485		
Fixed End Moments	-5114.02	4316.09	-144.02	140.60
Reaction Due to Unbalanced Moment	5114.02	2557.01	-70.30	-140.60
Distributed Reactions	0.00	-3429.27	-3229.51	0.00
Sum		3443.83	-3443.83	
Simple Support Reactions of:				
Vertical Load, P/L + wL/2 (AISC)	2425.16	2046.95	57.64	56.31
Distributed Moments, M/L	-405.16	405.16	335.98	-335.98
Sum	2020.00	2452.10	393.62	-279.68
Total Vertical Reactions at Supports	2020.00 lb	2845.72 lbs		-279.68 lbs

Client **WESTINGHOUSE HANFORD COMPANY**
Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP**
PROJECT **W-320 WASTE RETRIEVAL for TANK 241-C-106**
Location **C TANK FARM - HANFORD 200 EAST AREA**

WO/Job No. **ER4319**
Date **9/8/95**
Checked **9/8/95**
Revised

Filename **WIN-07.mcd**
By **D.L. STONE**
By **M.R. CUSTER *MRC***
By

Reel Shaft Forces

Determined by Moment Distribution Method for Indeterminant Beams

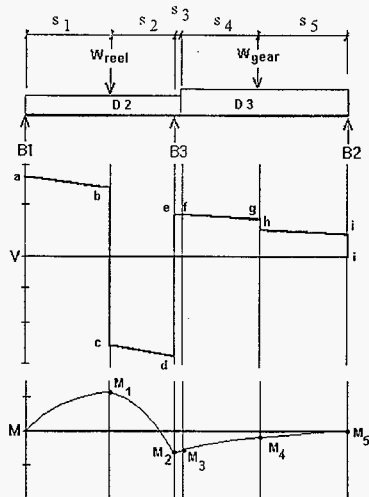
B1 := 2020.00·lbf B3 := 2845.72·lbf B2 := -279.68·lbf

Bending Moments for Reel Axle (12)

$D2 = 10.032 \cdot \frac{\text{lbf}}{\text{ft}}$ $D3 = 10.69 \cdot \frac{\text{lbf}}{\text{ft}}$ Weight per foot of Shaft

$W_{\text{reel}} = 4.465 \cdot 10^3 \cdot \text{lbf}$ $W_{\text{gear}} = 105 \cdot \text{lbf}$

a := B1 a = 2.02·10³ ·lbf
b := a - (s₁)·D2 b = 2.017·10³ ·lbf
c := b - W_{reel} c = -2.448·10³ ·lbf
d := c + (s₂)·D2 d = -2.452·10³ ·lbf
e := d + B3 e = 393.614 ·lbf
f := e - (s₃)·D2 f = 390.742 ·lbf
g := f - (s₄)·D3 g = 389.295 ·lbf
h := g - W_{gear} h = 284.295 ·lbf
i := h - (s₅)·D3 i = 279.671 ·lbf
j := i + B2 j = 0 ·lbf



Maximum Bending Moment on the Shaft

$$M_1 := b \cdot (s_1) + \frac{(a - b) \cdot (s_1)^2}{2}$$

$M_1 = 7.851 \cdot 10^3 \cdot \text{lbf} \cdot \text{in}$ ← Maximum Bending Moment

$$M_2 := M_1 + \left[c \cdot (s_2) + \frac{(d - c) \cdot (s_2)^2}{2} \right]$$

$M_2 = -3.444 \cdot 10^3 \cdot \text{lbf} \cdot \text{in}$

$$M_3 := M_2 + \left[f \cdot (s_3) + \frac{(e - f) \cdot (s_3)^2}{2} \right]$$

$M_3 = -2.097 \cdot 10^3 \cdot \text{lbf} \cdot \text{in}$

$$M_4 := M_3 + \left[g \cdot (s_4) + \frac{(f - g) \cdot (s_4)^2}{2} \right]$$

$M_4 = -1.463 \cdot 10^3 \cdot \text{lbf} \cdot \text{in}$

$$M_5 := M_4 + \left[i \cdot (s_5) + \frac{(h - i) \cdot (s_5)^2}{2} \right]$$

$M_5 = 0.554 \cdot \text{lbf} \cdot \text{in} \approx 0 \cdot (\text{lbf} \cdot \text{in})$

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-07.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Equivalent Radial load - Radial Ball Bearings

Shaft End (B1 and B2) Bearings

Pillow Block, RSAO (1-1/4" Shaft Diameter) Ref. 27, Pg 174
 Bearing, GN104KRRB Ref. 27, Pg 144

$C_o := 4500 \cdot \text{lbF}$ Published STATIC LOAD RATING (Maximum Permissible Radial Load) Ref. 27, Pg. 174

$i_B := 1$ Number of adjacently mounted bearings.

$R := B1$ Maximum APPLIED RADIAL LOAD on Bearings

$Thr := 0 \cdot \text{lbF}$ APPLIED THRUST LOAD on Bearing

$K_{Thr} := \frac{Thr}{i_B \cdot C_o}$ $K_{Thr} = 0$ Relative Thrust Load Factor for Ball Bearings Ref. 27, Pg. E-50, Table 1.1

$Y_1 := 0$ Thrust Load Factor for Ball Bearings (from Table 2) Ref. 27, Pg. E-51, Table 2

Formula for Single Row *WIDE INNER RING BALL BEARINGS*

Equivalent Radial Load on Bearing is the Larger result of the two following Formulas

$$P := 0.56 \cdot R + Y_1 \cdot Thr \quad P = 1.131 \cdot 10^3 \cdot \text{lbF} \quad \text{Ref. 27, Pg. E-50, Table 1.2}$$

$$P := R \quad P = 2.02 \cdot 10^3 \cdot \text{lbF}$$

$$SF := \frac{2.5 \cdot C_o}{P} = \frac{F_{all}}{F_{act}} \quad SF = 5.569$$

Given that actual bearing failure (ball fracture or disintegration) would require 2 to 3 times the Static Load Rating, a factor of 2.5 times C_o has been assumed to represent Ultimate Strength. Ref. 28

Safety > 5 **OK**

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DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-07.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Center Bearings (B3)

Flanges, RCJO (1-15/16" Shaft Diameter) Ref. 27, Pg 186
Bearing, GN115KRRB Ref. 27, Pg 144

$C_o = 8000 \cdot \text{lb}$ Published STATIC LOAD RATING (Maximum Permissible Radial Load)

$i_B = 2$ Number of adjacently mounted bearings. Ref. 27, Pg. 174

$R = B3$ Maximum APPLIED RADIAL LOAD on Bearings

$\text{Thr} = 0 \cdot \text{lb}$ APPLIED THRUST LOAD on Bearing

$K_{\text{Thr}} = \frac{\text{Thr}}{i_B C_o}$ $K_{\text{Thr}} = 0$ Relative Thrust Load Factor for Ball Bearings Ref. 27, Pg. E-50, Table 1.1

$Y_1 = 0$ Thrust Load Factor for Ball Bearings (from Table 2) Ref. 27, Pg. E-51, Table 2

Formula for Single Row *WIDE INNER RING BALL BEARINGS*

Equivalent Radial Load on Bearing is the Larger result of the two following Formulas

$$P = 0.56 \cdot R + Y_1 \cdot \text{Thr} \quad P = 1.594 \cdot 10^3 \cdot \text{lb} \quad \text{Ref. 27, Pg. E-50 Table 1.2}$$

$$P = R \quad P = 2.846 \cdot 10^3 \cdot \text{lb}$$

$$\text{SF} = \frac{i_B^{2.5} C_o}{P} = \frac{\Gamma_{\text{all}}}{F_{\text{act}}} \quad \text{SF} = 14.056 \quad \text{Safety factor} > 5 \quad \underline{\text{OK}} \quad \text{Ref. 28}$$

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Calc. No. W-320-27-019

Revision No. 0

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DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD CO.

WO/Job No. ER4319

Filename WIN-9.XLS

Subject WINCH ANALYSIS

Date 9/8/95

By D. L. STONE

PROJECT W-320 WASTE RETRIEVAL for Tank 241-106-C

Checked 9/8/95

By M. R. CUSTER M.I.C.

Location C TANK FARM- HANFORD 200 EAST AREA

Revised

By

Radial Distance from Reel Axis to Outermost Coil of Wire Rope on Reel

	Wrap Diameter d	Circumference c	Radius
Hub Diameter	11.75	36.91	5.875
Wrap 1	12.25	38.48	0.25
Wrap 2	12.75	40.06	0.25
Wrap 3	13.25	41.63	0.25
Wrap 4	13.75	43.20	0.25
Wrap 5	14.25	44.77	0.25
Wrap 6	14.75	46.34	0.25
Wrap 7	15.25	47.91	0.25
Wrap 8	15.75	49.48	0.25
Wrap 9	16.25	51.05	0.25
Wrap 10	16.75	52.62	0.25
Wrap 11	17.25	54.19	0.25
Wrap 12	17.75	55.76	0.25
Wrap 13	18.25	57.33	0.25
Wrap 14	18.75	58.90	0.25
Wrap 15	19.25	60.48	0.25
Wrap 16	19.75	62.05	0.25
Wrap 17	20.25	63.62	0.25
Wrap 18	20.75	65.19	0.25
	Sum	933.05	10.375

Length of Wire Rope 933.05 inches
 (Length of Wire Rope 77.75 feet)

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-10.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Axle 12 Center Bearing Stresses

Ref. 33.5

$B_3 = 2845.72 \text{ lbf}$	Vertical loading at Center Bearings
$M_2 = 3444 \cdot \text{lbf} \cdot \text{in}$	Moment at Center Bearings

Bearing Stress on Wall at the Screws

$t_h := \frac{1}{2} \text{ in}$	Plate Thickness	Ref. 33.5, Sh. 3
---------------------------------	-----------------	------------------

$d := \left(\frac{11}{16}\right) \cdot \text{in}$	Hole Diameter	Ref. 33.5, Sh. 3
---	---------------	------------------

$A := \left(\frac{\pi \cdot d}{2} \cdot t_h\right)$	$A = 0.54 \cdot \text{in}^2$	Bearing Surface Area.
---	------------------------------	-----------------------

$S_y := 36000 \frac{\text{lbf}}{\text{in}^2}$	Yield strength of ASTM A36
---	----------------------------

$S_{\text{all}} := (0.9) \cdot S_y$	$S_{\text{all}} = 3.24 \cdot 10^4 \cdot \frac{\text{lbf}}{\text{in}^2}$	Allowable Stress	Ref. 11, Page12
-------------------------------------	---	------------------	-----------------

$S_{\text{act}} := \frac{B_3}{4 \cdot A}$	$S_{\text{act}} = 1.318 \cdot 10^3 \cdot \frac{\text{lbf}}{\text{in}^2}$	Calculated stress on 1/2" wall plate.
---	--	---------------------------------------

$SF := \frac{S_{\text{all}}}{S_{\text{act}}}$	$SF = 24.591$	Safety factor > 3 <u>OK</u>
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DESIGN ANALYSIS

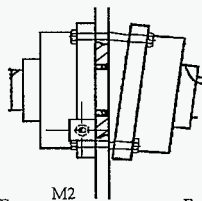
Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-10.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER M.L.C.
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check stress on 5/8 - 11 - UNC - 2B Hex Head Screws

Shear

$n_b := 4$		Number of Screws per Housing	Ref. 33.5, Sh. 3
$n_B := 1$		Number of Bearings	
$n := 11$		Number of threads per inch	
$D := \frac{5}{8} \text{ in}$		Major Diameter of Screw	
$A_T := 0.7854 \cdot \left[D - \left(\frac{0.9743}{n} \right) \cdot \text{in} \right]^2$	$A_T = 0.226 \cdot \text{in}^2$	Tensile Stress Area of Screw	Ref. 7, Pg. 4-147
$F_u := 100000 \cdot \text{psi}$		Ultimate Tensile Strength of A193 Class2 B8T Stainless Steel Fasteners	
$S_{all} := 0.17 \cdot F_u$	$S_{all} = 1.7 \cdot 10^4 \cdot \text{psi}$	Allowable shear strength of screws	Ref. 7, Pg. 4-5
$S_{act} := \frac{B_3}{n_b \cdot A_T}$	$S_{act} = 3.148 \cdot 10^3 \cdot \text{psi}$	Calculated shear stress	
$\frac{S_{act}}{S_{all}} := 5.4$		Safety factor > 5	<u>OK</u>

Tensile



$n_b := 2$		Number of Screws stressed by Moment	Ref. 33.5, Sh. 3
$d := 5.125 \text{ in}$		Center to center distance of housing bolts.	Ref. 27, Page 186
$y := \frac{d}{2}$		Housing base edge to screw C_L distance to determine maximum prying stress.	
$F_T := \frac{M_2}{n_B y}$	$F_T = 1.344 \cdot 10^3 \cdot \text{lb}$	Tension on Screws resulting from combined moment on housing	
$T_{act} := \frac{F_T}{n_b \cdot A_T}$	$T_{act} = 2.973 \cdot 10^3 \cdot \text{psi}$	Calculated Tensile stress at each screw	
$T_{all} := 0.33 \cdot F_u$		Allowable Yield Strength in Tension	Ref. 7, Pg. 4-3
$\frac{T_{all}}{T_{act}} = 11.098$		Safety factor > 5	<u>OK</u>

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-11.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

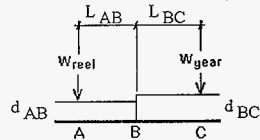
Design of Reel Axle

$W_{Hub} = 4290 \cdot \text{lb}f$	Weight on the Reel Hub
$R = 10.375 \cdot \text{in}$	Outer radius of reel plus 18 layers of 1/4" x 1-1/2" flat wire rope wrapped on the reel.
$T = W_{Hub} \cdot R = 4.451 \cdot 10^4 \cdot \text{in} \cdot \text{lb}f$	Applied Torsional Moment on winch axle
$M_b = 7851 \cdot \text{lb}f \cdot \text{in}$	Maximum Bending Moment

Determination of Torsion on a Shaft of Varying Diameters

Ref. 14, Page 74.

$L_{AB} = (4.61 + 3.435) \cdot \text{in}$	$L_{BC} = 1.625 \cdot \text{in}$	Shaft Section Lengths
$d_{AB} = 1.9375 \cdot \text{in}$	$d_{BC} = 2 \cdot \text{in}$	Shaft Diameters
$G_s = G_b$		Modulus of Rigidity
$\theta = \left(\frac{T \cdot L}{G \cdot \pi \cdot d^4} \right)$		Twist Angle



Equation 1

$$\theta_{AB} = \theta_{BC}$$

$$\left(\frac{T_{AB} \cdot L_{AB}}{G_s \cdot \pi \cdot d_{AB}^4} \right) \cdot 32 = \left(\frac{T_{BC} \cdot L_{BC}}{G_b \cdot \pi \cdot d_{BC}^4} \right) \cdot 32$$

$$\left(\frac{T_{AB} \cdot L_{AB}}{d_{AB}^4} \right) = \left(\frac{T_{BC} \cdot L_{BC}}{d_{BC}^4} \right)$$

$$\frac{T_{AB}}{T_{BC}} = x = \left(\frac{L_{BC}}{L_{AB}} \right) \cdot \left(\frac{d_{AB}^4}{d_{BC}^4} \right) \quad x = 0.178$$

$$T_{AB} = T_{BC} \cdot x$$

$$T_{AB} = T_{AB}$$

$$T_{BC} \cdot x = T_{\text{applied}} - T_{BC}$$

$$(T_{BC} \cdot x) + T_{BC} = T_{\text{applied}}$$

$$T_{BC} \cdot (x + 1) = T_{\text{applied}}$$

$$T_{BC} = \frac{T}{(x + 1)} \quad T_{BC} = 3.779 \cdot 10^4 \cdot \text{lb}f \cdot \text{in} \quad \text{Torsion on Section B}$$

$$T_{AB} = T_{BC} \cdot x \quad T_{AB} = 6.722 \cdot 10^3 \cdot \text{lb}f \cdot \text{in} \quad \text{Torsion on Section A}$$

Equation 2

$$T_{AB} + T_{BC} = T = 4.451 \cdot 10^4 \cdot \text{lb}f \cdot \text{in}$$

$$T_{AB} = T - T_{BC}$$

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Revision No.: 0
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DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-11.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Design of Reel Axle (continued.....)

$$S_y := 120000 \cdot \frac{\text{lb}}{\text{in}^2}$$

Yield Strength of ASTM A291-92, Class B Steel

$$S_{allC} := 0.30 \cdot S_y$$

$$S_{allC} = 3.6 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Shearing Stress for Combined Torsion and Bending

Ref. 8, Pg 270
(b), Below Table 2)

$$P_t := 0.75 \cdot S_{allC}$$

$$P_t = 2.7 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$$

Maximum Allowable Stress for Solid Shafts using Keyways

Ref. 8, Pg 270

$$B := 1$$

B defines hollow shafts and is assumed as 1 for solid shafts.

Ref. 8, Pg 270
(Below Table 3)

$$K_t := 1.0$$

Combined Shock and Fatigue factors for Steady, gradually applied, loading

Ref. 8, Pg 270, Table 1

$$K_m := 1.5$$

$$D_c := B \cdot \left[\frac{5.1}{P_t} \left[(K_m \cdot M_b)^2 + (K_t \cdot T_{BC})^2 \right] \right]^{\frac{1}{3}}$$

$$D_c = 1.955 \cdot \text{in}$$

Minimum Required Shaft Diameter for Combined Bending and Torsional Loading

Ref. 8, Pg. 269,(3a)

$$D_{act} := 1.9375 \cdot \text{in}$$

Design Shaft Diameter

$$D_{act} \geq D_c$$

OK

Acceptance Criteria for Minimum Shaft Diameter

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 Revision No.: 0
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DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP** Date **9/8/95**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked **9/8/95**
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised

Filename **WIN-11.mcd**
 By **D.L. STONE**
 By **M.R. CUSTER MKC**
 By

Key stress

Key, Reel and Spur Gear (typical) = 1/2" x 1/2" x 4.75".

Ref. 33.5, Page 5

$$F_{ky} := 100000 \cdot \frac{\text{lb}}{\text{in}^2}$$

Yield Strength for
 (ASTM A 276-94b, Type 304, Condiiton B)

$$SA_{ky} := 0.577 \cdot F_{ky} \quad SA_{ky} = 5.77 \cdot 10^4 \text{ psi}$$

Allowable Shear Stress

Ref. 11, Page 370

Reel

$$A_{kyr} := (0.50 \text{ in}) \cdot (3.75 \text{ in})$$

$$A_{kyr} = 1.875 \cdot \text{in}^2$$

$$d_{AB} := 1.9375 \text{ in}$$

$$r_r := \frac{d_{AB}}{2}$$

$$F_{kyr} := \frac{T_{AB}}{r_r} \quad F_{kyr} = 6.939 \cdot 10^3 \text{ lbf}$$

$$S_{kyr} := \frac{F_{kyr}}{A_{kyr}} \quad S_{kyr} = 3.701 \cdot 10^3 \cdot \frac{\text{lbf}}{\text{in}^2}$$

$$SF_r := \frac{SA_{ky}}{S_{kyr}}$$

$$SF_r = 15.591 > 3 \quad \text{OK}$$

Spur Gear

$$A_{kyg} := (0.50 \text{ in}) \cdot (4.75 \text{ in})$$

$$A_{kyg} = 2.375 \cdot \text{in}^2$$

$$d_{BC} := 2 \text{ in}$$

$$r_g := \frac{d_{BC}}{2}$$

$$F_{kyg} := \frac{T_{BC}}{r_g} \quad F_{kyg} = 3.779 \cdot 10^4 \text{ lbf} \quad \text{Force on key}$$

$$S_{kyg} := \frac{F_{kyg}}{A_{kyg}} \quad S_{kyg} = 1.591 \cdot 10^4 \cdot \frac{\text{lbf}}{\text{in}^2} \quad \text{Stress on key}$$

$$SF_g := \frac{SA_{ky}}{S_{kyg}}$$

$$SF_g = 3.627 > 3 \quad \text{OK} \quad \text{Safety factor}$$

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-12.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9 / 8 / 95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	4/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Loading on Circular Hub

Forces / Moments on Reel Hub

$W_{Hub} = 4290 \text{ lbf}$		Weight on Reel Hub	
$T = 44510 \text{ lbf in}$		Torsional Moment	
<hr/>			
Tension := W_{Hub}	Tension = $4.29 \cdot 10^3 \cdot \text{lbf}$	Tension on Rope	
$d = 1.5 \text{ in}$		Rope Diameter (Width of Flat Wire Rope)	Ref. 33.1
$D = 11.75 \text{ in}$		Hub (Drum) Diameter	Ref. 33.5, Page 6
$P = \frac{2 \cdot \text{Tension}}{D \cdot d}$	$P = 486.809 \frac{\text{lbf}}{\text{in}^2}$	Radial Pressure	Ref. 8, Pg. 334
$h = (2.255) \text{ in}$		Hub Thickness (Radial)	
$D_m = (11.75 \text{ in} - h)$	$D_m = 9.495 \text{ in}$	Hub Mean Diameter	
$\text{circ} = \pi \cdot D_m$	$\text{circ} = 29.829 \text{ in}$	Hub Circumference at mean Diameter	
$F_{rad} = P \cdot d$	$F_{rad} = 730.213 \frac{\text{lbf}}{\text{in}}$	Distributed Force due to Radial Pressure	Ref. 33.5, Sh. 4

Forces and Moment on Reel Hub

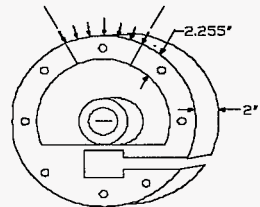
Stresses on curved beam are calculated using the standard straight beam formula, $S = Mc / I$, multiplied by a Stress Correction Factor, K . This calculation considers the initial straight beam length to be the distance between bolts, or 1/8 of the hub circumference.

$$\uparrow \sum F = 0 = W_{Hub} + F_{rad} \left(\frac{\text{circ}}{8} \right) - R_1 - R_2$$

$$\curvearrowright \sum M = 0 = R_2 \left(\frac{\text{circ}}{8} \right) - W_{Hub} \left(\frac{\text{circ}}{16} \right) - F_{rad} \left(\frac{\text{circ}}{8} \right) \left(\frac{\text{circ}}{16} \right)$$

$$R_2 = \frac{W_{Hub} \left(\frac{\text{circ}}{16} \right) + F_{rad} \left(\frac{\text{circ}}{8} \right) \left(\frac{\text{circ}}{16} \right)}{\left(\frac{\text{circ}}{8} \right)} \quad R_2 = 3.506 \cdot 10^3 \cdot \text{lbf}$$

$$R_1 = W_{Hub} + F_{rad} \left(\frac{\text{circ}}{8} \right) - R_2 \quad R_1 = 3.506 \cdot 10^3 \cdot \text{lbf}$$

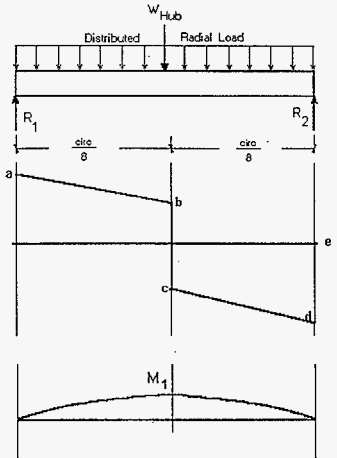


DESIGN ANALYSIS

Client WESTINGHOUSE HANFORD COMPANY WO/Job No. ER4319 Filename WIN-12.mcd
Subject WINCH ANALYSIS - SLURRY/SLUICE PUMP Date 9/8/95 By D.L. STONE
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked 9/8/95 By M.R. CUSTER *MRC*
Location C TANK FARM - HANFORD 200 EAST AREA Revised By

Bending Moments for Hub as a Straight Beam

$$\begin{aligned} a &:= R_1 & a &= 3.506 \cdot 10^3 \cdot \text{lb}f \\ b &:= a - F_{\text{rad}} \left(\frac{\text{circ}}{16} \right) & b &= 2.145 \cdot 10^3 \cdot \text{lb}f \\ c &:= b - W_{\text{Hub}} & c &= -2.145 \cdot 10^3 \cdot \text{lb}f \\ d &:= c - F_{\text{rad}} \left(\frac{\text{circ}}{16} \right) & d &= -3.506 \cdot 10^3 \cdot \text{lb}f \\ e &:= d + R_2 & e &= 0 \cdot \text{lb}f \end{aligned}$$

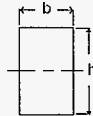


Maximum Bending Moment for Hub as a Straight Beam

$$M_1 := b \left(\frac{\text{circ}}{16} \right) + \frac{(a - b) \left(\frac{\text{circ}}{16} \right)}{2} \quad M_1 = 5.268 \cdot 10^3 \cdot \text{lb}f \cdot \text{in}$$

Calculated Straight Beam Stress

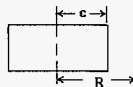
b := 2-in Hub "Length" (Axial)
h := 2.255-in Hub Width (Radial)
I := $\frac{b \cdot h^3}{12}$ I = 1.911 · in⁴ Moment of Inertia



Ref 8, Page 242

Parameters for determination of Stress Correction Factor, K

$$c := \frac{h}{2} \quad R := \frac{D}{2} - c \quad \frac{R}{c} = 4.211$$



Ref 8, Page 243

Interpolation of correction factor for inside fiber

$$\left\{ \begin{aligned} A_i &= 4 & a_i &= 1.2 & x_i &:= \frac{A_i - B_i}{A_i - C_i} (a_i - c_i) \\ B_i &= 4.211 & b_i & & b_i &:= a_i - x_i & b_i &= 1.192 \\ C_i &= 6 & c_i &= 1.12 & & & & \end{aligned} \right.$$

Ref 8, Page 242

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-12.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Calculated Curved Beam Stress

$$S := \frac{M_1 \cdot c}{I} \quad S = 3.108 \cdot 10^3 \text{ psi}$$

$$S_{act} := b_i \cdot S \quad S_{act} = 3.703 \cdot 10^3 \text{ psi} \quad \text{Maximum Stress for inner fiber of a Curved Beam}$$

$$S_y := 120000 \text{ psi} \quad \text{Yield Stress for ASTM A-291-92, Class 6, (BHN = 321)}$$

$$S_{all} := 0.45 \cdot S_y \quad S_{all} = 5.4 \cdot 10^4 \text{ psi} \quad \text{Allowable Tensile Stress} \quad \text{Ref. 11, Page 12}$$

$$\frac{S_{all}}{S_{act}} = 14.581 \quad \text{Safety Factor} > 3 \quad \text{OK}$$

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-13.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Analysis of Reel Hub Connections

$W_{Hub} := 4290 \cdot \text{lbf}$

Concentrated Vertical Load on Reel Hub

$P := 486.809 \cdot \frac{\text{lbf}}{\text{in}^2}$

Radial Pressure

$F_{rad} := 730.213 \cdot \frac{\text{lbf}}{\text{in}}$

Distributed Force due to Radial Pressure

$T := 44510 \cdot \text{lbf} \cdot \text{in}$

Torsional Moment

$b := 2 \cdot \text{in}$

Hub Length (Axial)

$h := (2.255) \cdot \text{in}$

Hub Thickness (Radial)

$D_m := (11.75 \cdot \text{in} - h)$

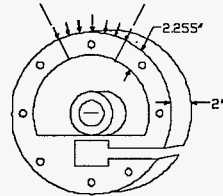
$D_m = 9.495 \cdot \text{in}$

Hub Mean Diameter

$\text{circ} = \pi \cdot D_m$

$\text{circ} = 29.829 \cdot \text{in}$

Hub Circumference



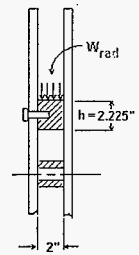
DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-13.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check stress on 1/2 - 13 - UNC - 2A x 1-1/4 L Hex Head Screws

Ref. 33.4, Pg. 4, Item 63

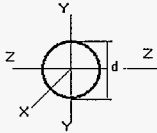
$n_b = 8$		Number of Screws	
$n = 13$		Number of threads per inch	
$D = \frac{1}{2}$ in		Major Diameter of Screw	
$A_T = 0.7854 \left[\left(D - \left(\frac{0.9743}{n} \right) \cdot \text{in} \right)^2 \right]$	$A_T = 0.142 \cdot \text{in}^2$	Tensile Stress Area of Screw	Ref. 7, Pg. 4-147
$F_u = 125000$ psi		Ultimate Tensile Strength of A193 Class2 B8T Stainless Steel Fasteners	
$S_{all} = 0.17 \cdot F_u$	$S_{all} = 2.125 \cdot 10^4$ psi	Allowable shear strength of screws	Ref. 7, Pg. 4-147
$W_{rad} = \frac{P \cdot \text{circ}}{n_b}$	$W_{rad} = 1.815 \cdot 10^3 \cdot \frac{\text{lb}}{\text{in}}$	Uniform Load resulting from Radial Pressure, per screw.	
$S_{act} = \frac{W_{Hub} + W_{rad} \cdot b}{n_b \cdot A_T}$	$S_{act} = 6.977 \cdot 10^3$ psi	Calculated Shear Stress	Ref. 7, Pg. 4-147
$M_{rad} = \frac{W_{rad} \cdot b^2}{12}$	$M_{rad} = 605.051 \cdot \text{lb} \cdot \text{in}$	Moment due to radial loading.	
$M_{con} = \frac{W_{Hub} \cdot b}{8}$	$M_{con} = 134.063 \cdot \text{lb} \cdot \text{in}$	Moment due to concentrated loading.	
$W_T = \frac{M_{rad} + M_{con}}{\left(\frac{h}{2} \right)}$	$W_T = 655.533 \cdot \text{lb}$	Tensile Force on Screw	
$T_{act} = \frac{W_T}{A_T}$	$S_{act} = 6.977 \cdot 10^3$ psi	Calculated Tensile stress	
$T_{all} = .33 \cdot F_u$	$T_{all} = 4.125 \cdot 10^4$ psi	Allowable Yield Strength in Tension	
$SF = \frac{S_{all}}{S_{act}} + \frac{T_{all}}{T_{act}}$	SF = 11.975	Safety factor > 5	<u>OK</u>



DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-13.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9 / 8 / 95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Weld Stresses Resulting from Radial and Vertical Loading



WELD PROPERTIES:

Ref. 9, Pg. 7.4 6 & 7

$d := 11.75 \cdot \text{in}$		Weld Diameter
$C_y := \frac{d}{2}$	$C_y = 5.875 \cdot \text{in}$	Location of center of gravity wrt b.
$C_z := \frac{d}{2}$	$C_z = 5.875 \cdot \text{in}$	Location of center of gravity wrt d.
$A_w := \pi \cdot d$	$A_w = 36.914 \cdot \text{in}$	Weld length
$S_{wy} := \frac{\pi \cdot d^2}{4}$	$S_{wy} = 108.434 \cdot \text{in}^2$	Section modulus
$S_{wz} := S_{wy}$	$S_{wz} = 108.434 \cdot \text{in}^2$	Section modulus
$J_w := \frac{\pi \cdot d^3}{4}$	$J_w = 1.274 \cdot 10^3 \cdot \text{in}^3$	Polar Moment of Inertia

JOINT LOADS:

Tensile	$F_x := \frac{F_{\text{rad}} \cdot 1 \cdot \text{in}}{2}$	$F_y := W_{\text{Hub}} \cdot \sin(61.1 \cdot \text{deg})$	$F_z := W_{\text{Hub}} \cdot \cos(61.1 \cdot \text{deg})$
Moment	$M_x := \frac{T}{2}$	$M_y := F_z \cdot \left(\frac{b}{2}\right)$	$M_z := F_y \cdot \left(\frac{b}{2}\right)$

WELD STRESS:

$$f_w := \left[\left(\frac{F_x}{A_w} + \frac{M_y}{S_{wy}} + \frac{M_z}{S_{wz}} \right)^2 + \left(\frac{F_y}{A_w} + \frac{M_x \cdot C_y}{J_w} \right)^2 + \left(\frac{F_z}{A_w} + \frac{M_x \cdot C_z}{J_w} \right)^2 \right]^{10.5} \quad f_w = 266.512 \cdot \frac{\text{lbf}}{\text{in}}$$

FILLET WELD SIZE REQUIRED:

$S := 15800 \cdot \text{psi}$	Allowable Weld Stress per AWS & AISC	Ref. 9, Pg. 7.4-8
$w := \frac{f_w}{0.707 \cdot S} = 0.024 \cdot \text{in}$	Minimum Allowable Fillet Weld Required by Design	
$w_{\text{Code}} := \left(\frac{5}{16}\right) \cdot \text{in}$	Minimum Size Fillet Weld Required by Code for material thickness > 3/4".	Ref. 7, Part 5, Section J.2, Table J2.4
$SF_{f_w} := \frac{w_{\text{Code}}}{w} = 13.098$	Safety factor > 3	<u>OK</u>

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-14.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRL</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Tooth Loading on Gears

Ref: 33.5, Sh. 1

$r_{reel} := 10.375$ -in Maximum Radius of Coil on Reel

$r_{25} := 9$ -in Pitch Radius of Gear 25

$r_{24} := 2$ -in Pitch Radius of Gear 24

$r_{26} := 6$ -in Pitch Radius of Gear 26 (Worm Gear)

$r_{worm} := 2.00$ -in Pitch Radius of the WORM

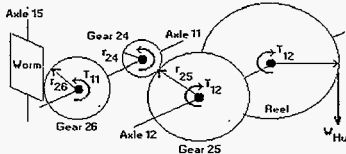
$W_{Hub} := 4290$ -lbf Weight on Reel Hub

$T_{12} := W_{Hub} \cdot r_{reel}$ $T_{12} = 4.451 \cdot 10^4$ ·lbf·in Torsional Moment on Axle 12 imposed by Winch Reel

$F_{tran} := \frac{T_{12}}{r_{25}}$ $F_{tran} = 4.945 \cdot 10^3$ ·lbf Force transmitted to Gear 24 by Gear 25

$T_{11} := F_{tran} \cdot r_{24}$ $T_{11} = 9.891 \cdot 10^3$ ·lbf·in Torsional Moment on Axle 11 Gear 25

$F_{worm} := \frac{T_{11}}{r_{26}}$ $F_{worm} = 1.648 \cdot 10^3$ ·lbf Axial force on Worm by Gear 25



Ref. 20, Page A 83

Ref. 20, Page A 83

Ref. 20, Page A 83

AGMA Strength Formula

Ref. 12, Page 407

$S_y := 120000$ Yield Strength for ASTM A-291-92, Class 6, BHN = 321

Interpolation of K_i

$$\left\{ \begin{array}{l} A_i = 250 \quad a_i = 2.4 \\ B_i = 321 \quad b_i \\ C_i = 450 \quad c_i = 3.4 \end{array} \quad x_i := \frac{A_i - B_i}{A_i - C_i} (a_i - c_i) \quad \left. \begin{array}{l} b_i = a_i - x_i \\ b_i = 2.755 \end{array} \right\}$$

$K_L := b_i$ $K_L = 2.755$ Life cycle Requirement

Temp = 220 Operating Temperature

$K_T := \frac{460 + Temp}{620}$ Temperature Requirement

$K_R := 1.5$ Reliability Requirement

$S := \frac{S_y \cdot K_L}{K_T \cdot K_R}$ $S = 2.01 \cdot 10^5$ Calculated Allowable Strength for Gear Teeth

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/18/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Lewis Beam Strength Formula (Barth Revision) for Tooth Loading on Spur Gears

Gear 25 - 54 tooth Spur Gear on Reel Axle (12)

$D_{25} := 18$ in	Pitch Diameter	Ref. 20. Pg. 15
$F := 3$ in	Face Width	
$P := 3$	Diametral Pitch	
$Y := 0.352$	Tooth Form Factor	Ref. 20. Pg. 137
$rpm := 1.08$	Reel Speed	
$V := 0.262 \cdot D_{25} \cdot rpm$ $V = 5.093 \frac{ft}{min}$	Pitch Line Velocity	Ref. 20. Pg. 137
$c := 600$	Material Constant for Metallic Gears	
$W := \left[\frac{S \cdot F \cdot Y}{P} \left(\frac{c}{c + V} \right) \right]$ $W = 7.014 \cdot 10^4$ lbf	Maximum Safe (Allowable) Loading	Ref. 20. Pg. 137
$F_{all} := W \cdot lbf$		
$F_{act} := \frac{T_{12}}{r_{25}}$ $F_{act} = 4.945 \cdot 10^3 \cdot lbf$		
$SF := \frac{F_{all}}{F_{act}}$ $SF = 14.183$	Safety factor > 3	OK

Lewis Beam Strength Formula (Barth Revision) for Tooth Loading on Spur Gears

Gear 24 - 12 tooth Spur Gear on Axle (11)

$D_{24} := 4$ in	Pitch Diameter	Ref. 20. Pg. 15
$GR := \frac{54}{12}$	Gear Ratio	
$F := 4$ in	Face Width	
$P := 3$	Diametral Pitch	
$Y := 0.210$	Tooth Form Factor	Ref. 20. Pg. 137
$V := 0.262 \cdot D_{24} \cdot (GR \cdot rpm)$ $V = 5.093 \frac{ft}{min}$	Pitch Line Velocity	Ref. 20. Pg. 137
$c := 600$	Material Constant	
$W := \frac{S \cdot F \cdot Y}{P} \left(\frac{c}{c + V} \right)$ $W = 5.579 \cdot 10^4$ lbf	Maximum Safe (Allowable) Loading	Ref. 20. Pg. 137
$F_{all} := W \cdot lbf$		
$F_{act} := \frac{T_{11}}{r_{24}}$ $F_{act} = 4.945 \cdot 10^3 \cdot lbf$	Actual Loading on Gear 24	
$SF := \frac{F_{all}}{F_{act}}$ $SF = 11.282$	Safety factor > 3	OK

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DESIGN ANALYSIS

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Worm and Worm Gear Analysis

Ref. 11

$l := 4.767 \text{ deg}$	Lead Angle	Ref. 20, Page A 83
$f_n := 14.5 \text{ deg}$	Pressure Angle	Ref. 20, Page A 83
$m := 0.06$	Coefficient of friction	Ref. 11, Fig. 13-42

$W_{Gt} := \frac{T_{11}}{r_{26}}$	$W_{Gt} = 1.648 \cdot 10^3 \cdot \text{lb}f$	Transmitted Load onto the GEAR Normal to the worm-tooth profile, neglecting friction.	Ref. 11, Edition 1, Page 453
$W := W_{Gt}$			

$W_x := W \cdot \cos(f_n) \cdot \sin(l)$	$W_x = 132.631 \cdot \text{lb}f$	Tangential component of force onto the worm.	Ref. 11, Edition 5, Page 565
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$W_y := W \cdot \sin(f_n)$	$W_y = 412.744 \cdot \text{lb}f$	Radial component of force onto the worm.	
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$W_z := W \cdot \cos(f_n) \cdot \cos(l)$	$W_z = 1.59 \cdot 10^3 \cdot \text{lb}f$	Axial component of force onto the worm.	
--	--	---	--

$W_{gt} := W \cdot (\cos(f_n) \cdot \cos(l) - m \cdot \sin(l))$	$W_{gt} = 1.582 \cdot 10^3 \cdot \text{lb}f$	Transmitted Load onto the GEAR Normal to the worm-tooth profile, friction considered.	Ref. 11, Edition 1, Page 454
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$W_{wa} := W_z - m \cdot W \cdot \sin(l)$	$W_{wa} = 1.582 \cdot 10^3 \cdot \text{lb}f$	Axial force on the WORM.	
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$W_{wt} := W_x + m \cdot W \cdot \cos(l)$	$W_{wt} = 231.197 \cdot \text{lb}f$	Tangential force acting on the WORM.	
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$W_{ga} := -W_{wt}$	$W_{ga} = -231.197 \cdot \text{lb}f$	Axial force acting on the GEAR.	Ref. 11, Edition 5, Page 565
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$W_{gr} := W_y = -W_{wr}$	$W_{gr} = 412.744 \cdot \text{lb}f$	Radial force acting on the GEAR.	
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$W_{gt} := -W_{wa}$	$W_{gt} = -1.582 \cdot 10^3 \cdot \text{lb}f$	Tangential force acting on the GEAR.	
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$T_w := r_{worm} \cdot W_{wt}$	$T_w = 462.394 \cdot \text{lb}f \cdot \text{in}$	Required Torsional input to the WORM.	Ref. 11, Edition 1, Page 454
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* Due to the greater contact area and inherent strength of the worm teeth over the gear teeth, a comparative analysis of the worm Gear will be considered as a conservative comparison for the worm for this calculation.

DESIGN ANALYSIS

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Gear 26 - 36 tooth Worm Gear on Gear Axle (11)

Lewis Beam Strength Formula (Barth Revision) for Tooth Loading on Spur Gears

$D_{26} := 12$ in		Pitch Diameter	Ref. 20, Page A 83
$F := 2$ in		Face Width	Ref. 20, Page A 83
$P := 3$		Diametral Pitch	Ref. 20, Page A 83
$Y := 0.223$		Tooth Form Factor	Ref. 20. Pg. 137
$V := 0.262 \cdot D_{26} \cdot (GR \cdot rpm)$	$V = 15.28 \frac{ft}{min}$	Pitch Line Velocity	Ref. 20. Pg. 137
$c := 600$		Material Constant	
$W := \frac{S \cdot F \cdot Y}{P} \cdot \left(\frac{c}{c + V} \right)$	$W = 2.913 \cdot 10^4$ lbf	Maximum Safe (Allowable) Loading	Ref. 20. Pg. 137
$F_{all} := W \cdot lbf$			
$F_{act} := \frac{T_{11}}{r_{26}}$	$F_{act} = 4.945 \cdot 10^3$ ·lbf	Actual Loading on Gear 26	
$SF := \frac{F_{all}}{F_{act}}$	$SF = 11.282$	Safety factor > 3 OK	

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-15.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP** Date **9/8/95** By **D.L. STONE**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked **9/8/95** By **M.R. CUSTER *MRC***
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

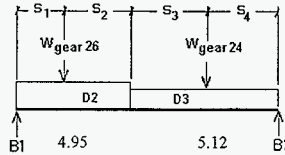
Shear on Gear Axle (11) Bearings

$D2 := \left(6.01 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{ft}}\right)$ $D3 := \left(3.77 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{ft}}\right)$ Weight per foot of Shaft

Ref. 7, Page 1-108

- $s_1 := 3.815 \text{ in}$
- $s_2 := 1.135 \text{ in}$
- $s_3 := 1.46 \text{ in}$
- $s_4 := 3.66 \text{ in}$

$W_{\text{gear26}} := 40 \cdot \text{lb} \cdot \text{ft}$ $W_{\text{gear24}} := 11 \cdot \text{lb} \cdot \text{ft}$



Gear Shaft Forces

$\uparrow \sum F = 0 = B1 + B2 - D2 \cdot (4.95 \text{ in}) - D3 \cdot (5.12 \text{ in}) - W_{\text{gear24}} - W_{\text{gear26}}$

$\circlearrowleft \sum M_{B1} = 0 = (B2 \cdot 10.07 \text{ in}) - \left[D2 \cdot \left(\frac{4.95 \text{ in}}{2} \right)^2 \right] - (D3 \cdot (5.12 \text{ in}) \cdot (7.51 \text{ in})) - W_{\text{gear24}} \cdot (6.41 \text{ in}) - (W_{\text{gear26}} \cdot 3.815 \text{ in})$

$B2 := \frac{\left[D2 \cdot \left(\frac{4.95 \text{ in}}{2} \right)^2 \right] + D3 \cdot (5.12 \text{ in}) \cdot (7.51 \text{ in}) + W_{\text{gear24}} \cdot (6.41 \text{ in}) + W_{\text{gear26}} \cdot (3.815 \text{ in})}{10.07 \text{ in}}$

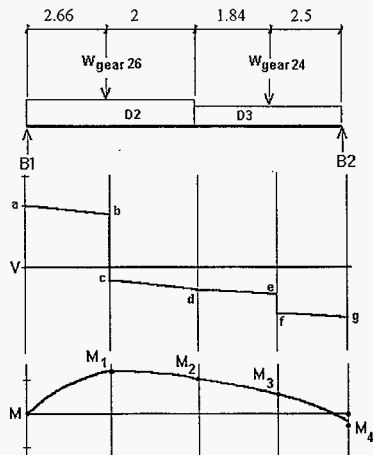
$B2 = 23.66 \cdot \text{lb} \cdot \text{ft}$

$B1 := D2 \cdot (4.95 \text{ in}) + D3 \cdot (5.12 \text{ in}) + W_{\text{gear24}} + W_{\text{gear26}} - B2$

$B1 = 31.427 \cdot \text{lb} \cdot \text{ft}$

Shear

- $a := B1$ $a = 31.427 \cdot \text{lb} \cdot \text{ft}$
- $b := a - (s_1) \cdot D2$ $b = 29.517 \cdot \text{lb} \cdot \text{ft}$
- $c := b - W_{\text{gear26}}$ $c = -10.483 \cdot \text{lb} \cdot \text{ft}$
- $d := c - (s_2) \cdot D2$ $d = -11.052 \cdot \text{lb} \cdot \text{ft}$
- $e := d - (s_3) \cdot D3$ $e = -11.51 \cdot \text{lb} \cdot \text{ft}$
- $f := e - W_{\text{gear24}}$ $f = -22.51 \cdot \text{lb} \cdot \text{ft}$
- $g := f - (s_4) \cdot D3$ $g = -23.66 \cdot \text{lb} \cdot \text{ft}$
- $h := -(g + B2)$ $h = -3.534 \cdot 10^{-15} \cdot \text{lb} \cdot \text{ft} \approx 0$



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 Subject WINCH ANALYSIS - SLURRY/SLUICE PUMP Date 9/8/95
 PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked 9/8/95
 Location C TANK FARM - HANFORD 200 EAST AREA Revised

Filename WIN-15.mcd
 By D.L. STONE
 By M.R. CUSTER MRC
 By

Maximum Bending Moment on the Shaft

$$M_1 := b \cdot (s_1) + \left[\frac{(a-b) \cdot s_1}{2} \right] \quad M_1 = 116.251 \cdot \text{lb} \cdot \text{ft} \cdot \text{in} \quad \leftarrow \text{Maximum Bending Moment}$$

$$M_2 := M_1 + \left[c \cdot (s_2) + \frac{(d-c) \cdot (s_2)}{2} \right] \quad M_2 = 104.03 \cdot \text{lb} \cdot \text{ft} \cdot \text{in}$$

$$M_3 := M_2 + \left[d \cdot (s_3) + \frac{(e-d) \cdot (s_3)}{2} \right] \quad M_3 = 87.56 \cdot \text{lb} \cdot \text{ft} \cdot \text{in}$$

$$M_4 := M_3 + \left[f \cdot (s_4) + \frac{(g-f) \cdot (s_4)}{2} \right] \quad M_4 = 3.068 \cdot \text{lb} \cdot \text{ft} \cdot \text{in}$$

Equivalent Radial load - Radial Ball Bearings

Pillow Block, RSAO (1-3/16" Shaft Diameter)

Ref. 27, Pg. 174

Bearing, GN103KRRE

Ref. 27, Pg. 144

$$C_o := 3550 \cdot \text{lb} \cdot \text{ft}$$

Published STATIC LOAD RATING (Maximum Permissible Radial Load)

Ref. 27, Pg. 174

$$i_B := 1$$

Number of adjacently mounted bearings.

$$R := B1 \quad \frac{221}{2} = 110.5$$

Maximum APPLIED RADIAL LOAD on Bearings

$$\text{Thr} := \frac{231.197}{2} \cdot \text{lb} \cdot \text{ft}$$

APPLIED THRUST LOAD on Bearing

$$K_{\text{Thr}} := \frac{\text{Thr}}{i_B \cdot C_o} \quad K_{\text{Thr}} = 0.033$$

Relative Thrust Load Factor for Ball Bearings

Ref. 27, Pg. E-50, Table 1

$$Y_1 := 2.00$$

Thrust Load Factor for Ball Bearings (from Table 2)

Ref. 27, Pg. E-51, Table 2

For Single Row WIDE INNER RING BALL BEARINGS

Equivalent Radial Load on Bearing is the Larger result of the two following Formulas

$$P := R \quad P = 31.427 \cdot \text{lb} \cdot \text{ft}$$

Ref. 27, Pg. E-50

$$P := 0.56 \cdot R + Y_1 \cdot \text{Thr} \quad P = 248.796 \cdot \text{lb} \cdot \text{ft}$$

Table 1

$$\text{SF} := \frac{C_o}{P} = \frac{F_{\text{all}}}{F_{\text{act}}} \quad \text{SF} = 14.269 \quad \text{Safety factor} > 3 \quad \underline{OK}$$

Due to low reactions at shaft ends, bearing stress on the wall at the Flange and bolt prying stresses have been considered negligible relative to material tensile strengths.

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>M.R.C.</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Design of Gear Axle (11)

$T_{11} := 9891 \cdot \text{lb} \cdot \text{ft} \cdot \text{in}$	$T_{11} = 9.891 \cdot 10^3 \cdot \text{in} \cdot \text{lb} \cdot \text{ft}$	Applied Torsional Moment on Axle 11	
$M_b := M_1$	$M_b = 116.251 \cdot \text{lb} \cdot \text{ft} \cdot \text{in}$	Maximum Bending Moment on Winch Axle	
$S_y := 120000 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^2}$		Yield Strength for (ASTM A-291-92, Class 6, BHN = 321)	
$S_{\text{allC}} := 0.30 \cdot S_y$	$S_{\text{allC}} = 3.6 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^2}$	Maximum Allowable Shearing Stress for Combined Torsion and Bending	Ref. 8, Pg 270 ((b), Below Table 2)
$P_t := 0.75 \cdot S_{\text{allC}}$	$P_t = 2.7 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^2}$	Maximum Allowable Stress for Solid Shafts using Keyways	Ref. 8, Pg 270
$B := 1$		B factor defines hollow shafts and is assumed as 1 for solid shafts.	Ref. 8, Pg 270 (Below Table 3)
$K_t := 1.0$	$K_m := 1.5$	Combined Shock and Fatigue factors for Steady, gradually applied, loading	Ref. 8, Pg 270, Table 1
$D_c := B \cdot \left[\frac{5.1}{P_t} \cdot \left[(K_m \cdot M_b)^2 + (K_t \cdot T_{11})^2 \right] \right]^{\frac{1}{3}}$	$D_c = 1.232 \cdot \text{in}$	Required Shaft Diameter for Combined Bending and Torsional Loading	Ref. 8, Pg. 269,(3a)
$D_{\text{act}} := 1.313 \cdot \text{in}$		Design Shaft Diameter	
$D_{\text{act}} \geq D_c$	<u>OK</u>	Acceptance Criteria for Minimum Shaft Diameter	

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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Key stress

Key, Spur and Worm Gear (typical) = 3/8" x 3/8" x 2.375".

$A_k := \left(\frac{3}{8}\text{ in}\right) \cdot 2.375\text{ in}$	$A_k = 0.891 \cdot \text{in}^2$	Area of key cross section	Ref. 33.5, Sh. 5, Item 11, Sh. 1, Item 78	
$F_y := 100000 \cdot \frac{\text{lb}}{\text{in}^2}$		Yield Strength for (ASTM A 276-94b, Conditon B, Type 304)		
$S_{all} := 0.577 \cdot F_y$	$S_{all} = 5.77 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$	Allowable Shear Stress	Ref. 11, Pg. 370	
Worm Gear (26)		Spur Gear (24)		
$r_w := \frac{1.5}{2} \text{ in}$		$r_s := \frac{1.1875}{2} \text{ in}$	Shaft Radius	
$F_{kw} := \frac{T_{11}}{r_w}$	$F_{kw} = 1.319 \cdot 10^4 \cdot \text{lb}$	$F_{ks} := \frac{T_{11}}{r_s}$	$F_{ks} = 1.666 \cdot 10^4 \cdot \text{lb}$	Force on key
$S_{kw} := \frac{F_{kw}}{A_k}$	$S_{kw} = 1.481 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$	$S_{ks} := \frac{F_{ks}}{A_k}$	$S_{ks} = 1.87 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$	Stress on key
$SF_w := \frac{S_{all}}{S_{kw}}$		$SF_s := \frac{S_{all}}{S_{ks}}$		Safety factor
$SF_w = 3.897$	> 3 OK	$SF_s = 3.085$	> 3 OK	

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-16.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Design of Worm Axle (15)

$W_{gt} := 1582 \cdot \text{lb}$ Transmitted Load onto the GEAR normal to the worm-tooth profile.

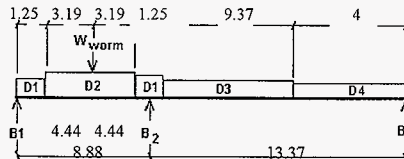
$W_w := 17 \cdot \text{lb}$ Weight of the worm

$W_{wa} := 1582 \cdot \text{lb}$ Axial force on the worm.

$W_{ax,max} := W_{wa} + W_w$ Maximum Thrust loading on Bearing

$W_{wt} := 231.197 \cdot \text{lb}$ Tangential force acting on the worm

$W_{wr} := 412.744 \cdot \text{lb}$ Radial force acting on the worm.



Load acting on bearings.

By inspection:

$M_b := 0.5 \cdot W_{wr} \cdot (4.44 \cdot \text{in})$ $M_b = 916.292 \cdot \text{lb} \cdot \text{in}$ Maximum Bending Moment on the Shaft

$R_w := 2 \cdot \text{in}$ Pitch Radius of the Worm

$T_w := R_w \cdot W_{wt}$ $T_w = 462.394 \cdot \text{lb} \cdot \text{in}$ Applied Torsional Moment on the Axle

Determine Minimum Diameter of Axle Required

$S_y := 120000 \cdot \frac{\text{lb}}{\text{in}^2}$ Safe Static Stress for 40 C. Alloy (Heat Treated)
(ASTM A-291-92, Class 6, BHN = 321)

$S_{allC} := 0.3 \cdot S_y$ $S_{allC} = 3.6 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$ Maximum Allowable Shearing Stress
for Combined Torsion and Bending Ref. 8, Pg 271
(b), Below Table 2)

$P_t := 0.75 \cdot S_{allC}$ $P_t = 2.7 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$ Maximum Allowable Stress for Solid Shafts
using Keyways Ref. 8, Pg 270

$B := 1$ B factor defines hollow shafts and is assumed
as 1 for solid shafts. Ref. 8, Pg 270
(Below Table 3)

$K_t := 1.0$ $K_m := 1.0$ Combined Shock and Fatigue factors for
Steady, gradually applied, loading Ref. 8, Pg 270, Table 1

$D_c := B \left[\frac{5.1}{P_t} \left[(K_m \cdot M_b)^2 + (K_t \cdot T_w)^2 \right] \right]^{\frac{1}{3}}$ $D_c = 0.579 \cdot \text{in}$ Required Shaft Diameter for Combined
Bending and Torsional Loading Ref. 8, Pg. 269, (3a)

$D_{act} := (1.50) \cdot \text{in}$ Shaft Design Diameter at Worm Centerline

$D_{act} > D_c$ **OK** Acceptance Criteria for Minimum Shaft Diameter

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DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-16.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Allowable Loads for Radial Tapered Roller Bearings

Bearing: Timken TDI , Cone: 17116D, Cup: 17244, Bore Diam. 1.1875", (Ref. 26, Pg 212)

$$F_t := W_{wt} \quad F_t = 231.197 \cdot \text{lbf}$$

$$F_r := W_{wr} \quad F_r = 412.744 \cdot \text{lbf}$$

$$F_a := \frac{W_{ax,max}}{2} \quad F_a = 799.5 \cdot \text{lbf} \quad \text{APPLIED THRUST LOAD per Bearing}$$

$$F_{rA} := \sqrt{\left(\frac{F_t \cdot 3.815 \cdot \text{in}}{7.63 \cdot \text{in}}\right)^2 + \left[\frac{F_r \cdot 3.815 \cdot \text{in}}{7.63 \cdot \text{in}} + \frac{F_a \cdot 4 \cdot \text{in}}{2 \cdot (7.63) \cdot \text{in}}\right]^2} \quad F_{rA} = 431.704 \cdot \text{lbf} \quad \text{Bearing Reactions (Typical)} \quad (\text{Ref. 26, Pg 49, 50})$$

K := 1.53 Relative Thrust Load Factor for Bearings

$$F_A := \frac{0.6 \cdot F_{rA}}{K} + F_a$$

P := 0.5 F_{rA} + 0.83 K F_A P = 1.446 · 10³ · lbf Equivalent Radial Load for ^{DOUBLE MRC DLS} Single Row RADIAL TAPERED ROLLER BEARINGS (Ref. 26, Pg 40)

P_{all} := 1.374 · (18100 · lbf) Maximum Permissible Radial Load at 175 rpm

$$SF := \frac{P_{all}}{P} \quad SF = 17.197 \quad \text{Safety factor} > 3 \quad \text{OK}$$

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 PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
 Location C TANK FARM - HANFORD 200 EAST AREA

WO/Job No. ER4319
 Date 9/8/95
 Checked 9/8/95
 Revised

Filename WIN-16.mcd
 By D.L. STONE
 By M.R. CUSTER *M.R.C.*
 By

Key stress

Key = 3/8 " x 3/8" x 2.00 "

$$A_k := \left(\frac{3}{8} \text{ in}\right) \cdot 2.00 \text{ in} \quad A_k = 0.75 \cdot \text{in}^2$$

Area of key cross section Ref. 33.5, Sh. 5, Item 15
 and Sh. 1, Item 78

$$F_y := 100000 \frac{\text{lb}}{\text{in}^2}$$

Yield Strength for
 (ASTM A 276-94b, Condiiton B, Type 304)

$$S_{all} := 0.577 \cdot F_y \quad S_{all} = 5.77 \cdot 10^4 \frac{\text{lb}}{\text{in}^2}$$

Allowable Shear Stress Ref. 11, Pg. 370

$$r_w := \frac{1.5}{2} \text{ in}$$

Shaft Radius

$$F_{kw} := \frac{T_w}{r_w} \quad F_{kw} = 616.525 \cdot \text{lb}$$

Force on key

$$S_{kw} := \frac{F_{kw}}{A_k} \quad S_{kw} = 822.034 \frac{\text{lb}}{\text{in}^2}$$

Stress on key

$$SF_w := \frac{S_{all}}{S_{kw}} \quad SF_w = 70.192 \quad > 3 \text{ OK}$$

Safety factor

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Stress on the worm axle Support Bracket

Ref. 33.5, Pg. 5, Item 20

$W_{wa} = 1.582 \cdot 10^3 \text{ lbf}$ Axial force transmitted to shaft at the worm

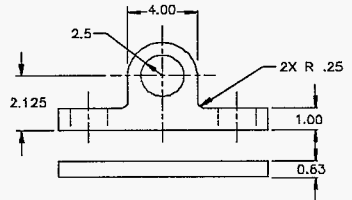
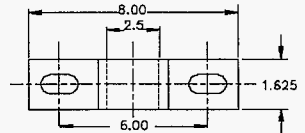
$F_{\text{housing}} = \frac{W_{wa}}{2}$ Force divided by number of bearings.

$w = 1.625 \text{ in}$ Width of Housing base.

$t = 1.00 \text{ in}$ Height of Housing base.

$b = 6 \text{ in}$ Distance between bolts

$C_L = 2.125 \text{ in}$ Distance from Mounting Block to Bearing Center Line (Moment Arm)



$M_B = W_{wa} \cdot C_L$ $M_B = 3.362 \cdot 10^3 \text{ lbf-in}$ Moment on the worm axle pillow blocks.

Check Stress on Bearing Housings

Torsion at Base

$T = \frac{M_B}{2}$

Torsion per section

$S_{act} = \frac{T}{w \cdot t^2} \left(3 + 1.8 \cdot \frac{t}{w} \right)$ $S_{act} = 4.249 \cdot 10^3 \text{ psi}$

Formula to approximate torsional stress on a rectangular area.

Ref. 11, Pg 55

$S_u = 145000 \text{ psi}$

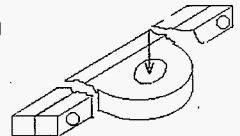
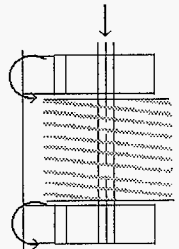
Ultimate stress for ASTM A291

$S_{sy} = 0.35 \cdot S_u$

Ref. P, Pg 423

$SF = \frac{S_{sy}}{S_{act}}$ $SF = 11.944$

Safety factor > 5 OK



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 Location **C TANK FARM - HANFORD 200 EAST AREA**

WO/Job No. **ER4319**
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 Checked **9/8/95**
 Revised

Filename **WIN-16.mcd**
 By **D.L. STONE**
 By **M.R. CUSTER MK**
 By

Shear and Bending at Housing Ring

cup = 2.46-in

Bearing cup Outer Diameter

$H_D = 4.00\text{-in}$

Bearing Housing Ring Outer Diameter

$th = \left(\frac{H_D - \text{cup}}{2} \right)$

$th = 0.77\text{-in}$

Metal Thickness at thinnest section of the bracket

$A_b = (w \cdot th) \cdot 2$

$A_b = 2.502 \cdot \text{in}^2$

Cross sectional Area at thinnest section of the bracket

$S_y = 120000 \frac{\text{lb}}{\text{in}^2}$

Yield stress for ASTM A291

$S_{all} = 0.4 \cdot S_y$

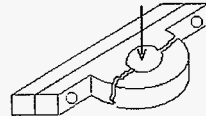
$S_{all} = 4.8 \cdot 10^4 \frac{\text{lb}}{\text{in}^2}$

Allowable Shear Stress

$S_{act} = \frac{F_{\text{housing}}}{A_b}$

$S_{act} = 316.084 \frac{\text{lb}}{\text{in}^2}$

Calculated Shear Stress



$Z = \frac{th \cdot w^2}{6}$

Section modulus of minimum Cross sectional area

$T_{act} = \frac{M_B + M_b}{2 \cdot Z}$

$T_{act} = 6.312 \cdot 10^3 \text{ psi}$

Tensile stress due to bending.

$T_{all} = 0.45 \cdot S_y$

$T_{all} = 5.4 \cdot 10^4 \text{ psi}$

$\frac{T_{all}}{T_{act}} = 8.555$

$\frac{S_{all}}{S_{act}} = 151.858$

Safety factor > 3 OK

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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check stress on 1/2 - 13 - UNC - 2A x 1-5/8 L Hex Head Screws

Shear

$n_b := 2$ Number of Screws per Housing Ref. 33.4, Pg. 1, Item 32

$n_B := 2$ Number of Bearing Housings

$n := 13$ Number of threads per inch

$D := \frac{5}{8} \text{ in}$ Major Diameter of Screw

$A_T := 0.7854 \cdot \left[\left[D - \left(\frac{0.9743}{n} \right) \cdot \text{in} \right] \right]^2$ $A_T = 0.238 \cdot \text{in}^2$ Tensile Stress Area of Screw Ref. 7, Pg. 4-147

$F_u := 125000 \cdot \text{psi}$ Ultimate Tensile Strength of A193 Class2 B8T Stainless Steel Fasteners

$S_{all} := 0.17 \cdot F_u$ $S_{all} = 2.125 \cdot 10^4 \cdot \text{psi}$ Allowable shear strength of screws Ref. 7, Pg. 4-5

$S_{act} := \frac{F_{\text{housing}}}{n_b \cdot A_T}$ $S_{act} = 1.664 \cdot 10^3 \cdot \text{psi}$ Calculated shear stress Ref. 7, Pg. 4-147

$\frac{S_{all}}{S_{act}} = 12.768$ Safety factor > 3 OK

Tensile

$w = 1.625 \cdot \text{in}$ Width of Housing base.

$y := \frac{w}{2}$ Housing base edge to screw C_L distance to determine maximum prying stress.

$F_T := \frac{(M_B + M_b)}{n_B \cdot y}$ $F_T = 2.633 \cdot 10^3 \cdot \text{lbF}$ Tension on Screws resulting from combined moment on housing

$T_{act} := \frac{F_T}{n_B \cdot A_T}$ $T_{act} = 5.539 \cdot 10^3 \cdot \text{psi}$ Calculated Tensile stress at screw

$T_{all} = 0.33 \cdot F_u$ $T_{all} = 4.125 \cdot 10^4 \cdot \text{psi}$ Allowable Yield Strength in Tension Ref. 7, Pg. 4-3

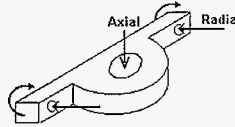
$\frac{T_{all}}{T_{act}} = 7.447$ Safety factor > 5 OK

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-17.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP** Date **9/8/95** By **D.L. STONE**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106 Checked **9/15/95** By **M.R. CUSTER**
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

Check Stresses in Gear Housing at Axle #15 Bearing Housings

$W_a := 1582 \cdot \text{lbf}$ Axial force on the WORM.
 $W_t := 231.197 \cdot \text{lbf}$ Tangential force acting on the WORM.
 $W_r := 412.744 \cdot \text{lbf}$ Radial force .
 $\text{Base}_w := 1.625 \cdot \text{in}$ Width of Bearing Housing Base
 $\text{Base}_L := 8.00 \cdot \text{in}$ Length of Bearing Housing Base
 $\text{th} := \left(\frac{1}{2} + 0.63 \right) \cdot \text{in}$ Plate Thickness
 $M := W_a \cdot (3.00 \cdot \text{in})$ $M = 4.746 \cdot 10^3 \cdot \text{lbf} \cdot \text{in}$



Ref. 33.5, Sh.5, Item 2

Shear on Housing wall due to Radial Loading

$A := \text{Base}_w \cdot \text{th}$ $A = 1.836 \cdot \text{in}^2$ Minimum Cross Sectional Area of Plate
 $S_{\text{act}} := \frac{\left(\frac{W_r}{2} \right)}{A}$ $S_{\text{act}} = 112.388 \cdot \text{psi}$ Calculated shear at plate resulting due to radial load on worm.
 $S_y := 36000 \cdot \text{psi}$ Yield Strength of A36 plate at at 250° F.
 $S_{\text{all}} := 0.4 \cdot S_y$ $S_{\text{all}} = 1.44 \cdot 10^4 \cdot \text{psi}$ Allowable shear stress
 $SF := \frac{S_{\text{all}}}{S_{\text{act}}}$ $SF = 128.128$ Safety factor > 3 OK

Stress in Housing Wall due to Torsional Moment resulting from axial force.

Ref. 7, Pg. 4-147

$y := \frac{\text{Base}_w}{2}$ Block edge to C_L distance to determine maximum prying stress.
 $I := \frac{\text{Base}_L \cdot \text{Base}_w^3}{12}$ Moment of Inertia of Housing base
 $B_1 := \frac{M \cdot y}{I}$ $B_1 = 1.348 \cdot 10^3 \cdot \text{psi}$ Calculated compressive shear at plate due to moment.

DESIGN ANALYSIS

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Subject **WINCH ANALYSIS - SLURRY/SUICE PUMP**
PROJECT **W-320 WASTE RETRIEVAL for TANK 241-C-106**
Location **C TANK FARM - HANFORD 200 EAST AREA**

WO/Job No. **ER4319**
Date **9/8/95**
Checked
Revised

Filename **WIN-17.mcd**
By **D.L. STONE**
By **M.R. CUSTER MRC**
By

Stress in Housing Wall due to Bending Moment resulting from radial force.

$$\uparrow \sum F = 0 = W_r - R_1 - R_2$$

$$\curvearrowleft \sum M = 0 = (R_2 \cdot 12.62 \text{ in}) - \left(\frac{W_r}{2}\right) \cdot 1.62 \text{ in} - \left(\frac{W_r}{2}\right) \cdot 7.62 \text{ in}$$

$$R_2 := \frac{\left(\frac{W_r}{2}\right) \cdot 1.62 \text{ in} + \left(\frac{W_r}{2}\right) \cdot 7.62 \text{ in}}{12.62 \text{ in}} \quad R_2 = 151.1 \cdot \text{lbf}$$

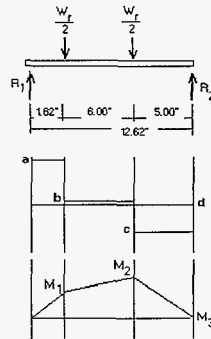
$$R_1 := W_r - R_2 \quad R_1 = 261.644 \cdot \text{lbf}$$

$$a := R_1 \quad a = 261.644 \cdot \text{lbf}$$

$$b := a - \left(\frac{W_r}{2}\right) \quad b = 55.272 \cdot \text{lbf}$$

$$c := b - \left(\frac{W_r}{2}\right) \quad c = -151.1 \cdot \text{lbf}$$

$$d := c + R_2 \quad d = 0 \cdot \text{lbf}$$



Maximum Bending Moment on the Wall

$$M_1 := (a \cdot 1.62 \text{ in}) \quad M_1 = 423.864 \cdot \text{lbf in}$$

$$M_2 := M_1 + (b \cdot 6.00 \text{ in}) \quad M_2 = 755.498 \cdot \text{lbf in} \quad \leftarrow \text{Maximum Bending Moment}$$

$$M_3 := M_2 + (c \cdot 5.00 \text{ in}) \quad M_3 = -1.258 \cdot 10^{-13} \cdot \text{lbf in} \approx 0$$

$$B_{\text{all}} = 0.6 \cdot S_y \quad B_{\text{all}} = 2.16 \cdot 10^4 \cdot \text{psi} \quad \text{Allowable stress due to Bending} \quad \text{Ref. 7, Pg. 4-147}$$

$$y := \frac{th}{2} \quad y = 0.565 \cdot \text{in} \quad C_L \text{ of plate.}$$

$$I := \frac{th^3 \cdot (10.01 \text{ in})}{12} \quad I = 1.204 \cdot \text{in}^4 \quad \text{Moment of Inertia of Plate cross section}$$

$$B_2 := \frac{M_2 \cdot y}{I} \quad B_2 = 354.645 \cdot \text{psi} \quad \text{Calculated stress at plate due to bending}$$

$$SF := \left(\frac{B_{\text{all}}}{B_1 + B_2} \right) \quad SF = 12.686 \quad \text{Safety factor} > 5 \quad \text{OK}$$

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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER M.L.C.
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Design of Drive Assembly Shaft

$T_w := 462.394 \cdot \text{lb} \cdot \text{ft} \cdot \text{in}$

Applied Torsional Moment on the Axle

$M_b := 0 \cdot \text{in} \cdot \text{lb} \cdot \text{ft}$

Maximum Bending Moment on the Shaft

Determine Minimum Diameter of Axle Required

$S_y := 120000 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^2}$

Safe Static Stress for 40 C. Alloy (Heat Treated)
 (ASTM A-291-92, Class 6, BHN = 321)

$S_{\text{allC}} := 0.3 \cdot S_y \quad S_{\text{allC}} = 3.6 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^2}$

Maximum Allowable Shearing Stress
 for Combined Torsion and Bending

Ref. 8, Pg 270
 ((b), Below Table 2)

$P_t := 0.75 \cdot S_{\text{allC}} \quad P_t = 2.7 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{ft}}{\text{in}^2}$

Maximum Allowable Stress for Solid Shafts
 using Keyways

Ref. 8, Pg 271

$B := 1$

B factor defines hollow shafts and is assumed
 as 1 for solid shafts.

Ref. 8, Pg 270
 (Below Table 3)

$K_t := 1.0 \quad K_m := 1.0$

Combined Shock and Fatigue factors for
 Steady, gradually applied, loading

Ref. 8, Pg 270, Table 1

$$D_c := B \cdot \left[\frac{S_y}{P_t} \left\{ (K_m \cdot M_b)^2 + (K_t \cdot T_w)^2 \right\} \right]^{\frac{1}{3}}$$

Ref. 8, Pg. 269,(3a)

$D_c = 0.444 \cdot \text{in}$

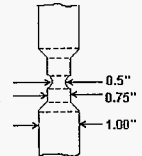
Required Shaft Diameter for Combined
 Bending and Torsional Loading

$D_{\text{act}} := 0.75 \cdot \text{in}$

Shaft Shaft Design Diameter

$D_{\text{act}} \geq D_c \quad \text{OK}$

Acceptance Criteria for Minimum Shaft Diameter



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Calculation of Force Required to Compress Nut Lock Springs

Weight of Upper Shaft Assembly

$w_{1.5} := 6.01 \cdot \frac{\text{lb}}{\text{ft}}$	Weight per ft of 1.5" A36 Rod	Ref. 16
$L_{\text{sec5}} := 14.44 \text{ in}$	Length of Upper Shaft	Ref. 33.3, Sh. 3, Item 5
$W_1 := w_{1.5} \cdot L_{\text{sec5}}$	$W_1 = 7.232 \cdot \text{lb}$	Weight of Upper Shaft

Weight of Drive Assembly

$w_2 := 2.67 \cdot \frac{\text{lb}}{\text{ft}}$	Weight per ft of 1" A36 Rod	Ref. 16
$L_{\text{sec6}} := 80.13 \text{ in}$	Length of Drive Assembly Shaft	Ref. 33.3, Sh. 3, Item 7
$W_2 := w_2 \cdot L_{\text{sec6}}$	$W_2 = 17.829 \cdot \text{lb}$	Weight of Drive Assembly Shaft

Estimated weight of Socket

$w_{1.25} := 4.17 \cdot \frac{\text{lb}}{\text{ft}}$	$W_3 := [(0.783 + 0.358) \cdot \text{in} \cdot w_{1.25}]$	$W_3 = 0.396 \cdot \text{lb}$	Upper Diameter of Socket
$w_2 := 10.68 \cdot \frac{\text{lb}}{\text{ft}}$	$W_4 := [(2.358 \cdot \text{in}) \cdot w_2]$	$W_4 = 2.099 \cdot \text{lb}$	Lower Diameter of Socket
$W_{\text{socket}} := W_3 + W_4$		$W_{\text{socket}} = 2.495 \cdot \text{lb}$	

Weight of Nut Lock Ref. 33.5, Sh.5, Item 19

$w_{\text{pl.375}} := 16.5 \cdot \frac{\text{lb}}{\text{ft}^2}$	Weight of 3/8" Stainless Steel Plate	Ref. 16
$d := 4 \text{ in}$	Nut Lock Diameter	
$c := 3.91 \cdot \text{in}^2$	Hex Cutout	
$A_{\text{NL}} := \frac{p \cdot d^2}{4} - c$	Total Plate Area of Nut Lock	
$W_{\text{NL}} := w_{\text{pl.375}} \cdot A_{\text{NL}}$	$W_{\text{NL}} = 0.992 \cdot \text{lb}$	

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Weight of Drive Guide Assembly

Ref. 33.5, Sh. 6, Item 21

$R_{\text{cone}} := 5 \text{ in}$		Cone Large Radius	
$r_{\text{cone}} := 1.5 \text{ in}$		Cone Small Radius	
$h_{\text{cone}} := 7.5 \text{ in}$		Cone Height	
$A_{\text{cone}} := \pi \cdot \sqrt{(R_{\text{cone}} - r_{\text{cone}})^2 + h_{\text{cone}}^2} \cdot (R_{\text{cone}} + r_{\text{cone}})$		Cone Area	Ref. 8, Pg 64
	$A_{\text{cone}} = 169.008 \cdot \text{in}^2$		
$d_{\text{cyl}} := 4 \cdot \text{in}$		Support Cylinder Diameter	
$h_{\text{cyl}} := 3.71 \text{ in}$		Support Cylinder Height	
$A_{\text{cyl}} := \pi \cdot d_{\text{cyl}} \cdot h_{\text{cyl}}$		Support Cylinder Area	
	$A_{\text{cyl}} = 46.621 \cdot \text{in}^2$		
$A_{\text{SecKK}} := \pi \cdot (4^2 - 2^2) \cdot \text{in}^2$		Area of Section K-K	
	$A_{\text{SecKK}} = 37.699 \cdot \text{in}^2$		
$A_{\text{DG}} := A_{\text{cone}} + A_{\text{cyl}} + A_{\text{SecKK}}$		Total Area of Guide Assembly	
	$A_{\text{DG}} = 253.329 \cdot \text{in}^2$		
$w_{569\text{pl}} := 7.5 \cdot \frac{\text{lb}}{\text{ft}^2}$		Weight per square ft of ASTM 569 Plate	Ref.16
$W_{\text{DG}} := A_{\text{DG}} \cdot w_{569\text{pl}}$		Total Weight of Guide Assembly	
	$W_{\text{DG}} = 13.194 \cdot \text{lb}$		
$W_{\text{DA}} := W_1 + W_2 + W_{\text{socket}} + W_{\text{NL}} + W_{\text{DG}}$		Total Weight of Drive Assembly	
	$W_{\text{DA}} = 41.742 \cdot \text{lb}$		

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Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-18.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER MRC.
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Spring Force

Ref. 33.5, Pgs 1, Item 36
 Pgs 5, Item 19

$L := 2.25 \cdot \text{in}$ Length of 1/2 -13 UNC-2B x 2-1/4 L Hex Screws

$th_{bp} := 0.5 \cdot \text{in}$ Thickness of baseplate.

$th_{nl} := 0.375 \cdot \text{in}$ Thickness of nutlock.

$SL_{req} := L - th_{bp} - th_{nl}$ $SL_{req} = 1.375 \cdot \text{in}$ Minimum Spring length required.

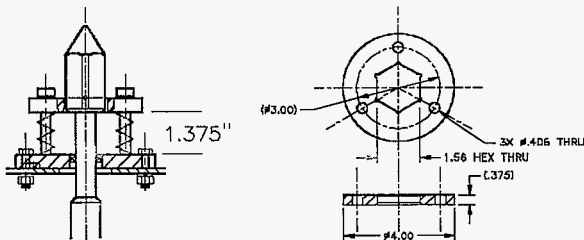
$k := 0.833 \cdot \left(53 \cdot \frac{\text{lb}f}{\text{in}} \right)$ $k = 44.149 \cdot \frac{\text{lb}f}{\text{in}}$ Spring Rate, Cat # CO600-081-2000S Ref. 21, Page 19
 Multiply Spring Rate by 0.833 for Stainless Steel

$y := \frac{3}{8} \cdot \text{in}$ $y = 0.375 \cdot \text{in}$ Length of compression (Plate thickness)

$n_s := 3$ Number of Springs

$F := n_s \cdot k \cdot y$ $F = 49.668 \cdot \text{lb}f > W_{DA}$ Force required to compress n Springs y inches Ref. 11, Page 9

Strength of Nutlock Bolts



$T := 0.5 \cdot T_w$ $T = 231.197 \cdot \text{lb}f \cdot \text{in}$ Estimated Torsional Moment applied at the drive shaft due to the transfer of load from Shaft 15 to the drive assembly.

$W := \frac{T}{1.5 \cdot \text{in}}$ $W = 154.131 \cdot \text{lb}f$ Force at bolt ring

$M := W \cdot (1.375 \cdot \text{in})$ $M = 211.931 \cdot \text{lb}f \cdot \text{in}$ Applied Moment on Bolts

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-18.mcd
Subject	WINCH ANALYSIS - SLURRY/SUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/5/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check stress on 1/2 - 13 - UNC - 2B Fasteners

Ref. 33.5, Pg. 5, Item 14

Shear

$n_b := 3$ Number of Fasteners Sharing Load

$n := 13$ Number of threads per inch

$D := \frac{1}{2} \text{ in}$ Major Diameter of Fastener

$A_T := 0.7854 \cdot \left[\left[D - \left(\frac{0.9743}{n} \right) \cdot \text{in} \right] \right]^2$ $A_T = 0.142 \cdot \text{in}^2$ Tensile Stress Area of Fastener Ref. 7, Pg. 4-147

$F_u := 125000 \text{ psi}$ Ultimate Tensile Strength of A193 Class2 B8T Stainless Steel Fasteners

$S_{all} := 0.17 \cdot F_u$ $S_{all} = 2.125 \cdot 10^4 \text{ psi}$ Allowable shear strength of Fasteners Ref. 7, Pg. 4-5

$S_{act} := \frac{W}{n_b \cdot A_T}$ $S_{act} = 362.069 \text{ psi}$ Calculated shear stress

$\frac{S_{all}}{S_{act}} = 58.691$ Safety factor > 5 OK

Tensile Stress due to bending moment

$D := \left(\frac{1}{2} \right) \text{ in}$ Major diameter of bolt

$r := \frac{D}{2}$ Radius of Bolt

$I := \frac{p \cdot D^4}{64}$ $I = 0.003 \cdot \text{in}^4$ Moment of Inertia of bolt

$T_{act} := \frac{M \cdot y}{n_b \cdot I}$ $T_{act} = 5.757 \cdot 10^3 \text{ psi}$ Calculated Tensile stress at Fasteners

$T_{all} := 0.33 \cdot F_u$ Allowable Yield Strength in Tension Ref. 7, Pg. 4-3

$\frac{T_{all}}{T_{act}} = 7.166$ Safety factor > 5 OK

DESIGN ANALYSIS

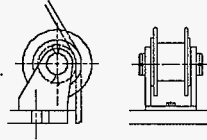
Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-19.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Design of Guide Idler

Ref. 33.5, Pg 1, View A-A

$W_T := 4290 \cdot \text{lbF}$ Tension on Wire Rope

Resolution of forces on idler axle due to loading from winch cable.
Dimension obtained from Autocad.



Ref. 31

$FH_{\text{empty}} := W_T \cos(61.1 \cdot \text{deg})$ $FH_{\text{empty}} = 2.073 \cdot 10^3 \cdot \text{lbF}$ Maximum load condition at empty reel

$FV_{\text{empty}} := W_T \sin(61.1 \cdot \text{deg})$ $FV_{\text{empty}} = 3.756 \cdot 10^3 \cdot \text{lbF}$

$FH_{\text{full}} := W_T \cos(76.66 \cdot \text{deg})$ $FH_{\text{full}} = 989.828 \cdot \text{lbF}$

$FV_{\text{full}} := W_T \sin(76.66 \cdot \text{deg})$ $FV_{\text{full}} = 4.174 \cdot 10^3 \cdot \text{lbF}$ Maximum load condition at full reel

$M_b := FH_{\text{empty}} \cdot 3.88 \cdot \text{in}$ $M_b = 8.044 \cdot 10^3 \cdot \text{lbF} \cdot \text{in}$ Moment on Bracket

Stress on sheave axles

Shear stress

$D := 1.5 \cdot \text{in}$

Axle Diameter

$A_a := \frac{\pi \cdot D^2}{4}$ $A_a = 1.767 \cdot \text{in}^2$

Cross Sectional Area

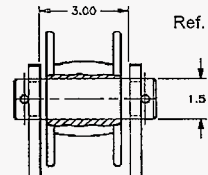
$W_{\text{axle}} := \left(\frac{W_T}{2} \right)$ $W_{\text{axle}} = 2.145 \cdot 10^3 \cdot \text{lbF}$ Force of sheave assembly felt by each shear plane.

$S_{\text{actS}} := \frac{W_{\text{axle}}}{A_a}$ $S_{\text{actS}} = 1.214 \cdot 10^3 \cdot \frac{\text{lbF}}{\text{in}^2}$ Calculated Shear Stress per axle.

$S_y := 120000 \cdot \frac{\text{lbF}}{\text{in}^2}$ Yield Strength for ASTM 291

$S_{\text{allS}} := 0.4 \cdot S_y$ $S_{\text{allS}} = 4.8 \cdot 10^4 \cdot \frac{\text{lbF}}{\text{in}^2}$ Allowable Shear Stress Ref. 11, Sec.1.8

$SF_S := \frac{S_{\text{allS}}}{S_{\text{actS}}}$ $SF_S = 39.545$ Safety factor > 3 OK



Ref. 33.5, Pg. 4, Item 8

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Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-19.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL FOR TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Bending stress

$D_m := 0.18 \text{ in}$ Thrust Washer Thickness (Moment arm on axle between sheave and housing.) Ref. 33.5, Pg. 4, Item 7

$M := W_{axle} \cdot D_m$ Moment applied to axle

$y := \frac{1.5}{2} \text{ in}$ Distance from centroidal axis to inside fiber.

$I := \frac{p \cdot D^4}{64}$ Axle Moment of Inertia

$B_{act} := \frac{M \cdot y}{I}$ $B_{act} = 1.165 \cdot 10^3 \cdot \frac{\text{lb} \cdot \text{f}}{\text{in}^2}$ Calculated Maximum Bending Stress per axle. Ref.8B, Pg. 177 (Table of Simple Stresses)

$B_{all} := 0.75 \cdot S_y$ $B_{all} = 9 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{f}}{\text{in}^2}$ Allowable Bending Stress Ref.7, Pg 5-48 (F2-1)

$SF_B := \frac{B_{all}}{B_{act}}$ $SF_B = 77.235$ Safety factor > 3 OK

Stress on Bracket

Shear, Horizontal

$w := 0.672 \text{ in}$ Width of Shear plane.

$th := \left(\frac{3}{8}\right) \cdot \text{in}$ $th = 0.375 \cdot \text{in}$ Plate thickness

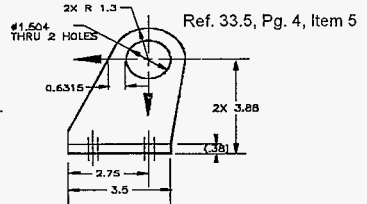
$A := 2 \cdot (w \cdot th)$ $A = 0.504 \cdot \text{in}^2$ Area of bracket loaded by the axle.

$S_y := 30000 \cdot \frac{\text{lb} \cdot \text{f}}{\text{in}^2}$ Yield strength of ASTM A240 Type 304

$S_{all} := (0.45) \cdot S_y$ $S_{all} = 1.35 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{f}}{\text{in}^2}$ Allowable Stress Ref.7, Part 5, Chap. D.3.1

$S_{act} := \frac{(FH_{empty})}{A}$ $S_{act} = 4.114 \cdot 10^3 \cdot \frac{\text{lb} \cdot \text{f}}{\text{in}^2}$ Calculated stress on brackets.

$SF := \frac{S_{all}}{S_{act}}$ $SF = 3.282$ Safety factor > 3 OK



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Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-19.mcd
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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Bearing

Ref. 33.5, Pg. 4, Item 5

$d = 1.054 \text{ in}$

Hole Diameter

$A = 2 \cdot \left(\frac{p \cdot d}{2} \cdot th \right)$

$A = 1.242 \cdot \text{in}^2$

Bearing Surface Area.

$S_y = 30000 \cdot \frac{\text{lb}}{\text{in}^2}$

Yield strength of ASTM A240 Type 304

$S_{all} = (0.9) \cdot S_y$

$S_{all} = 2.7 \cdot 10^4 \cdot \frac{\text{lb}}{\text{in}^2}$

Allowable Stress

$S_{act} = \frac{FV_{full}}{A}$

$S_{act} = 3.362 \cdot 10^3 \cdot \frac{\text{lb}}{\text{in}^2}$

Calculated stress on brackets.

$SF = \frac{S_{all}}{S_{act}}$

$SF = 8.032$

Safety factor > 3 OK

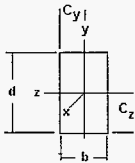
DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-19.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SUICE PUMP** Date **9/8/95** By **D.L. STONE**
 PROJECT **W-320 WASTE RETRIEVAL for TANK 241-C-106** Checked **9/8/95** By **M.R. CUSTER** *MRC*
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

Weld Strength at Base / Wall Connection

WELD PROPERTIES:

Ref. 9, Pg. 7.4 6 & 7



$b := 0.38 \cdot \text{in}$ Weld Outline Dimension - short side
 $d := 3.5 \cdot \text{in}$ Weld Outline Dimension - long side

$Cy := \frac{b}{2}$ $Cy = 0.19 \cdot \text{in}$ Location of center of gravity with respect to y axis.

$Cz := \frac{d}{2}$ $Cz = 1.75 \cdot \text{in}$ Location of center of gravity with respect to z axis.

$Aw := b \cdot d$ $Aw = 1.33 \cdot \text{in}^2$ Weld length

$Iy := \frac{b^3 \cdot d}{12}$ $Iy = 0.016 \cdot \text{in}^4$ Moment of Inertia about y axis

$Iz := \frac{b \cdot d^3}{12}$ $Iz = 1.358 \cdot \text{in}^4$ Moment of Inertia about z axis

JOINT LOADS:

Tensile $Fx := \frac{FV_{\text{full}}}{2}$ $Fy := \frac{FH_{\text{empty}}}{2}$ $Fz := 0 \cdot \text{lbf}$

Moment $Mx := 0 \cdot \text{in} \cdot \text{lbf}$ $My := 0 \cdot \text{in} \cdot \text{lbf}$ $Mz := \frac{M_b}{2}$

WELD STRESS:

$$S_{\text{act}} := \left[\left(\frac{Fx}{Aw} \right)^2 + \left(\frac{Fy}{Aw} \right)^2 + \left(\frac{Mz \cdot Cz}{Iz} \right)^2 \right]^{0.5} \quad S_{\text{act}} = 5.472 \cdot 10^3 \text{ psi}$$

FILLET WELD SIZE REQUIRED:

$S_{\text{all}} := 15800 \cdot \text{psi}$ Allowable Weld Stress per AWS & AISC Ref. 9, Pg. 7.4-8

$SF := \frac{S_{\text{all}}}{S_{\text{act}}} \quad SF = 2.887$ Safety factor based on full penetration weld $\approx > 3$ OK

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Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-19.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check stress on 3/4 - 10 - UNC - 2B Screws

Ref. 33.5, Pg. 3, Sec.C-C

Shear

$n_b := 2$ Number of Screws per Bracket

$n := 10$ Number of threads per inch

$D := \left(\frac{3}{4}\right) \text{ in}$ Major Diameter of Screw

$A_T := 0.7854 \cdot \left[\left[D - \left(\frac{0.9743}{n} \right) \cdot \text{in} \right] \right]^2$ $A_T = 0.334 \cdot \text{in}^2$ Tensile Stress Area of Screw Ref. 7, Pg. 4-147

$F_u := 125000 \cdot \text{psi}$ Ultimate Tensile Strength of A193 Class2 B8T Stainless Steel Fasteners

$S_{all} := 0.17 \cdot F_u$ $S_{all} = 2.125 \cdot 10^4 \cdot \text{psi}$ Allowable shear strength of screws Ref. 7, Pg. 4-5

$S_{act} := \frac{F_{H \text{ empty}}}{n_b \cdot A_T}$ $S_{act} = 3.099 \cdot 10^3 \cdot \text{psi}$ Calculated shear stress Ref. 7, Pg. 4-147

$\frac{S_{all}}{S_{act}} = 6.856$ Safety factor > 5 OK

Tensile

$y := 1.75 \cdot \text{in}$ Bracket base edge to screw C_L distance to determine maximum prying stress. Ref. 33.5, Pg. 4, Item 5

$M_b := 8.044 \cdot 10^3 \cdot \text{lb} \cdot \text{in}$ Moment on Bracket

$F_T := \frac{M_b}{y}$ $F_T = 4.597 \cdot 10^3 \cdot \text{lb} \cdot \text{f}$ Tension on Screws resulting from combined moment on housing

$T_{act} := \frac{F_T}{n_b \cdot A_T}$ $T_{act} = 6.872 \cdot 10^3 \cdot \text{psi}$ Calculated Tensile stress at screw

$T_{all} := 0.33 \cdot F_u$ Allowable Yield Strength in Tension Ref. 7, Pg. 4-3

$\frac{T_{all}}{T_{act}} = 6.003$ Safety factor > 5 OK

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP**
PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106
 Location **C TANK FARM - HANFORD 200 EAST AREA**

WO/Job No. **ER4319**
 Date **9/8/95**
 Checked **9/8/95**
 Revised

Filename **WIN-20.mcd**
 By **D.L. STONE**
 By **M.R. CUSTER MLL**
 By

Forces and Moments on Winch Baseplate

Weights of winch and winch housing components.

Ref. 33.5, Pg 3

T := 4290-lbf Weight of Pump and Sheave Assembly

Axle 12 := 610-lbf

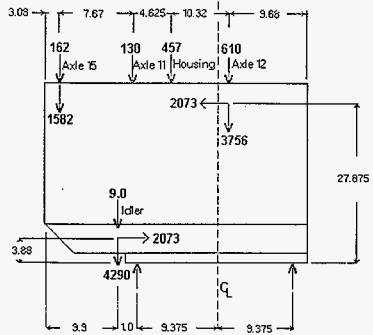
Axle 11 := 130-lbf

Axle 15 := 162-lbf

Housing := 457-lbf

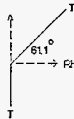
Idler := 9-lbf

F_{worm} := 1582-lbf



$$RH := T \cdot \cos(61.1 \cdot \text{deg}) \quad RH = 2.073 \cdot 10^3 \cdot \text{lbf}$$

$$RV := T \cdot \sin(61.1 \cdot \text{deg}) \quad RV = 3.756 \cdot 10^3 \cdot \text{lbf}$$



$$\downarrow \sum F = 0 = F_y := F_{\text{worm}} + T + \text{Axle } 12 + \text{Axle } 11 + \text{Axle } 15 + \text{Housing} + \text{Idler}$$

$$F_y = 7.24 \cdot 10^3 \cdot \text{lbf}$$

$$\circlearrowleft \sum M = 0 = M_{oL} := (T + \text{Idler}) \cdot (10.375 \cdot \text{in}) + ((RH) \cdot (27.375 - 3.88) \cdot \text{in}) + (\text{Housing} \cdot 5.945 \cdot \text{in}) \dots$$

$$+ [(Axle \ 11 \cdot 8.37 \cdot \text{in}) + [(Axle \ 15 + F_{\text{worm}}) \cdot 16.04 \cdot \text{in}]] - [(Axle \ 12 + (RV)) \cdot 1.95 \cdot \text{in}]$$

$$M_{oL} = 1.166 \cdot 10^5 \cdot \text{lbf} \cdot \text{in}$$

Analysis for Stress due to central couple on a simply supported plate.

Roark - Table 24 Case 20

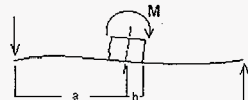
Ref. 10, Page 369

(Moment produced by eccentricity of winch is considered centrally located for this portion of the analysis and will produce conservative results.)

a := 22.5-in Radius to plate edge

b := 9.375-in Radial distance to winch baseplate edge

$\frac{b}{a} = 0.417$ Ratio for determination of proportionality constants.



DESIGN ANALYSIS

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MCC</i>
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Analysis for Stress due to central couple (continued...)

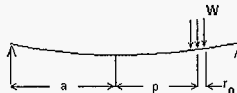
$b := 1.73$ Proportionality constant
 $a := 0.167$ Proportionality constant
 $M := M_{cL}$ Moment at winch baseplate centerline
 $t := 1.5\text{-in}$ Plate thickness
 $E := 29 \cdot 10^6 \text{ psi}$ Modulus of Elasticity of Stainless Steel

$s_w := \frac{b \cdot M}{a \cdot t^2}$ $s_w = 3.984 \cdot 10^3 \text{ psi}$ Stress Resulting from moment imposed by Winch

Analysis for Stress due to uniform loading over a small eccentric circular area of radius, r_o , for a simply supported plate.

Ref. 10, Page 367

Roark - Table 24 Case 18



$a = 22.5 \text{ in}$ Radius to plate edge
 $n = 0.3$ Poisson's Ratio
 $p = 14 \text{ in}$ Distance from plate center to center of eccentric circle
 $r_o = 3 \text{ in}$ Radius of eccentric circle

$W = (453 + 635) \text{ lbf}$ Weight at eccentric circle (Weight of hose assembly and Vertical Jumper Nozzle loading). Ref. 32

$C = 2 \cdot p \cdot a$ Winch plate Circumference

$M_r := \frac{W}{4 \cdot p} \left[1 + (1 + n) \cdot \ln \left[\frac{(a - p)}{r_o} \right] - \frac{(1 - n) \cdot r_o^2}{4 \cdot (a - p)^2} \right]$ $M_r = 201.913 \frac{\text{lbf} \cdot \text{in}}{\text{in}}$ Maximum Moment at the load.

$s_n := \frac{6 \cdot M_r}{t^2}$ $s_n = 538.435 \text{ psi}$ Stress resulting from Bending moment imposed at nozzle

$S_y = 25000 \text{ (psi)}$ Yield strength of ASTM A240 Type 304L

$B_{all} = 0.6 \cdot S_y$ $B_{all} = 1.5 \cdot 10^4 \text{ psi}$ Allowable stress due to Bending Ref. 7, Pg. 5-49

$SF = \left(\frac{B_{all}}{s_w + s_n} \right)$ $SF = 3.317$ Safety factor > 3 OK

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-20.mcd
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	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER MKC.
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Check Tensile Stress on 3/4 - 10 - UNC - 2B Fasteners

Ref. 33.4, Pg. 1, Item 33

$n_b := 2$

Number of Fasteners

$n := 10$

Number of threads per inch

$D := \frac{3}{4} \text{ in}$

Major Diameter of Fasteners

$A_T := 0.7854 \cdot \left[\left[D - \left(\frac{0.9743}{n} \right) \cdot \text{in} \right] \right]^2$

$A_T = 0.334 \cdot \text{in}^2$

Tensile Stress Area of Fasteners

Ref. 7, Pg. 4-147

$F_u := 125000 \text{ psi}$

Ultimate Tensile Strength of A193 Class2
B8T Stainless Steel Fasteners

$y := 18.75 \cdot \text{in}$

Distance between fasteners
to determine maximum prying stress.

Ref. 33.5, Pg.3

$F_T := \frac{(M_{cL})}{n_b \cdot y}$

$F_T = 3.109 \cdot 10^3 \cdot \text{lbf}$

Tension on each fastener resulting from
combined moment on housing.

$T_{act} := \frac{F_T}{A_T}$

$T_{act} = 9.295 \cdot 10^3 \cdot \text{psi}$

Calculated Tensile stress at each fastener

$T_{all} := 0.33 \cdot F_u$

$T_{all} = 4.125 \cdot 10^4 \cdot \text{psi}$

Allowable Tensile Strength of threaded fastener

Ref. 7, Pg. 4-3

$SF := \frac{T_{all}}{T_{act}}$

$SF = 4.438$

Safety factor ≈ 5 OK

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HANFORD COMPANY

Calc. No. **W-320-27-019**
 Revision No.: **0**
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DESIGN ANALYSIS

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Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL for TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER MAC
Location	C TANK FARM - HANFORD 200 EAST AREA	Revised		By	

Winchplate Lifting Brackets

$W_{Hub} := 4290 \cdot \text{lb}$	Weight of Pump and Sheave Assembly	Ref. 33.5, Pg 3
$W_{winch} := 1365 \cdot \text{lb}$	Weight of Winch Assembly	
$Hose := 494 \cdot \text{lb}$	Weight of hose and pipe assembly at Nozzle	
$W_{winchplate} := 700 \cdot \text{lb}$	Weight of Winchplate	
$W_{shieldplate} := 453 \cdot \text{lb}$	Weight of Shielding plate	

$$W := W_{Hub} + W_{winch} + Hose + W_{winchplate} + W_{shieldplate} \quad W = 7.302 \cdot 10^3 \cdot \text{lb}$$

Stress on Lifting Bracket Bars

Shear stress in Lifting Pins

Ref. 33.7, Pg. 2, Item 13

$D := 2 \cdot \text{in}$	Bar Diameter	
$A_a := \frac{\pi \cdot D^2}{4}$	$A_a = 3.142 \cdot \text{in}^2$	Cross Sectional Area
$W_{shear} := \left(\frac{W}{4} \right)$	$W_{shear} = 1.826 \cdot 10^3 \cdot \text{lb}$	Force of sheave assembly felt by each shear plane.
$S_{actS} := \frac{W_{shear}}{A_a}$	$S_{actS} = 581.075 \cdot \text{psi}$	Calculated Shear Stress per axle.
$S_y := 100000 \cdot \frac{\text{lb}}{\text{in}^2}$		Yield Strength for ASTM 276
$S_{allS} := 0.4 \cdot S_y$	$S_{allS} = 4 \cdot 10^4 \cdot \text{psi}$	Allowable Shear Stress
$SF_S := \frac{S_{allS}}{S_{actS}}$	$SF_S = 68.838$	Safety factor > 3 OK

Ref. 11, Sec.1.8

DESIGN ANALYSIS

Client	WESTINGHOUSE HANFORD COMPANY	WO/Job No.	ER4319	Filename	WIN-21.mcd
Subject	WINCH ANALYSIS - SLURRY/SLUICE PUMP	Date	9/8/95	By	D.L. STONE
	PROJECT W-320 WASTE RETRIEVAL FOR TANK 241-C-106	Checked	9/8/95	By	M.R. CUSTER <i>MRC</i>
Location	C TANK FARM - HANFORD	Revised		By	

Bending stress in Lifting Pin

$L := 3.0\text{-in}$ Span of Lifting Pin Bracket Ref. 33.7, Pg. 2, Item 2

$M := \left(\frac{W}{2}\right) \cdot \left(\frac{L}{2}\right)$ Maximum Moment potentially applied to axle

$y := 1\text{-in}$ Distance from centroidal axis to inside fiber.

$I := \frac{P \cdot D^4}{64}$ Axle Moment of Inertia

$B_{act} := \frac{M \cdot y}{I}$ $B_{act} = 6.973 \cdot 10^3 \cdot \frac{\text{lb} \cdot \text{in}}{\text{in}^2}$ Calculated Maximum Bending Stress per axle. Ref. 8, Pg. 177
(Table of Simple Stresses)

$B_{all} := 0.75 \cdot S_y$ $B_{all} = 7.5 \cdot 10^4 \cdot \frac{\text{lb} \cdot \text{in}}{\text{in}^2}$ Allowable Bending Stress Ref. 7, Pg 5-48, (F2-1)

$SF_B := \frac{B_{all}}{B_{act}}$ $SF_B = 10.756$ Safety factor > 3 OK

Stress on Bracket

Ref. 1, Pg. 4, Item 5

Shear

$w := 0.97\text{-in}$ Width of Shear plane.

$th := \left(\frac{1}{2}\right) \cdot \text{in}$ $th = 0.5 \cdot \text{in}$ Plate thickness

$A := 2 \cdot (w \cdot th)$ $A = 0.97 \cdot \text{in}^2$ Area of bracket loaded by the axle.

$S_y := 25000 \cdot \frac{\text{lb} \cdot \text{in}}{\text{in}^2}$ Yield strength of ASTM A240 Type 304L

$S_{all} := (0.45) \cdot S_y$ $S_{all} = 1.125 \cdot 10^4 \cdot \text{psi}$ Allowable Stress Ref. 7, Part 5, Chap. D.3.1

$S_{act} := \frac{W}{2 \cdot A}$ $S_{act} = 3.764 \cdot 10^3 \cdot \frac{\text{lb} \cdot \text{in}}{\text{in}^2}$ Calculated stress on brackets.

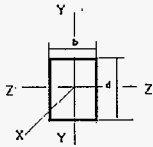
$SF := \frac{S_{all}}{S_{act}}$ $SF = 2.989$ Safety factor ≈ 3 OK

DESIGN ANALYSIS

Client **WESTINGHOUSE HANFORD COMPANY** WO/Job No. **ER4319** Filename **WIN-21.mcd**
 Subject **WINCH ANALYSIS - SLURRY/SLUICE PUMP** Date **9/8/95** By **D.L. STONE**
 PROJECT **W-320 WASTE RETRIEVAL for TANK 241-C-106** Checked **9/8/95** By **M.R. CUSTER /ARC-**
 Location **C TANK FARM - HANFORD 200 EAST AREA** Revised By

Weld Strength at Lifting Bracket Base

WELD PROPERTIES:



$b := 0.5 \text{ in}$ Weld Outline Dimension - short side Ref. 9, Pg. 7.4 6 & 7

$d := 8 \text{ in}$ Weld Outline Dimension - long side

$A_w := 2 \cdot 2 \cdot (b + d)$ $A_w = 34 \text{ in}$ Weld length (2 Plates)

JOINT LOADS:

Tensile $F_x := \frac{W}{2}$ $F_y := 0 \text{ lbf}$ $F_z := 0 \text{ lbf}$

Moment $M_x := 0 \text{ in-lbf}$ $M_y := 0 \text{ in-lbf}$ $M_z := 0 \text{ in-lbf}$

WELD STRESS:

$$f_w := \left(\frac{F_x}{A_w} \right) \quad f_w = 107.382 \cdot \frac{\text{lbf}}{\text{in}}$$

FILLET WELD SIZE REQUIRED:

$S := 15800 \text{ psi}$ Allowable Weld Stress per AWS & AISC Ref. 9, Pg. 7.4-8

$w := \frac{f_w}{0.707 \cdot S}$ $w = 0.01 \text{ in}$ Minimum Allowable Fillet Weld Required by Design

$w_{\text{Code}} := \left(\frac{3}{16} \text{ in} \right)$ Minimum Size Fillet Weld Required by Code Ref. 7, Part 5, Section J.2.2, Table J2.4

$w_{\text{act}} := \left(\frac{1}{4} \text{ in} \right)$ Weld size specified in design drawings. Ref. 33.6, Pg. 2

$SF_{f_w} := \frac{w_{\text{act}}}{w}$ $SF_{f_w} = 26.007$ Safety factor based on a 1/4" weld > 3 OK

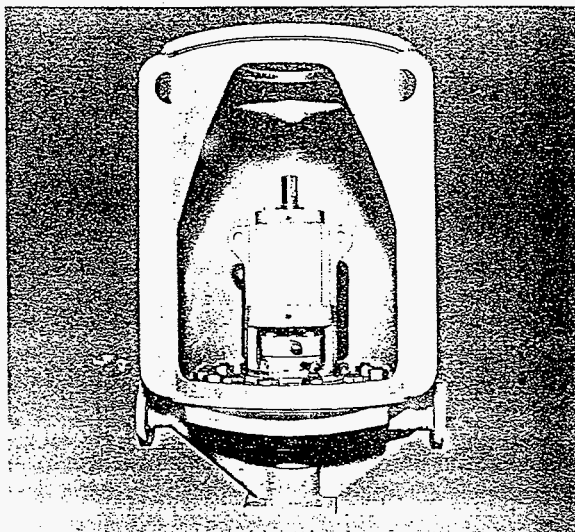
APPENDIX A

(NEW PUMP IN SUMP) as of 4/6/95
PIT

Series 1400

In-Line Bearing Frame True Line Pumps

- Partial emissions hydraulics for optimum efficiency
 - Heads to 700 FT (213m) at 3550 RPM with a single stage
- 700 lb. casing
- Back Pullout feature
- Mag Drive option
- High speed designs available
- Variable speed options available



Series 1400 2' x 2' - 13" Vertical In-Line Pump

Figure 1

Optimized Low Flow, High Head Performance

True Line pumps from Lawrence Pumps Inc. are designed to deliver long term mechanical and hydraulic reliability for your low flow, high head applications.

- Designed to API-610 Standards
- Mechanical seal life is extended because of the robust shaft design, combined with low radial loads minimizes shaft deflection
- Alignment between the pump and motor is guaranteed by a precision machined fit
- Liberal internal clearances and reduced radial thrust virtually eliminate the potential for any internal metal to metal contact
- Extreme temperature designs available (-50°F/-45°C to 700°F/370°C)

- Back Pullout feature allows ease of maintenance without disturbing the motor or connecting piping
- Seal chamber design accommodates Single, Double and Tandem mechanical seals
- Complete seal flush piping arrangements available

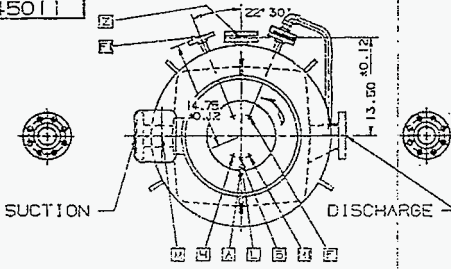
There are no close clearance wear rings or stuffing box bushing, and the axial clearances on both the front and back of the impeller are typically 0.035". Ultimately, the True Line design will withstand prolonged process upsets, or loss of suction, if seal lubrication is maintained.

Improved Process Pump Performance. Reduced Total Cost of Ownership.

- High efficiency at low flows
- Stable operations at all design points
- Minimum shaft deflection at off-design flows
- Reliable performance through process upsets
- Low maintenance from high seal reliability
- True In-Line with suction and discharge flanges on same plane
- Low NPSH characteristics



C 45011



CONNECTIONS:

- A 1/4-18 NPT - ALTERNATE LUBRICANT INLET TO THRUST BEARING (PLUGGED)
- B 1/4-18 NPT - OIL OUTLET FROM BEARING HOUSING
- F 1/2-14 NPT - INLET TO SEAL THRU API PLAN II FROM PUMP DISCHARGE
- G 1/2-300 LB. R.F. S.W. FLANGE - CASING VENT
- H 1/2-14 NPT - SEAL VENT
- I 1/2-14 NPT - SEAL DRAIN
- J 1/4-18 NPT - OIL MIST INLET TO BEARING HOUSING
- M MOTOR CONNECTION FOR POWER LEADS
- Z 3/4-300 LB. R.F. S.W. FLANGE - CASING DRAIN (COVERED WITH BLIND FLANGE)

WEIGHTS:

- PUMP 800 LBS.
- MOTOR 2700 APPROX. LBS.
- TOTAL 3500 APPROX. LBS. APPROXIMATE

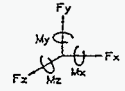
DRIVER DATA:

- 250 HP - 3500 RPM, T.E.F.C., VERTICAL ELECTRIC MOTOR, RELUCTANCE FRAME NO. 326TC

NOZZLE LOADS:

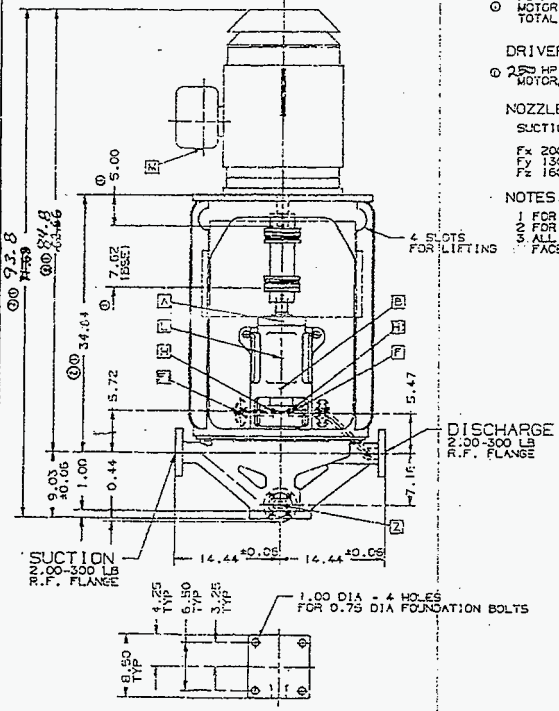
SUCTION & DISCHARGE

- Fx 200 LBS. Mx 340 FT.-LBS.
- Fy 130 LBS. My 250 FT.-LBS.
- Fz 160 LBS. Mz 170 FT.-LBS.



NOTES:

- 1 FOR ASSEMBLY SEE DRAWING NO. C45010.
- 2 FOR PARTS LIST SEE "LIST OF COMPONENTS".
- 3 ALL FLANGE CONNECTIONS TO CONFORM TO A FACE FINISH OF .125 RMS.



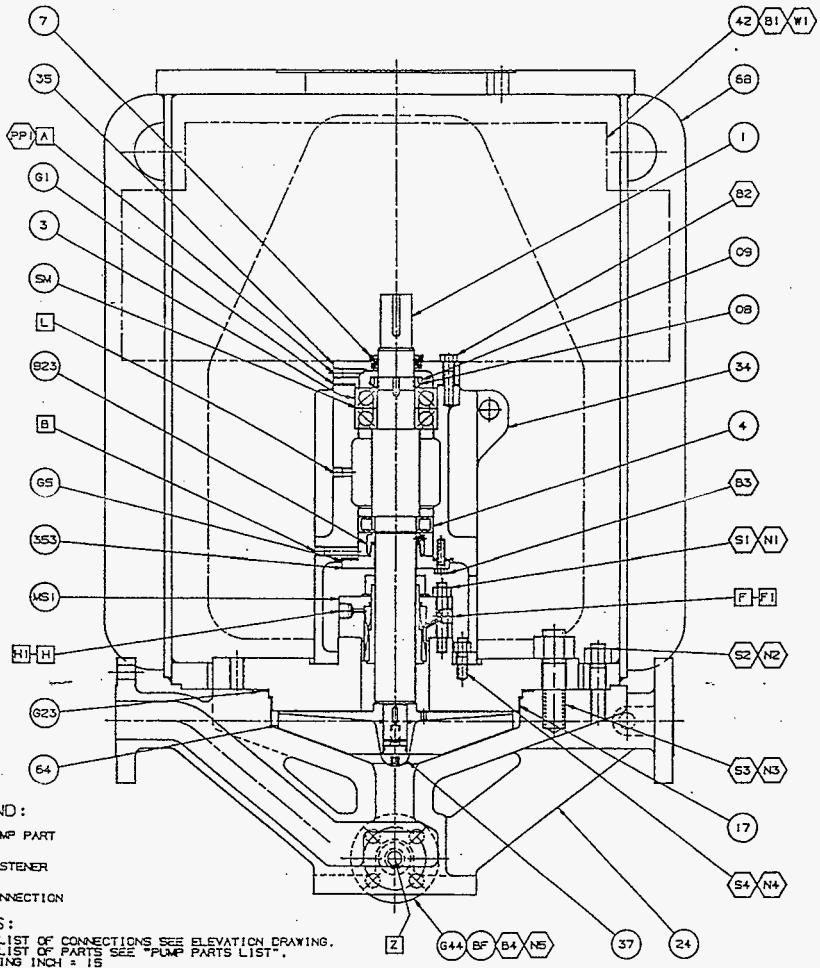
2	REV	RAM	DIMS DECREASED 0.50
1	MOD	RAM	MOTOR WAS 325 HP FRAME
NO.	BY	DATE	CHK. BY
1	REVISIONS		

LAWRENCE PUMPS INC.
PUMP SIZE 250/350/325/300 HP
SERIES: 1400
MODEL: 14-4A
DR. BY: 77
CHK. BY: RAM 11/28/88
APP. BY:
DATE: 2/11/89

TRUE-LINE SERIES
CUSTOMER:
PURCHASE ORDER NO.:
PUMP SERIAL NO.:
PUMP ITEM NO.:
CERTIFIED BY:
DATE:

PUMP ASSEMBLY:
PUMP PARTS LIST:
PUMP ELEVATION
DRAWING NO.: C 45011

C 45010





LEGEND:
 ○ PUMP PART
 ⬡ FASTENER
 □ CONNECTION

NOTES:
 1 FOR LIST OF CONNECTIONS SEE ELEVATION DRAWING.
 2 FOR LIST OF PARTS SEE "PUMP PARTS LIST".
 3 BEARING INCH = 15

		TRUE-LINE SERIES		CUSTOMER: _____		PUMP ELEVATION: _____	
				PURCHASE ORDER NO.: _____		PUMP PARTS LIST: _____	
PUMP SIZE: 20-24 1/2" 2-24 1/2" D SERIES: 14-00 MODEL: 14-4A		PUMP SERIAL NO.: _____		PUMP ITEM NO.: _____		CROSS-SECTIONAL PUMP ASSEMBLY	
DR. BY: JH CHK. BY: RAM 11/29/68 APPV. BY: _____		PUMP SERIAL NO.: _____		CERTIFIED BY: _____			
DATE: 21, NOV. 69		DATE: _____		DATE: _____			

NO.	DATE	BY	DESCRIPTION

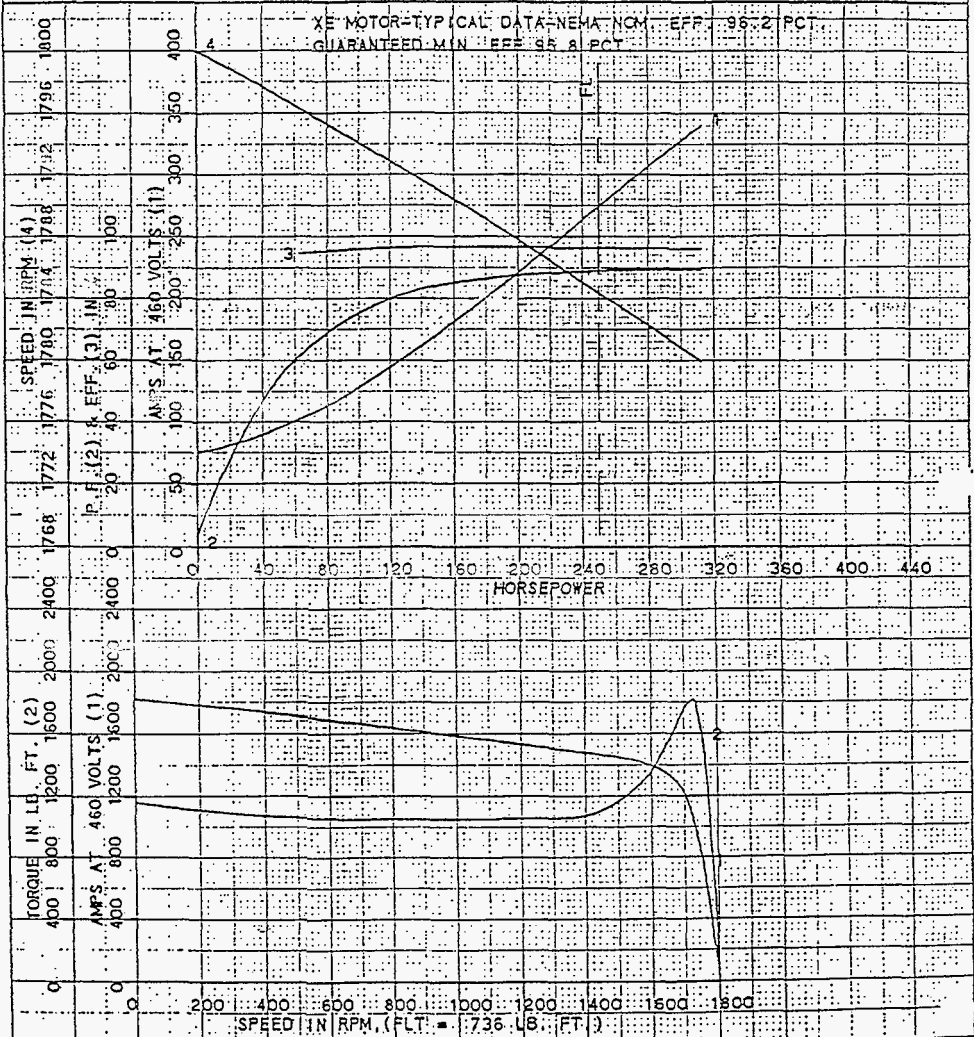
REL. S.O.	FRAME	HP	TYPE	PHASE/ HERTZ	RPM	VOLTS
	250	250	P	3/60	1785	460
AMPS	DUTY	AMB°C/ INSUL.	S.F.	NEMA DESIGN	CODE LETTER	ENCL. FCX P-10
273	CONT	40/F	1.15	B	G	FCX P-10
E/S	ROTOR	TEST S.O.	TEST DATE	STATOR RES. @25°C OHMS (BETWEEN LINES)		
533698	418143-31-GE	---	---	.0124		
PERFORMANCE						
LOAD	HP	AMPERES	RPM	% POWER FACTOR	% EFFICIENCY	
NO LOAD	0	75.6	1800	3.92	0	
1/4	62.6	100	1796	61.6	94.6	
2/4	125	150	1793	80.8	96.4	
3/4	187	209	1789	86.9	96.6	
4/4	250	273	1784	89.0	96.4	
5/4	312	341	1780	89.4	95.9	
SPEED TORQUE						
	RPM	TORQUE % FULL LOAD	TORQUE LB.-FT.	AMPERES		
LOCKED ROTOR	0	158	1160	1820		
PULL UP	700	143	1052	1660		
BREAKDOWN	1728	247	1816	1037		
FULL LOAD	1784	100	736	273		
<p>AMPERES SHOWN FOR 460. VOLT CONNECTION. IF OTHER VOLTAGE CONNECTIONS ARE AVAILABLE, THE AMPERES WILL VARY INVERSELY WITH THE RATED VOLTAGE</p> <p>REMARKS: XE MOTOR-TYPICAL DATA-NEMA NOM. EFF. 96.2 PCT. GUARANTEED MIN. EFF 95.8 PCT.</p>						
		DR. BY <u>D. MAY</u> CK. BY <u>J.P. SAO</u> APP. BY <u>J.P. SAO</u> DATE <u>01/21/81</u>			A-C MOTOR E06854-A-A001 PERFORMANCE DATA	
		CLEVELAND, OHIO 44117 U.S.A.			ISSUE DATE 06/05/84	

REL S.O.
FRAME 
HP 250
TYPE P
PHASE/HERTZ 3/60

RPM 1785
VOLTS 460
AMPS 273
DUTY CONT
AMBS/INSUL 40/F

S.F. 1.15
NEMA DESIGN B
CODE LETTER G
ENCLOSURE TEFC
E/S 533698

ROTOR 418143-31-GE
TEST S.O. ---
TEST DATE ---
STATOR RES. @ 25°C .0124
OHMS (BETWEEN LINES)



AMPERES SHOWN FOR 460 VOLT CONNECTION, IF OTHER VOLTAGE CONNECTIONS ARE AVAILABLE, THE AMPERES WILL VARY INVERSELY WITH THE RATED VOLTAGE.

RELIANCE ELECTRIC
CLEVELAND, OHIO 44117 U.S.A.

DR. BY	D. MAY
CK. BY	J.P. TSAO
APP. BY	J.P. TSAO
DATE	01/21/81

A-C MOTOR PERFORMANCE E06854-A-A001
CURVES ISSUE DATE 06/05/84

**Armco
NITRONIC 60
Stainless Steel**

**FIGHTS WEAR
AND GALLING**

- Best galling resistance of all stainless steels.
- Corrosion resistance better than Type 304.
- Pitting resistance better than Type 316.

**Applications
Potential**

Outstanding galling resistance at both ambient and elevated temperatures makes patented Armco NITRONIC 60 Stainless Steel a valuable material for valve stems, seats and trim; fastening systems, including nuts and bolts; screening; chain-drive systems; pins, bushings and roller bearings; and pump components such as wear rings and lobes.



Atlas Impeller Casting of Armco NITRONIC 60 Stainless Steel.

**ARMCO NITRONIC 60
STAINLESS STEEL**
(UNS S21800)



Product Data Bulletin No. S-45

Oxidation Resistance

NITRONIC 60 offers far superior oxidation resistance compared to AISI Types 304 and 316, and about the same oxidation resistance as AISI Type 309.

Table 28
Static Oxidation Resistance*

Test Temperature, F (C)	Weight Loss, mg/cm ²				
	RA 333	Type 310	NITRONIC 60	Type 304	
2100 (1149)	Before Descaling	3.1	4.6	16.5	1220
	After Descaling	12.2	15.7	23.2	1284
2200 (1204)	Before Descaling	10.1	10.1	26.1	2260
	After Descaling	16.7	20.6	35.4	2265

*240 hours at temperature, duplicate tests

Table 29
Cyclic Oxidation Resistance

Cycle	Alloy	Weight Change, mg/in ² , at number of cycles indicated					
		134 cycles	275 cycles	467 cycles	200 cycles	304 cycles	400 cycles
1600-1700 F (871-927 C) 25 minutes heat. 5 minutes cool - duplicate tests	RA 330	+ 3.4	+ 4.9	+ 6.4	—	—	—
	Type 310	+ 4.0	+ 6.7	22.7	—	—	—
	Type 309	+ 3.0	41.6	100.4	—	—	—
	NITRONIC 60	+ 1.5	69.2	167.6	—	—	—
	Type 316	-473.0	-970.8	-1287.0	—	—	—
		Weight Loss, mg/cm ²					
1900 F (1038 C) 30 minutes heat. 30 minutes cool	Type 446	—	—	—	1.47	1.72	1.97
	Type 310	—	—	—	2.70	15.95	17.22
	Type 309	—	—	—	22.53	26.34	33.69
	NITRONIC 60	—	—	—	42.99	60.40	74.80
	Type 316	—	—	—	93.04	135.34	178.27

Mechanical Properties

Table 30
Typical Room Temperature Tensile Properties*

Condition	Size	Hardness	UTS		Elongation % in 4XD.	Reduction of Area, %
			ksi (MPa)	0.2% YS ksi (MPa)		
Annealed	1" (25.4 mm) ϕ	95 HRB	103 (710)	60 (414)	64	74
Annealed	1-3/4" (44.4 mm) ϕ	100 HRB	101 (696)	56 (386)	62	73
Annealed	2-1/4" (57.2 mm) ϕ	100 HRB	101 (696)	60 (414)	60	76
Annealed	3" (76.2 mm) ϕ	97 HRB	113 (779)	65 (448)	55	67
Annealed	4-1/8" (104.8 mm) ϕ	95 HRB	106 (731)	56 (386)	57	67
10% Cold Drawn	442" (11.2 mm) ϕ	24 HRC	120 (827)	91 (627)	51	68
20% Cold Drawn	Start Size	31 HRC	140 (965)	112 (772)	35	65
30% Cold Drawn		34 HRC	161 (1110)	132 (910)	26	62
40% Cold Drawn		37.5 HRC	195 (1344)	153 (1055)	20	57
50% Cold Drawn		41 HRC	217 (1496)	174 (1200)	15	53
60% Cold Drawn		43 HRC	240 (1655)	195 (1344)	12	48
70% Cold Drawn		46 HRC	263 (1813)	217 (1496)	10	40

*Data based on duplicate tests.
(1) CG bar

Table 31

Typical Room Temperature Torsion and Shear Properties*

Condition	Size	Hardness HRB	Torsional	0.2% Torsional YS		Modulus of Rupture ksi (MPa)	Double Shear Strength ksi (MPa)
			Modulus, G ksi (MPa)	ksi (MPa)	γ		
Annealed	1" ϕ (25.4 mm)	95	8.83 x 10 ³ (61 x 10 ³)	48.9 (337)	50.7 (350)	124 (855)	—
Annealed	3/8" ϕ (9.6 mm)	95	—	—	—	—	86 (593)

*Data based on duplicate tests

Table 32

Double Shear Strength*

(Cold Drawn — 0.442" [11.23 mm] start size)

% Cold Drawn	Shear Strength, ksi (MPa)
10	89 (614)
20	98 (676)
30	106 (731)
40	113 (779)
50	122 (841)
60	130 (896)

*Triplicate tests

Table 33

Fatigue Strength

(R.R. Moore Machine)

Condition	Size	Hardness	Fatigue Limit, ksi (MPa) 10 ⁶ Cycles
Annealed	1" (25.4 mm) ϕ	95 HRB	37.5 (258)
Cold Worked 54.6%	0.70" (17.8 mm) ϕ	44 HRC	72.5 (500)

Table 34

Room Temperature Compression Strength

Condition	Size	0.2% Compressive YS, ksi (MPa)
Annealed	0.500" ϕ (12.7 mm)	67.6 (466)
Cold Drawn 39%	0.440" ϕ (11.2 mm)	121.0 (834)

Table 35

**Properties Acceptable for Material Specification
(Bar and Wire)**

Condition	Size	UTS ksi (MPa)	0.2% YS ksi (MPa)	Elongation % in 4XD	Reduction of Area, %	Hardness HRB
Annealed	1/2" ϕ + under (12.7 mm)	105 min (724)	55 min (379)	35 min	55 min	85 min
Annealed	Over 1/2" ϕ (12.7 mm)	95 min (655)	50 min (345)	35 min	55 min	85 min

Table 36

Typical Elevated Temperature Mechanical Properties*
(Annealed 3/4" and 1" [19.05 and 25.4 mm] Diameter Bar Stock)

G-33-05

Test Temperature F (C)	Reduction				
	UTS ksi (MPa)	0.2% YS ksi (MPa)	Elongation % in 4XD	of Area %	Hardness Brinell
Room Temperature	106.5 (734)	56.5 (389)	61.7	71.9	200
200 (93)	98.2 (677)	44.4 (306)	63.3	72.4	187
300 (149)	89.9 (620)	37.8 (260)	64.4	73.7	--
400 (204)	84.4 (580)	32.8 (227)	64.0	73.7	168
500 (260)	82.1 (566)	32.1 (222)	61.5	73.0	--
600 (316)	80.5 (555)	29.7 (205)	59.6	73.1	155
700 (371)	79.5 (548)	29.2 (201)	59.1	72.6	--
800 (427)	78.3 (540)	29.0 (200)	56.5	72.1	148
900 (482)	77.1 (532)	28.3 (195)	53.9	71.6	--
1000 (538)	75.4 (520)	28.0 (193)	52.2	70.4	145
1100 (593)	71.6 (494)	28.7 (198)	48.7	70.0	--
1200 (649)	66.6 (459)	28.1 (194)	48.2	69.6	144
1300 (704)	59.0 (407)	27.5 (189)	41.4	50.0	--
1400 (760)	49.8** (344)	25.3 (174)	47.1	53.9	143
1500 (816)	37.0** (255)	23.8 (164)	72.8	75.0	--
1600 (871)	30.2** (208)	16.4 (113)	72.8	--	110

*Triplicate tests of 2 heats and single tests of 1 heat

**Single tests of 1 heat

Table 37

Elevated Temperature Tensile Properties
(Cold Swaged 54% to 0.700" [17.8 mm] ϕ)

Test Temp. F (C)	Reduction			
	UTS ksi (MPa)	0.2% YS ksi (MPa)	Elongation % in 4XD	of Area. %
RT	230 (1586)	216 (1489)	12	55
200 (93)	215 (1482)	205 (1442)	12	54
300 (149)	206 (1420)	199 (1372)	11	52
400 (204)	200 (1379)	194 (1338)	11	51
500 (260)	195 (1344)	191 (1317)	11	48
600 (316)	193 (1331)	188 (1296)	11	47
700 (371)	191 (1317)	176 (1213)	10	47
800 (427)	190 (1310)	184 (1269)	9	46
900 (482)	187 (1289)	177 (1220)	11	44
1000 (538)	179 (1234)	166 (1145)	11	47
1100 (593)	162 (1117)	144 (993)	13	52
1200 (649)	112 (772)	72 (496)	11	25

Table 38

Elevated Temperature Stress Rupture Strength
(Annealed Bars 5/8" to 1" [16.0 to 25.4 mm] Diameter)

Temperature F (C)	Number of Heats	Stress Rupture Strength, ksi (MPa)		
		100 hr. life	1000 hr. life	10,000 hr. life
1000 (538)	3	72 (496)	52 (359)	35 (241)
1100 (593)	3	49 (338)	31 (214)	20 (138)
1200 (649)	4	29 (200)	17 (117)	10* (69)
1350 (732)	1	14 (97)	8 (55)	--
1500 (816)	1	6.7 (46)	4 (28)	--

*Extrapolated

Table 39
Cryogenic Tensile Properties*

Condition	Size	Temperature, F (C)	UTS		Elongation % in 4XD	Reduction of Area, %
			ksi (MPa)	0.2% YS ksi (MPa)		
Annealed	3/8" (9.5 mm) ϕ	-100 (-73)	155 (1069)	76 (524)	57	69
	3/8" (9.5 mm) ϕ	-200 (-129)	170 (1172)	87 (600)	56	71
	1" (25.4 mm) ϕ	-320 (-196)	213 (1469)	109 (752)	60	67
Cold Swaged 54%	700" (17.8 mm) ϕ	-320 (-196)	322 (2220)	272 (1875)	10	53
	700" (17.8 mm) ϕ	-200 (-129)	287 (1979)	250 (1724)	13	62

*Duplicate tests

Table 40
Low Temperature Mechanical Properties of
NITRONIC 60 Stainless Steel Longitudinal Tensile Specimens*

Test Temperature F (C)	UTS ksi (MPa)	0.2% Offset YS ksi (MPa)	Elongation % in 1" (25.4 mm) or 4XD	Reduction of Area %	Fracture Strength ksi (MPa)	Modulus psi (MPa)	N/U** Tensile Ratio	Charpy V-Notch Impact ft-lbs (J)
0 (-18)	128.1 (883)	67.3 (464)	71.3	79.7	433.4 (2988)	23.7x10 ⁶ (163,000)	1.37	216 (292)
-100 (-73)	148.4 (1023)	77.9 (537)	70.5	80.9	447.1 (3083)	24.2x10 ⁶ (167,000)	1.45	197 (267)
-200 (-129)	167.8 (1155)	87.4 (602)	57.4	79.4	457.0 (3151)	24.2x10 ⁶ (167,000)	1.46	170 (231)
-320 (-196)	217.9 (1502)	101.4 (699)	55.5	65.8	594.0 (4095)	24.8x10 ⁶ (171,000)	1.26	138 (188)
-423 (-253)	203.8 (1405)	125.3 (864)	23.5	26.6	277.6 (1914)	24.8x10 ⁶ (171,000)	1.33	—

*0.250" (6.35 mm) diameter, machined from a 1" (25.4 mm) diameter annealed and straightened bar. Four specimen average.
**Average Stress Concentration Factor $K_t = 7.0$

Data taken with permission from NASA TM X-73359, Jan. 1977.

Table 41
Impact Properties**

Condition	Size	Test Temperature, F (C)	Charpy V-Notch Impact, ft-lbs (J)
Annealed	1" ϕ (25.4 mm)	Room Temperature	240* (325)
		-100 (-73)	229 (310)
		-320 (-196)	144 (195)
Annealed	2-1/4" ϕ (54.2 mm)	Room Temperature	240* (325)
		-100 (-73)	240* (325)
		-320 (-196)	160 (217)
Cold Swaged 18% Hardness R_C 29	.932" ϕ (23.7 mm)	-320 (-196)	67 (91)
Cold Swaged 40% Hardness R_C 37	.795" ϕ (20.2 mm)	-320 (-196)	40 (54)
Cold Swaged 54% Hardness R_C 42	.700" ϕ (17.8 mm)	-320 (-196)	26 (35)
Cold Swaged 18% Hardness R_C 29	.932" ϕ (23.7 mm)	-200 (-129)	90 (122)
		-200 (-129)	44 (60)
		-200 (-129)	30 (41)

*Did not fracture completely

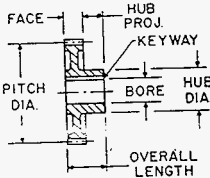
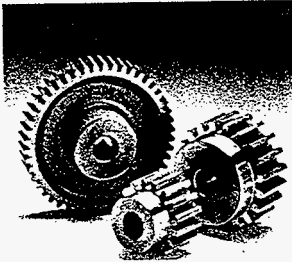
**Data based on duplicate tests

SPUR GEARS

A25

4 AND 3 DIAMETRAL PITCH
STEEL AND CAST IRON

14½° PRESSURE ANGLE
(Will not operate with 20° spurs)



ALL DIMENSIONS IN INCHES
ORDER BY CATALOG NUMBER OR ITEM CODE

No. of Teeth	Pitch Dia.	Bore	Hub		Style See Page A93	Without Keyway or Setscrew		
			Dia.	Proj.		Catalog Number	Item Code	
DIAMETRAL PITCH								
STEEL								
12	3.000*	1-1/8	2-17/64	7/8	A	NL11B	09860	
15	3.000		2-17/64			NL12B	09862	
15	3.500		2-49/64			NL14B	09864	
15	3.750		3-1/64			NL15B	09866	
15	4.000		3-17/64			NL16B	09868	
15	4.500		3-49/64			NL18B	09870	
20	5.000		4-17/64			NL20B	09872	
22	5.500		4-49/64			NL22B	09874	
CAST IRON								
24	6.000	1-1/8	3-1/2	1-1/2	A	NL24B	10484	
24	7.000				B	NL28B	10486	
24	7.500					NL30B	10488	
24	8.000				C	NL32B	10490	
24	9.000		NL36B			10492		
24	10.000		NL40B			10494		
24	12.500		NL42			10496		
24	11.000		1-1/4		4	D	NL44B	10498
24	12.000	NL48B		10500				
24	13.500	NL54		10502				
24	14.000	NL56B		10504				
24	15.000	NL60		10506				
24	16.000	NL64B		10508				
24	18.000	NL72B		10510				
24	20.000	1-3/8		4-1/2			1-1/2	NL80B
24	21.000		NL84		10514			
24	22.000		NL88B		10516			
24	24.000		NL96B		10518			
DIAMETRAL PITCH								
STEEL								
40	4.000*	1-5/16	-	-	A	NO11B†	09876	
40	4.000					NO12B†	09878	
40	4.867					NO14B	09880	
40	5.000					NO15B	09882	
40	5.333					NO16B	09884	
40	6.000					NO18B	09886	
40	6.867					NO20	09888	
40	7.000					NO21B	09890	
CAST IRON								
40	8.000	1-7/16	4-1/2	1-3/4	B	NO24B	10524	
40	10.000		5-1/4			1-1/4	NO30B	10526
40	12.000					NO36B	10528	
40	14.000		C			NO42	10530	
40	18.000	1-9/16	5-1/4	1-3/4	D	NO48B	10532	
40	18.000					5-1/2	NO54	10534
40	20.000						NO60B	10536
40	24.000	1-11/16	5-3/4	1-3/4	D	NO72B	10538	
40	28.000					NO84B	10540	
40	32.000	1-15/16	5-3/4	1-3/4	D	NO96B	10542	
40	36.000					NO108B	10544	

STANDARD TOLERANCES

DIMENSION	TOLERANCE
BORE	± .0005

REFERENCE PAGES

- Alterations — A92
- Horsepower Ratings — A9, A10
- Lubrication — A92
- Materials — A93
- Selection Procedure — A2

* Special Pitch Diameter, used for calculating Center Distance only, not Ratio.
† NO11B and NO12B have 4" Face.

APPROXIMATE HORSEPOWER AND TORQUE* RATINGS
FOR CLASS I SERVICE (Service Factor = 1.0)

4 DIAMETRAL PITCH CAST IRON 14½° PRESSURE ANGLE 2" FACE REFERENCE PAGE A25.

	25 RPM		50 RPM		100 RPM		200 RPM		300 RPM		600 RPM		900 RPM		1200 RPM		1800 RPM		3600 RPM		
	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	
2.00	5042	3.77	4750	6.75	4257	11.19	3525	14.32	3008	19.88	2089	22.84	1600								
2.43	6129	4.54	5723	8.02	5053	12.99	4094	16.38	3441			22.16	2328								
2.62	6599	4.87	6135	8.53	5378	13.69	4314	17.14	3601			22.92	2408								
2.86	7211	5.30	6675	9.22	5811	14.65	4616	18.22	3929			24.11	2534								
3.25	8187	5.96	7514	10.24	6454	15.98	5034	19.64	4126			25.49	2677								
3.64	9177	6.63	8354	11.24	7085	17.24	5433	20.97	4406	26.77	2812										
3.80	9588	6.90	8694	11.63	7328	17.69	5575	21.42	4499	27.12	2849										
4.04	10161	7.29	9195	12.22	7703	18.46	5816	22.24	4672	27.97	2938										
4.36	10999	7.82	9858	12.95	8163	19.28	6074	23.02	4837	28.58	3002										
4.97	12530	8.81	11104	14.35	9045	20.94	6598	24.72	5193												
5.13	12933	9.06	11419	14.68	9253	21.29	6708	25.04	5261												
5.44	13727	9.55	12034	15.32	9652	21.94	6915	25.65	5388												
5.86	14763	10.20	12852	16.20	10209	22.95	7233	26.65	5601												
6.47	16305	11.11	14005	17.33	10923	24.07	7585	27.65	5810												
7.18	18101	12.18	15350	18.68	11772	25.48	8029														
7.47	18838	12.60	15877	19.17	12079	25.93	8170														
7.76	19561	13.00	16387	19.63	12372	26.35	8304														
8.32	20970	13.78	17365	20.50	12922	27.13	8548														

3 DIAMETRAL PITCH STEEL 14½° PRESSURE ANGLE 3" FACE REFERENCE PAGE A25.

	25 RPM		50 RPM		100 RPM		200 RPM		300 RPM		600 RPM		900 RPM		1200 RPM		1800 RPM		3600 RPM		
	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	
2.94	7421	5.67	7146	10.56	6652	18.55	5846	24.82	5213	37.47	3936	45.15	3162	50.30	2642						
3.20	8067	6.14	7743	11.37	7167	19.80	6239	26.30	5524	39.14	4111	46.74	3273	51.78	2719						
4.15	10462	7.92	9979	14.49	9134	24.79	7812	32.48	6824	47.09	4947	55.40	3880								
4.61	11618	8.76	11046	15.96	10056	27.06	8528	35.24	7403	50.49	5304	59.01	4132								
5.09	12825	9.64	12156	17.47	11008	29.38	9259	38.03	7989	53.89	5661	62.60	4383								
6.03	15214	11.37	14333	20.38	12845	33.75	10527	43.20	9078	60.00	6303	58.93	4827								
6.97	17570	13.05	16454	23.16	14599	37.80	11913	47.89	10062	65.33	6863										
7.46	18795	13.92	17549	24.59	15495	39.64	12556	50.24	10554	67.96	7139										

3 DIAMETRAL-PITCH CAST IRON 14½° PRESSURE ANGLE 3" FACE REFERENCE PAGE A25.

	25 RPM		50 RPM		100 RPM		200 RPM		300 RPM		600 RPM		900 RPM		1200 RPM		1800 RPM		3600 RPM		
	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	H.P.	Torque	
5.23	13175	9.67	12195	16.84	10617	26.76	8433	33.30	6995	44.05	4527										
6.81	17164	12.40	15626	21.03	13251	32.25	10162	39.23	8241	50.07	5259										
8.41	21199	15.07	18998	24.96	15732	37.15	11707	44.37	9322	55.08	5786										
9.81	24721	17.32	21828	28.06	17687	40.69	12822	47.87	10056												
11.20	28241	19.50	24586	30.99	19530	43.91	13838	51.00	10715												
12.71	32043	21.83	27523	34.06	21466	47.30	14906	54.35	11417												
13.87	34965	23.52	29651	36.08	22740	49.22	15509														
16.35	41223	27.08	34136	40.30	25402	53.32	16803														
18.76	47293	30.40	38322	44.08	27782	56.87	17923														
20.75	52300	32.96	41545	46.71	29437	59.02	18557														
22.99	57968	35.87	45212	49.81	31395	61.64	19486														

Ratings are based on strength calculation. Basic static strength rating, or for hand operation of above gears is approximately 3 times the 100 RPM rating.

NOTE: Ratings to right of heavy line are not recommended, as pitch line velocity exceeds 1000 feet per minute. They should be used for interpolation purposes only.

*Torque Rating (Lb. Ins.).

WORMS AND WORM GEARS

A77

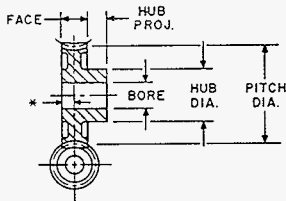
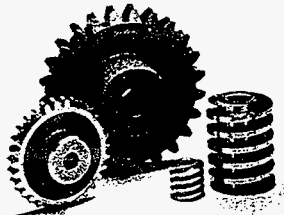
3 DIAMETRAL PITCH
CAST IRON WORM GEARS
STEEL WORMS — UNHARDENED AND HARDENED

PRESSURE ANGLE — 14½°

RATIO = Gear Teeth ÷ Worm Threads

All Worm and Worm Gears stocked RIGHT HAND ONLY

57
 12 =



ALL DIMENSIONS IN INCHES
 ORDER BY CATALOG NUMBER OR ITEM CODE

3 DIAMETRAL PITCH					WORM GEARS			FACE OF CENTER LINE WORM TO FLUSH END ±.0005"			
No. of Teeth	Pitch Dia.	Bore	Hub		SINGLE Thread		DOUBLE Thread		FOUR Thread		
			Dia.	Proj.	Catalog Number	Item Code	Catalog Number	Item Code	Catalog Number	Item Code	
18	6.000	1	3	1-1/2	B	G1110	13248	—	—	—	—
24	8.000		3-1/2	1-1/2		G1111	13250	—	—	—	—
30	10.000					G1112	13252	—	—	—	—
36	12.000	1-1/2			G1113	13254	—	—	—	—	
48	16.000		4		C	G1114	13256	—	—	—	—
54	18.000					D	G1115	13258	—	—	—

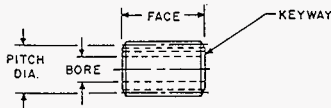
#26

STANDARD TOLERANCES

DIMENSION	TOLERANCE
Bore	All ±.0005

WORM LEAD and LEAD ANGLE

LEAD	1.0472"
LEAD ANGLE	4°45'



3 DIAMETRAL PITCH					WORMS FOR ABOVE GEARS					
Pitch Dia.	Face	Bore	Hub		SINGLE Thread		DOUBLE Thread		QUAD Thread	
			Dia.	Proj.	Catalog Number*	Item Code	Catalog Number	Item Code	Catalog Number	Item Code
UNHARDENED										
4.000	4	5-1/2	1-1/2	—	—	G1116K	12880	—	—	—
						L1116	12910	—	—	—
HARDENED										
4.000	4	5-1/2	1-1/2	—	—	H1116	12978	—	—	—
						HL1116	13016	—	—	—

#27

* All Worms furnished with 3/8" keyway.
 Hardened Worms have ground and polished threads.

REFERENCE PAGES

- Alterations — A92
- Horsepower Ratings — A66, A67
- Lubrication — A92
- Materials — A93
- Selection Procedure — A65

STEEL-HARDENED, GROUND AND POLISHED WORMS BRONZE WORM GEARS

APPROXIMATE HORSEPOWER AND TORQUE * RATINGS
FOR CLASS I SERVICE (Service Factor = 1.0)

Worm RPM	1800		600		100		Worm Cat. No.	Gear Cat. No.	DP		
	Center Distance	Input HP	Output Torque	Input HP	Output Torque	Input HP				Output Torque	
10	1.000	.52	50	.27	72	.06	83	H1607	QB1212	12	
	1.500	1.19	109	.66	183	.15	227	H1627	QB812	8	
	1.167	.78	99	.40	143	.08	166	H1607	QB1216	12	
	1.425	1.11	142	.61	223	.13	267	H1618	QB1016	10	
	1.750	1.77	216	.98	361	.22	454	H1627	QB816	8	
2.333	3.01	392	1.84	689	.45	933	H1638	QB616	6		
15	1.333	.68	109	.34	158	.07	180	H1607	DB1600	12	
	1.625	1.03	165	.57	257	.12	309	H1618	DB1610	10	
	2.000	1.73	264	.96	441	.22	551	H1627	DB1620	8	
	2.667	3.02	693	2.40	1124	.59	1512	H1638	QB620	6	
	3.000	3.82	745	2.34	1317	.57	1777	H1638	DB1630	6	
20	1.750	1.04	247	.53	355	.11	411	H1607	DB1601	12	
	2.125	1.59	381	.87	599	.19	714	H1618	DB1611	10	
	2.625	2.65	607	1.47	1015	.33	1276	H1627	DB1621	8	
	3.500	4.80	1174	2.94	2064	.72	2789	H1638	DB1631	6	
	1.333	.44	130	.23	189	.05	208	H1407	DB1400	12	
25	1.625	.67	196	.38	305	.09	366	H1418	DB1410	10	
	2.000	1.05	318	.63	525	.15	649	H1427	DB1320	8	
	2.167	1.39	441	.71	641	.15	756	H1607	DB1302	12	
	2.667	2.01	616	1.26	1071	.32	1450	H1438	DB620	6	
	2.625	2.11	672	1.16	1061	.25	1267	H1618	DB1612	10	
	3.250	3.54	1082	1.98	1806	.44	2270	H1627	DB1622	8	
	4.333	6.43	2094	3.94	3685	.96	4980	H1638	DB1632	6	
	3.000	2.39	882	1.50	1537	.38	2042	H1438	DB1430	6	
	2.583	1.72	683	.87	985	.18	1134	H1607	DB1603	12	
	3.125	2.81	1042	1.44	1641	.31	1961	H1618	DB1613	10	
3.875	4.40	1681	2.44	2810	.55	3466	H1627	DB1623	8		
30	1.750	.64	284	.33	410	.07	463	H1407	DB1401	12	
	2.125	.98	436	.55	678	.13	804	H1418	DB1411	10	
	2.625	1.54	699	.92	1150	.22	1428	H1427	DB1421	8	
	3.000	2.04	966	1.03	1402	.22	1617	H1607	QB1260	12	
	3.333	2.94	1355	1.84	2364	.47	3120	H1438	DB1431	6	
3.000	2.27	1039	1.33	2373	.41	4198	H1116	G11101	3		
35	1.333	.28	140	.15	210	.04	227	H1056	GB1050	12	
	1.625	.42	217	.25	336	.06	391	H1066	GB1060	10	
	2.000	.65	343	.41	567	.10	706	H1076	GB1070	8	
	2.167	.83	483	.43	693	.09	794	H1407	DB1402	12	
	2.667	1.22	695	.60	1156	.22	1550	H1086	GB1077	6	
40	2.625	1.25	742	.71	1156	.16	1374	H1418	DB1412	10	
	3.250	1.98	1191	1.18	1974	.28	2433	H1427	DB1422	8	
	3.833	2.61	1660	1.32	2395	.28	2773	H1607	DB1604	12	
	4.000	2.92	1667	1.99	3025	.64	4663	H1106	GB1100	4	
	4.333	3.77	2318	2.36	4034	.60	5420	H1438	DB1432	6	
	3.000	1.42	933	.93	1613	.26	2163	H1086	GB1080	6	
	6.000†	3.23	2218	1.81	4020	.53	7109	H1116	G11111	3	
	4.500	3.41	2336	2.32	4235	.75	6504	H1106	GB1101	4	
	45	2.583	.99	726	.52	1048	.11	1197	H1407	DB1403	12
		3.125	1.50	1112	.85	1730	.19	2048	H1418	DB1413	10
3.875		2.39	1794	1.43	2862	.34	3671	H1427	DB1423	8	
4.667		3.14	2495	1.59	3591	.33	4096	H1607	DB1605	12	
5.167		4.53	3475	2.84	6055	.72	8035	H1438	DB1433	6	
50	1.750	.40	294	.21	410	.05	473	H1056	GB1051	12	
	2.125	.59	452	.35	693	.09	831	H1066	DB1061	10	
	2.625	.90	725	.57	1197	.13	1286	H1076	GB1071	8	
	3.000	1.15	1008	.60	1450	.13	1663	H1407	DB1260	12	
	3.500	1.69	1386	1.12	2426	.31	3233	H1086	GB1081	6	
	3.625	1.74	1544	.98	2995	.22	2636	H1418	DB1414	10	
	4.500	2.75	2489	1.65	4128	.39	5105	H1427	DB860	8	
7.000†	4.23	3326	2.53	6002	.76	10683	H1116	G11122	3		
52	5.500	4.26	3909	2.92	7092	.94	10890	H1106	GB1102	4	
	4.000	1.95	1915	1.29	3366	.36	4470	H1086	GB1082	6	
	8.000†	3.87	3990	1.33	4130	.68	12816	H1116	G1113 T	3	

* Torque in Lb. Ins.

† Cast Iron Gear Rating with Hardened Worm shown.

All Worm and Worm Gear Ratings are based on a Hardened Steel Worm used with a Bronze Worm Gear.

1. For a Hardened Steel Worm used with a Cast Iron Gear, multiply the listed Rating by .50.

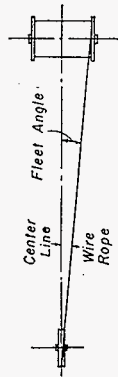
2. For an Unhardened Steel Worm used with a Cast Iron Gear, multiply the listed Rating by .25.

Table 7. Weight and Strengths of Standard Flat Wire Ropes, Not Preformed.

Wire Rope Flat Wire Rope	This rope consists of a number of strand rope units placed side by side and stitched together with soft steel sewing wire.			
	With Thickness, Inches	No. of Ropes	Approx. Weight per Ft. of Rope, Pounds	Breaking Strength, Tons of 2,000 Lbs.
1 1/2 x 1 1/2	7	6	6.69	46.8
1 1/2 x 2 1/2	11	1.35	26.5	21.6
1 1/2 x 3 1/2	13	3.3	31.3	27.2
1 1/2 x 4 1/2	5	0.77	18.5	16.0
1 1/2 x 5 1/2	7	1.55	35.8	28.8
1 1/2 x 6 1/2	11	1.61	40.5	35.3
1 1/2 x 7 1/2	13	2.19	47.9	48.1
1 1/2 x 8 1/2	4	2.71	58.3	48.1
1 1/2 x 9 1/2	6	1.43	31.4	36.3
1 1/2 x 10 1/2	8	1.84	47.1	40.9
1 1/2 x 11 1/2	9	2.43	67.2	54.6
1 1/2 x 12 1/2	14	2.83	73.2	63.7
1 1/2 x 13 1/2	15	3.43	88.6	77.3
1 1/2 x 14 1/2	18	3.63	94.1	81.9
1 1/2 x 15 1/2	6	2.13	54.5	47.4
1 1/2 x 16 1/2	7	2.47	63.6	55.4
1 1/2 x 17 1/2	8	2.82	72.7	63.3
1 1/2 x 18 1/2	9	3.17	81.8	71.2
1 1/2 x 19 1/2	10	3.52	90.9	79.1
1 1/2 x 20 1/2	11	3.87	100.	87.1
1 1/2 x 21 1/2	12	4.22	109.	95.1
1 1/2 x 22 1/2	13	4.57	118.	103.
1 1/2 x 23 1/2	14	4.92	127.	111.
1 1/2 x 24 1/2	15	5.27	136.	119.
1 1/2 x 25 1/2	16	5.62	145.	127.
1 1/2 x 26 1/2	17	5.97	154.	135.
1 1/2 x 27 1/2	18	6.32	163.	143.
1 1/2 x 28 1/2	19	6.67	172.	151.
1 1/2 x 29 1/2	20	7.02	181.	159.
1 1/2 x 30 1/2	21	7.37	190.	167.
1 1/2 x 31 1/2	22	7.72	199.	175.
1 1/2 x 32 1/2	23	8.07	208.	183.
1 1/2 x 33 1/2	24	8.42	217.	191.
1 1/2 x 34 1/2	25	8.77	226.	199.
1 1/2 x 35 1/2	26	9.12	235.	207.
1 1/2 x 36 1/2	27	9.47	244.	215.
1 1/2 x 37 1/2	28	9.82	253.	223.
1 1/2 x 38 1/2	29	10.17	262.	231.
1 1/2 x 39 1/2	30	10.52	271.	239.

Source: Rope diagram, Bethlehem Steel Co.; all data, U. S. Simplified Practice Recommendation 198-50.

When the rope is underdressed on the drum, a right lay rope should be started from the left end and a left lay rope from the right end. When this is done, the rope will spool evenly and the turns will lie snugly together.



Sheaves and drums should be properly aligned to prevent undue wear. The proper position of the main or lead sheave for the rope as it comes off the drum is governed by what is called the fleece angle or angle between the rope as it stretches from drum

to sheave and an imaginary center-line passing through the center of the sheave groove and a point halfway between the ends of the drum. When the rope is at the drum, this angle should not exceed one and a half to two degrees. With the lead sheave, the angle should not exceed one and a half to two degrees. With the lead sheave, the angle should not exceed one and a half to two degrees.

Sheave and Drum Dimensions: Sheaves and drums should be as large as possible to obtain maximum rope life. However, factors such as the need for lightweight equipment with consequent sacrifice in rope life in the interest of overall economy. No hard and fast rule can be laid down for any particular rope life if the utmost in economical performance is to be obtained. Where maximum rope life is of prime importance, sheave or drum dimensions of Federal Specification RLK-R-5714 for minimum sheave or drum diameters are recommended. $D = 2d$; for 6 X 19 rope, $D = 45d$; for 6 X 25 ropes, $D = 45d$; for 6 X 30 ropes, $D = 30d$; for 6 X 37 ropes, $D = 27d$; and for 8 X 19 ropes, $D = 31d$.

Too small a groove for the rope is to carry its weight, proper seating of the rope in the bottom of the groove and consequently uneven distribution of load on the rope will result. Too large a groove will not give the rope sufficient side support.

Larger specifications RLK-R-5714 recommend that sheave groove diameters be larger than the rope diameters. For ropes of 18 to 36-inch diameters, the groove diameter for 36- to 48-inch ropes of 18- to 36-inch diameters, 36-inch larger; for 36- to 55-inch ropes, 48-inch larger; for 55-inch ropes, 36-inch larger; for 36- to 55-inch ropes, 48-inch larger; for 55-inch ropes, 36-inch larger; and for 55-inch and larger diameter ropes, 36-inch larger. For new or regrooved sheaves these values should be doubled; in other words for 36- to 55-inch diameter ropes, the groove diameter should be 72-inch larger, etc.

Drum or Reel Capacity: The length of wire rope, in feet, that can be spooled onto a drum or reel, is computed by the following formula, where

D = drum or reel diameter, in feet, that can be spooled onto a drum or reel, is computed by the following formula, where

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Table 8. Factors K Used in Calculating Wire Rope Drum and Reel Capacities

Rope Diam., In.	Factor K	Rope Diam., In.	Factor K	Rope Diam., In.	Factor K
3/8	23.4	3/4	0.925	1 1/2	0.127
1/2	23.4	5/8	0.741	1 3/4	0.160
5/8	8.8	3/4	0.688	1 3/4	0.688
3/4	8.72	1 1/8	0.596	1 3/4	0.688
5/8	6.14	3/4	0.438	1 3/4	0.688
3/4	4.59	1 1/8	0.354	1 3/4	0.688
3/4	3.29	3/4	0.368	1 3/4	0.688
3/4	2.21	1	0.239	1 3/4	0.688
3/4	1.58	1 1/4	0.191	1 3/4	0.688
3/4	1.19	1 1/2	0.152	1 3/4	0.688

Note: The values of "K" allow for normal oversize of ropes, and the fact that it is practically impossible to "thread-wind" ropes of small diameter. However, the formula is based on uniform rope winding and will not give correct figures if rope is wound in a zig-zag fashion. The formula is based on the same number of rope layers on each layer, which is not strictly correct, but which does not result in appreciable error unless the width (B) of the reel is quite small compared with the flange diameter (H).

STEEL-HARDENED, GROUND AND POLISHED WORMS BRONZE WORM GEARS

APPROXIMATE HORSEPOWER AND TORQUE * RATINGS
FOR CLASS I SERVICE (Service Factor = 1.0)

Worm RPM	1800		600		100		Worm Cat. No.	Gear Cat. No.	DP	
	Center Distance	Input HP	Output Torque	Input HP	Output Torque	Input HP				Output Torque
40	2.167	.48	490	.26	672	.06	782	H1056	GB1052	12
	2.625	.72	742	.43	1134	.11	1361	H1066	GB1062	10
	3.250	1.12	1190	.71	1974	.18	2420	H1076	GB1072	8
	3.833	1.43	1680	.74	2395	.16	2748	H1047	DB1404	12
	4.333	2.11	2310	1.39	4034	.38	5345	H1086	GB1078	6
45	4.625	2.16	2562	1.22	3991	.28	4790	H1418	DB1080	10
	5.750	3.42	4130	2.05	6807	.48	8319	H1427	DB1424	8
	3.750	1.26	1614	.79	2622	.20	3267	H1076	GB1073	8
	5.000	2.37	3110	1.56	5445	.43	7260	H1086	GB1083	6
	10.000	4.06	5320	1.68	6608	.72	17088	H1116	G11147	3
50	2.583	.55	700	.30	998	.07	1134	H1056	GB1053	12
	3.125	.83	1068	.51	1733	.12	1954	H1066	GB1063	10
	3.875	1.30	1716	.82	2836	.21	3498	H1076	GB850	8
54	5.167	2.43	3327	1.80	5777	.44	7563	H1086	GB1079	6
	11.000	4.34	5985	1.79	7434	.77	19224	H1116	G11151	3
	3.330	.60	924	.33	1323	.08	1664	H1056	GB1260	12
60	3.825	.91	1408	.54	2142	.13	2571	H1066	GB1064	10
	4.500	1.42	2263	.89	3718	.23	4538	H1076	GB860	8
	6.000	2.66	4370	1.75	7625	.49	10210	H1086	GB1087	6
72	7.000	2.79	5521	1.84	9605	.51	12705	H1086	GB1084	6
	3.833	.64	1288	.35	1843	.08	2118	H1056	GB1054	12
	4.625	.96	1961	.57	3042	.14	3830	H1066	GB1067	10
90	5.750	1.49	3165	.94	5210	.24	6555	H1076	GB1074	8
	4.667	.60	1505	.33	2206	.08	2458	H1056	GB1055	12
100	5.625	.90	2310	.54	3571	.13	4223	H1066	GB1065	10
	7.000	1.40	3711	.88	6092	.22	7563	H1076	GB8100	8

* Torque in Lb. Ins.

† Cast Iron Gear Rating with Hardened Worm shown.

All Worm and Worm Gear Ratings are based on a Hardened Steel Worm used with a Bronze Worm Gear.

1. For a Hardened Steel Worm used with a Cast Iron Gear, multiply the listed Rating by .50.

2. For an Unhardened Steel Worm used with a Cast Iron Gear, multiply the listed Rating by .25.

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CENTER DISTANCES AND RATIOS AVAILABLE WITH STOCK WORM GEARING

Center Distance (Inches)	Pitch	No. of Teeth in Gear	Worm Thread			Center Distance (Inches)	Pitch	No. of Teeth in Gear	Worm Thread			Center Distance (Inches)	Pitch	No. of Teeth in Gear	Worm Thread		
			Single	Double	Quad				Ratio	Single	Double				Quad	Ratio	Single
48	20	20	10	5	7.5	48	20	100	100	50	25	48	20	40	20	10	15
48	30	30	15	7.5	5	48	30	50	50	25	12.5	48	30	60	30	15	10
32	20	20	10	5	7.5	32	20	80	80	40	20	32	20	24	—	—	—
40	20	40	20	10	5	40	20	80	20	10	5	40	20	80	40	20	20
24	20	20	10	5	7.5	24	20	40	20	10	5	24	20	100	—	—	—
48	50	50	25	12.5	7.5	48	50	60	60	30	15	48	50	30	18	—	—
32	30	30	15	7.5	5	32	30	60	30	15	7.5	32	30	18	—	—	—
48	60	60	30	15	7.5	48	60	96	96	48	24	48	60	50	25	12.5	—
32	40	40	20	10	5	32	40	100	100	50	25	32	40	32	—	—	—
24	30	30	15	7.5	5	24	30	50	50	25	12.5	24	30	100	50	25	20
12	20	20	10	5	7.5	12	20	40	20	10	5	12	20	80	40	20	15
48	80	80	40	20	10	48	80	20	30	15	7.5	48	80	40	20	—	—
32	50	50	25	12.5	7.5	32	50	60	60	30	15	32	50	60	30	15	—
24	40	40	20	10	5	24	40	80	40	20	5	24	40	24	—	—	—
48	100	100	50	25	12.5	48	100	120	120	60	30	48	100	40	—	—	—
16	30	30	15	7.5	5	16	30	60	30	15	7.5	16	30	96	96	—	—
24	50	50	25	12.5	7.5	24	50	50	50	25	12.5	24	50	100	50	25	18
12	20	20	10	5	7.5	12	20	40	20	10	5	12	20	30	30	—	—
32	80	80	40	20	10	32	80	100	100	50	25	32	80	48	48	—	—
24	60	60	30	15	7.5	24	60	60	30	15	7.5	24	60	80	40	20	—
16	40	40	20	10	5	16	40	48	48	—	—	16	40	36	—	—	—
10	20	20	10	5	7.5	10	20	80	40	20	5	10	20	100	50	25	—
32	96	96	48	24	12.5	32	96	50	50	25	12.5	32	96	64	—	—	—
24	72	72	36	18	7.5	24	72	36	36	—	—	24	72	48	—	—	—
12	30	30	15	7.5	5	12	30	20	20	—	—	12	30	54	54	—	—

Example: Given a center distance of 2.625", Table lists Worm and Worm Gear Ratios available:

10 Pitch, 40 tooth, single = 40 to 1

10 Pitch, 40 tooth, double = 20 to 1

10 Pitch, 40 tooth, quad = 10 to 1

8 pitch, 30 tooth, single = 30 to 1

8 pitch, 30 tooth, double = 15 to 1

8 pitch, 30 tooth, quad = 7.5 to 1

SPUR GEARS

LEWIS FORMULA (Barth Revision)

Gear failure can occur due to tooth breakage (tooth stress) or surface failure (surface durability) as a result of fatigue and wear. Strength is determined in terms of tooth-beam stresses for static and dynamic conditions, following well established formula and procedures. Satisfactory results may be obtained by the use of Barth's Revision to the Lewis Formula, which considers beam strength but not wear. The formula is satisfactory for commercial gears at Pitch Circle velocities of up to 1500 FPM. It is this formula that is the basis for all Boston Spur Gear ratings.

METALLIC SPUR GEARS

- $W = \frac{SFY}{P} \left(\frac{600}{600 + V} \right)$
 W = Tooth Load, Lbs. (along the Pitch Line)
 S = Safe Material Stress (static) Lbs. per Sq. In. (Table II)
 F = Face Width, In.
 Y = Tooth Form Factor (Table I)
 P = Diametral Pitch
 D = Pitch Diameter
 V = Pitch Line Velocity, Ft. per Min. = .262 x PD x RPM

For NON-METALLIC GEARS, the modified Lewis Formula shown below may be used with (S) values of 6000 PSI for Phenolic Laminated material.

$$W = \frac{SFY}{P} \left(\frac{150}{200 + V} + .25 \right)$$

TABLE II—VALUES OF SAFE STATIC STRESS (s)

Material	(s) Lb. per Sq. In.	
Bronze	10000	
Cast Iron	12000	
Steel	.20 Carbon (Untreated)	20000
	.20 Carbon (Case-hardened)	25000
	.40 Carbon (Untreated)	25000
	.40 Carbon (Heat-treated)	30000
.40 C. Alloy (Heat-treated)	40000	

Max. allowable torque (T) that should be imposed on a gear will be the safe tooth load (W) multiplied by $\frac{D}{2}$ or $T = \frac{W \times D}{2}$

The safe horsepower capacity of the gear (at a given RPM) can be calculated from $HP = \frac{T \times RPM}{63,025}$ or directly from (W) and (V);

$$HP = \frac{WV}{33,000}$$

For a known HP, $T = \frac{63025 \times HP}{RPM}$

TABLE I Y FACTORS

Number of Teeth	14½° Full Depth Involute	20° Full Depth Involute
10	0.176	0.201
11	0.192	0.226
12	0.210	0.245
13	0.223	0.264
14	0.236	0.276
15	0.245	0.289
16	0.255	0.295
17	0.264	0.302
18	0.270	0.308
19	0.277	0.314
20	0.283	0.320
22	0.292	0.330
24	0.302	0.337
26	0.308	0.344
28	0.314	0.352
30	0.318	0.358
32	0.322	0.364
34	0.325	0.370
36	0.329	0.377
38	0.332	0.383
40	0.336	0.389
45	0.340	0.399
50	0.346	0.408
55	0.352	0.415
60	0.355	0.421
65	0.358	0.425
70	0.360	0.429
75	0.361	0.433
80	0.363	0.436
90	0.366	0.442
100	0.368	0.446
150	0.375	0.458
200	0.378	0.463
300	0.382	0.471
Rack	0.390	0.484

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SPUR GEARS

BACKLASH

Stock spur gears are cut to operate at standard center distances. The standard center distance being defined by:

$$\text{Standard Center Distance} = \frac{\text{Pinion PD} + \text{Gear PD}}{2}$$

When mounted at this center distance, stock spur gears will have the following average backlash:

Diametral Pitch	Backlash (Inches)	Diametral Pitch	Backlash (Inches)
3	.013	8-9	.005
4	.010	10-13	.004
5	.008	14-32	.003
6	.007	33-64	.0025
7	.006		

An increase or decrease in center distance will cause an increase or decrease in backlash.

Since, in practice, some deviation from the theoretical standard center distance is inevitable and will alter the backlash, such deviation should be as small as possible. For most applications, it would be acceptable to limit the deviation to an increase over the nominal center distance of one half the average backlash. Varying the center distance may afford a practical means of varying the backlash to a limited extent.

The approximate relationship between center distance and backlash change of $14\frac{1}{2}^\circ$ and 20° pressure angle gears is shown below:

For $14\frac{1}{2}^\circ$ — Change in Center Distance = $1.933 \times$ Change in Backlash
For 20° — Change in Center Distance = $1.374 \times$ Change in Backlash

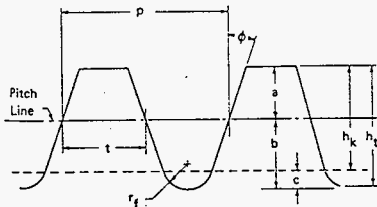
From this, it is apparent that a given change in center distance, $14\frac{1}{2}^\circ$ gears will have a smaller change in backlash than 20° gears. This fact should be considered in cases where backlash is critical.

UNDERCUT

When the number of teeth in a gear is small, the tip of the mating gear tooth may interfere with the lower portion of the tooth profile. To prevent this, the generating process removes material at this point. This results in loss of a portion of the involute adjacent to the tooth base, reducing tooth contact and tooth strength.

On $14\frac{1}{2}^\circ$ PA gears undercutting occurs where number of teeth is less than 32 and for 20° PA less than 18. Since this condition becomes more severe as tooth numbers decrease, it is recommended that the minimum number of teeth be 16 for $14\frac{1}{2}^\circ$ PA and 13 for 20° PA.

In a similar manner INTERNAL Spur Gear teeth may interfere when the pinion gear is too near the size of its mating internal gear. The following may be used as a guide to assure proper operation of the gear set. For $14\frac{1}{2}^\circ$ PA, the difference in tooth numbers between the gear and pinion should not be less than 15. For 20° PA, the difference in tooth numbers should not be less than 12.



- a = Addendum
- b = Dedendum
- c = Clearance
- h_k = Working Depth
- h_t = Whole Depth
- p = Circular Pitch
- r = Fillet Radius
- t = Circular Tooth Thickness
- ϕ = Pressure Angle

SPUR GEAR FORMULAS

FOR FULL DEPTH INVOLUTE TEETH

To Obtain	Having	Formula
Diametral Pitch (P)	Circular Pitch (p)	$P = \frac{3.1416}{p}$
	Number of Teeth (N) & Pitch Diameter (D)	$P = \frac{N}{D}$
	Number of Teeth (N) & Outside Diameter (D _o)	$P = \frac{N + 2}{D_o}$ (Approximate)
Circular Pitch (p)	Diametral Pitch (P)	$p = \frac{3.1416}{P}$
Pitch Diameter (D)	Number of Teeth (N) & Diametral Pitch (P)	$D = \frac{N}{P}$
	Outside Diameter (D _o) & Diametral Pitch (P)	$D = D_o - \frac{2}{P}$
Base Diameter (D _b)	Pitch Diameter And Pressure Angle	$D_b = D \cos \phi$
Number of Teeth (N)	Diametral Pitch (P) & Pitch Diameter (D)	$N = P \times D$
Tooth Thickness (t) @ Pitch Diameter (D)	Diametral Pitch (P)	$t = \frac{1.5708}{P}$
Addendum (a)	Diametral Pitch (P)	$a = \frac{1}{P}$
Outside Diameter (D _o)	Pitch Diameter (D) & Addendum (a)	$D_o = D + 2a$
Whole Depth (h _t) (20P & Finer)	Diametral Pitch (P)	$h_t = \frac{2.2}{P} + .002$
Whole Depth (h _t) (Coarser than 20P)	Diametral Pitch (P)	$h_t = \frac{2.157}{P}$
Working Depth (h _k)	Addendum	$h_k = 2(a)$
Clearance (c)	Whole Depth (h _t) & Addendum (a)	$c = h_t - 2a$
Dedendum (b)	Whole Depth (h _t) & Addendum (a)	$b = h_t - a$
Contact Ratio (M _c)	Outside Radii, Base Radii, Center Distance and Pressure Angle	$M_c = \frac{\sqrt{R_o^2 - R_b^2} + \sqrt{r_o^2 - r_b^2} - C \sin \phi}{P_c \cos \phi}$
Root Diameter (D _r)	Pitch Diameter and Dedendum	$D_r = D - 2b$
Center Distance (C)	Pitch Diameter or No. of Teeth and Pitch	$C = \frac{D_1 + D_2}{2}$ or $\frac{N_1 + N_2}{2P}$

- R_o = Outside Radius, Gear
- r_o = Outside Radius, Pinion
- R_b = Base Circle Radius, Gear
- r_b = Base Circle Radius, Pinion

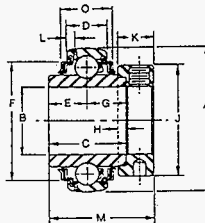
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WIDE INNER RING BEARINGS

N-KRRB Heavy Series Relubricatable Type

The heavy series R-Seal wide inner ring bearings are similar to the standard series described on page 141 but are capable of withstanding continuous, heavy or shock loads. The GN-KRRB series utilizes a heavier outer ring than the standard series, as well as, a considerably thicker sealing member in the contact-type diaphragm seal. This design assures complete retention of the lubricant and positive exclusion of all contaminants. These bearings are especially effective at slow-to-moderate speeds under the severest conditions of dirt and corrosion.



Recommended shaft tolerances: $1/32$ "- $2/32$ " \pm , nominal to -0.005 ", -0.13 mm;
 $2/32$ "- $3/32$ " \pm , nominal to -0.001 ", -0.25 mm.

ORDER, SPECIFY BEARING NUMBER FOLLOWED BY "AND COLLAR." Example: GN303KRRB and Collar.

Bearing Number	Collar Number	Basic Outer Ring Size	Bore ^B	O.D.		Ring Widths		E	G	H	J	K	L	M	F	O	Brg. & Collar Wt.	Static Load Rating C ₀	Extended Dynamic Load Rating C _e		
				A	A	Inner	Outer														
				in.	in.	mm.	mm.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	lbs.
1103KRRB	SN103K	G306	1 1/16	2.8345 72	1 7/16 36.51	.787 20	1 1/16 15.5	3/4 19.1	3/32 4.0	1 13/16 45.2	1 1/16 17.5	.142 3.61	1 1/32 50.0	2.369 60.17	.925 23.50	1.22 29.50	553	1.22 15600	7500 33500		
1104KRRB	SN104K		1 1/4														1.68 4500	762 9150			
1105KRRB	SN105K		1 5/16	3.1496	1 1/2 38.10	.866 22	2 1/32 18.3	2 1/32 19.6	3/32 4.0	2 3/16 51.6	1 1/16 17.5	.156 3.96	2 1/32 51.6	2.638 67.01	1.063 27.00	1.64 36.50	1.64 20000	7144 2726	9150 46500		
1106KRRB	SN106K	G307	1 3/8														1.56 15600	708 31500			
1107KRRB	SN107K		1 7/16														1.56 15600	708 31500			
1108KRRB	SN108K	G308	1 1/2	3.5433 90	1 5/8 41.28	.984 25	2 1/32 15.6	2 1/32 21.4	3/16 4.8	2 1/2 63.5	1 3/16 25.6	.182 4.62	2 1/4 51.2	2.525 75.06	1.650 26.67	2.54 61.00	1.52 15200	5500 24500	11000 49000		
1110KRRB	SN110K		1 5/8	3.6570	1 11/16 42.85	1.063 27	2 1/32 19.8	2 1/32 23.0	3/16 4.8	2 3/4 69.9	1 3/16 20.6	.197 5.00	2 1/16 53.7	3.251 82.58	1.123 28.52	3.65 2.95	1.656 1388	6700 30000	13200 59300		
1111KRRB	SN111K	G309	1 11/16														3.21 7100	1.456 66000			
1112KRRB ^Q	SN112K		1 5/8														3.21 7100	1.456 66000			
1114KRRB	SN114K	G310	1 7/8	4.3307 110	1 11/16 42.21	1.142 29	2 1/32 24.6	2 1/32 24.5	3/16 4.8	2 9/16 75.8	1 3/8 22.2	.211 5.36	2 3/8 56.7	3.654 82.87	1.215 30.86	4.35 4.20	1.973 1.955	8000 35500	15300 68000		
1115KRRB	SN115K		1 15/16														4.20 9300	1.955 88000			
1200KRRB	SN200K	G311	2	4.7244 120	2 1/16 55.36	1.220 31	1 1/32 27.8	1 1/32 27.6	3/16 4.8	3 1/4 82.6	1 3/4 22.2	.216 5.49	2 1/4 70.0	4.007 101.76	1.475 37.47	4.70 5.22	2.132 2.398	9300 41500	18000 83000		
1203KRRB	SN203K		2 1/8														6.26 13800	2.839 126000			
1207KRRB	SN207K	G312	2 7/16	5.1181 130	2 7/16 61.91	1.299 33	1 1/32 31.0	1 1/32 31.0	3/8 9.4	3 7/8 88.9	1 3/4 23.8	.230 5.84	3 1/8 77.4	4.312 109.52	1.535 36.99	6.26 13800	2.839 126000	10800 49000	23400 105000		
1211KRRB	SD211K	G314	2 11/16	5.9055 150	2 11/16 66.26	1.457 37	1 1/32 34.1	1 1/32 34.1	1/2 12.7	4 101.6	1 1/4 27.0	.255 6.43	3 1/2 88.9	4.973 126.31	1.770 44.96	9.54 212.00	4.509 10000	14300 63000	26000 115000		
1215KRRB	SN215K	G315	2 15/16	6.2992 160	2 15/16 74.61	1.535 39	1 1/32 37.3	1 1/32 37.3	1/2 12.7	4 101.6	1 1/4 27.0	.255 6.43	3 1/8 95.0	5.273 133.02	2.013 51.13	12.42 276.00	5.634 12500	16000 71000	28500 125000		
1303KRRB	SN303K	G316	3 1/16	6.6929 170	3 1/16 80.96	1.614 41	1 1/32 45.5	1 1/32 45.5	1/2 12.7	4 1/16 104.4	1 1/4 31.8	.266 6.76	4 1/16 106.4	5.823 142.82	2.040 51.05	15.71 348.00	7.126 15800	18000 80000	35500 157000		
1307KRRB	SN307K	G318	3 7/16	7.4803 190	3 7/16 87.31	1.772 45	1 1/32 42.1	1 1/32 42.1	3/8 9.4	5 1/4 133.4	1 3/4 36.5	.302 7.68	4 3/16 111.9	6.353 161.57	2.072 52.63	20.26 453.00	9.190 20800	24400 98000	35500 155000		
1315KRRB	SN315K	G320	3 5/16	8.4846 215	3 5/16 100.01	1.939 49	1 1/32 50.0	1 1/32 50.0	1/2 12.7	5 1/4 136.1	1 3/4 36.5	.308 7.82	5 1/16 125.6	7.199 182.85	2.337 59.36	26.97 600.00	12.233 27000	22000 132000	43000 193000		

Q: tolerances: $1/32$ "- $2/32$ " \pm , nominal to -0.005 ", -0.13 mm; $2/32$ "- $3/32$ " \pm , nominal to -0.006 ", -0.15 mm.
B: tolerances: $1/32$ "- $3/32$ " \pm , nominal to -0.007 ", -0.18 mm.

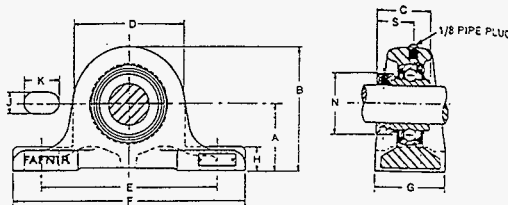
ISO available with cylindrical O.D. Delete suffix "B" Example: GN106KRR



RSAO Heavy Series

Fafnir RSAO units are ideally suited for installations where the load is heavy in proportion to the shaft diameter or where considerable shock loads exist. Incorporating a high capacity GN-KRRB (R-Seal) wide inner ring bearing, these units have the same base-to-center height and bolt hole spacing as Fafnir heavy series SAO pillow blocks.

These pillow blocks are prelubricated and ready for immediate use. For wet or extremely dirty conditions, a grease fitting installed in place of the standard 1/8 pipe plug will provide means of relubrication through a groove in the inside surface of the housing. All units are supplied with self-locking collars.



Recommended shaft tolerances: 1 1/4" - 1 1/2" shaft, nominal to -.0005", -.013 mm
2" - 3 1/2" shaft, nominal to -.0010", -.025 mm.

Bearing Data

Unit	Bearing Number	Dimensions and Load Ratings
RSAO	GN...KRRB	Page 144

TO ORDER, SPECIFY UNIT AND SHAFT DIAMETER. Example: RSAO 1 1/2" x F.

Unit	Shaft Diam.	A	B	C	D	E	F	G	H	J	K	N	S	Bolt Size	Bearing Number	Collar Number	Housing Number	Unit Wt.
	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.				lbs.
		mm	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm				kg
RSAO	1 1/4	2 1/4	4 1/4	1 3/4	2 3/4	5 1/4	5 1/4	2 3/4	7/8	5/8	1	1 1/4	1 1/2	1/2	GN101KRRB	SK101K	T-2267B	6.47
RSAO	1 1/4	85.23	108.0	55.0	69.2	135.3	135.3	69.2	22.2	15.9	25.4	45.2	38.5	12.7	GN102KRRB GN103KRRB	SK102K SK103K	T-2267B T-2267B	2.537
RSAO	1 1/2														GN104KRRB	SK104K	T-2249B	
RSAO	1 1/2	2 3/4	4 3/4	2 1/2	4 1/4	8 1/4	10 1/4	2 3/4	1 3/4	3/4	1 1/4	2 1/4	1 1/4	3/4	GN105KRRB	SK105K	T-2249B	9.15
RSAO	1 1/2	65.85	122.2	51.5	104.8	209.6	265.9	69.8	22.8	19.0	28.6	55.9	38.3	19.9	GN106KRRB GN107KRRB	SK106K SK107K	T-2249B T-2249B	4.154
RSAO	1 1/2														GN108KRRB	SK108K	T-2267E	12.80
RSAO	1 1/2	3 1/8	5 3/8	2 5/8	4 1/2	9	11 3/4	3 1/4	1 3/4	3/4	1 1/4	2 1/2	1 3/4	3/4	GN109KRRB	SK109K	T-2267E	1.637
RSAO	1 1/2	75.38	138.6	57.2	114.3	228.6	288.9	79.4	27.0	19.0	28.6	65.5	37.3	19.9				
RSAO	1 5/8														GN110KRRB	SK110K	T-2249B	
RSAO	1 5/8	3 3/4	5 3/4	2 5/8	5	9	11 3/4	3 3/4	1 3/4	3/4	1 1/4	2 3/4	1 3/4	3/4	GN111KRRB	SK111K	T-2249B	14.45
RSAO	1 5/8	75.38	142.9	58.7	127.0	228.6	288.9	79.4	33.2	19.0	28.6	65.6	38.9	19.9	GN112KRRB	SK112K	T-2249B	8.560
RSAO	1 3/4														GN113KRRB	SK113K	T-2260E	
RSAO	1 3/4	3 3/4	5 7/8	2 3/4	5 7/8	9	11 3/4	3 3/4	1 3/4	3/4	1 1/4	3	1 3/4	3/4	GN114KRRB	SK114K	T-2260E	15.96
RSAO	1 3/4	75.38	145.4	58.7	138.1	228.6	288.9	79.4	33.3	19.0	28.6	76.2	42.1	19.9	GN115KRRB	SK115K	T-2260E	7.245
RSAO	2														GN200KRRB	SK200K	T-2260B	
RSAO	2	3 3/4	6 3/4	2 3/4	5 1/4	10 1/4	12 1/4	3 3/4	1 3/4	3/4	1 3/4	3 1/4	1 3/4	3/4	GN201KRRB	SK201K	T-2260B	22.45
RSAO	2 1/4	85.25	170.7	73.0	135.8	265.4	320.7	81.9	35.5	22.2	34.9	62.6	45.2	19.0	GN202KRRB	SK202K	T-2260B	11.192
RSAO	2 1/4														GN203KRRB	SK203K	T-2260B	
RSAO	2 1/4														GN204KRRB	SK204K	T-2245A	
RSAO	2 1/4	4 1/4	7 1/4	3 1/4	6 1/4	11 1/4	13 1/4	4	1 1/2	7/8	1 3/4	3 1/2	1 3/4	3/4	GN205KRRB	SK205K	T-2245A	33.56
RSAO	2 1/4	104.78	185.5	75.4	163.5	285.8	345.2	101.6	38.1	22.2	34.9	85.9	45.4	19.0	GN206KRRB	SK206K	T-2245A	15.144
RSAO	2 1/4														GN207KRRB	SK207K	T-2245A	
RSAO	2 3/4														GN210KRRB	SK210K	T-2245E	
RSAO	2 3/4	4 3/4	8 1/4	3 1/2	7 1/4	12	15 1/4	4 3/4	1 3/4	1	1 3/4	4	2 3/4	7/8	GN211KRRB	SK211K	T-2245E	42.50
RSAO	2 3/4	115.89	210.3	85.9	181.9	304.8	395.5	111.1	44.4	25.4	34.9	101.6	54.8	22.2	GN212KRRB	SK212K	T-2245E	15.295
RSAO	2 3/4														GN213KRRB	SK213K	T-2249B	
RSAO	2 3/4	4 3/4	8 1/4	3 1/4	8	12 3/4	15 1/4	4 3/4	1 3/4	1	1 3/4	4 1/4	2 1/4	7/8	GN214KRRB	SK214K	T-2249B	44.25
RSAO	2 3/4	115.89	217.5	100.0	203.2	314.3	395.5	111.1	47.6	25.4	34.9	117.7	62.7	22.2	GN215KRRB	SK215K	T-2249B	20.090
RSAO	3 1/4														GN300KRRB	SK300K	T-2244A	
RSAO	3 1/4	5 1/4	9 1/4	4 1/4	8 1/4	12 3/4	15 1/4	4 3/4	1 3/4	1	1 3/4	4 1/4	2 1/4	7/8				50.25
RSAO	3 1/4	135.18	235.8	115.5	211.3	325.7	425.6	125.6	57.2	28.6	54.0	133.4	72.2	25.4	GN307KRRB	SK307K	T-2244E	68.25
RSAO	3 1/4																	22.124
RSAO	3 1/4	5 1/4	11 1/4	5 1/4	10 3/4	14 3/4	17 1/4	5 1/4	2 1/4	1 1/4	1 3/4	5 1/4	3 3/4	1	GN315KRRB	SK315K	T-2244E	85.50
RSAO	3 1/4	144.45	281.0	128.6	273.0	374.5	438.7	132.2	65.1	28.6	44.4	145.0	76.6	25.4				41.633

RCJO Heavy Series

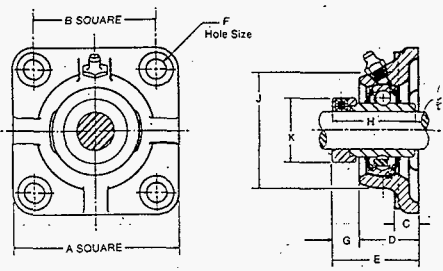
RCJO type heavy series flange cartridges are similar in design to the standard series. They are for applications where a minimum amount of machining is to be done and are supplied completely assembled, being mounted in place by means of bolts through the flange.

RCJO units are equipped with GN-KRRB (R-Seal) wide inner ring bearings.

A grease fitting provides for relubrication if required. Units are supplied with self-locking collars and are dimensionally interchangeable.

Recommended shaft tolerances: 1 1/4" - 1 1/2", nominal to -0.005", -0.03 mm;
2" - 3 1/4", nominal to -0.010", -0.25 mm.

Bearing Data		
nit	Bearing Number	Dimensions and Load Ratings
CJO	GN...KRRB	Page 144



ON ORDER, SPECIFY UNIT AND SHAFT DIAMETER. Example: RCJO 1 1/2".

Unit	Shaft Diam.	A	B	C	D	E	F	G	H	J	K	Bolt (4 req'd)	Bearing Number	Collar Number	Housing Number	Unit WL
	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.				lbs.
	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm	mm				kg
RCJO	1 1/4"	4 3/4	3 1/2	9/16	1 1/2	2 1/32	9/16	1 1/2	1 1/2	5 19/32	1 19/16	1/2	GN103KRRB	SN103K	T-19165	4.00
		120.6	92.08	14.3	38.1	53.2	14.3	15.1	50.0	50.0	96.8	45.2				1.816
RCJO	1 1/2"	5 1/2	4	5/8	1 11/32	2 1/32	9/16	1 1/2	1 1/2	4 1/4	2 1/4	1/2	GN107KRRB	SN107K	T-19167	5.50
		138.2	101.60	15.9	45.5	54.8	14.3	14.3	51.5	104.8	58.7					2.497
RCJO	1 1/2"	5 3/8	4 1/8	5/8	1 3/4	2 3/8	9/8	1 1/2	1 1/2	4 1/2	2 1/2	3/8	GN108KRRB	SN108K	T-19169	6.50
		136.5	104.78	15.9	44.4	60.3	15.9	15.9	51.2	114.3	63.5					3.323
RCJO	1 11/16"	5 5/8	4 3/8	11/16	1 7/32	2 7/16	5/8	1 1/2	1 1/2	4 7/8	2 3/4	3/8	GN111KRRB	SN111K	T-19171	7.87
		142.9	111.12	17.5	43.2	61.9	15.9	15.1	58.7	123.8	66.8					3.573
RCJO	1 5/8"	6 1/2	5 1/8	11/16	2 1/32	2 3/4	11/16	1 1/2	1 1/2	5 1/8	3	3/8	GN115KRRB	SN115K	T-19173	11.42
		165.1	130.18	17.5	53.2	69.9	17.5	15.7	66.7	141.3	76.2					5.185
RCJO	2"	7	5 5/8	11/16	2 5/16	3	11/16	1 1/2	1 1/2	6 1/8	3 3/4	3/8	GN203KRRB	SN203K	T-19175	14.15
		177.8	142.88	17.5	58.7	76.2	17.5	17.5	73.0	154.0	92.6					6.424
RCJO	2 1/8"	7 1/2	5 7/8	3/4	2 1/8	3 1/8	11/16	1 1/2	1 1/2	6 3/8	3 1/2	3/8	GN207KRRB	SN207K	T-19177	16.32
		190.5	145.22	19.0	65.1	84.1	20.8	17.9	75.4	150.3	86.5					7.439
RCJO	2 11/16"	8 1/8	7	3/4	2 7/32	3 11/16	11/16	1 1/2	1 1/2	7 1/16	4	3/8	GN211KRRB	SN211K	T-19179	21.00
		205.4	177.80	22.2	72.2	93.7	25.8	21.4	88.9	185.7	101.6					9.534
RCJO	2 15/16"	8 1/4	7 1/4	7/8	3 1/16	4 1/8	11/16	1 1/2	1 1/2	7 19/32	4 1/8	3/8	GN215KRRB	SN215K	T-19181	31.12
		231.8	184.15	22.2	77.8	104.8	23.8	22.0	100.0	195.4	112.7					14.128
RCJO	3"	11	8 1/2	1 1/8	3 1/8	4 3/4	1 1/4	1 1/2	1 1/2	4 1/8	9	1	GN307KRRB	SN307K	T-24425	47.30
		279.4	215.90	26.6	64.1	120.6	27.0	36.5	115.9	226.6	132.2					21.474
RCJO	3 1/4"	12 1/2	9 1/2	1 1/4	3 13/16	5 1/4	1 1/4	1 1/2	1 1/2	5 1/2	9 1/4	1 1/8	GN315KRRB	SN315K	T-24477	67.50
		317.5	241.30	31.8	95.8	132.4	36.2	36.5	126.5	266.7	145.7					30.645

LM67000 SERIES

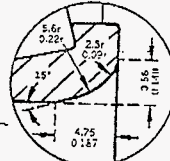
BORE RANGE
31.750
1.2500

d *R* *B*
Any cone in this series may be used with any cup in this series.

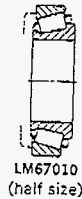
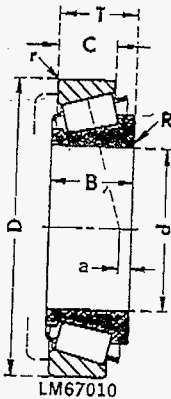
		cup no. LM67010						
		outside diameter	max housing fillet radius	width	bearing width			
		D	r	C	T	T		
31.750	**	LM67048	16.764 0.6600	-3.0 -0.12	59.131	1.3 0.05	11.811 0.4650	15.875 0.6250

a

** Compound radius—see illustration



Compound Radius



LM67010
(half size)



LM67010
(1:2.5 size)

THE TIMKEN COMPANY

No. of Pages 1

To: MIKE CUSTER
Co. _____
Dept. _____
Fax No. _____

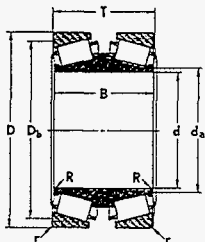
From: JIM LEE
Co. _____
Phone No. _____
Fax No. _____

x 1.374 for 175 RPM

basic rating @ 500 rpm for 3000 hours L ₁₀	
radial	9460 N 2130 lbf
thrust	6680 N 1500 lbf
K	1.42

use modifying factors to compare Timken bearing capacity-ratings with other makes of bearings.

TWO-ROW ASSEMBLY - DOUBLE CORE; SINGLE CUPS



cone	cup	width		rating at 500 rpm for 3000 hours L ₁₀			factor K	part numbers		cone	cup		
		over cups	T	one-row radial	thrust	two-row radial		cone	cup				
												N _{tot}	N _{tot}
30.162	62.000	33.797	34.925	10400	6800	18100	1.53	17116D	17244	0.8	35.5	1.5	54.0
1.1875	2.4409	1.3306	1.3750	2330	1530	4050				0.03	1.40	0.06	2.13
36.512	68.262	36.810	38.100	12200	9260	21200	1.31	19145D	19268	0.8	42.5	1.5	61.0
1.4375	2.6875	1.4492	1.5000	2740	2080	4760				0.03	1.67	0.06	2.40
36.512	71.438	36.810	38.100	12200	9260	21200	1.31	19145D	19281	0.8	42.5	1.0	63.0
1.4375	2.8125	1.4492	1.5000	2740	2080	4760				0.03	1.67	0.04	2.48
36.512	72.000	39.096	38.100	12200	9260	21200	1.31	19145D	19283	0.8	42.5	1.5	63.0
1.4375	2.8346	1.5392	1.5000	2740	2080	4760				0.03	1.67	0.06	2.48
42.862	85.000	50.267	52.375	19600	10300	34200	1.91	358D	354A	1.5	51.0	1.3	77.0
1.6875	3.3465	1.9790	2.0620	4420	2320	7690				0.06	2.01	0.05	3.03
42.862	90.119	54.991	52.375	19600	10300	34200	1.91	358D	352	1.5	51.0	2.3	78.0
1.6875	3.5480	2.1650	2.0620	4420	2320	7690				0.06	2.01	0.09	3.07
44.450	80.962	34.825	31.750	12200	11000	21200	1.11	13176D	13318	SPCL	50.0	1.5	72.0
1.7500	3.1675	1.3750	1.2500	2740	2480	4770					1.57	0.06	2.83
50.800	93.264	50.013	53.188	21100	12200	36800	1.73	375D	374	0.8	57.0	1.3	85.0
2.0000	3.6718	1.9690	2.0940	4750	2750	8260				0.03	2.24	0.05	3.25
†54.988	136.525	66.091	65.989	41100	60900	71500	0.67	78216D	78537	2.3	79.0	3.3	115.0
†2.1649	5.3750	2.6020	2.5980	9230	13700	16100				0.09	3.11	0.13	4.53
†54.988	140.030	66.091	65.989	41100	60900	71500	0.67	78216D	78551	2.3	79.0	2.3	117.0
†2.1649	5.5130	2.6020	2.5980	9230	13700	16100				0.09	3.11	0.09	4.61
61.912	110.000	55.550	55.550	23700	16300	41300	1.45	399D	394A	0.8	70.0	1.3	101.0
2.4375	4.3307	2.1870	2.1870	5340	3670	9290				0.03	2.76	0.05	3.98
61.912	110.000	55.550	55.550	23700	16300	41300	1.45	399D	394AS	0.8	70.0	3.3	99.0
2.4375	4.3307	2.1870	2.1870	5340	3670	9290				0.03	2.76	0.13	3.90
63.500	112.712	60.325	60.325	40100	23300	69800	1.72	39585D	39520	0.8	72.0	3.3	101.0
2.5000	4.4375	2.3750	2.3750	9010	5230	15700				0.03	2.83	0.13	3.98
63.500	112.712	60.325	60.325	40100	23300	69800	1.72	39585D	39521	0.8	72.0	0.8	103.0
2.5000	4.4375	2.3750	2.3750	9010	5230	15700				0.03	2.83	0.03	4.06
63.500	136.525	66.091	65.989	41100	60900	71500	0.67	78251D	78537	2.3	79.0	3.3	115.0
2.5000	5.3750	2.6020	2.5980	9230	13700	16100				0.09	3.10	0.13	4.53
63.500	140.030	66.091	65.989	41100	60900	71500	0.67	78251D	78551	2.3	79.0	2.3	117.0
2.5000	5.5130	2.6020	2.5980	9230	13700	16100				0.09	3.10	0.09	4.61
†64.988	136.525	66.091	65.989	41100	60900	71500	0.67	78255D	78537	1.5	79.0	3.3	115.0
†2.5586	5.3750	2.6020	2.5980	9230	13700	16100				0.06	3.11	0.13	4.53
†64.988	140.030	66.091	65.989	41100	60900	71500	0.67	78255D	78551	1.5	79.0	2.3	117.0
†2.5586	5.5130	2.6020	2.5980	9230	13700	16100				0.06	3.11	0.09	4.61
76.200	115.868	53.975	53.975	29900	13900	52100	2.15	LM114848D	LM114811	0.8	83.0	1.5	107.0
3.0000	4.5625	2.1250	2.1250	6720	3130	11700				0.03	3.27	0.06	4.21

† These maximum fillet radii will be cleared by the bearing corners.
 † For standard class ONLY, the maximum metric size is a whole millimetre value.
 * Cone is extended on one side.

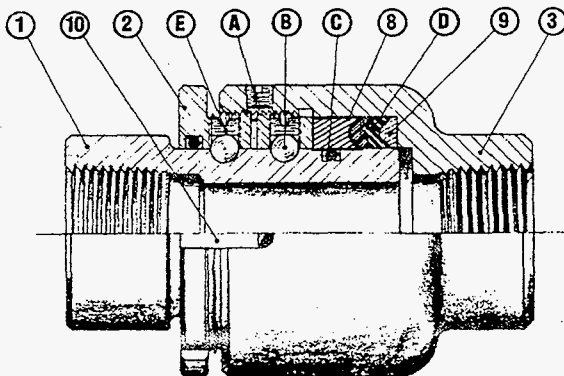
Date: * June 6, 1995

Teleconference with engineer (Jim *Good*) at Torrington Bearing Co., 206-455-4466 regarding Safety factor incorporated into their published Static Load Ratings. Loading less than the Static Load Rating will result in less than 1/10 inch deformation of the ball, and would correspond to a yield strength. Actual bearing failure (ball fracture or disintegration) would require 2 to 3 times that force, which would correspond to ultimate strength. Since we are designing to safety factors of 5 to ultimate and 3 to yield, a calculated SF of 3 based on the Static Load Rating should be sufficient.

- OK but still require heavy duty bearings, resulting in enlargement of winch box.

New pumps issued - Submersible pump now weighs 3000 lbs. Three bearings will be required.

E-21-15

T-M**SWIVEL JOINTS****ENGINEERING SPECIFICATIONS**

Manufactured in eight styles. Sizes from $\frac{3}{8}$ " through 16" pipe size in steel, and $\frac{3}{8}$ " to 4" in aluminum and stainless steel. The basic components of T-M Union Assembly Joints are as follows:

- 1** Swivel Nipple: Rotates on 2 rows of chrome alloy steel balls to ensure ease of rotation under severe thrust and radial loads.
- 2** Holding Nut: Provides half of bearing race and utility of (1) Providing initial compression on packing (2) Take-up on worn packing (3) Quick disassembly for repacking. Ball races of 6000 PSI and 15,000 PSI series are through hardened to 42 to 58 Rockwell "C" scale, for maximum strength and wear.
- 3** Body: Parts 1 and 3 are changed in configuration to provide eight different styles.
- 8** Back-up Ring: Furnished in similar metal to part 1 to back up packing and act as an outboard bearing during severe radial loads. Optional materials: Aluminum, stainless steel, teflon.
- 9** Male Adaptor: Acts as wedge to expand chevron packing and effect seal. Furnished in Buna N as standard, but may be furnished in aluminum, steel, stainless steel or aramid fiber.
- 10** Lock Bar: Locks assembly of parts 1 and 2 to part 3. Removal of lock bar allows holding nut to be rotated to (1) Take-up on packing in increments of .005" or (2) Remove assembly of parts 1 and 2 from body for repacking.
- A** Grease Plug: Hole is tapped to receive Alemite fitting but since T-M joints usually do not require lubrication until repacked, a flush set screw is inserted at factory.

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E-21-15

Basic Components:

- (B)** Balls: Hardened chrome alloy steel balls are furnished as standard. Optional material: stainless steel.
- (C)** "O" Rings: Act as grease seals to (1) Seal grease in assembly to insure proper lubrication of bearings (2) Seal against grit and dirt from outside sources (3) Seal bearings against line contaminants. Standard 6227 and 6230 sizes used. Optional materials: Silicone, teflon, viton.
- (D)** Chevron Packing: The most effective packing design known, used in standard JIC and AN-6225 sizes in Buna N. Optional materials: teflon, aramid fiber, viton.
- (E)** Ball Plugs: Retain balls in races. Assembly of parts 1 and 2 is made at factory and need not be disassembled to repack the joint. The T-M joints can be repacked without handling balls.

When ordering specify Size, Style, Material, e.g.: steel, stainless steel, etc., working pressure, working temperature, material to be handled, and end connections.

Standard end connections are threaded, (TE), bored for welding, (BR), or flanged (F); see price list for prices. Other end connections on special order.

Component parts can be ordered to suit individual requirements, e.g.: 2" style 100 1000 PSI MWP, steel with bronze male adaptor (9) teflon packing (D), bronze back-up ring (8), and stainless steel balls (B).

T-M Swivel Joints Out-perform All Others:

They can be repacked in the line or in the field without the handling of balls. Result: Less down time.

Union Assembly allows for take up on worn packing. Result: Up to 400% longer packing life. Provides positive pressure on packing to insure against low pressure and low temperature leaks. Result: No leaks, less maintenance.

Bearing assembly is protected against outside and inside contaminants, and grease is sealed in. Result: Longer life, less maintenance.

They will withstand heavier radial loads and thrust loads while maintaining seal due to high quality bearing races, outboard bearing, and modern engineering design. Result: Lower final cost.

All packing, "O" rings, used in T-M Swivel Joints are standard to JIC and AN sizes. Commercially available. Result: Less inventory.



SWIVEL JOINTS

E-21-15

STYLE	ESTIMATED WEIGHTS								
	100	200	300	400	500	600	700	800	
3/8" & 1/2" Alum.	1	1	1	1	2	2	2	2	
	Steel	2	2	2	3	4	4	5	6
1" & 3/4" Alum.	2	2	2	3	4	4	6	6	
	Steel	4	5	6	9	9	10	13	13
1 1/4" Alum.	2	3	3	4	5	5	7	7	
	Steel	5	6	7	10	11	12	15	16
1 1/2" Alum.	3	4	4	5	6	6	8	9	
	Steel	9	11	12	16	18	19	23	24
2" Alum.	6	8	9	12	13	15	18	19	
	Steel	13	19	23	31	34	38	48	51
	15000 PSI Steel	18	21	24	34	39	44	52	57
2 1/2" Alum.	8	9	12	15	17	19	23	25	
	Steel	21	25	31	38	43	47	59	65
3" Alum.	10	12	15	21	23	26	30	33	
	Steel	25	29	37	49	57	65	82	90
4" Alum.	19	23	27	31	42	47	58	62	
	Steel	52	60	72	85	98	110	136	144
6" Steel	95	110	125	205	220	235	315	330	
8" Steel	115	146	177	262	292	323	407	438	
10" Steel	140	195	250	335	390	445	530	585	
12" Steel	155	233	311	388	466	544	621	699	

Note: All weights given in pounds for threaded or welded ends. (Flanged weights not included.)

0

TERMS AND CONDITIONS OF SALES:

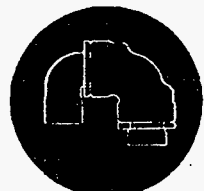
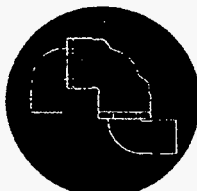
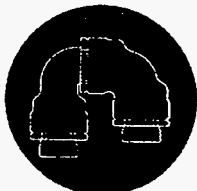
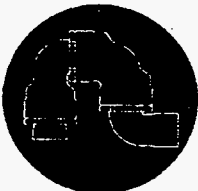
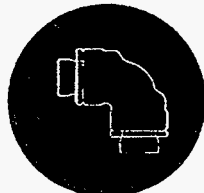
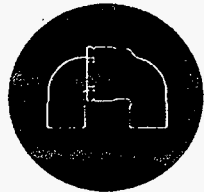
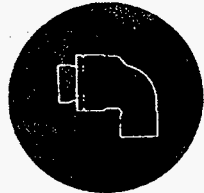
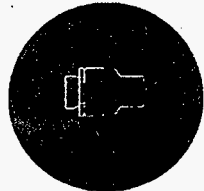
Prices and conditions of sale are subject to change without notice. Prices of all T-M Products are f. o. b., Sunnyvale, California. All material for export is subject to a 4% charge for export boxing. Material will be accepted for credit when returned with permission, and material returned for credit is subject to a restocking charge. Orders for products incorporating variations from catalog listed items are special; are subject to cancellation at extra charge and cannot be accepted for credit upon return. We insure to the common carrier but not normally beyond the point of delivery to the common carrier unless otherwise instructed. We maintain responsibility to the extent of the insurance carried . . . to the common carrier . . . or beyond when so instructed.

GUARANTEE:

All T-M Products are unconditionally guaranteed for one year against defective material or workmanship. Materials returned, f. o. b. Sunnyvale, if found defective will be repaired. Material found damaged, but not defective, will be repaired at nominal service charge.

WHEN ORDERING SPECIFY:

1. Pipe Size.
2. Style number of joints.
3. Material (aluminum, steel, stainless steel).
4. Product to be handled
5. Operating pressure.
6. Operating temperature.
7. End connections (threaded, flanged, bored or beveled for weld).





- THE FINAL COST OF AN ITEM IS DETERMINED BY ITS PERFORMANCE
- T-M SWIVEL JOINTS ARE OUTPERFORMING ALL OTHERS
- SPECIFY AND BUY T-M SWIVEL JOINTS FROM

REPRESENTATIVE

T-M

SWIVEL JOINTS

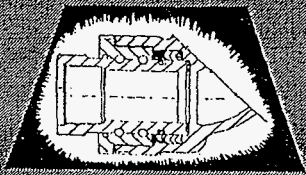
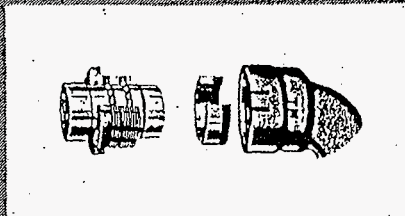
T-M MFG. CO., 695 E. AROUES AVE., SUNNYVALE, CALIF. 94086 (408) 736-5202 FAX (408) 736-3743

REV 4/92

PRINTED IN U.S.A.

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*Complete
Union Assembly
Speeds Maintenance*



Easy Repack!

The T-M Swivel Joints are the result of modern engineering methods applied to the design and manufacture of Ball-Bearing Swivel Joints. Over 400 different types, styles, and sizes have been developed for pressures and services from 28" vacuum to 15,000 p.s.i., and temperatures to 550 degrees F.

Along with the development of new types of Swivel Joints for many different services, T-M also carries on continuous research for the purpose of developing and perfecting Swivel Joints which will meet your performance specifications.

All T-M Ball Bearing Swivel Joints employ the basic principle that all rotating movement takes place on double rows of Ball-Bearings. Although each rotating member turns in a single plane, unlimited flexibility is secured by combining swivels in the proper sequence.

The T-M Swivel Joint arrangement bridges the gap between the working parts so that abrasives are filtered out. Packing seals last longer, lubrication is acted in, which maintains the life of the Ball-Bearings. T-M swivel joints are grease packed when assembled, and no further greasing is required unless the joint is in constant rotation.

T-M Swivel Joints are superior because:

They will withstand greater radial and thrust loads while maintaining seal.

The union assembly allows for repacking in minutes, while in the line.

Union assembly allows take-up for worn packing, thus extending the usable life of packing.

Union assembly allows take-up for pressure variations and for extremely volatile materials.

Union assembly permits repacking without the handling of balls.

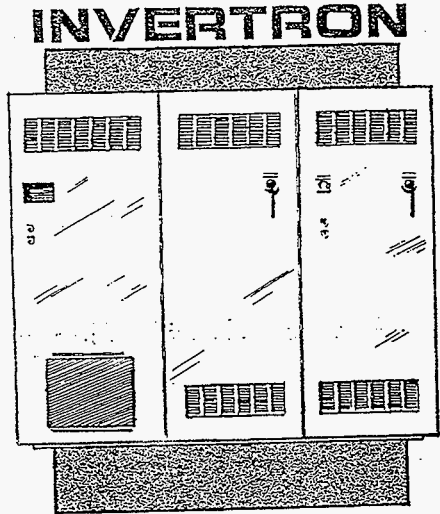
Standard packing is used throughout T-M Swivel Joints. Available from T-M Manufacturing, their Dealers, or from your local supplier.

T-M
SWIVEL JOINTS
 T-M MFG. CO., 695 E. ARQUES AVE., SUNNYVALE, CALIF. 94086 (408) 736-5202

4362 D-03

PROBLEM HARD COPY

250HP AC Drive



VGI and VCI
Digital PWM Inverters

INVERTRON

A-C DRIVES

250HP AC DRIVE

Standard Specifications

Inverter Model	VCI								Sales Office Engineering Models							
	VCI	980	980	980	980	980	980	980	2000	2000	2000	2000	3000	3000	3000	
Output Ratings	HP Rating @ 460V	150	200	250	300	350	400	500	600	700	800	900	1000	1200		
	Output Current	150	240	300	330	420	480	625	750	875	960	1080	1200	1380		
Output Ratings	Overload Capacity	Up to 150% for 1 minute					125%	110%	110% @ 930 A							
	Output Frequency	0-300 Hertz														
Input Ratings	Power Supply	3-Phase 380/400/415/460V														
	Allow. Variation	Plus or minus 10 percent														
Control Specs.	Control Method	VCI: Flux Regulated Sine Wave VCI: Flux Vector Regulation														
	Speed Regulation	VCI: 0.5% of base frequency with slip compensation VCI: 0.05% of base frequency														
	Freq. Regulation	0.1 Hertz														
	Volts/Hertz Ratio	VCI: Multi-point V/Hz curve shaping either linear or quadratic VCI: Programmable motor voltage regulation														
	Torque Boost	VCI: Up to 30 Volts at a programmable output frequency VCI: Calculated from motor input data														
	Carrier Frequency	Full-spectrum switching carrier frequency (between 800 Hz and 4 KHz) to lower motor noise														
	Auxiliary Functions	VCI: Three critical frequency avoidance bands, D-C braking, Start into a rotating motor, Slip compensation VCI: 8 selectable preset speeds, Torque limit, Programmable for wider applications														
	Operating Specs.	Operating Method	Keypad, or hardwired pilot devices connected to control terminal strip													
Speed Reference/ Input Signals		Up/Down arrow keys on keypad; 0-10 VDC (+/-), 0-20 or 4-20 mA analog signals wired to terminal strip, Static MOP inputs wired to terminal strip, 0-1 MHz (Pulse Train), and preset speeds														
Accel. and Decel. Time		VCI: Linear or "J" curve 1 to 3277 seconds; acceleration and deceleration set independently VCI: Linear 0.1 to 3277 seconds, acceleration and deceleration set independently														
External Output Signals (analog)		0-10 VDC analog outputs proportional to motor voltage, motor current, and motor speed; Optional 4-20 mA output signals														
Digital Outputs		Fault contact, 1-N.O.; Run contact, 1-N.O.; 2-Programmable output relay contacts each 1-N.O.														
Protective Functions	Serial Communications	RS232C serial port with optional MMI software provides ability to upload/download drive parameters, and monitor register values														
	Inverter Trip	Run permissive open, heatsink overtemperature, line dip ride-through time exceeded, Overspeed, D-C bus overcurrent, Motor overcurrent, Ground fault current, Microprocessor fault, Motor overcurrent														
	Fault Warning	D-C Bus Voltage less than 85%, plus two programmable input contacts from user.														
	Ride Through	Six cycles with a programmable voltage ride-through level														
Indication and Keypad	Fault Reset	Fault reset by contact closure or keypad														
	Display Mode	Motor RPM, Input Speed Reference, Motor Voltage, Motor Current, D-C Bus Voltage, Slip Frequency, and Output Frequency; Units are scalable														
	Reference Mode	Displays the method of speed reference input such as analog input, MOP, preset speeds or speed pot														
	Control Mode	Displays the method of drive control either keypad, or terminal strip														
	Diagnostic Mode	Fault codes with English messages are displayed; As each fault occurs it is stored with an occurrence time in NVRAM; If several faults occur at once, they will be shown flashing														
Operating Conditions	Code Mode	Provides the ability to enter and display the three levels of password protection														
	Storage Temp.	-40° C to 65° C (-40° F to 149° F)														
	Ambient Temp.	0° C to 40° C (32° F to 104° F)														
	Humidity	Non-condensing, relative humidity to 95%														
	Installation	Indoor not more than 1000 m (3300 ft.) above sea level														
	Enclosure	NEMA 1 force-ventilated														
Weight (kg)	980	980	980	980	980	980	2000	2000	2000	2000	3000	3000	3000			
	Up to 400 HP						500-800 HP				900-1200 HP					



RELANCE ELECTRIC

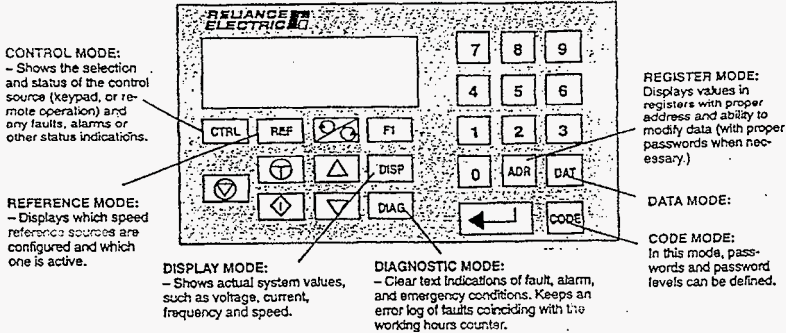
250 HP AC DRIVE

INVERTRON
A-C DRIVES

A multi-function alphanumeric keypad and display provides operation, programming, and monitoring of running condition.

The VGI/VCI keypad display is comprised of two lines and 40 characters. This backlit, LCD,

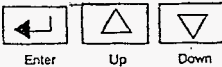
English message display is easily visible when viewing - even from a distance.



OPERATOR KEYS:



NON-STATE KEYS:

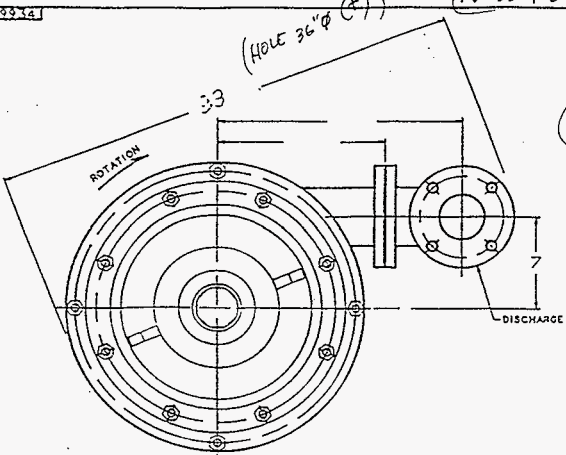


RS232 PROGRAMMING PORT IS ALSO AVAILABLE - AS STANDARD.

Using Reliance MMI Software all parameters can be downloaded, uploaded, monitored, and saved as a file for transferring to future additional drives.

FAULT DISPLAYS (Sample List)

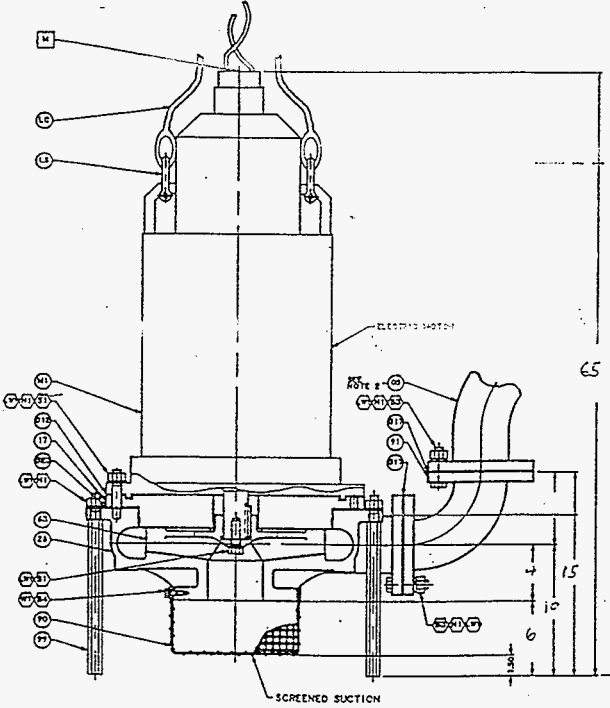
DISPLAY	DESCRIPTION
LOW_LINE_FLT	Line Dip Ride Thru Exceeded
BUS_OVERVOLT	Bus Overvoltage
OVERTEMP_INV	Overtemperature on Heatsink
BASE_DRV_FAULT	Power Supply Fault
BUS_OVERCURR	Bus Overcurrent Trip
MOT_OVERCURR	Motor Overcurrent
EARTH_CURRNT	Ground Fault Trip
LVOLT_TIMEOUT	Coast Thru too Long
PROC:A-B_FAULT	Processor 'A' Fault
MCR_KPS_OPEN	Main Contactor Auxiliary Open
MCR_KPS_CLSD	Main Contactor Auxiliary Closed
±15/5V_FAULT	Logic Power Not Ready
PROC: B.W. DOG	Processor Watchdog Time Out
BUS_ASYM_VLT	D-C Bus Asym. Charged



CONNECTIONS:
 (1) MAIN POWER LEAD TO MOTOR

WEIGHTS:
 PUMP 1500 LBS
 MOTOR 1500 LBS
 TOTAL 3000 LBS (APPROXIMATE)
 DRIVER DATA

NOTES:



LEGEND

- PUMP PART
- PARTNER
- CONNECTION

TITLE	16047-311-EX	2	REVISED	10-20-94
NO	1	1	REVISED	10-20-94
DATE	10-20-94	1	REVISED	10-20-94
BY	10-20-94	1	REVISED	10-20-94
CHKD	10-20-94	1	REVISED	10-20-94
APP'D	10-20-94	1	REVISED	10-20-94
PROJECT				

APPROVED AND IN CHARGE

DATE: 10/20/94

BY: [Signature]

FOR THE CONTRACTOR

PROJECT: [Blank]

PRELIMINARY

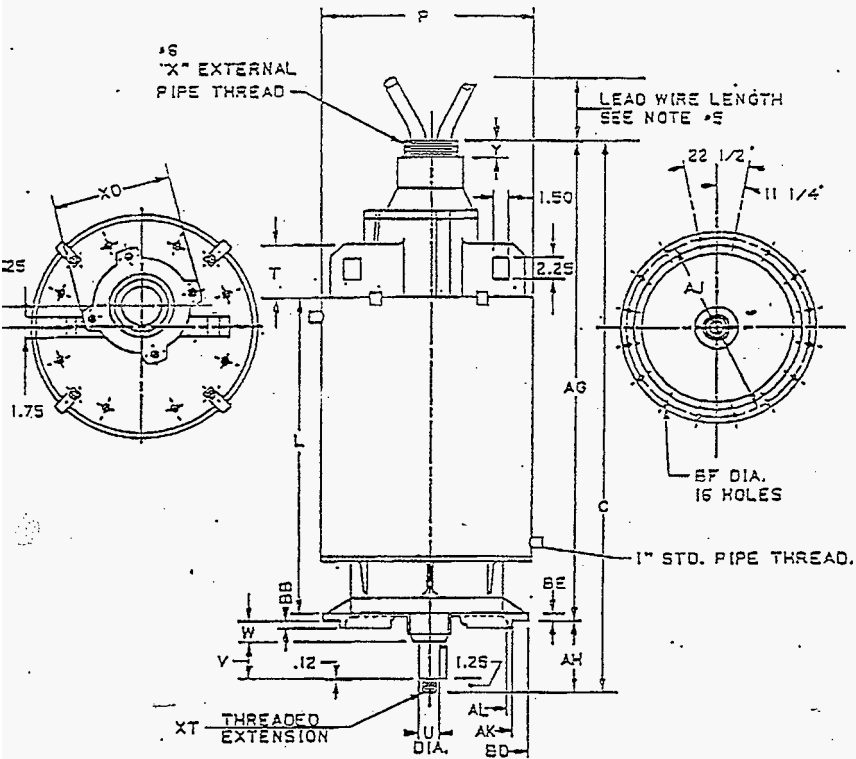
NO	1	DESCRIPTION	
DATE	10-20-94	REVISIONS	
3 X 4 X 13			
CON	10/20/94	BY	
CHKD	10/20/94	BY	
APP'D	10/20/94	BY	
DATE	10/20/94	BY	
LAWRENCE PUMPS INC.		PROJECT NO. 16047-311-EX	
49934 1st			

TY MASTER ALTERNATING CURRENT MOTORS

SCREWDRIVER INDUCTION

ENCLOSURE: TOTALLY ENCLOSED SUBMERSIBLE

FRAME L360TY



DIMENSIONS ARE IN INCHES

FRAME	C	L	P	T	U (3)	V	W	AG	AH	AJ	AK (4)	AL
360TY	51.81	33.25	20.38	4.25	2.4997	3.19	2.12	45.25	6.56	18.75	17.500	15.25

FRAME	BB	BD (2)	BE	BF	XT	LEAD CONNECTION		B.E. KEY			
						X	Y	X0	SO.	LGTH.	
360TY	.25	20.25	1.12	.69	1-1/2-12	3	8	1.25	10.88	.50	3.00

- 1J SPECIAL DIMENSIONS APPLYING TO THIS ORDER OR THIS LINE.
- 2J "BD" VARIES $\pm .010$.
- 3J "U" VARIES $\pm .0000$, $\pm .0005$.
- 4J "AK" VARIES $\pm .000$, $\pm .002$.
- 5J LEAD WIRE LENGTH AS SPECIFIED ON SALES ORDER.
- 6J 3/12" ϕ PIPE THREAD SUPPLIED WITH CABLE STRIP LARGER THAN #10

FACE RUNOUT AND ECCENTRICITY .005 MAX. T.I.R.

PROTECTIVE CONDUIT FURNISHED BY CUSTOMER.

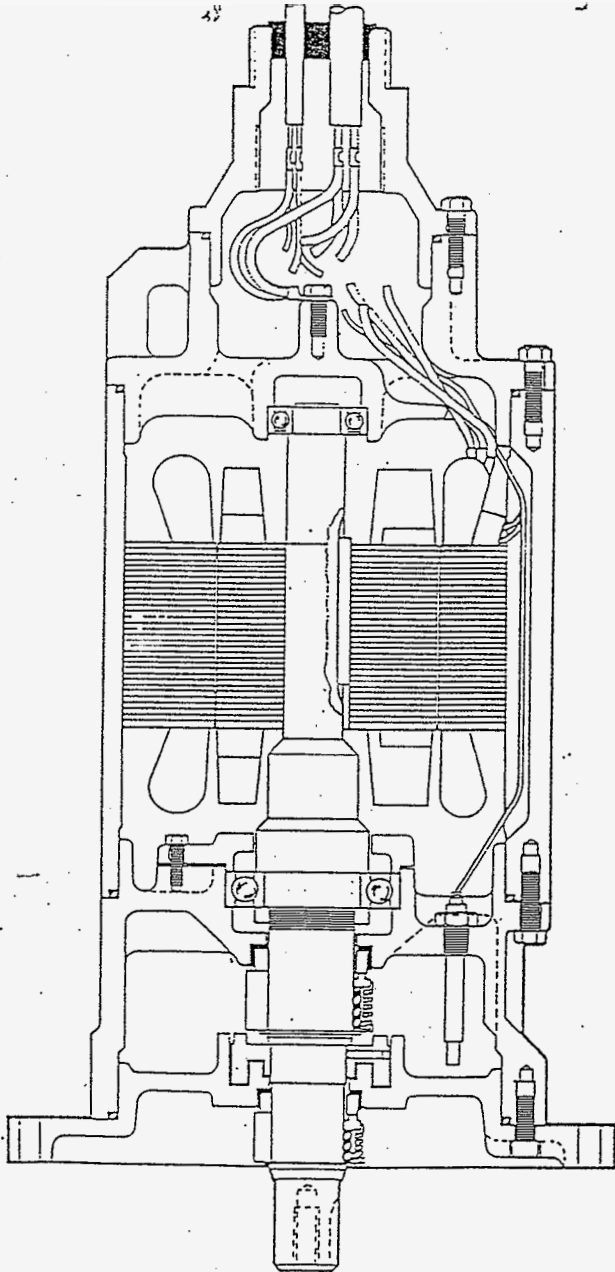
TYPE: L360TY CERTIFIED FOR:

ORDER NO. _____ ITEM _____ QTY _____ PH _____ HZ _____ SD _____ VOLTS 480
 APPROVED BY: _____ DATE: 7-15-62



CO. OF D.L. FULLER
 CO. OF D.L. MORGENTHAU
 CO. OF D.L. BERRY

DIMENSION 66930-120



matl

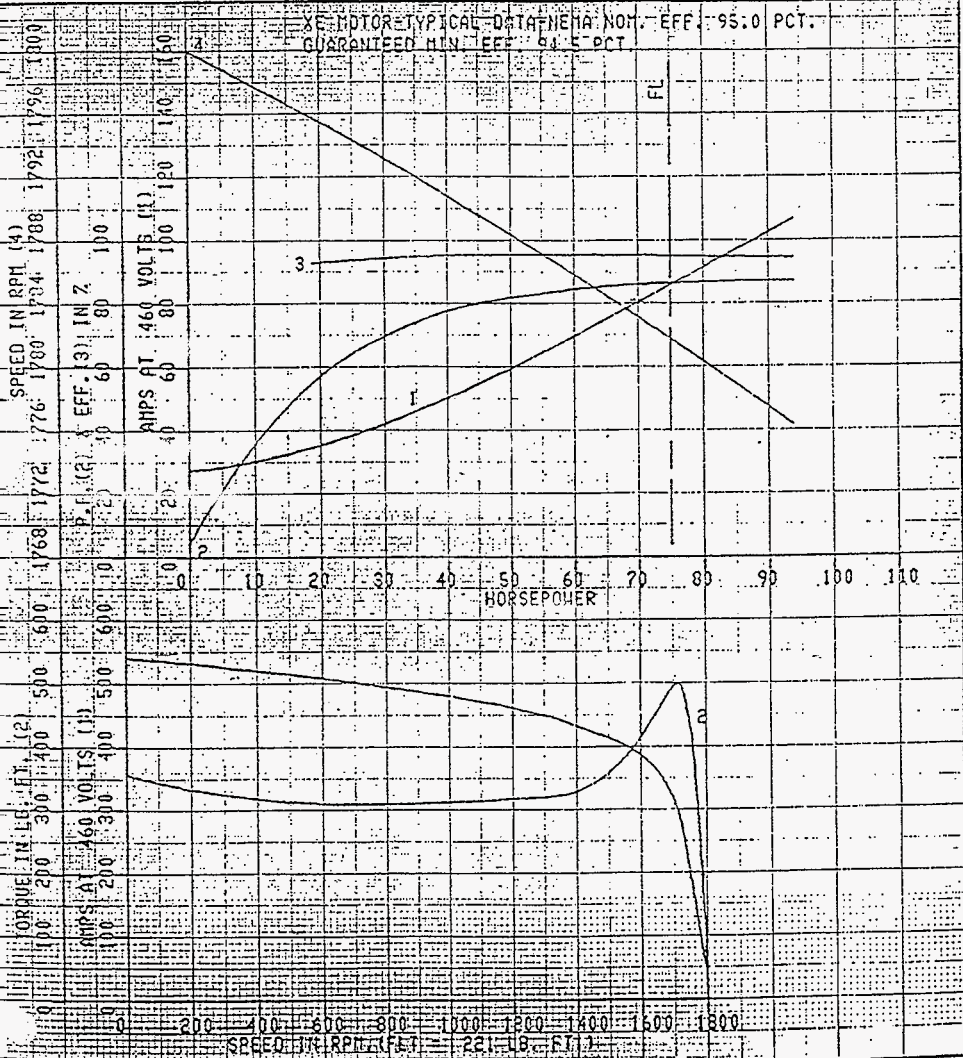
DEL S.O. -
S
TYPE P
BASE/HERTZ 3/60

RPM 1780
VOLTS 460
AMPS 85.5
DUTY CONT
AMB/INSUL 40/F

S.F. 1.15
NEMA DESIGN B
CODE LETTER G
ENCLOSURE TEFC-XT
E/S 592966

ROTOR 418141-33-RE
TEST S.O. ---
TEST DATE ---
STATOR RES. @ 25°C .0806
ORNS (BETWEEN LINES)

THIS MOTOR TYPICAL DATA NEMA NOM. EFF. 95.0 PCT.
GUARANTEED MIN. EFF. 94.5 PCT.



AMPERES SHOWN FOR 460 VOLT CONNECTION. IF OTHER VOLTAGE CONNECTIONS ARE AVAILABLE, THE AMPERES WILL VARY INVERSELY WITH THE RATED VOLTAGE.

RELIANCE
ELECTRIC

DR. BY P. B. GREENE
CK. BY D. M. BYRD
APP. BY D. M. BYRD

A-C MOTOR PERFORMANCE
CHARTS
E3669A-A-002
REVISION DATE 09/13/84

REL. S.O.	FRAME	HP	TYPE	PHASE/HERTZ	RPM	VOLTS
-	3038	75	P	3/60	1780	460
AMPS	DUTY	AMB°C/ INSUL.	S.F.	NEMA DESIGN	CODE LETTER	ENCL.
85.5	CONT	40/F	1.15	B	G	TEFC-XT
E/S	ROTOR	TEST S.O.	TEST DATE	STATOR RES. @25°C OHMS (BETWEEN LINES)		
592966	418141-33-RE	---	---	.0806		

PERFORMANCE

LOAD	HP	AMPERES	RPM.	% POWER FACTOR	% EFFICIENCY
NO LOAD	0	27.5	1800	4.41	0
1/4	18.8	34.3	1796	55.3	92.8
2/4	37.5	48.6	1791	76.0	95.1
3/4	56.2	66.1	1787	83.6	95.4
4/4	74.9	85.5	1782	86.4	95.0
5/4	93.8	107	1776	87.0	94.4

SPEED TORQUE

	RPM	TORQUE % FULL LOAD	TORQUE LB.-FT.	AMPERES
LOCKED ROTOR	0	162	357	540
PULL UP	700	140	309	501
BREAKDOWN	1715	226	499	290
FULL LOAD	1782	100	221	85.5

AMPERES SHOWN FOR 460. VOLT CONNECTION. IF OTHER VOLTAGE CONNECTIONS ARE AVAILABLE, THE AMPERES WILL VARY INVERSELY WITH THE RATED VOLTAGE

REMARKS: XE MOTOR-TYPICAL DATA-NEMA NOM. EFF. 95.0 PCT.
GUARANTEED MIN. EFF. 94.5 PCT.


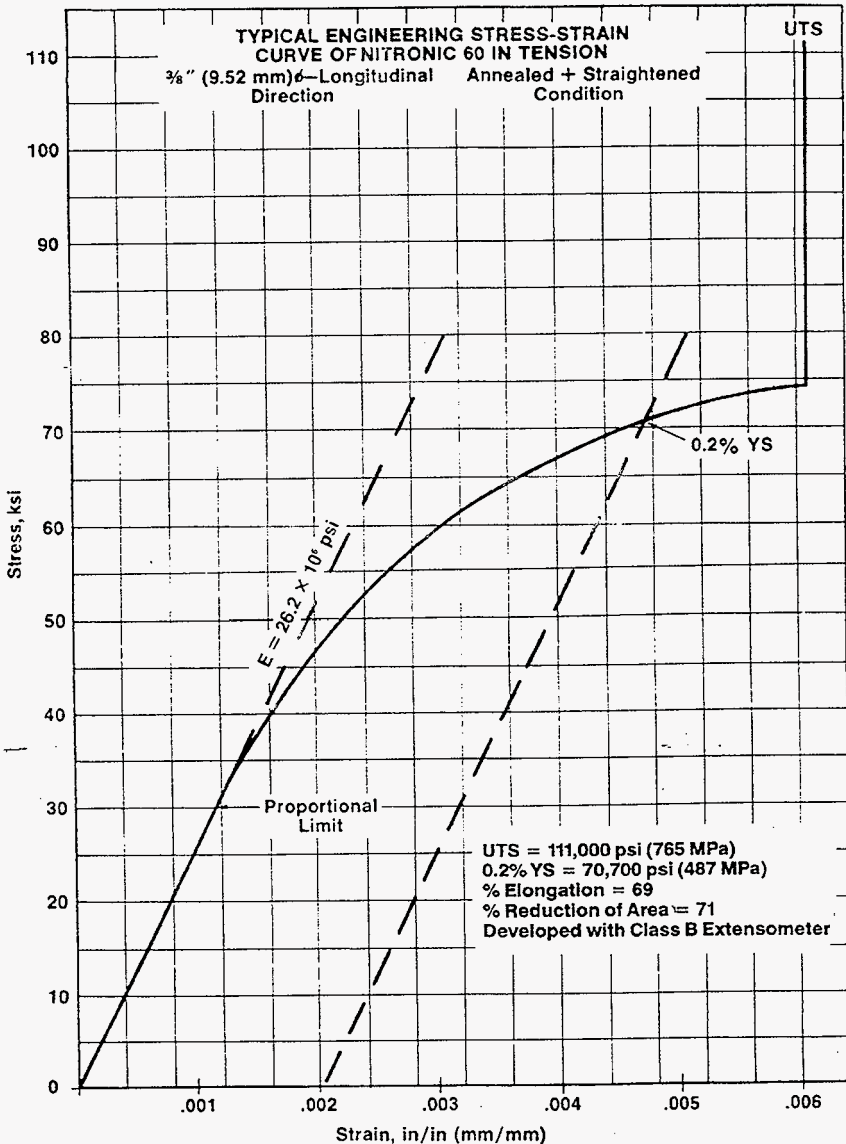
 CLEVELAND, OHIO 44117 U.S.A.	DR. BY P. B. GREENE	A-C MOTOR E3669A-A-002 PERFORMANCE DATA DATE 09/13/84
	CK. BY U. H. BYRD	
	APP. BY U. H. BYRD	
	DATE 09/17/84	

FIGURE 7

G-33-05



WIRE ROPE Stainless Steel

Stainless Steel Wire Rope

Stainless steel wire rope is especially suited for frequent, continuous exposure to extremely corrosive conditions. Stainless steel provides high resistance to corrosion from marine environments, petrochemicals, oxidizing acids, and food processing. It is available in the following five grades. TYPE 316—Provides chemical resistance along with low magnetic properties ideal for use around sensitive instruments. TYPE 302 AND TYPE 304—Breaking strength conforms to the latest revision to applicable Federal Specification RR-W-410 (except 1 x 7 Type 304 construction) or MIL-W-85420.

Controlled chemistry of Type 302 provides exceptionally high resistance, resulting in a high performance cable that far exceeds the endurance requirements of the applicable specification. TYPE 316 PLUS SUPER ALLOY WIRE ROPE—See page 491. Devoted to combat spreading crevice corrosion and pitting that attack conventional Type 302 and Type 304 stainless steel in a marine sphere. TYPE 305 NONMAGNETIC—See page 484. Meets Federal Specification MIL-C-16375A and is ideal for applications which require good corrosion resistance and low magnetic permeability.

Type 316 Stainless Steel Wire Rope

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
1 x 19 SINGLE STRAND			
1/8"	1,780	8908712	\$0.68 \$0.55
1/4"	4,000	8908716	1.13 0.91
3/8"	6,900	8908722	1.60 1.28
1/2"	10,600	8908724	2.14 1.72
5/8"	14,500	8908728	3.41 2.93
3/4"	19,300	8908758	\$6.57 \$5.80
6 x 19 IWRC			
1/8"	1,780	8908712	\$0.68 \$0.55
1/4"	4,000	8908716	1.13 0.91
3/8"	6,900	8908722	1.60 1.28
1/2"	10,600	8908724	2.14 1.72
5/8"	14,500	8908728	3.41 2.93
3/4"	19,300	8908758	\$6.57 \$5.80

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
7 x 7 STRAND CORE			
1/8"	550	8908732	\$0.38 \$0.30
1/4"	1,300	8908734	.61 .49
3/8"	2,100	8908736	.86 .69
1/2"	3,000	8908744	\$1.38 \$1.11
5/8"	4,000	8908748	1.54 1.24
3/4"	5,000	8908752	2.05 1.69
1"	7,600	8908754	3.64 2.93
1 1/8"	11,000	8908756	4.56 3.67
7 x 15 STRAND CORE			
1/8"	1,500	8908744	\$1.38 \$1.11
1/4"	3,200	8908748	1.54 1.24
3/8"	4,900	8908752	2.05 1.69
1/2"	6,600	8908754	3.64 2.93
5/8"	8,300	8908756	4.56 3.67

Stainless

Type 302 Stainless Steel

Nonmagnetic type 302 stainless steel provides excellent tensile strength and elongation properties, plus high fatigue resistance. The breaking strength conforms to the latest revision of applicable Specification RR-W-410, MIL-W-85420, and MIL-W-87161 to which it applies. This high performance cable far exceeds the endurance requirements of the applicable specification. Commonly used in the chemical, food processing, petroleum, and

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
SINGLE STRAND—Minimum sizes are used as cable and standard sizes for guy wire.			
1/8"	25	3458711	\$0.07 \$0.06
1/4"	55	3458712	.06 .05
3/8"	90	3458713	.06 .05
1/2"	140	3458714	.06 .05
5/8"	200	3458715	.07 .06
3/4"	250	3458716	.09 .08
Break Sizes			
4/8"	40	3458711	\$0.05 \$0.05
5/8"	150	3458712	.08 .06
3/4"	500	3458714	.12 .10
1"	1,200	3458716	.24 .19
1 1/8"	2,100	3458717	.40 .33
1 1/4"	3,300	34587171	.69 .57
1 1/2"	4,700	3458719	.82 .67

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
SINGLE STRAND—Used for guying, bracing and standing rigging.			
1/8"	335	3458722	.12 .10
1/4"	500	3458723	.18 .13
3/8"	800	3458724	.27 .21
1/2"	1,200	3458725	.35 .30
5/8"	2,100	3458726	.59 .50
3/4"	3,300	3458727	.75 .64
1"	4,700	3458728	.99 .84
1 1/8"	6,200	3458731	1.40 1.18
1 1/4"	8,000	3458732	1.87 1.59
1 1/2"	11,500	3458734	3.16 2.68

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
7 HOLLOW CORE			
1/8"	110	3458751	\$0.16 \$0.13
6 x 19 IWRC			
1/8"	16,300	3458721	\$5.28 \$4.60
1/4"	22,800	3458712	6.03 5.26
6 x 19 IWRC—More flexible than 6 x 19 construction if it is not as durable.			
1/8"	3,000	3458711	\$1.94 \$1.61
1/4"	5,400	3458754	3.10 2.56
3/8"	6,300	3458755	4.28 3.55
1/2"	11,700	3458756	5.27 4.95
5/8"	20,400	3458757	7.38 6.09

Type 304 Stainless Steel—Wire Rope

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
1 x 19 SINGLE STRAND			
1/8"	1,500	3461714	\$0.16 \$0.13
1/4"	3,200	3461715	.30 .25
3/8"	2,100	3461716	.49 .40
1/2"	3,300	3461717	.62 .51
5/8"	4,700	3461718	.85 .69
3/4"	6,200	3461721	1.30 1.06
1"	12,500	3461723	1.87 1.59
1 1/8"	17,500	3461724	2.72 2.33
6 x 19 IWRC			
1/8"	16,300	3461774	\$4.06 \$3.32
1/4"	22,800	3461775	5.04 4.12
3/8"	35,000	3461777	7.17 5.87
6 x 37 IWRC			
1/8"	5,400	3375731	\$2.10 \$1.72
1/4"	6,300	3375732	2.89 2.44
3/8"	11,700	3375733	3.71 3.03

Dia.	Lbs. No.	NET/FOOT	
		1-99	100-299 300-Up
7 x 7 STRAND CORE			
1/8"	490	3461744	\$0.26 \$0.21
1/4"	920	3461745	.40 .32
3/8"	1,700	3461746	.59 .49
1/2"	2,400	3461747	.70 .57
5/8"	3,700	3461748	.98 .80
3/4"	6,700	3461751	1.37 1.12
7 x 19 STRAND CORE			
1/8"	920	3461764	\$0.73 \$0.64
1/4"	1,760	3461765	.81 .75
3/8"	2,400	3461766	.87 .79
1/2"	3,700	3461767	1.08 .88
5/8"	4,000	3461768	1.44 1.18
3/4"	6,400	3461769	1.61 1.32
1"	8,000	3461772	2.11 1.87
1 1/8"	12,000	3461773	3.41 2.95

Construction	Core	Characteristics			
		Strength	Abrasion Resistance	Flexibility	Corrosion Resistance
1 x 7	Strand	Excellent	Very Good	Low	Excellent
1 x 19	Strand	Excellent	Very Good	Low	Excellent
6 x 7	Hollow	Very Good	Good	Excellent	Excellent
1 x 3	Hollow	Very Good	Good	Excellent	Excellent
6 x 7	Strand	Excellent	Good	Good	Excellent
6 x 7	Strand	Very Good	Moderate	Very Good	Excellent
6 x 19	IWRC	Excellent	Excellent	Very Good	Excellent
6 x 19	Strand	Excellent	Good	Excellent	Excellent
6 x 37	IWRC	Good	Very Good	Excellent	Excellent
6 x 42	Fiber	Moderate	Low	Excellent	Excellent

All descriptions of wire rope applications listed on this page are general, and are provided with the intention of making it easier for you to select the proper wire rope for your individual needs. To select wire rope correctly, be sure to apply sound engineering practices.

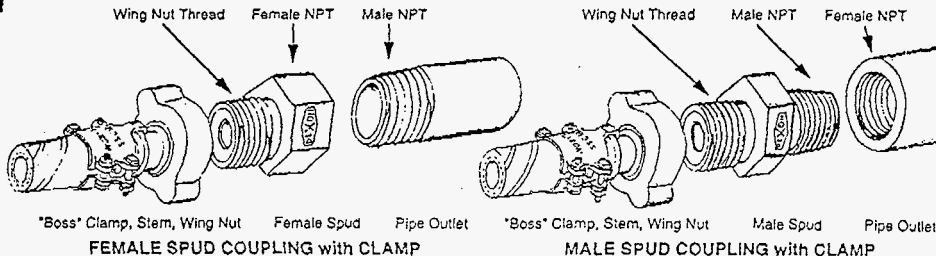
Descriptions of wire rope applications listed on this page are general, and are provided with the intention of making it easier for you to select the proper wire rope for your individual needs. To select wire rope correctly, be sure to apply sound engineering practices.

McMASTER-CARR

BOSS COUPLINGS

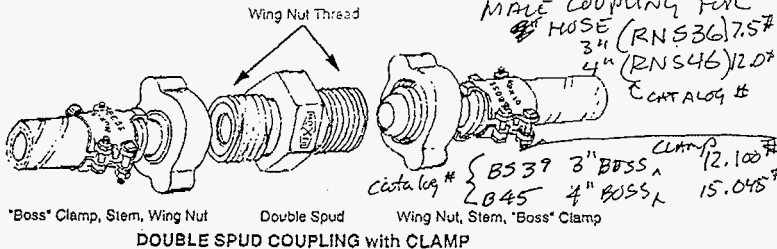
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BOSS™ ASSEMBLY DIAGRAM



*Boss' Clamp, Stem, Wing Nut Female Spud Pipe Outlet
FEMALE SPUD COUPLING with CLAMP

*Boss' Clamp, Stem, Wing Nut Male Spud Pipe Outlet
MALE SPUD COUPLING with CLAMP



DOUBLE SPUD COUPLING with CLAMP

"EZ-BOSS" GROUND JOINT SEAL

"EZ-Boss" non-metallic polymer seal is not subject to corrosion and offers resistance to chemicals and steam found in manufacturing facilities. It precisely machined recess holds the seal firmly in place, allowing it to absorb harsh punishment. Recommended for steam service up to 450° F; easy to seal; works with existing "Boss" Ground Joint Stems; Made in the USA. Fittings will be supplied with Polymer seats as stock is depleted

PLATED STEEL &/or MALLEABLE IRON

Sizes	Coupling With Female Spud		Stem		Wing Nut		Female Spud		Male Spud		Double Spud		
	Part #	Price	Part #	Price	Part #	Price	Part #	Price	Part #	Price	Part #	Price	
1/4"	GF1	\$24.25	GBA	\$7.75	100	BA2	\$6.00	100	GBC	\$10.50	100	GMB	\$10.50
3/8"	GF3	22.25	GCA	5.25	100	CB	6.00	100	GCC	11.00	100	GMC	10.50
1/2"	GF6	15.40	GB1	4.00	100	B2	3.75	100	GB3	7.65	100	GM3	12.00
3/4"	GF20	21.85	GB6	9.00	50	B12	4.70	25	GB8	7.95	50	GM8	13.00
1"	GF36	22.20	GB11	8.25	50	B12	4.70	25	GB13	8.25	50	GM13	14.00
1 1/4"	GF51	36.80	GB16	11.00	25	B17	8.00	25	GB18	17.80	25	GM18	30.00
1 1/2"	GF81	37.05	GB21	11.25	25	B17	8.00	25	GB23	17.80	25	GM23	28.30
2"	GF81	43.80	GB26	12.00	10	B27	14.00	10	GB28	17.80	10	GM28	30.85
2 1/2"	GF96	123.15	GB31	32.15	5	B32	35.00	5	GB33	56.00	5	GM33	80.00
3"	GF111	132.00	GB36	36.00	5	B37	35.00	5	GB38	61.00	5	GM38	85.00
4"	GF141	292.00	GB46	87.00	5	B47	95.00	5	GB48	110.00	5		
5"	GF171	765.00	GB56	\$40.00	2	B67	175.00	2	GB56	250.00	2		
6"	GF201	875.00	GB66	250.00	2	B67	175.00	2	GB68	400.00	2		

1/4", 3/8", 5" & 6" come with Copper seat spuds. Copper seats are available in other sizes - consult the Factory for pricing and availability.

**"Boss" Couplings And The Steam Quick Disconnects On P. 6
 Are The Only Dixon Couplings Recommended For Steam Service**

Industrial Hose Products/List Prices

Steam Hose

200MB

 Steam Pile Driver
(Specification 3601)

Item No.	Nom. I.D. (In.)	Nom. O.D. (In.)	Maximum W.P. (psi)	Suction (In. Hg)	Min. Bend Radius (In.)	Wt. Per Ft. (Lbs.)	Standard Pack	Stock (X) or Minimum Order Qty.	Gates Prod. No.	List Price Per Ft.
303500	2	2.95	200	10.0	25.0	3.43	50 ft	500 ft	3601-0410	\$48.438
303501	2 1/2	3.45	200	10.0	30.0	3.88	50 ft	500 ft	3601-0420	65.208
303502	3	3.97	200	10.0	36.0	4.47	50 ft	X	3601-0430	77.516
303503	4	5.20	200	10.0	48.0	7.47	50 ft	X	3601-0440	125.364

RECOMMENDED FOR: Saturated steam up to 200 PSI and 388° F for pile driver applications.
Service life will be severely shortened if steam without lubricating oil is used.

NOTE: NOT FOR USE WITH STEAM ONLY.

TEMPERATURE: Maximum to 388° F (198° C) continuous service.

CONSTRUCTION: Tube: Type A (Neoprene) Black.

Reinforcement: Braided high tensile steel wire.

Cover: Type A (Neoprene) Black. All sizes are perforated.

COUPLINGS: [14] or [15]. Reference pages 130-141.

PACKAGING: 50' coiled in a carton.

BRANDING: Rubber patch label every 5-10 ft. Example: "200MB Steam Pile Driver 200 PSI (1.38MPa) WP at 388 Degrees F Made In U.S.A. Caution: Tighten Clamps Before Using."

SPECIAL ORDER

REQUIREMENTS: Special production runs require minimum order quantities of 500 feet per size.

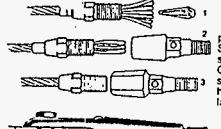
REMNANT LENGTHS: Not Available.

Due to continual product improvements, Gates reserves the right to alter specs and prices without prior notice.



The world's most trusted name in belts and hose.

Positive Grip Wire Rope Fittings Information



Fittings consist of three parts: A sleeve, plug and covering socket. To assemble: (1) Slip sleeve over end of wire rope. (2) Open rope strands and drive plug into sleeve. (3) Close strands over plug and secure on covering socket. For safety, covering socket on fittings has inspection hole for checking correct installation. Fittings are reusable by replacing plug. Recommend use of kit listed below for easier, more efficient assembly of fitting.

Machined Standard Clevis End Fittings

Material: Machined from bar stock. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for all standard industrial rigging jobs.

Wire Rope Dia.	Approx. Overall Length	Jaw Opening	Pin Dia.	Self-Colored Steel		Cadmium-Plated Steel		Stainless Steel	
				No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	3 1/2"	1 1/2"	3/8"	3473175	12.96	3473235	16.20	3473375	22.80
1/2"	4 1/2"	2 1/2"	1/2"	3473716	13.15	3473236	16.44	3473717	22.80
5/8"	5 1/2"	3 1/2"	5/8"	3473717	15.76	3473237	22.59	3473718	32.28
3/4"	6 1/2"	4 1/2"	3/4"	3473718	18.19	3473238	22.59	3473719	32.28
7/8"	7 1/2"	5 1/2"	7/8"	3473721	20.26	3473241	31.58	3473720	44.16
1"	8 1/2"	6 1/2"	1"	3473722	40.74	3473242	40.74	3473721	64.16
1 1/8"	9 1/2"	7 1/2"	1 1/8"	3473724	45.86	3473244	51.59	3473723	84.16
1 1/2"	10 1/2"	8 1/2"	1 1/2"	3473726	53.73	3473246	67.16	3473725	112.16

Forged Heavy Duty Clevis End Fittings

Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3473.

Wire Rope Dia.	Approx. Overall Length	Jaw Opening	Pin Dia.	Self-Colored Steel		Galvanized Steel	
				No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	3 1/2"	1 1/2"	3/8"	3480714	\$23.70	3480703	\$24.16
1/2"	4 1/2"	2 1/2"	1/2"	3480715	25.20	3480704	25.84
5/8"	5 1/2"	3 1/2"	5/8"	3480716	30.69	3480705	31.16
3/4"	6 1/2"	4 1/2"	3/4"	3480717	41.46	3480706	41.92
7/8"	7 1/2"	5 1/2"	7/8"	3480718	45.33	3480707	45.84
1"	8 1/2"	6 1/2"	1"	3480719	67.10	3480708	67.61
1 1/8"	9 1/2"	7 1/2"	1 1/8"	3480721	74.45	3480710	74.96
1 1/2"	10 1/2"	8 1/2"	1 1/2"	3480722	126.44	3480711	126.95
1 3/4"	11 1/2"	9 1/2"	1 3/4"	3480723	181.84	3480712	182.35

Machined Standard Duty Hook Assemblies

Material: Machined from bar stock. Available in alloy steel. Socket assembly machined from alloy steel. Hook forged alloy steel. Finish: Self-colored. Special Features: Available with rigid or swivel hook. Uses: For hoisting and lifting operations.

Wire Rope Dia.	Hook Dia.	Overall Length	Rigid Hooks		Swivel Hooks		Safety Latches	
			No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	1 1/4"	3 1/2"	3511714	341.53	7116"	429"	3511734	348.86
1/2"	1 3/4"	4 1/2"	3511716	65.76	7116"	429"	3511736	86.44
5/8"	1 7/8"	5 1/2"	3511718	87.94	7116"	429"	3511738	114.76

Machined Standard Eye End Fittings

Material: Machined from bar stock. Special Features: Positive Grip Sockets are ideal for most industrial application especially where a smooth machined finish is desired for ease of handling and better appearance.

Wire Rope Dia.	Approx. Overall Length	Socket Dia.	Hole Dia.	Eye Dia.	Self-Colored Steel		Zinc Plated Steel		Stainless Steel	
					No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	2 1/2"	3/8"	3/8"	3/8"	3474714	\$10.04	3474734	\$12.55	3474774	\$21.16
1/2"	3 1/2"	1/2"	1/2"	1/2"	3474715	10.28	3474735	12.84	3474775	22.16
5/8"	4 1/2"	5/8"	5/8"	5/8"	3474716	10.51	3474736	13.10	3474776	22.16
3/4"	5 1/2"	3/4"	3/4"	3/4"	3474717	11.56	3474737	14.45	3474777	26.16
7/8"	6 1/2"	7/8"	7/8"	7/8"	3474718	12.51	3474738	15.77	3474778	26.16
1"	7 1/2"	1"	1"	1"	3474719	16.44	3474739	18.44	3474779	26.16
1 1/8"	8 1/2"	1 1/8"	1 1/8"	1 1/8"	3474721	22.08	3474741	27.59	3474781	46.16
1 1/2"	9 1/2"	1 1/2"	1 1/2"	1 1/2"	3474722	24.15	3474742	30.19	3474782	46.16
1 3/4"	10 1/2"	1 3/4"	1 3/4"	1 3/4"	3474726	45.86	3474746	55.83	3474786	84.16

Assembly Kits for Use with Positive Grip Wire Rope Fittings

Material: Extruded alloy steel. Finish: Self-colored. Special Features: Provide easier, more efficient assembly of Positive Grip Wire Rope Fittings listed on this page and on the following pages. How It Works: Steel block holds wire strands firmly that plug can be driven to a solid seat. Assembly block prevents wires of the rope strands from being nicked by the jaws of the wire. Note: Hollow tubing is used for terminating IWRC wire rope.

Wire Rope Dia.	No. NET EACH
3/8"	8929212
1/2"	8929214

Positive Grip Wire Rope Fittings

Machined Standard Stud End Fittings. Material: Machined from bar stock. Special Feature: Positive grip socket has stud end. Uses: Ideal for most industrial rigging jobs. A quick method of attaching lines to beams, equipment or to various lifting devices.

Wire Rope Dia.	Threaded Stud Length	Approx. Overall Length	Socket Dia.	Self-Colored Steel		Cadmium Plated Steel		Stainless Steel	
				No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	10-32	1 1/2"	3/8"	3475714	\$11.00	3475713	\$13.75	3475712	\$16.50
1/2"	1/2-20	1 3/4"	1/2"	3475715	12.85	3475714	16.06	3475713	26.99
5/8"	3/4-24	2 1/4"	5/8"	3475716	14.7	3475715	18.39	3475714	30.90
3/4"	7/16-24	2 3/4"	3/4"	3475717	17.35	3475716	21.69	3475715	35.64
7/8"	1/2-24	3 1/4"	7/8"	3475718	20.98	3475717	26.22	3475716	44.05
1"	9/16-24	3 3/4"	1"	3475719	23.63	3475718	31.14	3475717	60.72
1 1/8"	5/8-24	4 1/4"	1 1/8"	3475720	33.38	3475719	41.2	3475718	70.93
1 1/2"	3/4-24	5 1/4"	1 1/2"	3475726	49.88	3475724	62.34	3475723	104.74

Forged Heavy Duty Stud End Fittings

End fittings for wire rope diameter up to 1 1/2" available, specify 3479399- on Request.

Wire Rope Dia.	Threaded Stud Length	Approx. Overall Length	Socket Dia.	Self-Colored Steel		Galvanized Steel	
				No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	10-32	1 1/2"	3/8"	3479711	\$21.50	3479710	\$26.88
1/2"	1/2-20	1 3/4"	1/2"	3479712	22.59	3479711	30.65
5/8"	3/4-24	2 1/4"	5/8"	3479713	28.39	3479712	35.49
3/4"	7/16-24	2 3/4"	3/4"	3479714	35.18	3479713	43.97
7/8"	1/2-24	3 1/4"	7/8"	3479715	42.97	3479714	52.86
1"	9/16-24	3 3/4"	1"	3479716	52.86	3479715	62.75
1 1/8"	5/8-24	4 1/4"	1 1/8"	3479717	77.56	3479716	96.95

Machined Standard Duty Turnbuckles

Socket & Jaw Turnbuckle. Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3473.

Wire Rope Dia.	Approx. Overall Length	Jaw Opening	Pin Dia.	Self-Colored Steel		Galvanized Steel	
				No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	3 1/2"	1 1/2"	3/8"	3480714	\$23.70	3480703	\$24.16
1/2"	4 1/2"	2 1/2"	1/2"	3480715	25.20	3480704	25.84
5/8"	5 1/2"	3 1/2"	5/8"	3480716	30.69	3480705	31.16
3/4"	6 1/2"	4 1/2"	3/4"	3480717	41.46	3480706	41.92
7/8"	7 1/2"	5 1/2"	7/8"	3480718	45.33	3480707	45.84
1"	8 1/2"	6 1/2"	1"	3480719	67.10	3480708	67.61
1 1/8"	9 1/2"	7 1/2"	1 1/8"	3480721	74.45	3480710	74.96
1 1/2"	10 1/2"	8 1/2"	1 1/2"	3480722	126.44	3480711	126.95
1 3/4"	11 1/2"	9 1/2"	1 3/4"	3480723	181.84	3480712	182.35

Forged Heavy Duty Eye End Fittings

Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3473.

Wire Rope Dia.	Approx. Overall Length	Eye Dia.	Hole Dia.	Eye Width	Self-Colored Steel		Galvanized Steel	
					No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	2 1/2"	3/8"	3/8"	3/8"	3482713	\$23.23	3482733	\$22.55
1/2"	3 1/2"	1/2"	1/2"	1/2"	3482714	23.23	3482734	22.55
5/8"	4 1/2"	5/8"	5/8"	5/8"	3482715	33.88	3482735	40.72

Machined Standard Duty Drum Sockets

Material: Alloy steel. Machined from alloy steel. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Used for attaching lines to slotted drums. See # with No. 3475.

Wire Rope Dia.	Approx. Overall Length	Socket Dia.	Hole Dia.	Eye Dia.	Self-Colored Steel		Zinc Plated Steel		Stainless Steel	
					No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH	No. NET EACH
3/8"	1 1/2"	3/8"	3/8"	3/8"	3470714	\$9.00	3470713	\$11.25	3470712	\$15.03
1/2"	2 1/2"	1/2"	1/2"	1/2"	3470715	10.43	3470714	12.99	3470713	16.69
5/8"	3 1/2"	5/8"	5/8"	5/8"	3470716	11.75	3470715	14.31	3470714	19.16
3/4"	4 1/2"	3/4"	3/4"	3/4"	3470717	12.93	3470716	15.49	3470715	20.16
7/8"	5 1/2"	7/8"	7/8"	7/8"	3470718	13.68	3470717	16.24	3470716	22.05
1"	6 1/2"	1"	1"	1"	3470719	17.00	3470718	19.56	3470717	26.16
1 1/8"	7 1/2"	1 1/8"	1 1/8"	1 1/8"	3470721	31.40	3470720	39.25	3470719	52.86

Spark resistant Bronze in sizes listed available, specify 3473099. Prices on Request.

Note: Nos. 34737, 34807, 35117 and 34747 are for 3 strand fiber core wire rope. Fittings for other construction see top of this page.



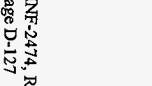
Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3473.



Zinc-plated finish is available, specify 3511799. Prices on Request. See # with No. 3473.



Spark Resistant Bronze in sizes listed available, specify 3474799. Prices on Request. See # with No. 3473.



Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3473.

Spark Resistant Bronze in sizes listed available, specify 3475799. Prices on Request. See # with No. 3473.



Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3475.



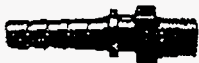
Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3475.



Material: Drop forged steel. Finish: Self-colored or galvanized. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Ideal for tough rigging jobs which demand maximum strength and durability. See # with No. 3475.



Material: Alloy steel. Machined from alloy steel. Special Feature: Positive grip socket equipped with clevis pin for easy, quick removal and changing of end fittings. Uses: Used for attaching lines to slotted drums. See # with No. 3475.



STEEL BAR STOCK

MALLEABLE IRON

316 STAINLESS STEEL

BRASS

Hose x NPT Size	Part #	Price	Pkg Qty	Size	Part #	Price	Pkg Qty	Size	Part #	Price	Size	Part #	Price
1/4" x 1/8"	MS4X2	\$5.00	25					1/2"	RMS1	\$40.00			
1/4" x 1/4"	MSA	4.15	50					3/4"	RMS6	47.00	3/4"	BMS6	\$13.50
1/4" x 3/8"	MSB	4.23	25					1"	RMS11	47.00	1"	BMS11	25.00
3/8" x 1/4"	MS6X4	4.23	25	1 1/4"	MS16	\$11.00	20	1 1/4"	RMS16	105.00	1 1/4"	BMS16	31.00
3/8" x 3/8"	MSC	3.50	100	1 1/2"	MS21	11.25	10	1 1/2"	RMS21	105.00	1 1/2"	BMS21	42.00
3/8" x 1/2"	MS6X8	4.40	25	2"	MS26	16.80	10	2"	RMS26	110.00	2"	BMS26	58.00
1/2" x 1/4"	MS6X4	4.40	25	2 1/2"	MS31	36.00	5	2 1/2"	RMS31	300.00			
1/2" x 3/8"	MS6X6	4.40	50	3"	MS36	48.00	5	3"	RMS36	390.00			
1/2" x 1/2"	MS7	3.25	25	4"	MS46	84.00	2				3"	BMS36	145.00
1/2" x 3/4"	MS6X12	8.85	25										
3/4" x 1/2"	MS12X8	7.75	25										
3/4" x 3/4"	MS6	5.50	25										
3/4" x 1"	MS12X16	13.50	25										
1" x 3/4"	MS16X12	12.50	25										
1"	MS11	8.70	25										

"LP-BOSS" MALE STEM

Non-porous tubular steel for volatile materials

Plated, Male NPT End
Unplated, Beveled End

Size	Part #	Price	Part #	Price
1 1/4"	LP16	\$35.00		
1 1/2"	LP21	35.00		
2"	LP26	24.00	LPB26	\$28.00
2 1/2"	LP31	80.00		
3"	LP36	80.00	LPB36	82.00
4"	LP46	80.00		
6"	LP66	175.00		

"BOSS" WASHER SEAL

PLATED STEEL &/or MALLEABLE IRON



Coupling With Female Spud

Stem

Wing Nut

Female Spud

Male Spud

Double Spud

Washer*

Sizes	Part #	Price	Part #	Price	Pkg Qty	Part #	Price	Pkg Qty	Part #	Price	Pkg Qty	Part #	Price	Part #	Price	
3/8"	WF3	\$15.00	CA	\$3.50	100	CB	\$8.00	100	CC	\$5.50	100	WMC	\$10.00	DB3	WBC	\$35
1/2"	WF6	12.00	B1	4.25	100	B2	3.75	100	BC	4.00	100	WM3	10.00	DB2	W2	35
3/4"	WF9	19.45	B6	9.00	50	B12	4.70	25	B8	5.75	25	WM8	11.50	DB13	W12	.45
1"	WF96	19.70	B17	8.25	50	B12	4.70	25	B13	5.75	25	WM13	12.00	DB13	W12	.45
1 1/4"	WF51	35.50	B16	12.50	25	B17	3.00	25	B18	18.00	25	WM18	28.00	DB23	W17	.75
1 1/2"	WF61	35.00	B21	12.50	25	B17	3.00	25	B23	14.50	25	WM23	28.00	DB23	W17	.75
2"	WF81	43.00	B26	12.50	10	B27	14.00	10	B28	18.50	10	WM28	32.00	DB28	W27	.85
2 1/2"	WF96	99.00	B31	30.00	5	B32	35.00	5	B33	34.00	5	WM33	60.00	DB33	W32	1.10
3"	WF111	124.00	B38	39.00	5	B37	35.00	5	B38	60.00	5	WM38	60.00	DB38	W37	1.10
4"	WF147	292.00	B46	105.00	5	B47	95.00	5	B48	92.00	5				W47	5.25
5"	WF171	735.00	B56	340.00	2	B67	175.00	2	B58	220.00	2				W67	5.25
6"	WF207	860.00	B66	265.00	2	B67	175.00	2	B68	220.00	2				W67	5.25

* Washer is Klinger nylon, Nitrile bonded, Kevlar reinforced material.

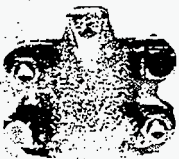
BOSS COUPLINGS

"BOSS" CLAMPS

A



2-BOLT TYPE



4-BOLT TYPE
2 GRIPPING FINGERS



4-BOLT TYPE
4 GRIPPING FINGERS



THREE-PIECE
8-BOLT TYPE

Hose I.D.	Hose O.D.		Plated Malleable Iron			Brass Shipped from factory only.		Stainless Steel Shipped from factory only.		
	From:	To:	Part #	Price	Qty	Part #	Price	Part #	Price	Torque**
1/4"	9/16"	21/32"	BD	\$3.95	100					6
3/8"	11/16"	7/8"	CD	3.95	100					6
1/2"	13/16"	15/16"	DD	3.95	100					6
1/2"	15/16"	1 1/16"	B4	3.95	25	BB4	\$13.00	RB4	\$90.00	12
1/2"	1 1/16"	1 3/16"	B5	4.25	25					12
3/4"	1 5/32"	1 9/32"	BU9S	5.30	50					21
3/4"	1 3/16"	1 5/16"	BU9	5.30	50	BBU9	16.00	RBU9	99.00	21
3/4"	1 5/16"	1 1/2"	B9	5.30	25	BB9	16.00	RB9	99.00	21
3/4"	1 1/2"	1 11/16"	B10	6.00	25					21
1/2"	29/32"	1 1/32"	968	6.00	50					6
1"	1 13/32"	1 5/16"	156*	9.45	20					21
1"	1 17/32"	1 23/32"	BU14	9.45	25	BBU14	45.00	RBU14	126.00	21
1"	1 11/16"	1 15/16"	B14	9.65	25	BB14	49.00	RB14	126.00	21
1"	1 15/16"	2 1/8"	B15	11.00*	20					21
1 1/4"	1 1/2"	1 25/32"	BU18	18.00	10					40
1 1/4"	1 11/16"	1 7/8"	187*	12.50	10					21
1 1/4"	1 25/32"	2 3/32"	BU19	13.00	10					40
1 1/4"	1 7/8"	2 1/16"	206*	13.00	20					21
1 1/4"	2 1/8"	2 3/8"	B19	14.00	10	BB19	49.00	RB19	128.00	40
1 1/2"	1 13/16"	2"	BU22	17.50	10					40
1 1/2"	2"	2 7/32"	B22	18.00	10					40
1 1/2"	2"	2 1/8"	212*	15.00	10					21
1 1/2"	2 1/16"	2 1/4"	225*	16.50	10					40
1 1/2"	2 3/16"	2 3/8"	BU24	15.00	10	BBU24	90.00	RBU24	135.00	40
1 1/2"	2 3/8"	2 9/16"	B24	15.00*	10	BB24	90.00	RB24	135.00	40
2"	2 1/4"	2 3/4"	B25	17.50*	10					40
2"	2 1/4"	2 1/2"	250*	17.50	10					40
2"	2 11/32"	2 17/32"	BU28	18.00	10					60
2"	2 1/2"	2 3/4"	275*	21.50	10					40
2"	2 1/2"	2 25/32"	BU29	17.50	10	BBU29	90.00	RBU29	200.00	60
2"	2 3/4"	3 1/16"	B29	18.50	10	BB29	90.00	RB29	200.00	60
2"	2 3/4"	3 1/16"	306*	24.00	10					60
2"	3 3/32"	3 7/16"	B30	35.50*	5					60
2 1/2"	3 1/16"	3 1/2"	350*	25.00	5					60
2 1/2"	3 3/32"	3 7/16"	BU34	40.00	5					60
2 1/2"	3 1/2"	3 15/16"	B34	40.00	5					150
3"	3 1/2"	3 3/4"	375*	35.50	5					60
3"	3 1/2"	3 15/16"	BU35	63.00	5					150
3"	3 3/4"	4"	401*	57.00	5					150
3"	3 13/16"	4 1/16"	B35	45.00	5					150
3"	4"	4 3/16"	418*	76.50	4					200
3"	4 1/16"	4 7/16"	B39	70.00	5					200
3"	4 3/16"	4 1/2"	450*	70.00	2					200

Hose I.D.	Hose O.D.		Plated Malleable Iron			Qty	Torque**
	From:	To:	Part #	Price			
3"	4 1/4"	4 13/16"	BS39	\$130.00	2	150	
4"	4 5/8"	5"	B45	140.00	2	150	
4"	4 7/8"	5 1/4"	BS49	190.00	2	200	
4"	5 1/4"	5 19/32"	BU49	200.00	2	200	
4"	5 17/32"	5 15/16"	B49	200.00	2	200	
6"	6 7/8"	7 3/8"	750	240.00	1	200	
6"	7 1/2"	8"	850	240.00	1	200	

* 4 Gripping Fingers
 ** Recommended torque rating in FT. LBS.
 (for malleable iron & stainless steel clamps only)

VE 8555 (Pg 574) 0

39496-000

March 30, 1992

Supersedes 39496-000

January 2, 1990

Industrial Hose Products/List Prices

How To Use The Coupling Section

Couplings are designated by Block Numbers, and are described as to construction and source. Adapters and fittings are available for special applications.

Each coupling has a pressure rating of Low, Medium or High. These terms have a general meaning as follows:



Coupling ratings apply only where the recommended clamps or bands are used. Use this information as a guide since different tube stocks affect coupling retention.

Common shank couplings or nipples with bands or clamps should be used only on straight or plain end water or material handling hoses up to the following rated working pressures:

1/2" - 1 1/2" I.D. — 315 psi	
2 - 3" I.D. — 100 psi	6" I.D. — 50 psi
4" I.D. — 75 psi	8" I.D. — 40 psi
5" I.D. — 60 psi	10" I.D. — 30 psi



Listed are all of the clamps, bands and ferrules used to secure the couplings to the hose. The proper clamp size should be selected carefully, since many clamps will fit only one hose size or a narrow range of sizes.

For certain critical fluids, specific couplings are used, as follows:

Critical Applications

- Use only the couplings recommended in this catalog for conveying:
 - Steam
 - LP Gas
 - Corrosive Chemicals
 - Petroleum Products
- For any high temperature application, use only interlocking type couplings — 14 or 15.
- For conveying flammable fluids, use couplings made of nonsparking materials, such as brass or aluminum.
- For ground fueling of aircraft, use certified coupled assemblies only.



- If more than one coupling is recommended, make a final selection for type of coupling on basis of cost, sizes available, thread type available and application.
- Determine how the coupling is to be fastened to the hose—ferrule, band or clamp.

Due to continual product improvements, Gates reserves the right to alter specs and prices without prior notice.



Industrial Hose Products/List Prices

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January 2, 1990

Coupling Installation

Before making any hose assembly, check over these general instructions and precautions:

1. **Square Ends**—Before attaching a coupling to the hose, make sure the end is cut square. This will help prevent leakage of the coupling.
2. **Proper Fit of Ferrules and Clamps**—This is important. Make sure you are using the correct sizes for the hose being coupled. Never buff the cover, except when specified for certain hydraulic hose couplings, and you should not enlarge the hose tube in any way to make the coupling insert fit. Occasionally, a nipple pusher tool will be helpful in pushing the insert into the hose.
3. **Installation Lubricants**—Use of lubricants on coupling inserts and ferrules will simplify coupling installation and reduce possibility of damage to the hose. Some lubricants to use are water, soap solution or lactol. Or, you can use a solution of glycerin and alcohol made up of one (1) part glycerin and 15 parts wood alcohol by volume.
4. **Appearance of the Hose Assembly**—A properly installed coupling significantly improves customer acceptance of the product. It also provides added assurance of satisfactory service and safety of assembly.
5. **Seal the Ends**—Where the hose is intended for use in petroleum transfer applications and hose ends will be exposed after couplings are attached, apply a sufficient amount of quality Neoprene cement or shellac to seal the ends.
6. **Critical Applications.**
 - (a) Use only the couplings recommended on pages 134-140 for conveying:
 - Steam
 - LP Gas
 - Corrosive Chemicals
 - Petroleum Products
 - Edible Products
 - (b) For any high temperature applications, use only interlocking type couplings—
14 or 15.
 - (c) For conveying flammable fluids, use couplings made of nonsparking materials; i.e., brass or aluminum.
 - (d) For aircraft ground fueling hose, use factory coupled assemblies only.



Due to continual product improvements, Gates reserves the right to alter specs and prices without prior notice.



The world's most trusted name in belts and hose.

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39496-000


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**Industrial Hose
Products/List Prices**


Couplings for Gates Hoses

1 Long Shank 

Description: Cast brass, serrated shank, set has hex male and hex swivel female couplings, with a washer seal. Threads are GHT or NPSM.
Sources: Dixon Valve & Coupling Co.
John W. Moon, Inc.
H.B. Sherman Mfg. Co.
Seal-Fast Bobo, Inc.

2 Lightweight Water Hose 


Description: Spun or wrought brass serrated shank, round swivel female and solid male, both GHT, washer seal.
Sources: American Coupling Co.
L.R. Nelson Corp.
H.B. Sherman Mfg. Co.

3 Cast Brass Short Shank 


Description: Cast brass, serrated shank, hex swivel female with washer, solid male, GHT threads, 3/4-11 1/2.
Sources: J.C. Gadd Co.
John W. Moon, Inc.
H.B. Sherman Mfg. Co.
Seal-Fast Bobo, Inc.
Stayput Clamp & Coupling Co.

4 Machined Brass Short Shank 

Description: Machined brass, serrated shank. Each set has round swivel female and solid male, GHT, washer seal. Octagon nut male and female.
Sources: American Coupling Co.
Anchor-Lenz Div.
Royal Brass, Inc.
H.B. Sherman Mfg. Co.

5 Brass Pin-Lug 

Description: Cast brass shank and swivel, shank is serrated, NPSM threads in female and male, with washer seal. Sizes under 3" have lugs on female only.
Sources: John W. Moon, Inc.
Dixon Valve & Coupling Co.

6 Malleable Iron Pin Lug 

Description: Malleable iron shank and swivel with serrated shank, cadmium plated, NPSM threads in male and female, washer seal. Pin lugs on female only.
Sources: Dixon Valve & Coupling Co.
H.B. Sherman Mfg. Co.
Seal-Fast Bobo, Inc.

7 Combination Nipple 




Victaulic Nipple

Description: Swaged steel with scored or serrated shank, NPT threads same nominal size as I.D. of hose. Special plastic materials available.
Sources: Band-It
Dixon Valve & Coupling Co.
Marin Brass Works, Inc.
Sorew Machine Products Co.
Seal-Fast Bobo, Inc.
Stayput Clamp & Coupling Co.

8 Tri-Lokt 

Description: Machined steel insert, male NPT threads only. Held in place with steel yoke and Band-It Jr. clamps. Insert is reusable—can be reinstalled with new yoke and clamps.
Source: Band-It

10 Single or Double Bolt Malleable Iron Clamp 


Description: Cast malleable iron, cadmium plated.
Source: Dixon Valve & Coupling Co.

14 Interlocking, Ground Joint 

Boss Coupling

Part # S650 BU49 clamp

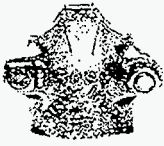

Description: Malleable iron swivel. Inserts and spud may be either steel or malleable iron. All parts are cadmium plated. Male and female threads both NPT with same nominal size as hose I.D. Ground joint between spud and female insert. Available with 2- and 4-bolt clamps.
Sources: Dixon Valve & Coupling Co.
Seal-Fast Bobo, Inc.


15 Interlocking, Washer Joint 

Description: Malleable iron swivel. Insert and spud may be either malleable iron or steel. All parts are cadmium plated. Female thread in spud is NPT with same nominal size as hose I.D. Washer joint between insert and spud. Available with 2- and 4-bolt clamps.
Source: Dixon Valve & Coupling Co.

Due to continual product improvements, Gates reserves the right to alter specs and prices without prior notice.



Hose I.D.		Hose O.D.		Plated Malleable Iron			Brass Shipped from factory only.		Stainless Steel Shipped from factory only.		
Size	From	To	Part #	Price	Qty	Part #	Price	Part #	Price	Torque**	
											
2-BOLT TYPE											
1/4"	9/16"	21/32"	BD	\$3.95	100					6	
3/8"	1 1/16"	7/8"	CD	3.95	100					6	
1/2"	1 3/16"	15/16"	DD	3.95	100					6	
1/2"	1 5/16"	1 1/16"	B4	3.95	25	BB4	\$13.00	RB4	\$90.00	12	
1/2"	1 1/16"	1 3/16"	B5	4.25	25					12	
3/4"	1 5/32"	1 9/32"	BUSS	5.30	50					21	
3/4"	1 3/16"	1 5/16"	BU9	5.30	50	BBU9	16.00	RBU9	99.00	21	
3/4"	1 5/16"	1 1/2"	B9	5.30	25	BB9	16.00	RB9	99.00	21	
3/4"	1 1/2"	1 11/16"	B10	6.00	25					21	
											
4-BOLT TYPE 2 GRIPPING FINGERS											
1/2"	29/32"	1 1/32"	968	6.00	50					6	
1"	1 13/32"	1 9/16"	156*	9.45	20					21	
1"	1 17/32"	1 23/32"	BU14	9.45	25	BBU14	45.00	RBU14	126.00	21	
1"	1 11/16"	1 15/16"	B14	9.65	25	BB14	49.00	RB14	126.00	21	
1"	1 15/16"	2 1/8"	B15	11.00	20					21	
1 1/4"	1 1/2"	1 25/32"	BU18	18.00	10					40	
1 1/4"	1 11/16"	1 7/8"	187*	12.50	10					21	
1 1/4"	1 25/32"	2 3/32"	BU19	18.00	10					40	
1 1/4"	1 7/8"	2 1/16"	206*	13.00	20					21	
1 1/4"	1 5/8"	2 3/8"	B19	14.00	10	BB19	49.00	RB19	128.00	40	
1 1/2"	1 13/16"	2"	BU22	17.50	10					40	
1 1/2"	2"	2 7/32"	B22	18.00	10					40	
1 1/2"	2"	2 1/8"	212*	15.00	10					21	
1 1/2"	2 1/16"	2 1/4"	225*	16.50	10					40	
1 1/2"	2 3/16"	2 3/8"	BU24	15.00	10	BBU24	90.00	RBU24	135.00	40	
1 1/2"	2 3/8"	2 9/16"	B24	15.00	10	BB24	90.00	RB24	135.00	40	
1 1/2"	2 9/16"	2 3/4"	B25	17.50	10					40	
2"	2 1/4"	2 1/2"	250*	17.50	10					40	
2"	2 11/32"	2 17/32"	BU28	17.50	10					60	
2"	2 1/2"	2 3/4"	275*	21.50	10					40	
2"	2 1/2"	2 25/32"	BU29	17.50	10	BBU29	90.00	RBU29	200.00	60	
2"	2 3/4"	3 1/16"	B29	17.50	10	BB29	90.00	RB29	200.00	60	
2"	2 3/4"	3 1/16"	306*	24.00	10					60	
2"	3 3/32"	3 7/16"	B30	35.50	5					60	
2 1/2"	3 1/16"	3 1/2"	350*	25.00	5					60	
2 1/2"	3 3/32"	3 7/16"	BU34	40.00	5					60	
2 1/2"	3 1/2"	3 15/16"	B34	40.00	5					150	
3"	3 1/2"	3 3/4"	375*	35.50	5					60	
3"	3 1/2"	3 15/16"	BU35	63.00	5					150	
3"	3 3/4"	4"	401*	57.00	5					150	
3"	3 13/16"	4 1/16"	B35	45.00	5					150	
3"	4"	4 3/16"	418*	76.50	4					200	
3"	4 1/16"	4 7/16"	B39	70.00	5					200	
3"	4 3/16"	4 1/2"	450*	70.00	2					200	

Hose I.D.		Hose O.D.		Plated Malleable Iron			* 4 Gripping Fingers ** Recommended torque rating in FT. LBS. (for malleable iron & stainless steel clamps only)	
Size	From	To	Part #	Price	Qty	Torque**		
								
THREE-PIECE 6-BOLT TYPE								
3"	4 1/4"	4 13/16"	BS39	\$130.00	2	150		
4"	4 5/8"	5"	B45	140.00	2	150		
4"	4 7/8"	5 1/4"	BS49	190.00	2	200		
4"	5 1/4"	5 19/32"	BU49	200.00	2	200		
4"	5 17/32"	5 15/16"	B49	200.00	2	200		
6"	6 7/8"	7 3/8"	750	240.00	1	200		
6"	7 1/2"	8"	850	240.00	1	200		

GF-141 (4")
WF-141 (4")

FAX MEMO

#PAGES 3 DATE 7/16 FAX# _____

CO. _____

ATTN: Mike Custer

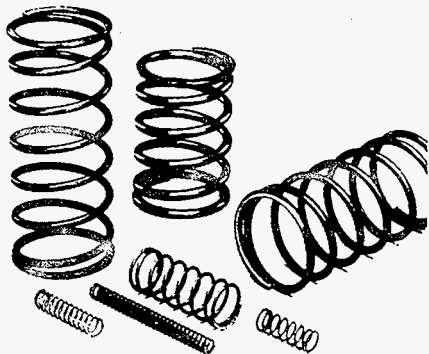
FROM: AMERICAN HOSE & FITTINGS

PH# (206) 872-8080 FAX# (206) 395-3410

J-25-05



Compression Springs



Stock sizes in music wire and stainless steel

Associated Spring offers a broad variety of helical compression springs in the SPEC selection. They are reliable, inexpensive and efficient — the right combination for general-purpose use throughout industry.

COMPRESSION
SPRINGS

Material

Music wire

ASTM-A228 or AMS 5112

Stainless steel

Commercial Type 302, ASTM-A313 or
AMS 5688 spring temper. (chemical & physical only)

No charge for certificate of compliance when requested; certificate of chemical analysis available, see price book.

Music wire will be furnished unless stainless steel is specified. When inquiring or ordering, use letter "M" or letter "S" as suffix on catalog numbers to designate music wire or stainless steel wire, respectively.

Music wire springs are not recommended for applications where the temperature exceeds 250 deg F (121 deg C). Stainless steel springs are not recommended for applications where the temperature exceeds 500 deg F (260 deg C).

Direction of Helix

Right hand.

Ends

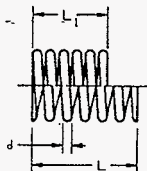
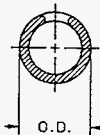
Squared and ground. Ends to be square within 3° with axis
O.D. sizes 0.057-0.088 in (1.45-2.24 mm) squared ends
not ground.

Free length L is for reference use only. Load P is attained
at length L₁. For stainless steel multiply P by 0.833.

Load values shown are for music wire.

For normal service, springs should not be compressed
below L₁.

To determine load P at any length other than L₁, multiply
the proposed deflection by the rate R. * $[P + (L-L_1) \times R]$
When stainless steel is used the value for rate R must
be corrected by multiplying R by 0.833.



Finishes

Standard finish is that of the normal wire. Shot-peened and
plated finishes furnished on request. Allow sufficient
additional time for special finishes.

Tolerances

O.D. (English)	O.D. (Metric)
0.057 to 0.118 in ± 0.003 in	1.45 to 3.02 mm ± 0.08 mm
0.120 to 0.240 in ± 0.005 in	3.05 to 6.10 mm ± 0.13 mm
0.241 to 0.500 in ± 0.008 in	6.12 to 12.70 mm ± 0.20 mm
0.501 to 1.000 in ± 0.015 in	12.73 to 25.40 mm ± 0.38 mm
1.001 to 1.225 in ± 0.020 in	25.43 to 31.12 mm ± 0.51 mm
1.226 to 1.480 in ± 0.030 in	31.14 to 37.08 mm ± 0.76 mm
1.481 to 2.000 in ± 0.040 in	37.11 to 50.80 mm ± 1.02 mm

STOCK COMPRESSION SPRINGS

Music Wire and Stainless Steel



Associated Spring
Raymond BARNES

CATALOG NUMBER*	Outside Diameter		Wire Diameter		Free Length L ₁ , Approx.		Load, P at L ₁		Length, L ₁		Solid Height, Approx.		Spring Rate, R			
	in	mm	in	mm	in	mm	lb†	N†	in	mm	in	mm	lb/in†	N/mm†		
C0600-063-2000	0.600	15.24	0.063	1.60	2.00	50.80	18.00	80.07	1.059	26.90	0.700	17.78	18.0	3.15		
C0600-063-2250					2.25	57.15			1.184	30.07	0.768	19.51	15.9	2.78		
C0600-063-2500					2.50	63.50			1.308	33.22	0.836	21.23	14.2	2.49		
C0600-063-2750					2.75	69.85			1.433	36.40	0.863	21.92	13.7	2.40		
C0600-063-3000					3.00	76.20			1.558	39.57	0.927	23.55	12.5	2.19		
C0600-063-3500			3.50	88.90	1.807	45.90	1.055	26.80	10.6	1.856						
C0600-067-0625			0.067	1.70			0.62	15.75	21.00	93.41	0.360	9.14	0.300	7.62	80.0	14.01
C0600-067-0750							0.75	19.05			0.430	10.92	0.336	8.53	66.0	11.56
C0600-067-0880							0.88	22.35			0.455	11.56	0.401	10.19	50.0	8.76
C0600-067-1000							1.00	25.40			0.530	13.46	0.430	10.92	45.0	7.88
C0600-067-1250							1.25	31.75			0.665	16.89	0.505	12.83	36.0	6.30
C0600-067-1500			1.50	38.10	0.780	19.81	0.594	15.09	29.0	5.08						
C0600-067-1750			1.75	44.45	0.830	21.08	0.715	18.16	23.0	4.03						
C0600-067-2000			2.00	50.80	1.106	28.09	0.771	19.58	22.5	3.94						
C0600-067-2250			2.25	57.15	1.236	31.39	0.847	21.51	19.8	3.47						
C0600-067-2500			2.50	63.50	1.366	34.70	0.923	23.44	17.7	3.10						
C0600-067-2750			2.75	69.85	1.496	38.00	0.966	24.54	16.7	2.92						
C0600-067-3000			3.00	76.20	1.626	41.30	1.040	26.42	15.3	2.68						
C0600-072-0620			0.600	15.24	0.072	1.83	0.62	15.75	24.00	106.76	0.405	10.29	0.381	9.68	114.5	20.05
C0600-072-0750							0.75	19.05			0.445	11.30	0.396	10.06	78.0	13.66
C0600-072-0880	0.88	22.35					0.520	13.21			0.433	11.00	68.0	11.91		
C0600-072-1000	1.00	25.40					0.565	14.35			0.502	12.75	55.0	9.83		
C0600-072-1250	1.25	31.75					0.710	18.03			0.581	14.76	45.0	7.88		
C0600-072-1500	1.50	38.10			0.830	21.08	0.691	17.55	36.0	6.30						
C0600-072-1750	1.75	44.45			0.950	24.13	0.801	20.35	30.0	5.25						
C0600-072-2000	2.00	50.80			1.140	28.98	0.848	21.54	28.0	4.90						
C0600-072-2250	2.25	57.15			1.301	33.05	0.946	24.03	25.9	4.54						
C0600-072-2500	2.50	63.50			1.439	36.53	1.033	26.24	23.1	4.05						
C0600-072-2750	2.75	69.85			1.601	40.67	1.119	28.42	20.9	3.66						
C0600-072-3000	3.00	76.20			1.742	44.25	1.206	30.63	19.1	3.24						
C0600-081-0620	0.600	15.24			0.081	2.06	0.62	15.75	32.69	145.41	0.466	11.84	0.412	10.46	212.6	37.33
C0600-081-0750							0.75	19.05			0.553	14.05	0.459	11.66	165.6	29.00
C0600-081-0880							0.88	22.35			0.639	16.23	0.507	12.88	135.6	23.74
C0600-081-1000							1.00	25.40			0.719	18.26	0.552	14.02	116.2	20.35
C0600-081-1250							1.25	31.75			0.885	22.43	0.644	16.36	89.5	15.67
C0600-081-1500					1.50	38.10	1.051	26.70	0.736	18.69	72.8	12.75				
C0600-081-1750					1.75	44.45	1.217	30.91	0.828	21.03	61.3	10.73				
C0600-081-2000					2.00	50.80	1.383	35.13	0.920	23.37	53.0	9.28				
C0600-081-2250			2.25	57.15	1.549	39.34	1.012	25.70	46.6	8.16						
C0600-081-2500			2.50	63.50	1.715	43.56	1.104	28.04	41.7	7.30						
C0600-081-2750			2.75	69.85	1.881	47.78	1.196	30.38	37.6	6.58						
C0600-081-3000			3.00	76.20	2.047	51.99	1.288	32.72	34.3	6.01						
C0600-081-3250			3.25	82.55	2.213	56.21	1.380	35.05	31.5	5.52						
C0600-081-3500			3.50	88.90	2.379	60.43	1.472	37.39	29.2	5.11						
C0600-081-3750			3.75	95.25	2.545	64.64	1.564	39.73	27.1	4.75						
C0600-081-4000			4.00	101.60	2.711	68.86	1.656	42.06	25.4	4.45						
C0600-085-0620			0.600	15.24	0.085	2.16	0.62	15.75	37.62	167.33	0.477	12.12	0.433	11.00	262.5	45.96
C0600-085-0750							0.75	19.05			0.565	14.35	0.484	12.29	203.6	35.65
C0600-085-0880							0.88	22.35			0.654	16.61	0.536	13.61	166.4	29.14
C0600-085-1000							1.00	25.40			0.736	18.69	0.583	14.81	142.3	24.92
C0600-085-1250	1.25	31.75					0.906	23.01			0.682	17.32	109.4	19.16		
C0600-085-1500	1.50	38.10			1.076	27.33	0.781	19.84	88.8	15.55						
C0600-085-1750	1.75	44.45			1.247	31.67	0.880	22.35	74.8	13.10						
C0600-085-2000	2.00	50.80			1.417	35.99	0.978	24.84	64.5	11.29						
C0600-085-2250	2.25	57.15			1.588	40.34	1.077	27.36	56.8	9.95						
C0600-085-2500	2.50	63.50			1.758	44.65	1.176	29.87	50.7	8.88						
C0600-085-2750	2.75	69.85			1.928	48.97	1.275	32.39	45.8	8.02						
C0600-085-3000	3.00	76.20			2.099	53.31	1.374	34.90	41.7	7.30						
C0600-085-3250	3.25	82.55			2.269	57.63	1.473	37.41	38.3	6.71						
C0600-085-3500	3.50	88.90			2.439	61.95	1.572	39.93	35.5	6.22						
C0600-085-3750	3.75	95.25			2.610	66.29	1.670	42.42	33.0	5.78						
C0600-085-4000	4.00	101.60			2.780	70.61	1.769	44.93	30.8	5.39						
C0600-092-0750	0.092	2.34					0.75	19.05	46.46	206.65	0.591	15.01	0.524	13.31	291.5	51.04
C0600-092-0880							0.88	22.35			0.684	17.37	0.611	14.77	237.0	42.50
C0600-092-1000							1.00	25.40			0.770	19.56	0.633	16.08	202.2	35.41

*For stainless steel, multiply values by 0.833.

†When inquiring or ordering, use letter "M" or letter "S" as suffix on catalog numbers to designate music wire or stainless-steel wire, respectively.

COMPRESSION SPRINGS

T-M MANUFACTURING CO

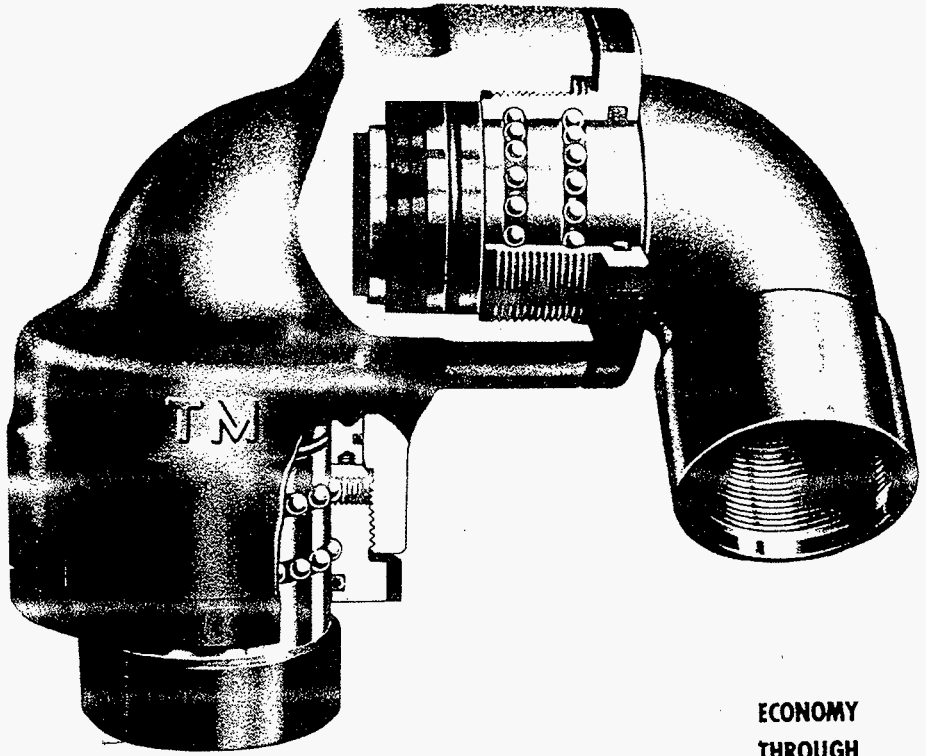
PHONE (408) 736-5202

53E D ■ 8822969 0000099 485 ■

VE 7954

NPT (NATIONAL PIPE THREAD)
(SINGLE POINT THREAD)

E-21-15



ECONOMY
THROUGH
QUALITY

T-M

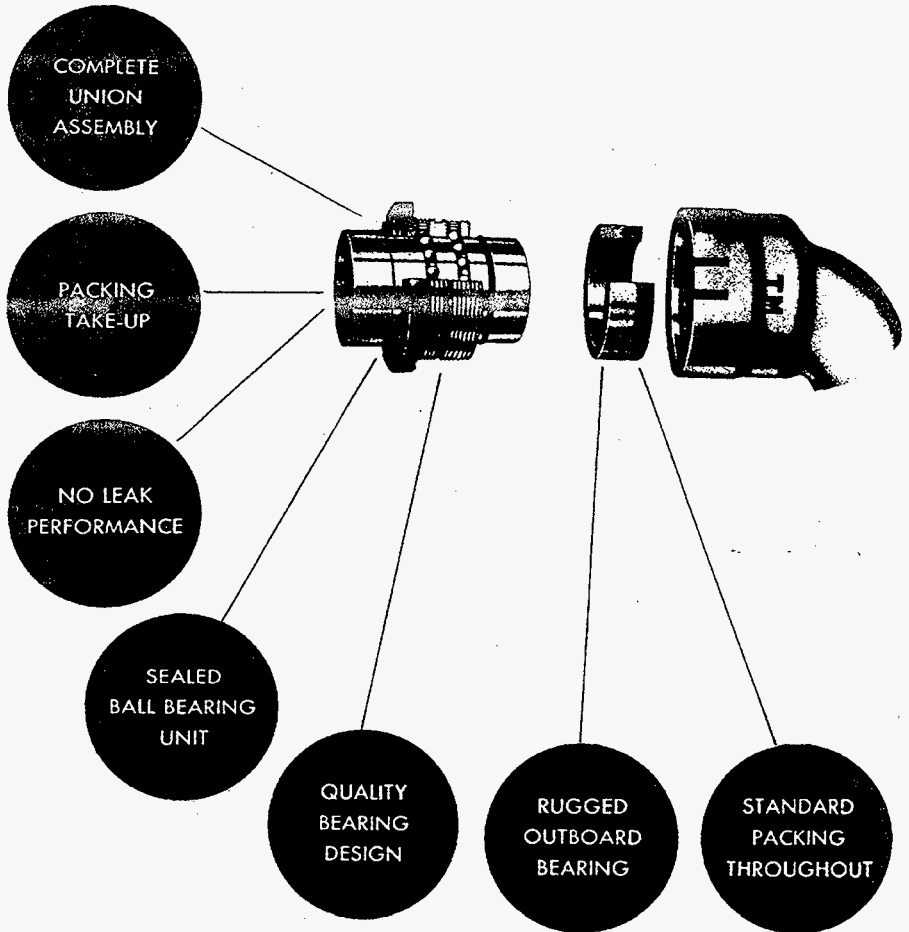
SWIVEL JOINTS

T-M MFG. CO. 695 E. ARQUES AVE., SUNNYVALE, CALIF. 94086-3832

A

T-M

SWIVEL JOINTS



B

COMPLETE UNION ASSEMBLY

Repack anywhere, anytime, without handling ball bearings, even when welded into the line. Reduces service and maintenance to an absolute minimum.

PACKING TAKE-UP

Chevron packing compensates for normal wear, varying temperatures, extremely volatile materials. Extends packing life as much as 400 %.

NO - LEAK PERFORMANCE

Superior engineering design enable guaranteed, no-leak performance from gases to acids to drilling cement.

SEALED BALL BEARING UNIT

Factory assembled rotating unit sealed from outside contaminants and line fluids by "o" rings. Lubricants are sealed in and foreign matter sealed out. Reduces maintenance costs by eliminating daily lubrication. Quality design will cut costs, extend life of joints.

QUALITY BEARING DESIGN

The finest alloy steels used in manufacture of T-M Joints. Through hardened ball races in 6000 and 15,000 steel series coupled with hardened chrome alloy steel balls assure longer life, lower maintenance costs and fewer replacements.

RUGGED OUTBOARD BEARING

The packing backup ring acts as a solid bearing during severe or shock loads. Protects the ball races from brinnelling. Eliminates misalignment leaks. Guarantees top performance under all conditions.

STANDARD PACKING THROUGHOUT

No special packing worries with T-M. Packing for all services is available in standard sizes and materials.



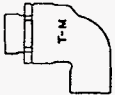
SWIVEL JOINTS

E-21-15
STYLES

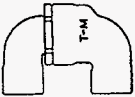
No. 100
STRAIGHT SINGLE SWIVEL



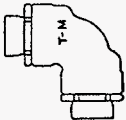
No. 200
ELL SINGLE SWIVEL



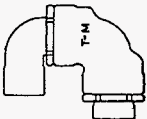
No. 300
2 ELL SINGLE SWIVEL



No. 400
1 ELL DOUBLE SWIVEL



No. 500
2 ELL DOUBLE SWIVEL



MATERIALS

125 PSI Aluminum Alloy
Color Code: Aluminum

1000 PSI Steel
Color Code: Blue

6000 PSI Steel
Color Code: Red

15,000 PSI Steel
Color Code: Yellow

125 PSI Type 316 Stainless Steel
Color Code: #316

Special Alloys
Color Code by Type

PSI = Maximum Working Pressure

TEMPERATURES

225° Hycar Packing
450° Aramid Fiber
550° Aramid Fiber
300° Teflon
400° Viton

SIZES

$\frac{3}{8}$ " - Thru 16" Pipe Size
Larger on request

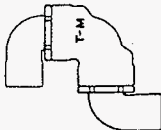
END CONNECTIONS

Threaded Ends (TE) (Female)
Beveled for Weld (BV) (Beveled)
Bored for Weld (BR) (Bored)
Flanged
Special --- On Order

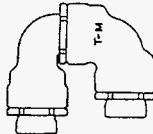
SPECIAL ORDERS

We will be pleased to submit prices on your special requirements. Manufactured on Order.

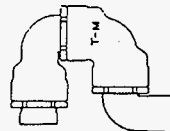
No. 600
3 ELL DOUBLE SWIVEL



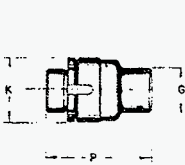
No. 700
2 ELL TRIPLE SWIVEL



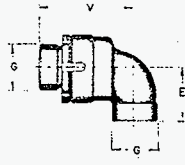
No. 800
3 ELL TRIPLE SWIVEL



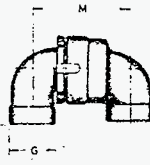
THREADED AND BORED FOR WELDING



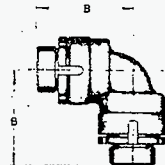
STYLE 100



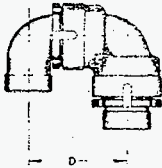
STYLE 200



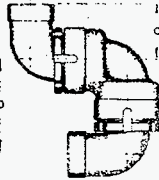
STYLE 300



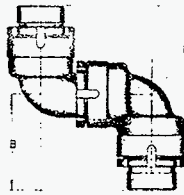
STYLE 400



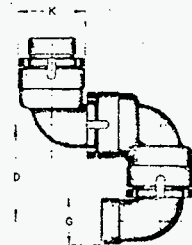
STYLE 500



STYLE 600



STYLE 700



STYLE 800

D I M E N S I O N S

SIZES	B	C	D	E	G	K	M	N	P	R	V	SIZE
3/8 & 1/2	3 1/2	2	3 3/8	1 1/8	1 1/4	2	3	3 3/8	3 3/8	3 3/8	3 3/8	3/8 & 1/2
3/4 & 1	4 1/4	2 3/8	4 1/4	2 1/4	1 3/4	2 7/8	4 1/8	4 1/8	4 3/8	5	4 1/8	3/4 & 1
1 1/4	4 3/4	2 3/4	4 3/4	2 3/4	2 1/8	3 1/8	4 3/8	4 3/8	5 1/8	5 1/8	5 1/4	1 1/4
1 1/2	5 1/2	3	6	3 1/4	2 1/2	3 1/2	6	5 1/4	5 3/4	5 3/8	5 1/8	1 1/2
2	6 3/8	3 1/2	6 3/8	4 1/4	3 3/8	4 3/8	6 3/8	6	6 3/8	7 3/8	6 3/4	2
2 1/2	6 7/8	4	7 3/8	4	3 3/4	5 1/4	6 7/8	6 3/8	6 7/8	8 1/4	6 3/8	2 1/2
3	7 3/4	4 3/8	9 1/8	4 1/4	4 1/4	5 7/8	8 3/8	7 1/4	7 3/8	10 1/4	7	3
4	9 1/4	5 1/2	10 1/8	5 1/2	5 1/2	7 1/4	10 3/8	9 1/4	8 3/4	11 1/2	8 3/8	4

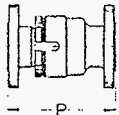
All dimensions given in inches $\pm 1/16$ "

For sizes larger than 4" refer to section on fabricated joints.

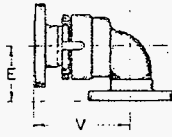


SWIVEL JOINTS

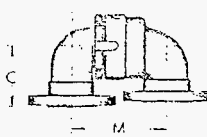
FLANGED (SERIES 15)



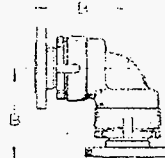
STYLE 100



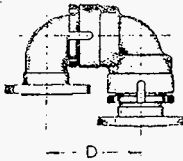
STYLE 200



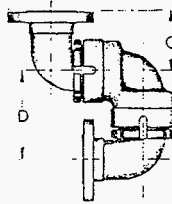
STYLE 300



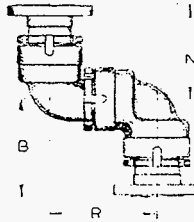
STYLE 400



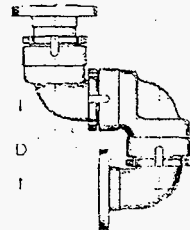
STYLE 500



STYLE 600

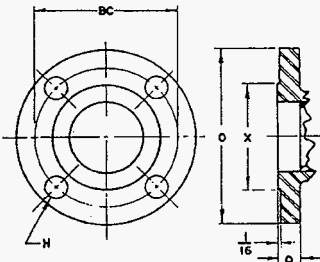


STYLE 700



STYLE 800

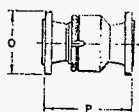
JOINT DIMENSIONS										FLANGE DIMENSIONS							
SIZES	B	C	D	E	M	N	P	R	V	No. Holes	BC	H	Bolt Size	O	Q	X	SIZES
2	6½	4	6½	4¾	6½	6½	7¾	7¾	6¾	4	4¾	¾	¾	6	¾	3¾	2
2½	7½	4½	7¾	4¾	6¾	7	8½	8¼	7	4	5½	¾	¾	7	¾	4¾	2½
3	8½	5½	9¾	4¾	8¾	7¾	8¾	10½	7¾	4	6	¾	¾	7½	1¾	5	3
4	9¾	6¾	10¾	6¾	10¾	10	10¾	11¾	9¾	8	7½	¾	¾	9	1¾	6¾	4



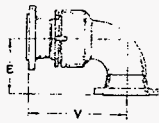
All dimensions given in inches ±1/8"

For sizes larger than 4" refer to section on fabricated joints.

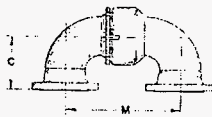
For dimensions of other series flanged joints consult factory.

E-21-15
FABRICATED

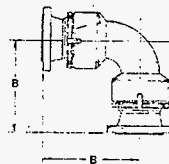
STYLE 100



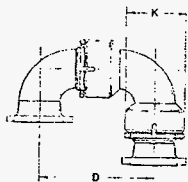
STYLE 200



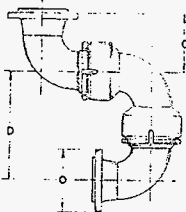
STYLE 300



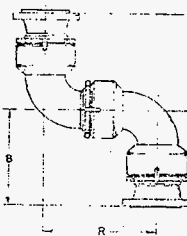
STYLE 400



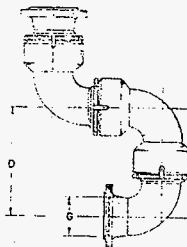
STYLE 500



STYLE 600



STYLE 700



STYLE 800

D I M E N S I O N S




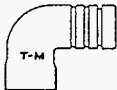



SIZES		B	C	D	E	G	K	M	H	O	P	R	V	SIZES
BEVELED WELDING	6	13½	6	19½	6	—	9%	19¾	13½	—	7¼	19¾	13½	6
	8	16	8	24¼	8	—	12	24¼	16	—	7¾	24¼	16	8
	10	18	10	28¾	10	—	14	28¾	18	—	7¾	28¾	18	10
	12	20	12	32¾	12	—	16	32¾	20	—	7¾	32¾	20	12
FLANGED (SERIES 15)	6	16	8¾	19	8¾	7½	9%	19%	16¼	—	12¾	19¾	16¼	6
	8	19	11	24¼	11	9%	12	24¼	19	—	13¾	24¼	19	8
	8	17¼	9%	19½	9%	—	9%	19%	17	11	14¾	19¾	17¼	6
	8	20¼	12¼	24¼	12¼	—	12	24¼	20¼	13½	16¼	24¼	20¼	8
	10	22¼	14¼	28¾	14¼	—	14	28¾	22¼	16	16¼	28¾	22¼	10
	12	24¼	16¼	33¼	16¼	—	16	32¼	24¼	19	17¼	32¼	24¼	12

All dimensions given in inches ±1/16"
For dimensions of other series
flanged joints consult factory.



SWIVEL JOINTS

E-21-15
REPLACEMENT PARTS

<p>Part 1 STRAIGHT SWIVEL NIPPLE</p> 	<p>Part 2 HOLDING NUT</p> 	<p>Part 3 STRAIGHT ONE SWIVEL BODY</p> 	<p>Part 4 ELL SWIVEL NIPPLE</p> 
<p>Part 5 ELL TWO SWIVEL BODY</p> 	<p>Part 6 ELL ONE SWIVEL BODY</p> 	<p>Part 7 ELL TWO SWIVEL NIPPLE</p> 	<p>ACCESSORIES:</p> <p>Ball Bearings Lock Bar O-Ring - V-Ring Adaptors (Upper and Lower) Counterbalance Lug; designated by "X" after part number.</p>

SERVICING

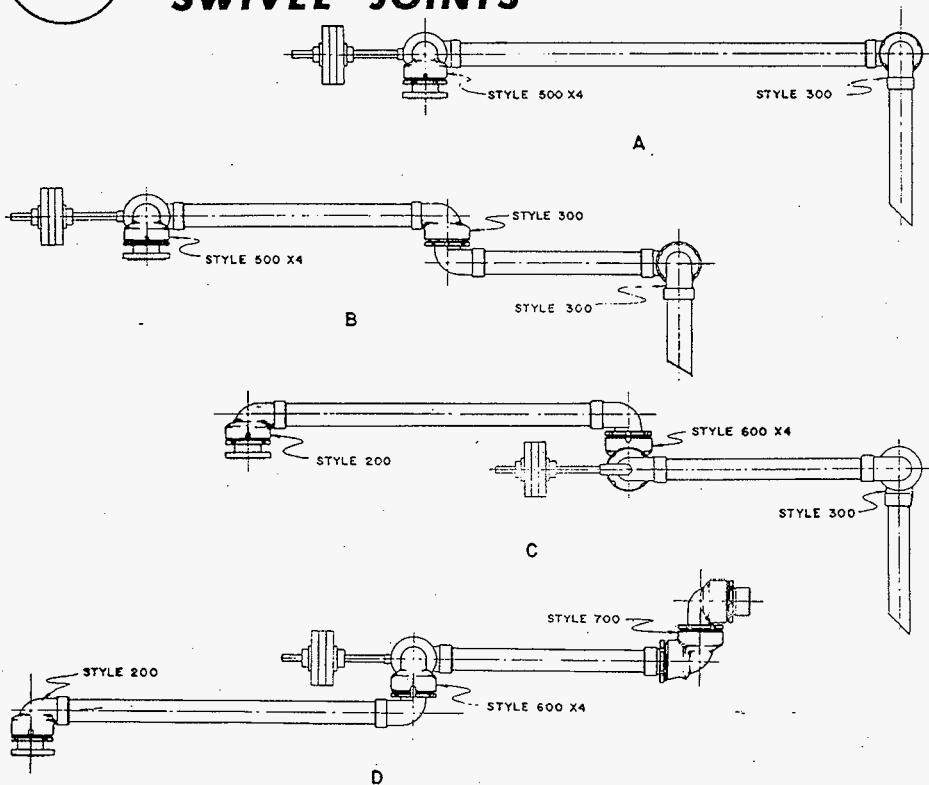
- (1) T-M Swivel Joints are greased at the factory and require only occasional greasing unless joint is in constant rotation or high temperature service. When greasing, remove slotted plug on outside of body, and insert alemite fitting.
- (2) Provision has been made for take-up on the packing. After packing wears, remove lock bar, turn nut clockwise to next locking slot and reinstall lock bar. Process may be repeated until packing has worn too badly to effect seal.

REPACKING

- (1) Remove lock bar.
- (2) Remove nut and nipple assembly from body by turning nut counter clockwise.
- (3) Remove back-up ring from body.
- (4) Remove 2 packing rings.
- (5) Remove male adaptor.
- (6) Clean packing chamber and reinstall (1) male adaptor (2) new packing and (3) back-up ring. Grease all parts thoroughly.
- (7) Insert nut and nipple assembly in body and reassemble. Rotate nipple occasionally as the nipple passes through the packing.
- (8) Screw nut clockwise until compression on packing causes nipple to rotate with a mild amount of torque.
- (9) You do not need to handle the balls or disassemble the nut and nipple to repack the T-M Swivel Joint.



SWIVEL JOINTS



T-M Swivel Joints assembled into the 4 basic tank car and tank truck designs will enable you to transfer your product with economy and safety.

Even when welded into the line T-M Swivel Joints can be taken up to compensate for packing wear or separated for repacking. Overhung loads do not cause leaks on T-M loading arms due to the rugged design.

Furnished in aluminum, steel and stainless steel and packed to fit your individual requirements. Other designs on request.

COUNTER BALANCE DESIGN

Since counterbalance design necessarily varies with each installation, we have shown no standard design.

Please consult the factory for prices and furnish the following information (1) Clearance restrictions and (2) Overhung loading.

PACKING AND COMPONENT PARTS

SWIVEL JOINT SIZE	3/8" & 1/2"	3/4" & 1"	1 1/4"	1 1/2"	2"	2 1/2"	3"	4"	6"	8"	10"	12"
SIZE 6225 CHEVRON PACKING	-20	-28	-30	-32	-39	-44	-49	-54	-63	-68	-72	-76
SIZE 6229 MALE ADAPTOR	-20	-28	-30	-32	-39	-44	-49	-54	-63	-68	-72	-76
FEMALE ADAPTOR p/n 8	50-8	100-8	125-8	150-8	200-8	250-8	300-8	400-8	600-8	800-8	1000-8	1200-8
*SIZE 6227 AND 6230 "O" RING p/n-1	7-15	7-25	0-01	0-03	0-09	0-14	0-18	0-28	0-40	30-47	7-77	7-81
*SIZE 6227 AND 6230 "O" RING p/n 2	7-19	0-01	0-03	0-05	0-11	0-16	0-20	0-30	0-41	30-49	7-79	7-83
BALL SIZE	3/16	1/4	1/4	5/16	5/16	3/8	3/8	7/16	9/16	9/16	9/16	9/16
NUMBER OF BALLS PER ROTATING ELEMENT	32	40	46	42	56	58	66	74	80	104	126	148
SIZE DESIGNATION LOCK BAR p/n 10	1/4	1/4	1/4	3/8	3/8	3/8	3/8	3/8	1/2	1/2	1/2	1/2
SIZE DESIGNATION BALL PLUGS	1/4	5/16	5/16	3/8	3/8	7/16	7/16	1/2	5/8	5/8	5/8	5/8

* ALL "O" RINGS PRECEDED BY 7 INDICATES 6227 SERIES; PRECEDED BY O INDICATES 6230 SERIES.

STANDARD AND OPTIONAL MATERIALS

CHEVRON PACKING

Standard Hycar: To 225°F working temperature
 Optional Teflon: To + 300°F Acid Services
 Optional Aramid Fiber: To 450°F working temp.
 Optional Aramid Fiber: To 550°F working temp.
 Optional Viton: To 400°F working temperature

MALE ADAPTOR

Standard Buna N: To 225°F working temperature
 Optional Aluminum: Acid and corrosive services
 Optional Teflon: Acid Services
 Optional Stainless Steel: Alloy to suit application
 Optional Hi-Temp: To 550°F working temperature

FEMALE ADAPTOR

Standard Steel: To 550°F working temperature
 Optional Aluminum: Acid and corrosive services
 Optional Stainless Steel: Alloy to suit application

ALL "O" RINGS

Standard Buna-N: To 225°F working temperature
 Optional Teflon: Acid Services
 Optional Silicon: To 450°F working temperature
 Optional Viton: To 450°F working temperature

DISTRIBUTION SHEET

To	From	Page 1 of 1			
Distribution	JW Bailey	Date 23 July 1998			
Project Title/Work Order		EDT No. 622232			
Project W-320 Tank 106-C, HNF-2474, Piping Calculations, Vol. 4		ECN No. n/a			
Name	MSIN	Text With All Attach.	Text Only	Attach./ Appendix Only	EDT/ECN Only
JW Bailey	S2-48				
W-320 Project Files	RI-29				